

June 11, 2020

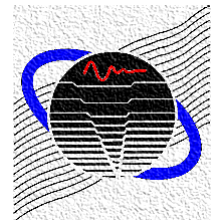
# Impedance Tube Measurements

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Vibro-Acoustics Consortium Web Meeting  
University of Kentucky

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**Vibro-Acoustics Consortium**



# Plane Waves in a Pipe

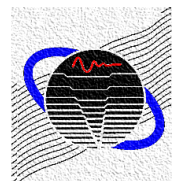
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Daniel Russell, Penn State University  
<https://www.acs.psu.edu/drussell/Demos/waves-intro/waves-intro.html>

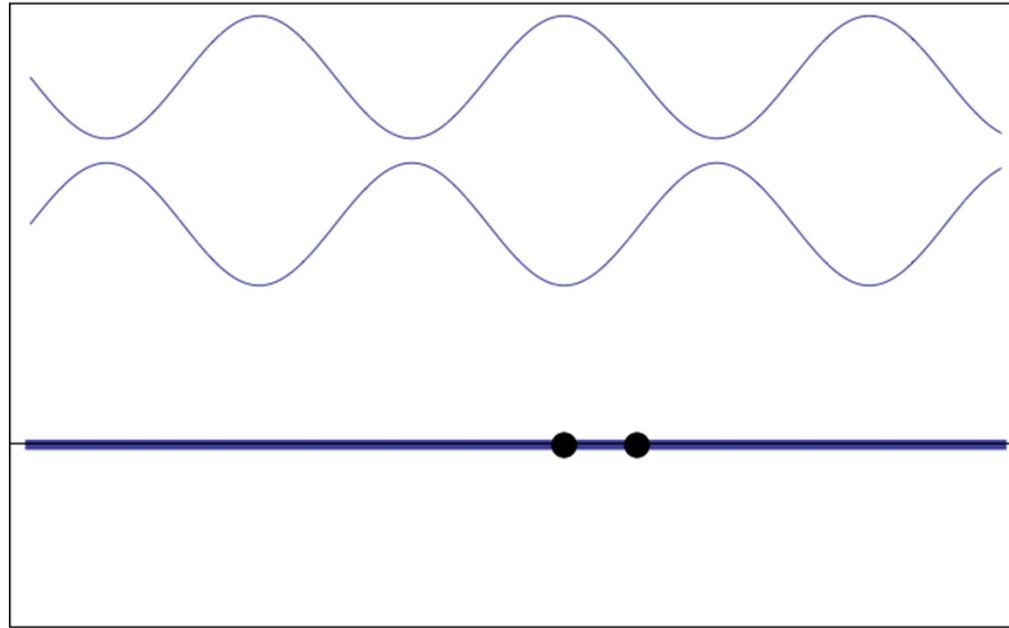
For a pipe, the cutoff frequency is defined as:

$$f_{cutoff} = \frac{c}{1.71d}$$



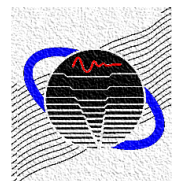
# Plane Waves in a Pipe

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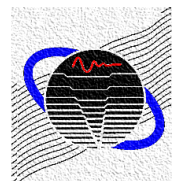
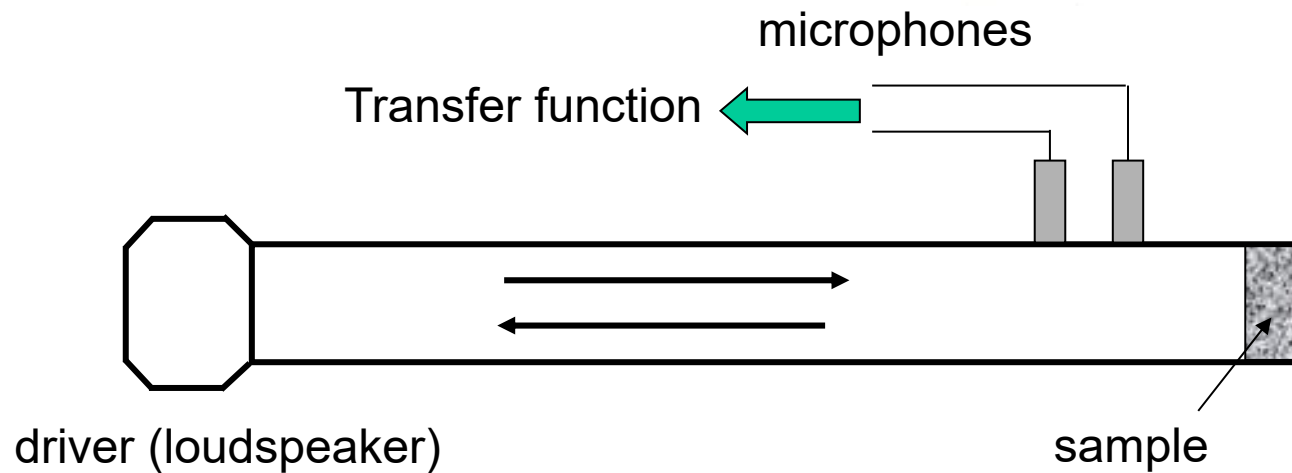
Daniel Russell, Penn State University

<https://www.acs.psu.edu/drussell/Demos/superposition/superposition.html>



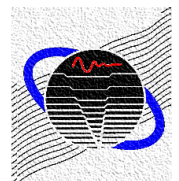
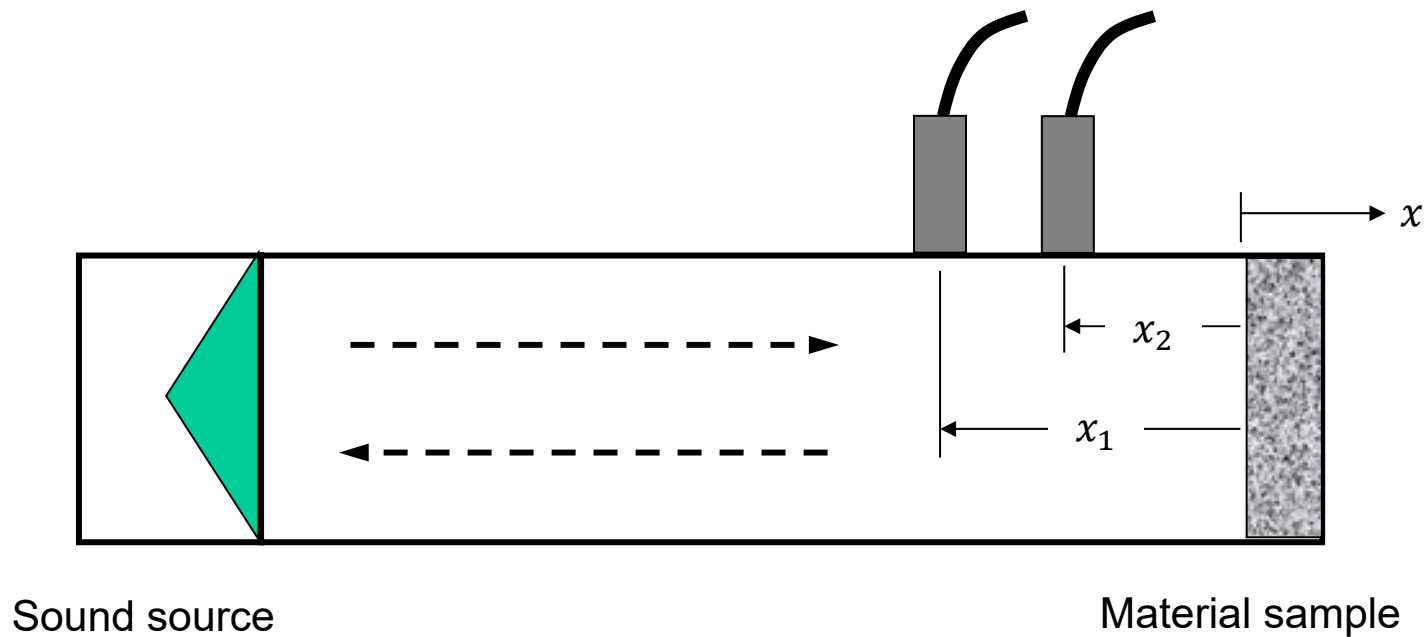
# Part 1 Measurement of Sound Impedance

ASTM E1050-95 (ISO 10534-2) Test Method



# Coordinate System and Microphone Locations

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# Solving for Material Properties

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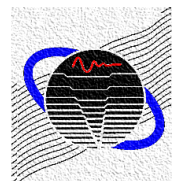
Solving for  $R$ : 
$$R = \frac{e^{-jkx_2} - H_{12}e^{-jkx_1}}{H_{12}e^{jkx_1} - e^{jkx_2}}$$

Normalized specific boundary impedance:

$$\frac{z}{\rho c} = \frac{1 + R}{1 - R}$$

Normal incident sound absorption

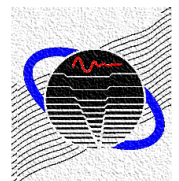
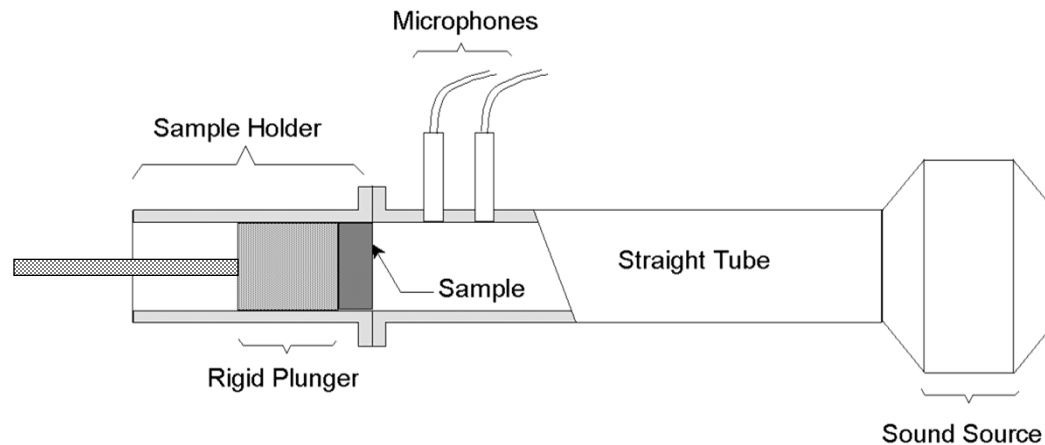
$$\alpha = 1 - |R|^2$$



# Two-Microphone Standards

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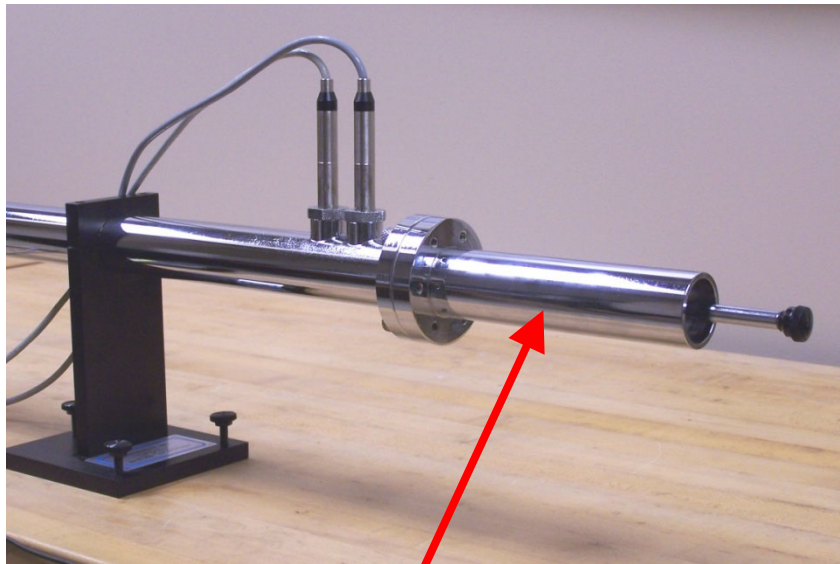
1. ISO 10534-2, *Acoustics-Determination of sound absorption coefficient and impedance in impedance tubes - Part 2: Transfer-function method*
2. ASTM E1050-10, *Standard Test Method for Impedance and Absorption of Acoustical Material Using a Tube, Two Microphones and a Digital Frequency Analysis System*



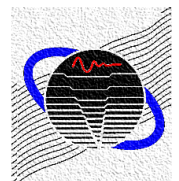
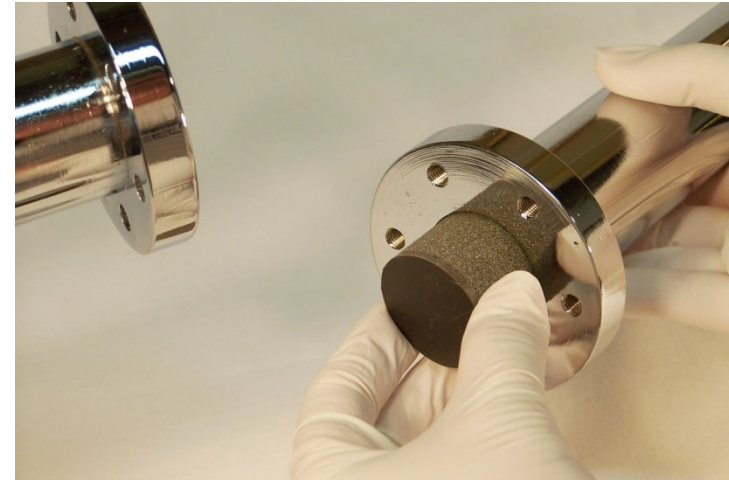


# Sound Absorption Measurement

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Sample holder  
with rigid piston



# Cutting with Rotating Blade

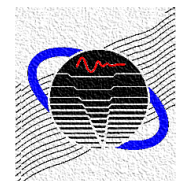
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Inexpensive and accurate if kept sharpened



Stanley, Internoise 2012

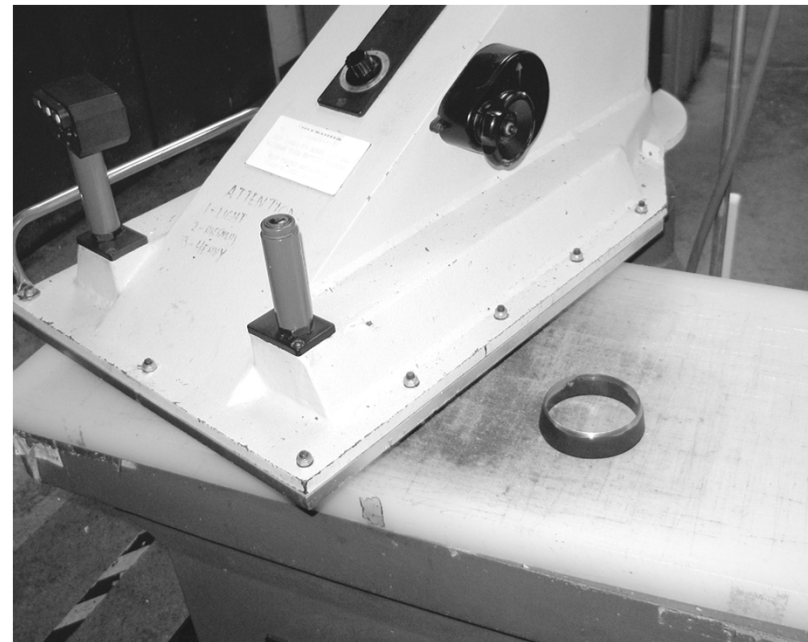


# Stamping Press System

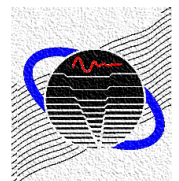
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Used for low-density fibrous materials



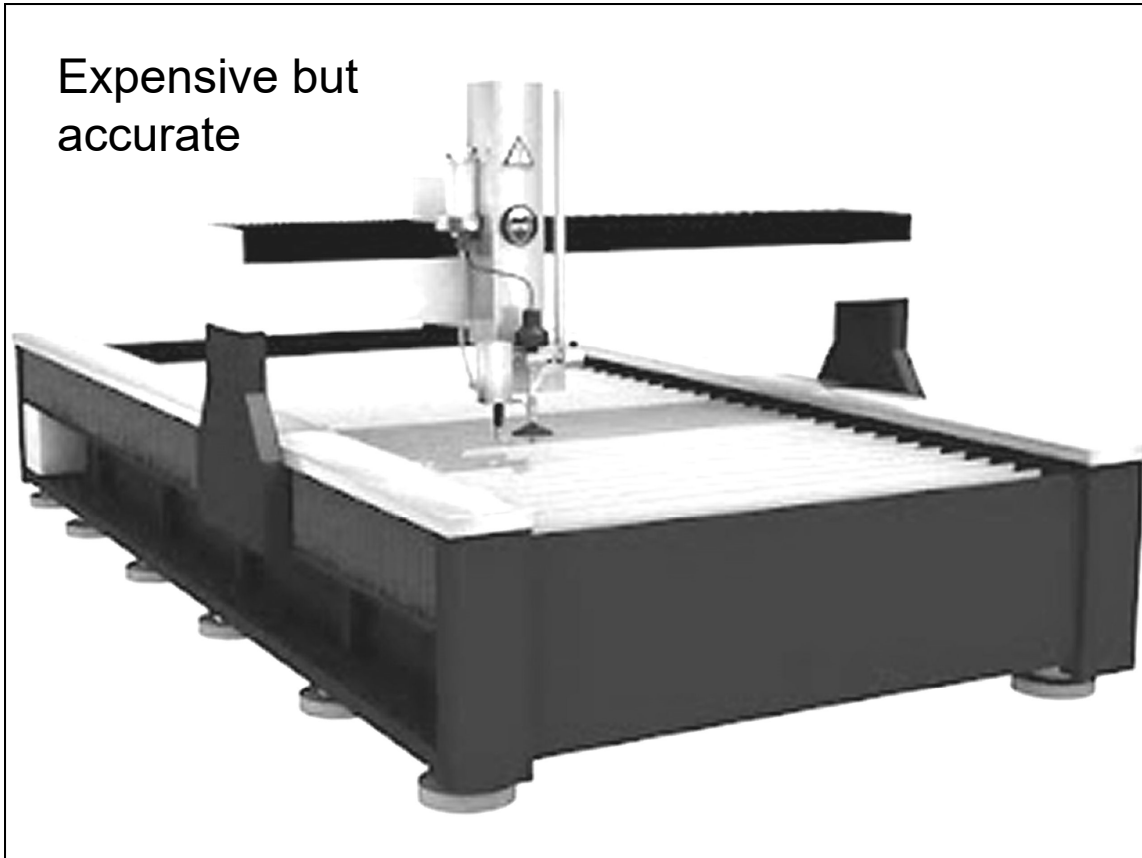
Stanley, Internoise 2012



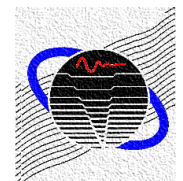
# Water Jet Cutting

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Expensive but  
accurate



Stanley, Internoise 2012

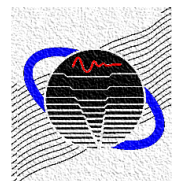


# “Not so Good” and “Good” Specimens

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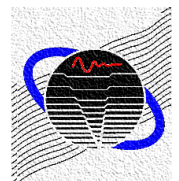
Stanley, Internoise 2012



# Stanley, 2012 Specimen Preparation

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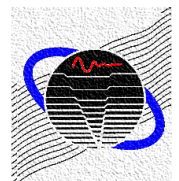
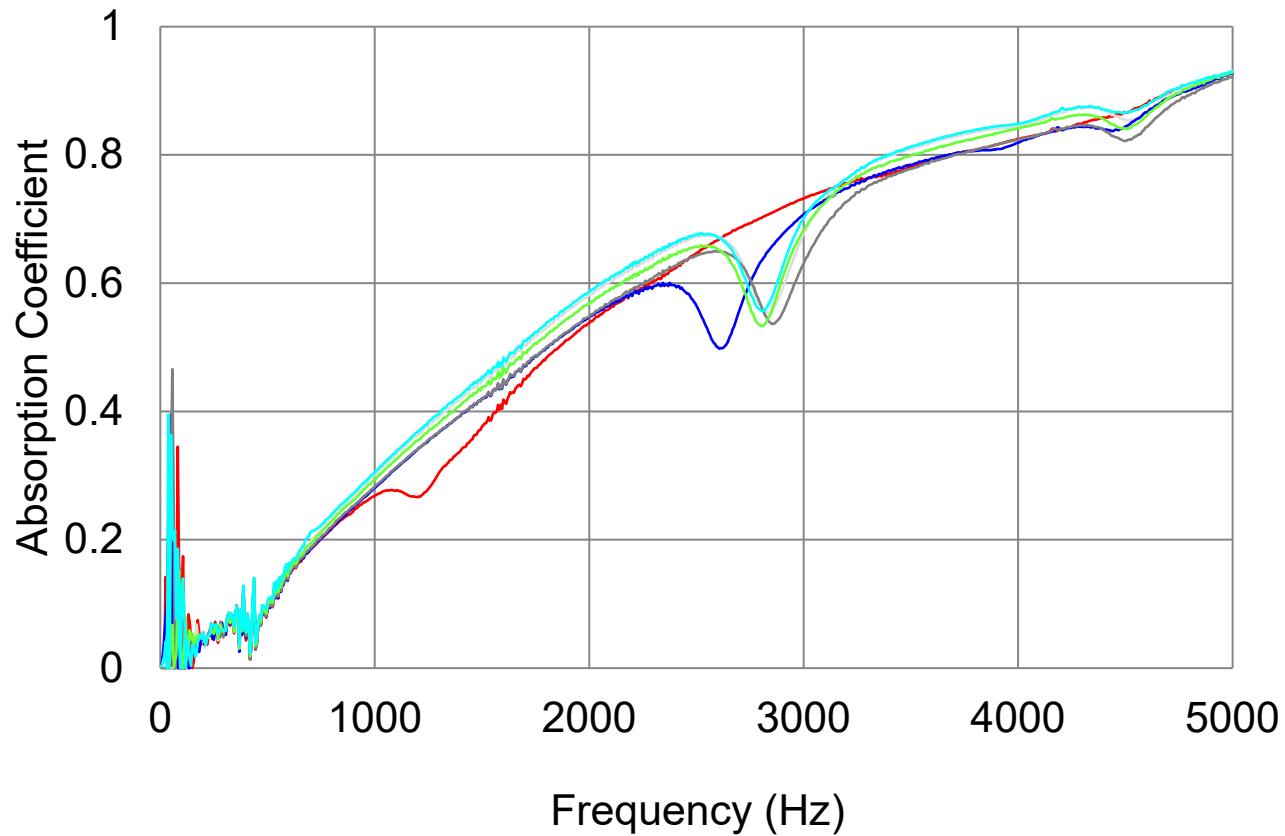
- Face uniformly flush with cell lip
- Front surface even across lip of sample holder
- Extremely small (at most) and consistent gap between specimen and sample holder
- No specimen compression in the holder



# Variability of Melamine

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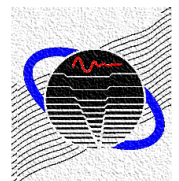
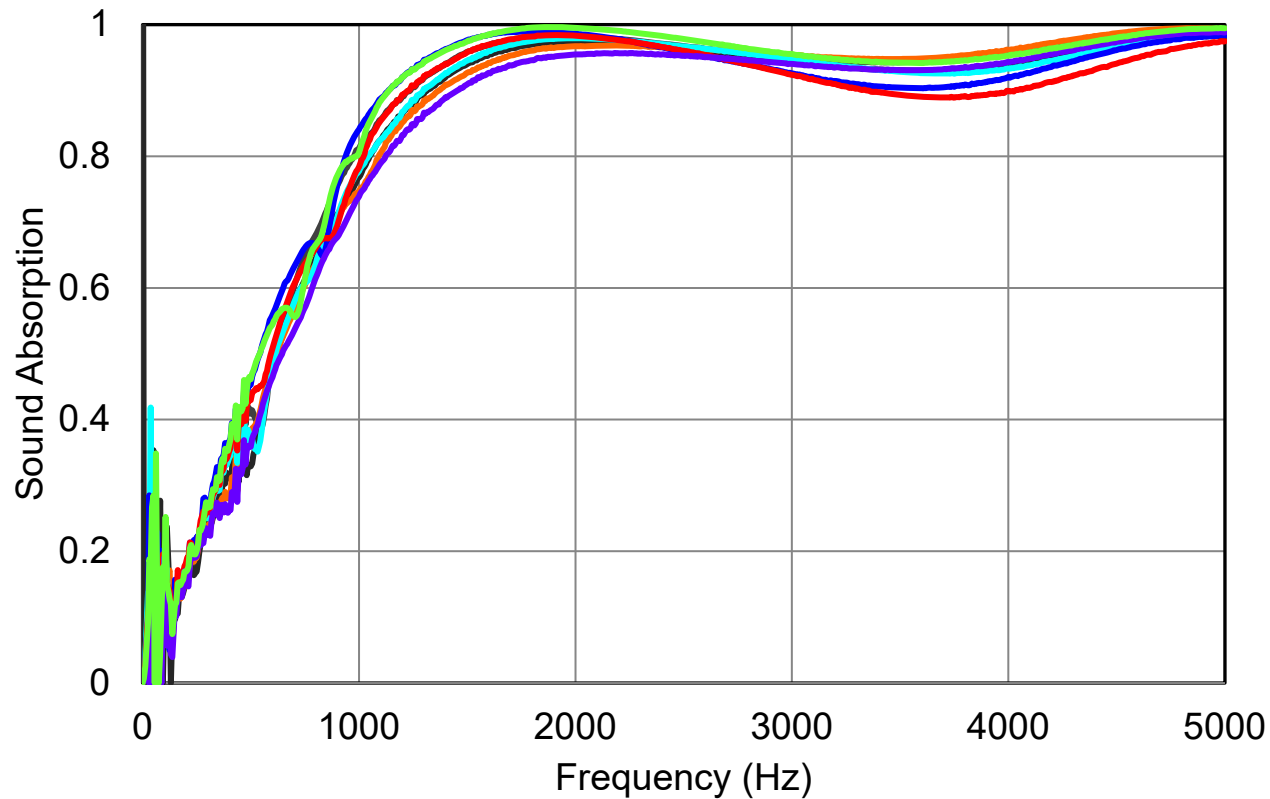
6 Samples of 1.91 cm Melamine



# Variability Glass Fiber

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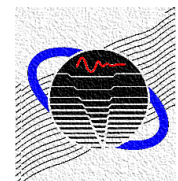
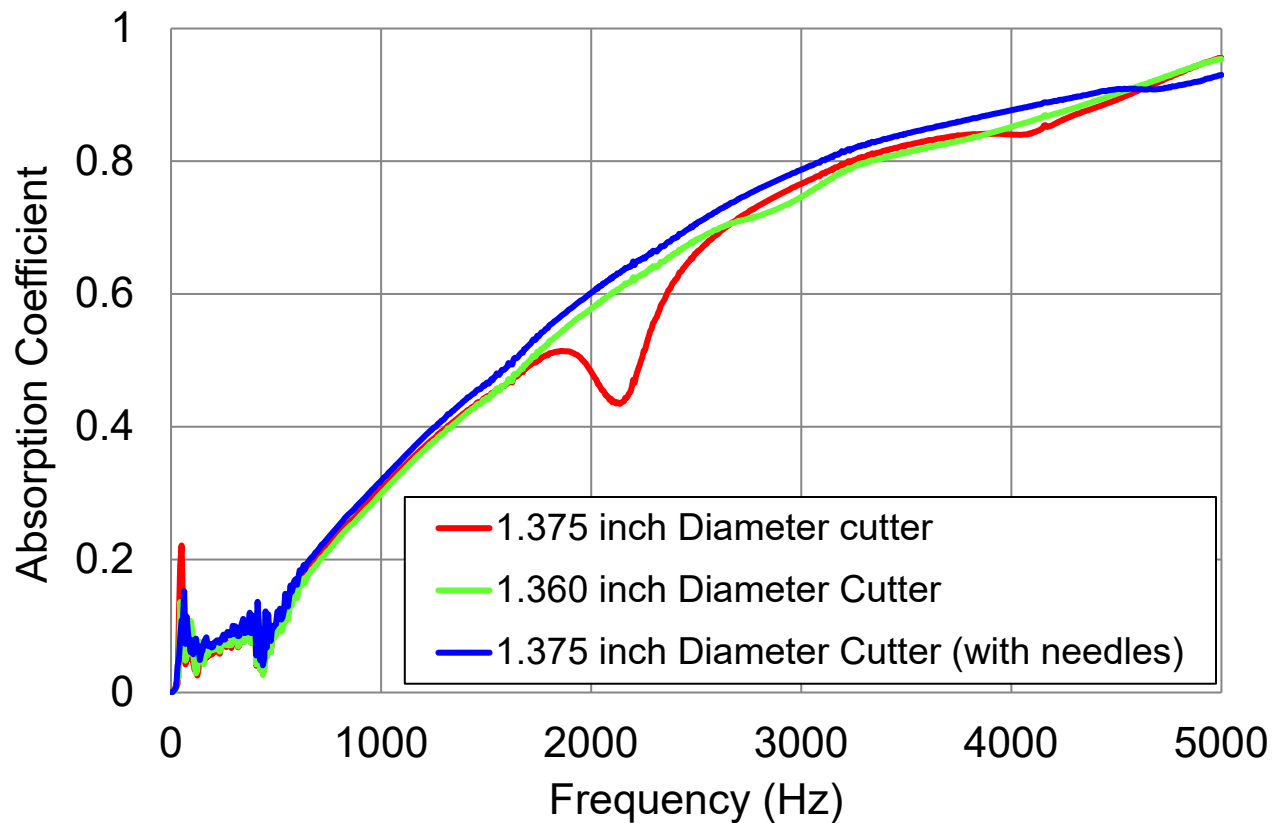
8 Samples of 5 cm Glass Fiber





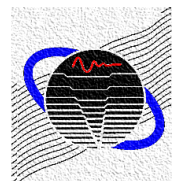
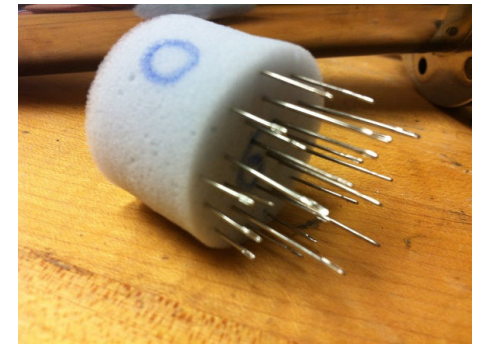
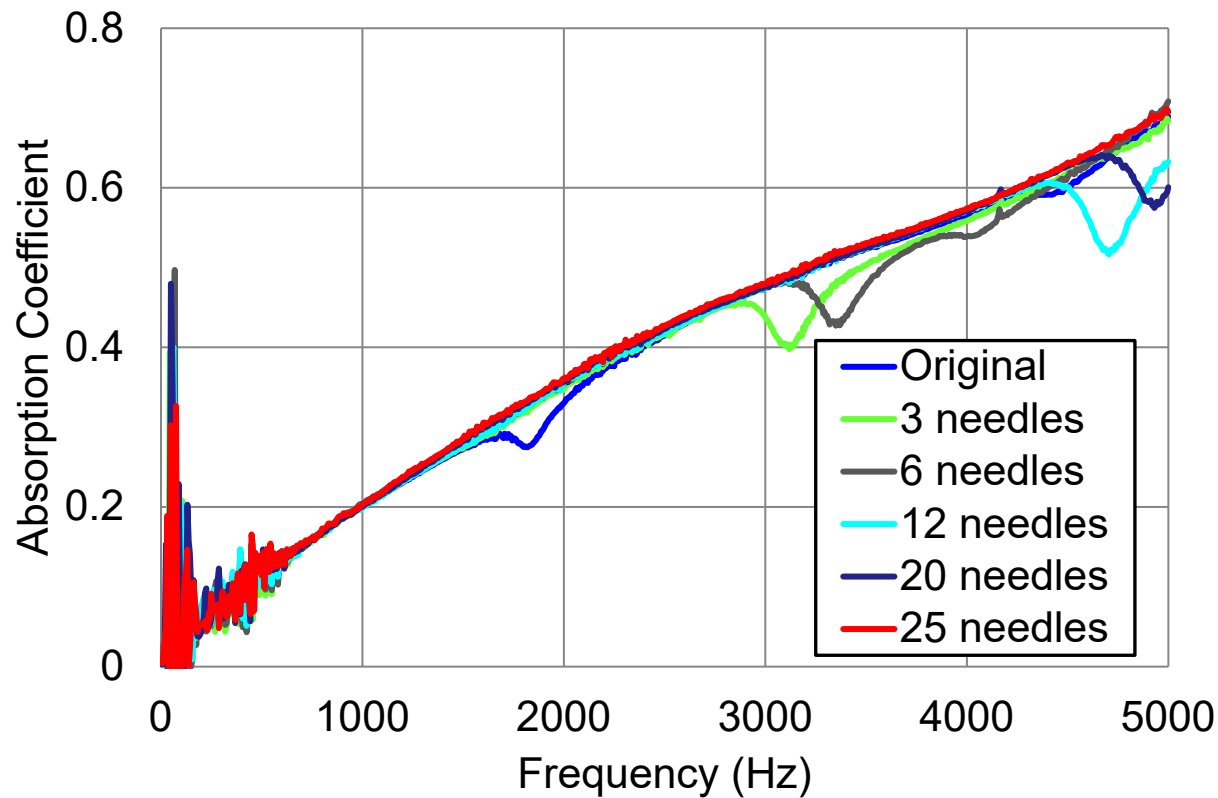
# Effect of Cutter Size

8 Samples of 1 inch thick 0.6 lbs/ft<sup>3</sup> Melamine



# Effect of Adding Needles

1.25 cm thick Melamine

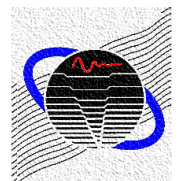


# Summary

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“While the use of an impedance tube system to measure acoustic absorption is not an extremely precise and repeatable process due to unavoidable variations of specimen cutting and cell fit, the disciplined use of the guidelines stated in this paper will help to insure that test results maintain a consistent level of accuracy and validity. The *experience gained with repeated preparation and testing* will also contribute to a better feel for more subtle aspects of preparation and specimen fitting for testing.”

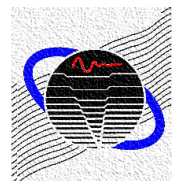
Stanley, *Internoise 2012*



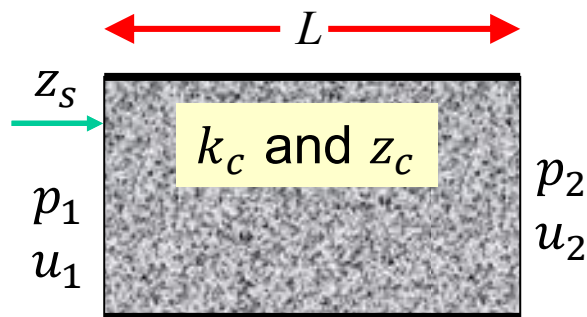
## Part 2 Determination of Bulk Properties

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- Determining the bulk properties
  - ✓ Complex wave number and characteristic impedance
  - ✓ Complex speed of sound and density
- Bulk properties are used
  - ✓ For designing layered absorbers
  - ✓ In FEM and BEM models



# Porous Absorbers Property Determination



$$k_c = \frac{\omega}{c'}$$

$$z_c = \rho' c'$$

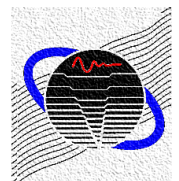
## Determination of Sound Absorption

$$\begin{Bmatrix} p_1 \\ u_1 \end{Bmatrix} = \begin{bmatrix} \cos(k_c L) & jz_c \sin(k_c L) \\ j/z_c \sin(k_c L) & \cos(k_c L) \end{bmatrix} \begin{Bmatrix} p_2 \\ u_2 \end{Bmatrix}$$

$$\rightarrow z_s = \frac{p_1}{u_1} = -jz_c \cot(k_c L)$$

$$R = \frac{z_s - \rho c}{z_s + \rho c}$$

$$\rightarrow \alpha = 1 - |R|^2$$



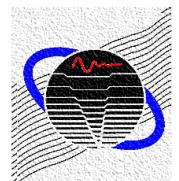
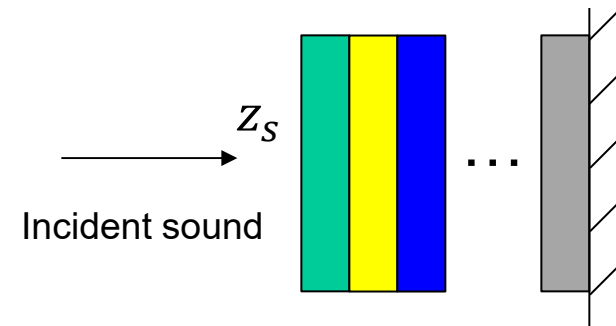
# Porous Absorbers Layered

$$[T] = \begin{bmatrix} T_{11} & T_{12} \\ T_{21} & T_{22} \end{bmatrix} = [T_1][T_2][T_3]\dots[T_n]$$

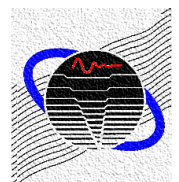
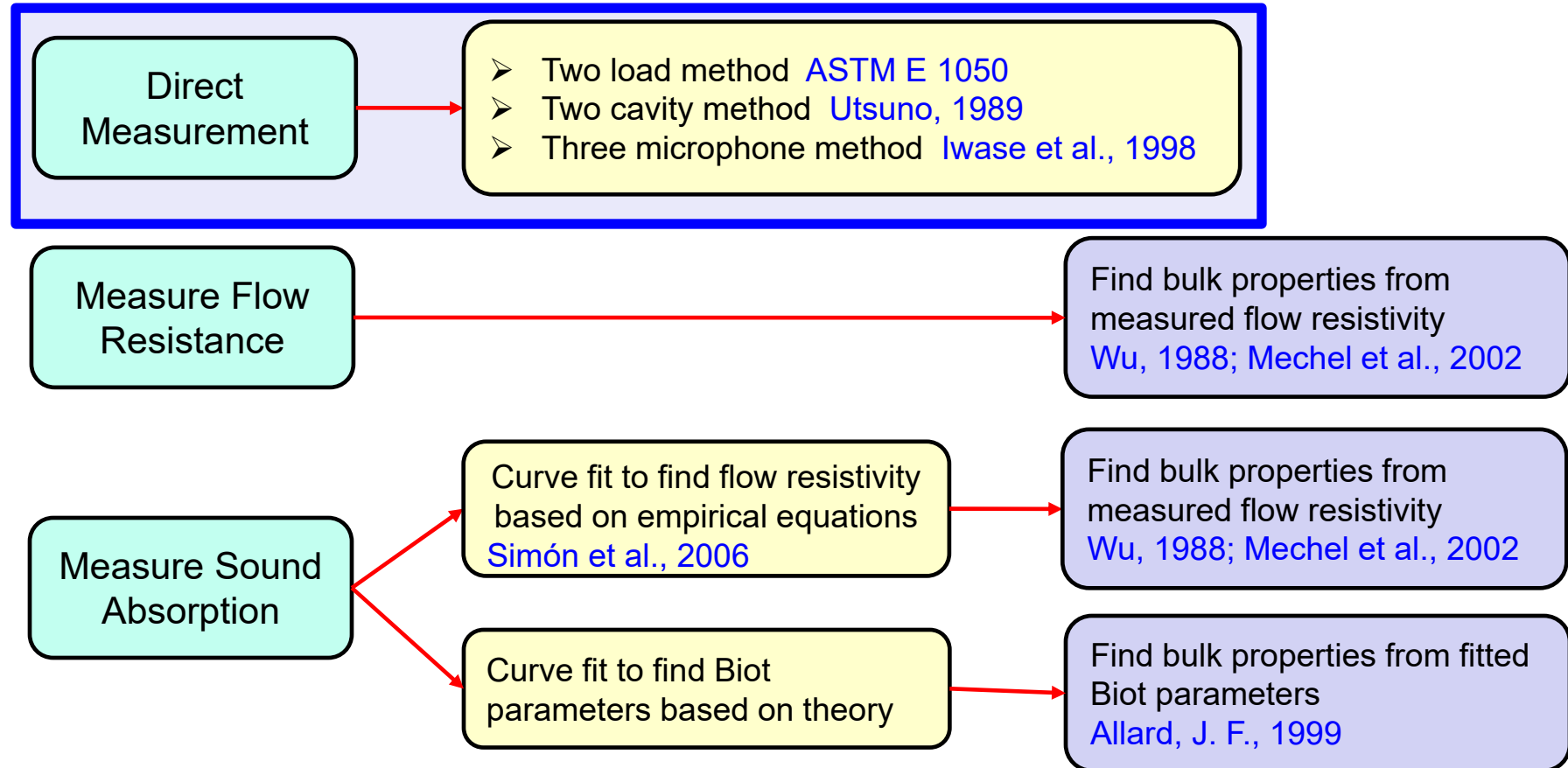
$$\rightarrow z_s = \frac{T_{11}}{T_{21}}$$

$$R = \frac{z_1 - \rho c}{z_1 + \rho c}$$

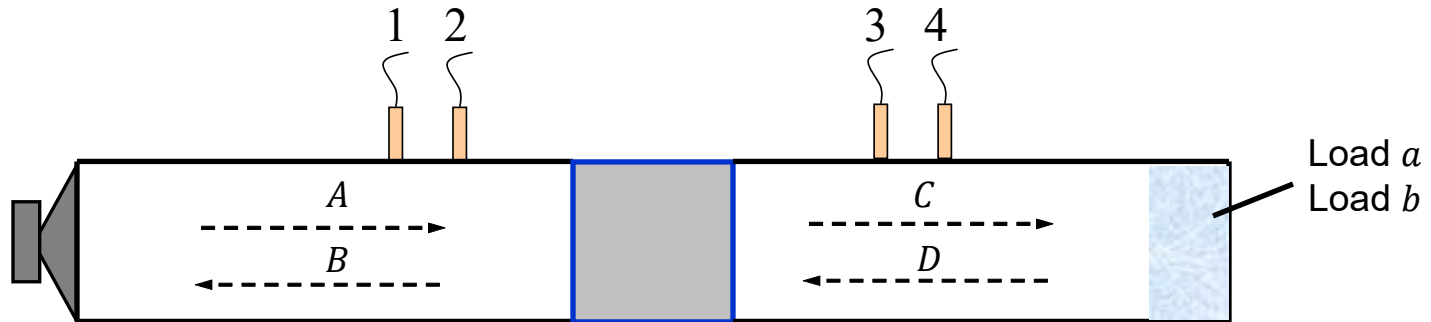
$$\rightarrow \alpha = 1 - |R|^2$$



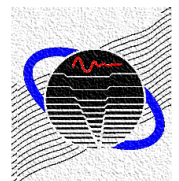
# Overview of Approaches



# Why Two-Loads?

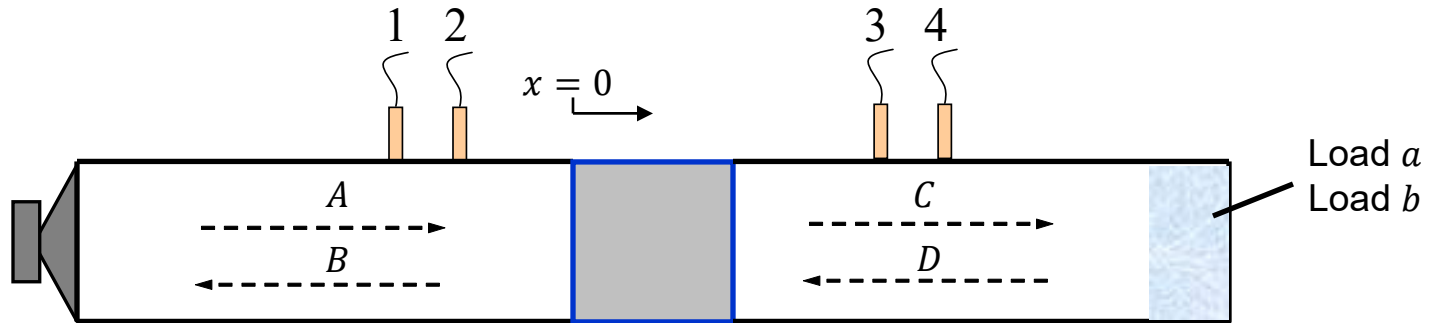


$$\begin{Bmatrix} p_1 \\ u_1 \end{Bmatrix} = \begin{bmatrix} T_{11} & T_{12} \\ T_{21} & T_{22} \end{bmatrix} \begin{Bmatrix} p_2 \\ u_2 \end{Bmatrix}$$

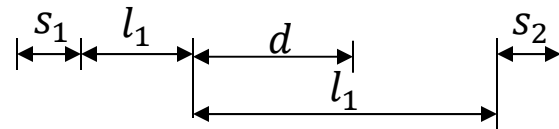




# Two-Load Method



For each load:

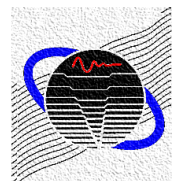


$$A = j \frac{H_{1,ref} e^{-jkl_1} - H_{2,ref} e^{-jk(l_1+s_1)}}{2 \sin k s_1}$$

$$C = j \frac{H_{3,ref} e^{jk(l_2+s_2)} - H_{4,ref} e^{jkl_2}}{2 \sin k s_2}$$

$$B = j \frac{H_{2,ref} e^{jk(l_1+s_1)} - H_{1,ref} e^{jkl_1}}{2 \sin k s_1}$$

$$D = j \frac{H_{4,ref} e^{-jkl_2} - H_{3,ref} e^{-jk(l_2+s_2)}}{2 \sin k s_2}$$



# Transmission Loss Measurement

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Pressures and particle velocities at two ends of the sample:

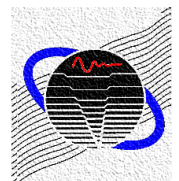
$$p_0 = A + B \qquad p_d = Ce^{-jkd} + De^{jkd}$$

$$u_0 = \frac{(A - B)}{\rho c} \qquad u_d = \frac{(Ce^{-jkd} - De^{jkd})}{\rho c}$$

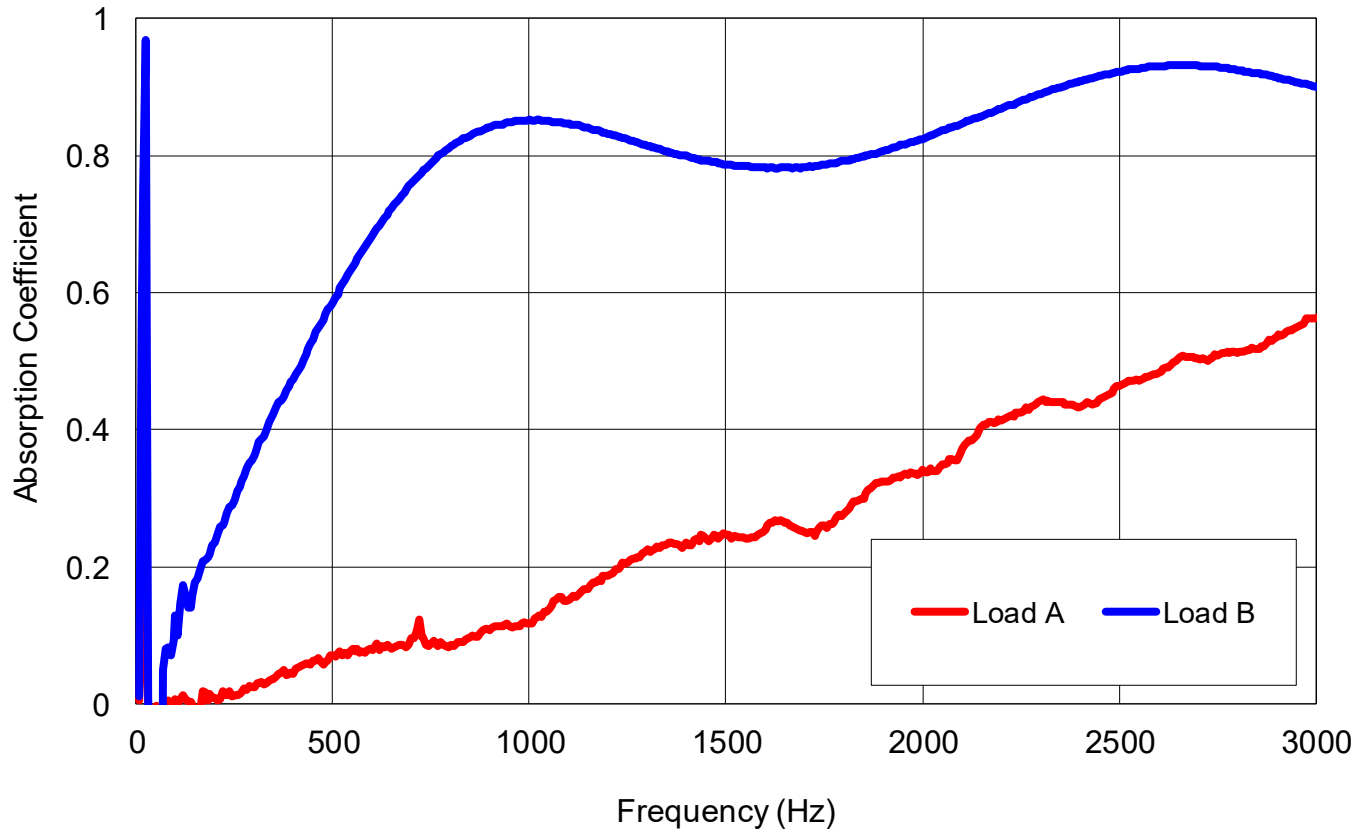
Four-pole matrix (subscripts  $a$  and  $b$  indicate acoustic loads)

$$T = \begin{bmatrix} \frac{p_{0a}u_{db} - p_{0b}u_{da}}{p_{da}u_{db} - p_{db}u_{da}} & \frac{p_{0b}p_{da} - p_{0a}p_{db}}{p_{da}u_{db} - p_{db}u_{da}} \\ \frac{u_{0a}u_{db} - u_{0b}u_{da}}{p_{da}u_{db} - p_{db}u_{da}} & \frac{p_{da}u_{0b} - p_{db}u_{0a}}{p_{da}u_{db} - p_{db}u_{da}} \end{bmatrix}$$

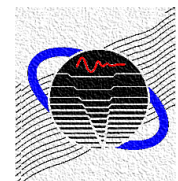
$$TL = 20 \log_{10} \left| \frac{1}{2} \left( T_{11} + \frac{T_{12}}{\rho c} + \rho c T_{21} + T_{22} \right) \right|$$



# Two Loads



Load A: Open tube. Load B: 10 cm sound absorbing material.



## Two Load Method

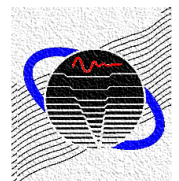
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$$\begin{bmatrix} T_{11} & T_{12} \\ T_{21} & T_{22} \end{bmatrix} = \begin{bmatrix} \cos(k_c d) & jz_c \sin(k_c d) \\ \frac{j \sin(k_c d)}{z_c} & \cos(k_c d) \end{bmatrix}$$

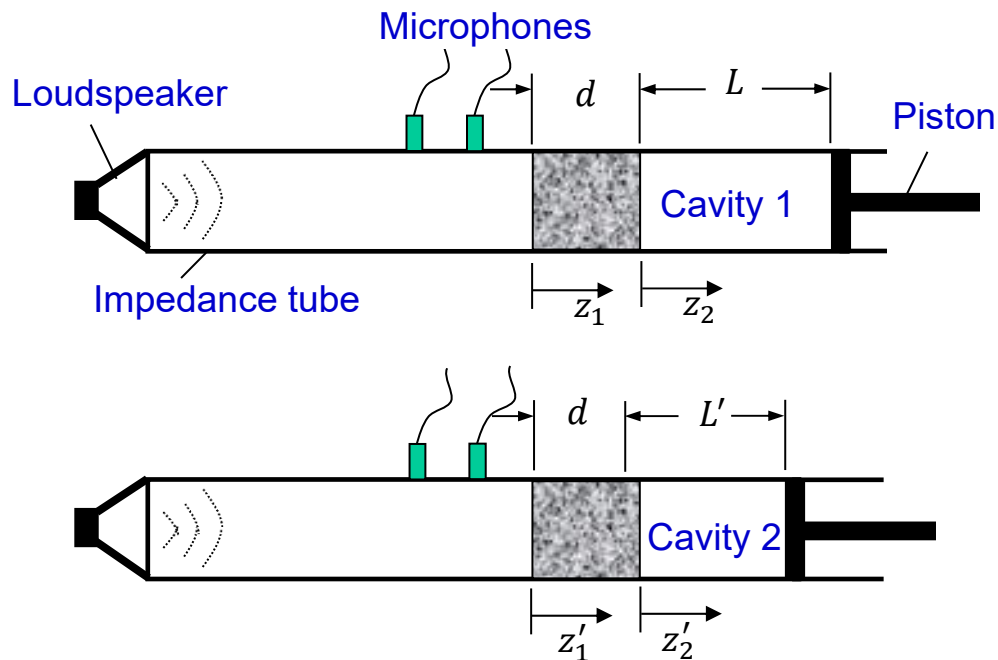


$$z_c = \sqrt{\frac{T_{12}}{T_{21}}}$$

$$k_c d = \arctan\left(\frac{T_{12}}{jT_{11}z_c}\right)$$

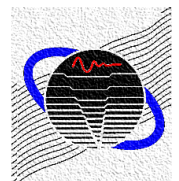


# Two Cavity Method

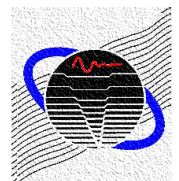
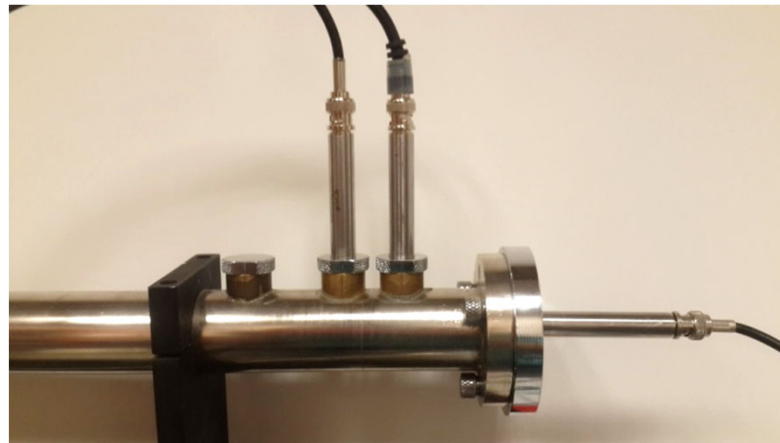
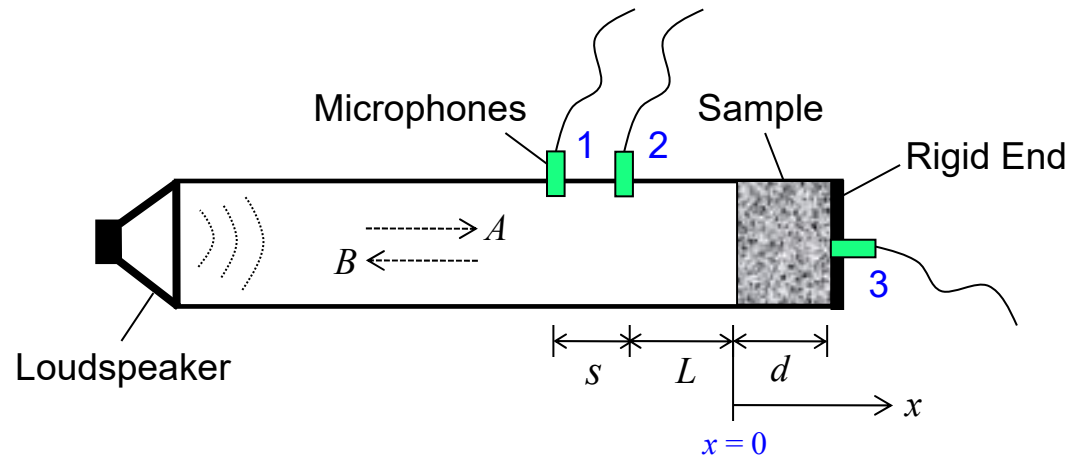


$$z_c = \pm \sqrt{\frac{z_1 z'_1 (z_2 - z'_2) - z_2 z'_2 (z_1 - z'_1)}{(z_2 - z'_2) - (z_1 - z'_1)}}$$

$$k_c = \left(\frac{1}{2jd}\right) \ln \left(\frac{(z_1 + z_c)(z_2 - z_c)}{(z_1 - z_c)(z_2 + z_c)}\right)$$



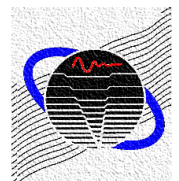
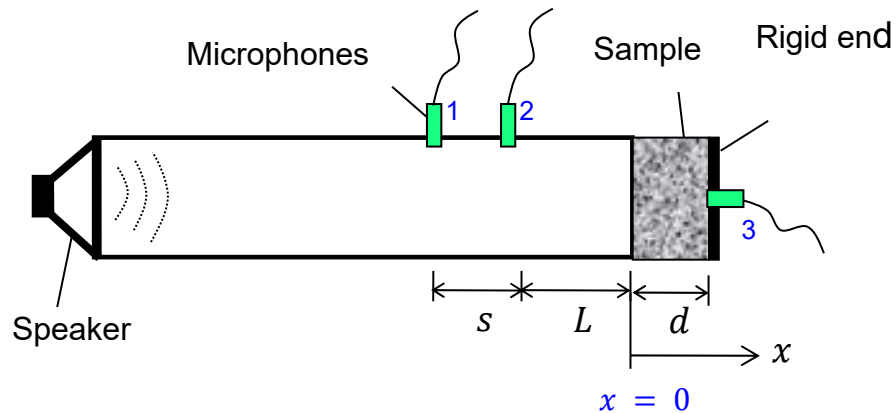
# Three Microphone Method



# Three Microphone Method

Reflection coefficient:

$$R = \frac{e^{jks} - H_{12}}{H_{12} - e^{-jks}} e^{2jkL}$$



# Three Microphone Method

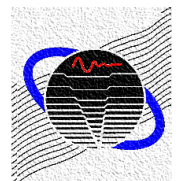
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The complex wave number of the sample is:

$$k_c = \frac{1}{d} \arccos \left( \left( \frac{1 + R}{e^{jkL} + R e^{-jkL}} \right) H_{23} \right)$$

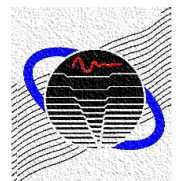
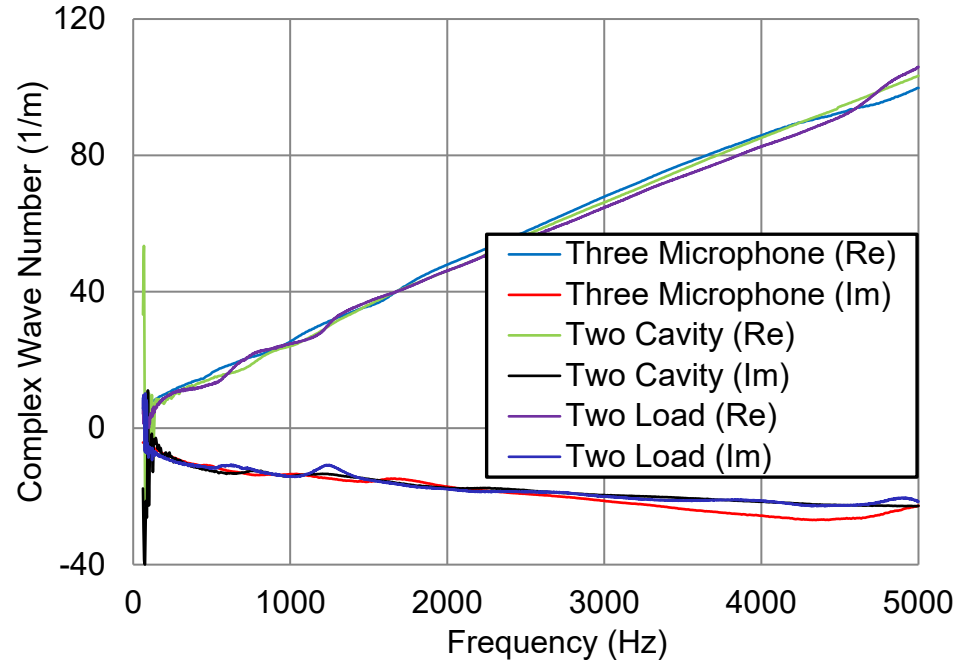
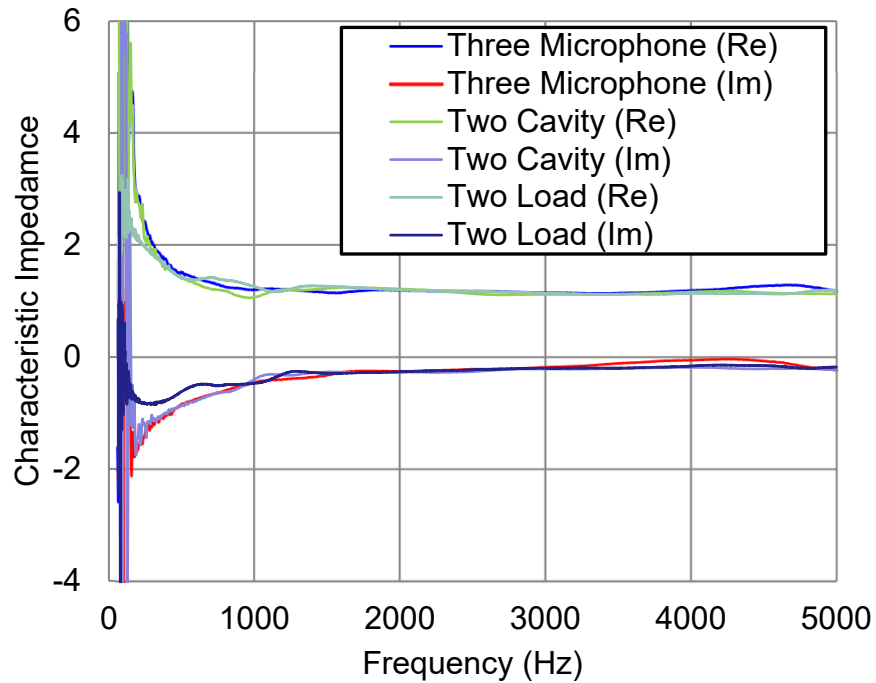
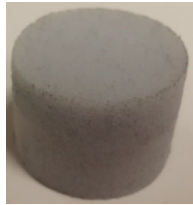
The characteristic impedance of the sample is:

$$z_c = jz_0 \frac{1 + R}{1 - R} \tan(k_c d)$$



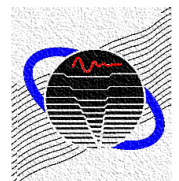
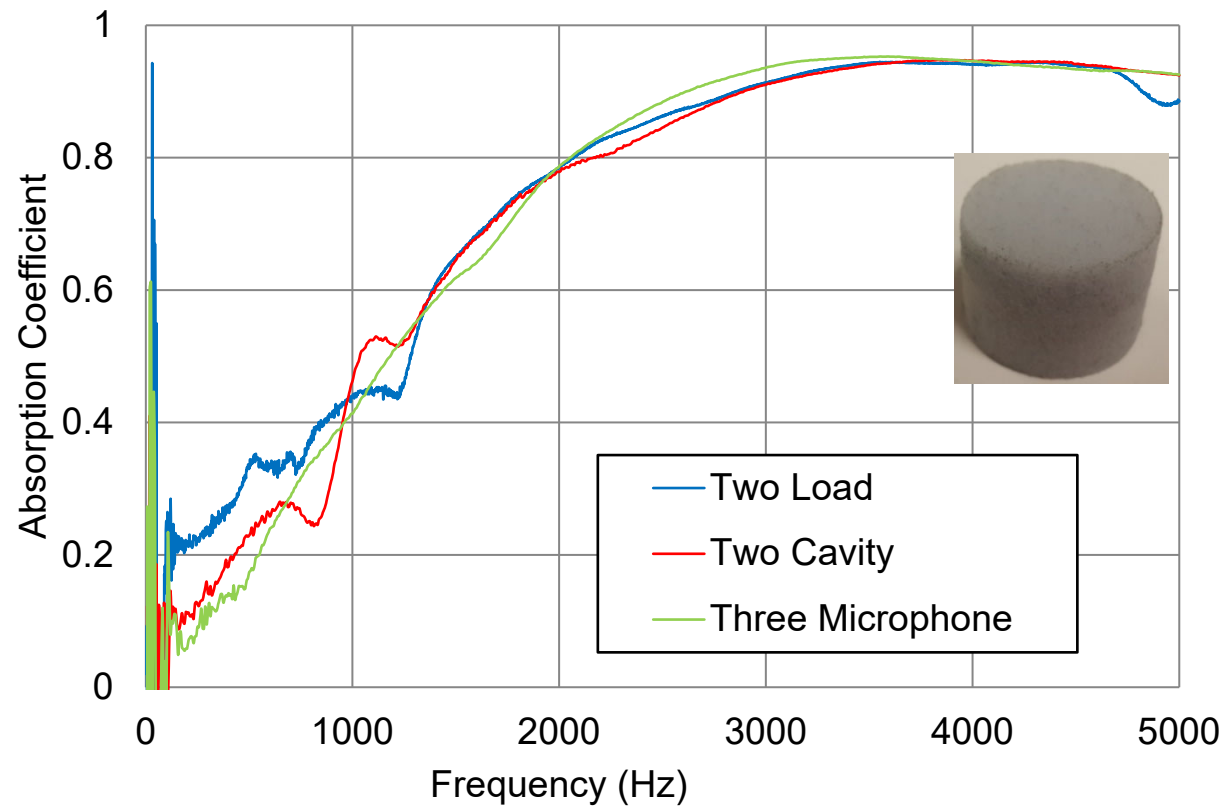


# 2.5 cm Melamine Bulk Properties



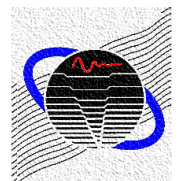
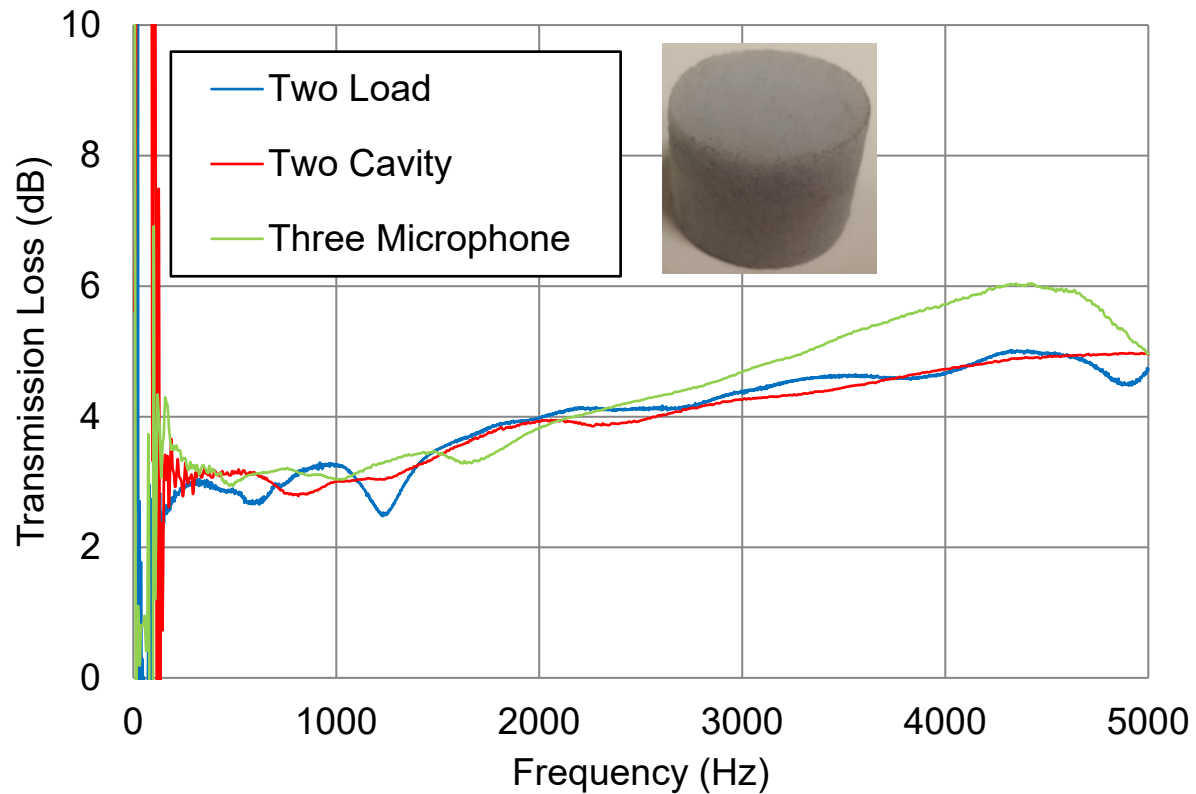
# Melamine Sound Absorption Coefficient

2.5 cm Melamine Foam

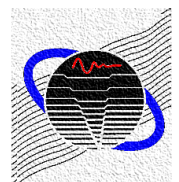
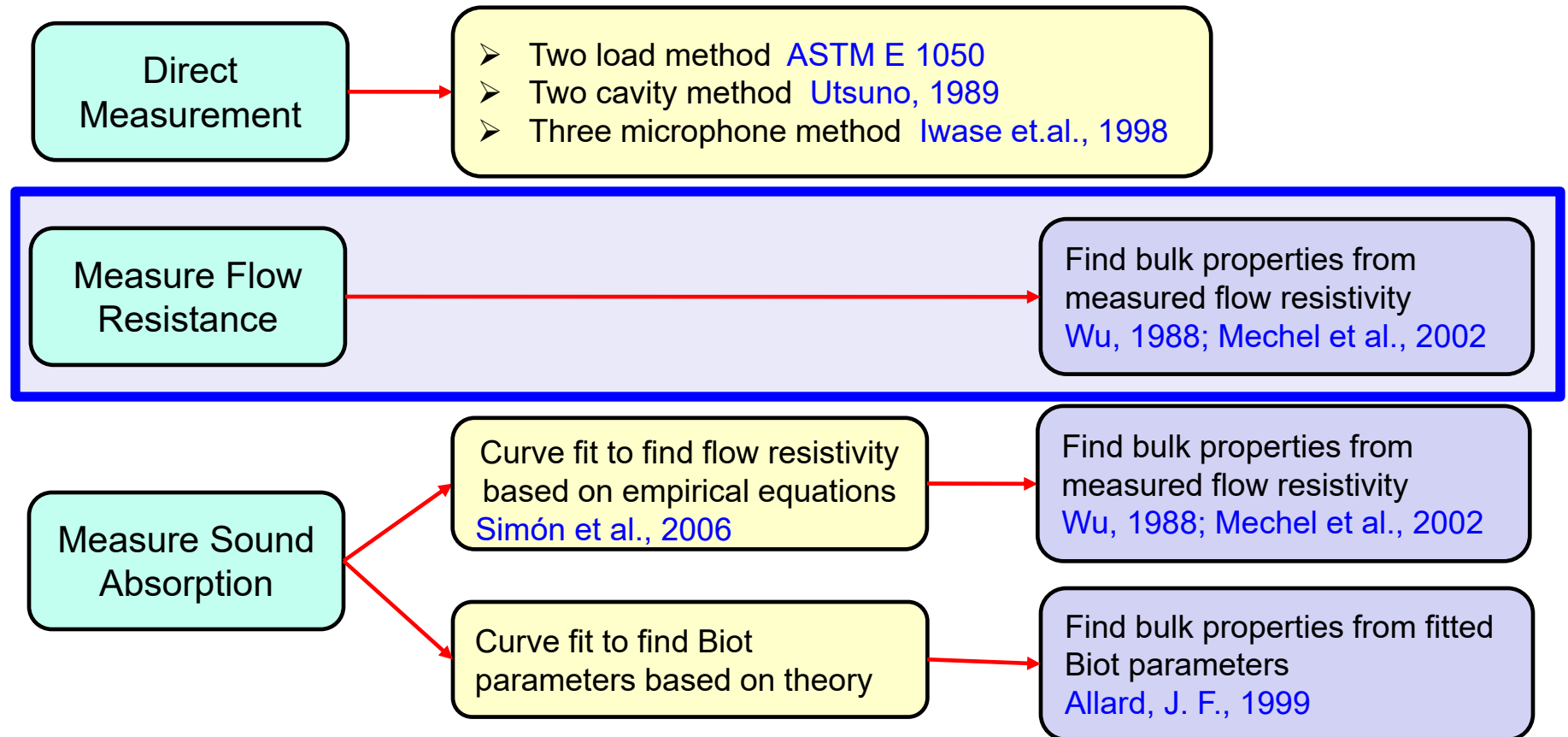


# Melamine Transmission Loss

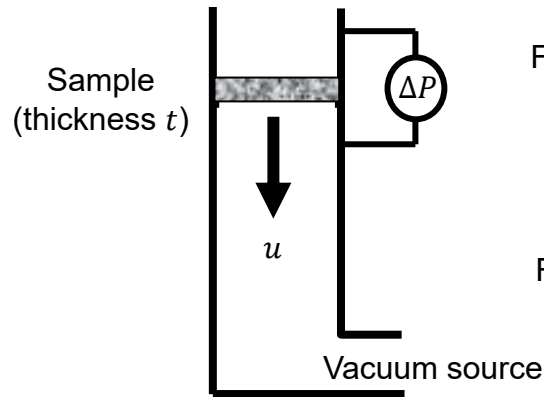
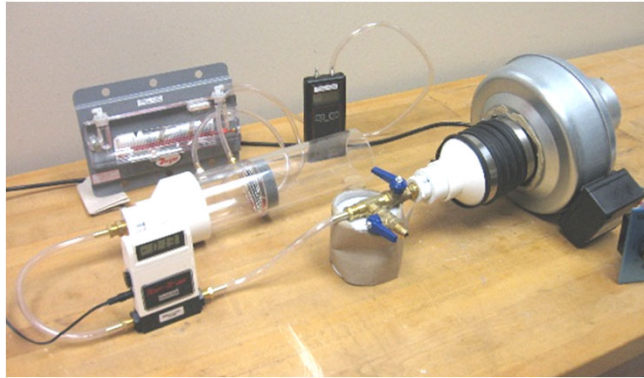
## 2.5 cm Melamine Foam



# Overview of Approaches



# Measure Flow Resistance



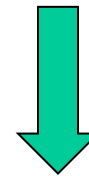
Flow resistance:

$$r_s = \frac{\Delta P}{u}$$

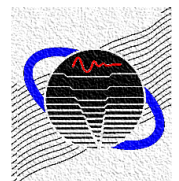
Flow resistivity:

$$\sigma = \frac{r_s}{t}$$

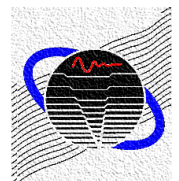
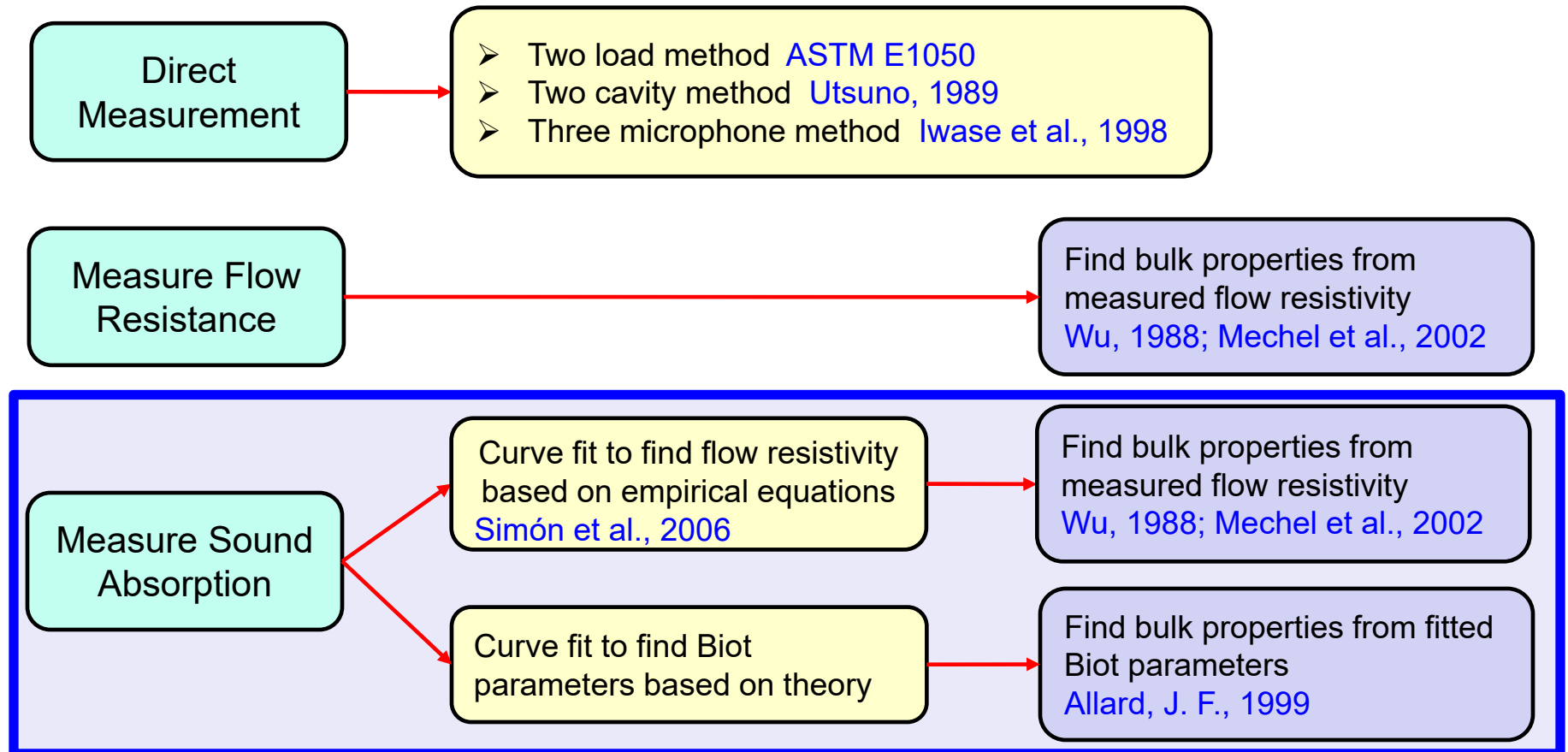
Measure  
Flow Resistivity Using  
ASTM C522



Plug into Empirical Models  
[See Sound Absorptive  
Material Webinar](#)

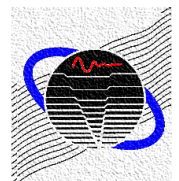
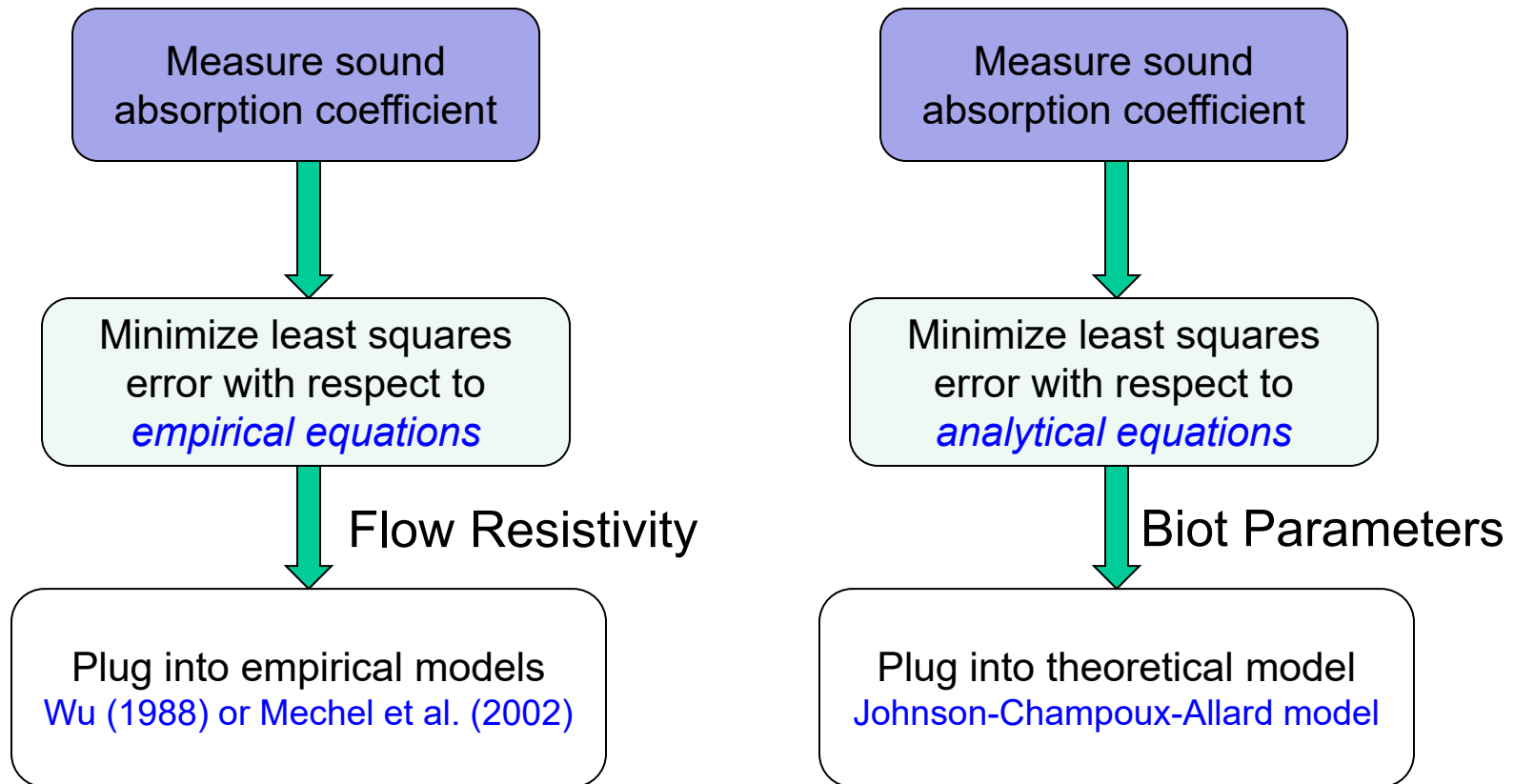


# Overview of Approaches

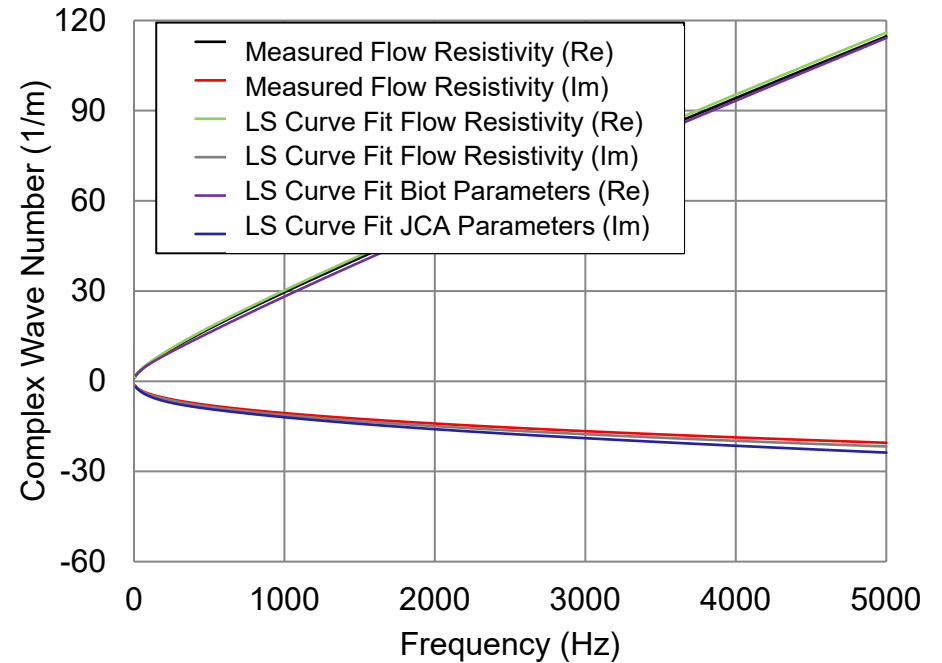
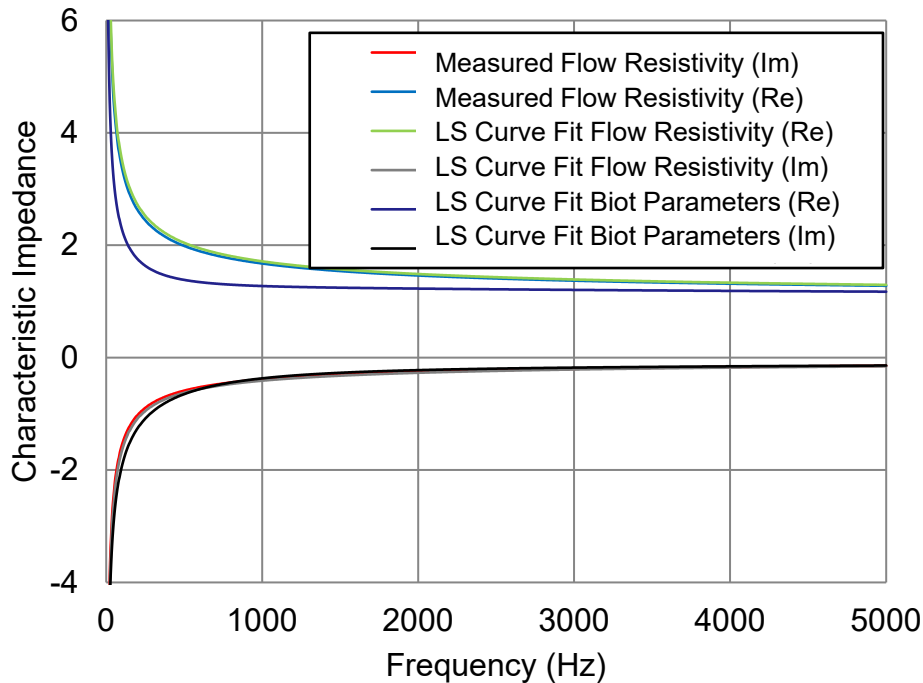


# Curve Fitting Methods

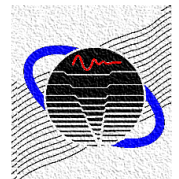
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# 2.5 cm Melamine Bulk Properties



Measured	12,100 Rays/m
Simón Curve Fitting	11,400 Rays/m

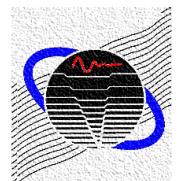




# Summary

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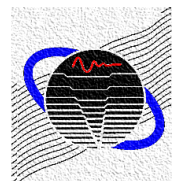
- Compared the various methods for determining the bulk properties of sound absorbers.
- Curve fitting approaches are adequate for common sound absorbing materials.



## Part 3 Covers and Adhesives

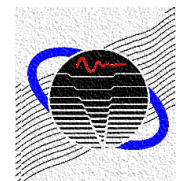
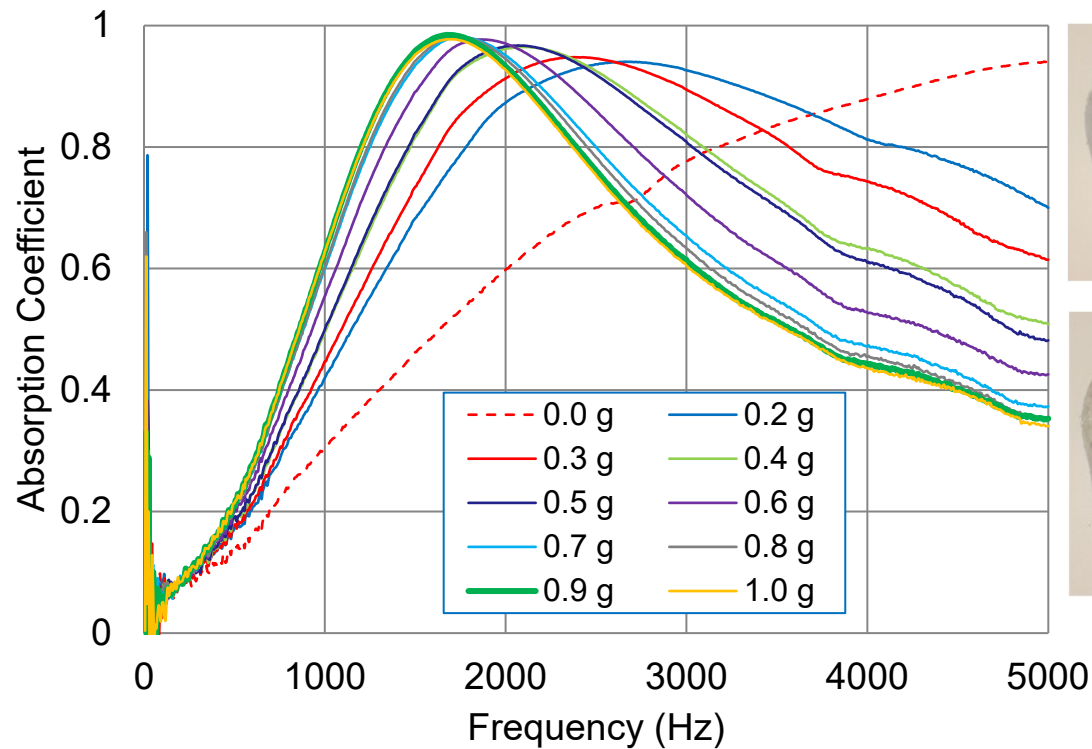
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- How do adhesives and covers between sound absorbing layers affect the overall performance?
- How can we determine the acoustic properties of an adhesive or a cover?



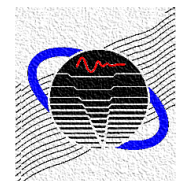
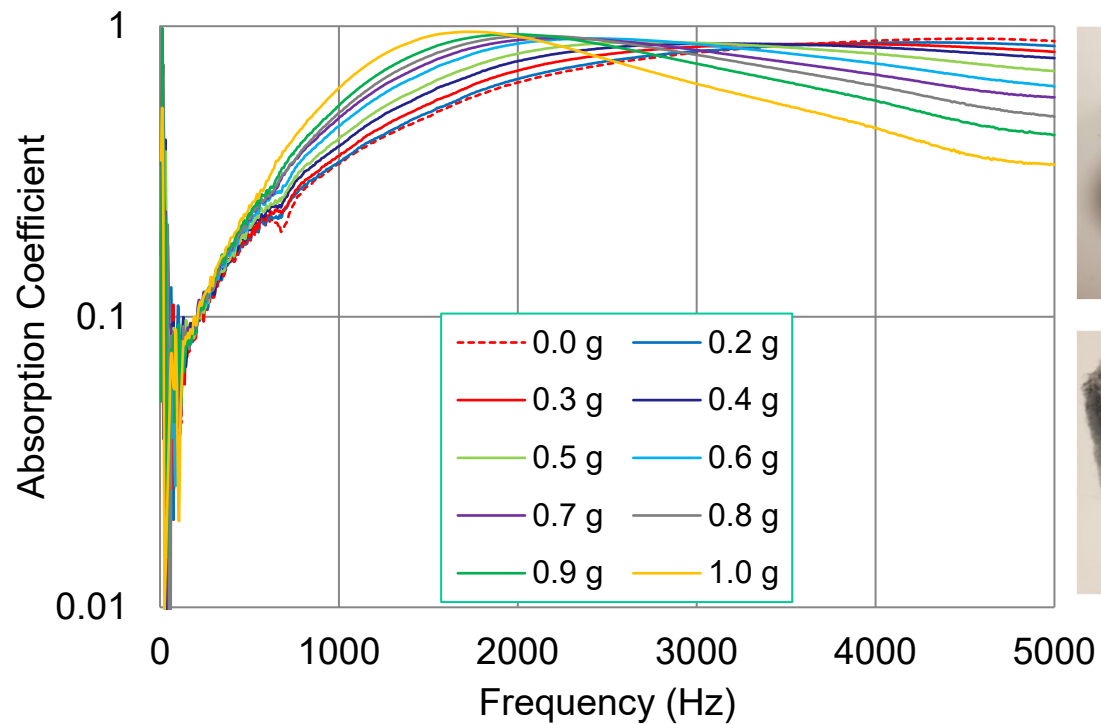
# Melamine Effect of Adhesives

1.8 cm thick 9.6 kg/m<sup>3</sup> Melamine Foam



# Fiber Effect of Adhesives

2.5 cm thick 19.2 kg/m<sup>3</sup> Fiber



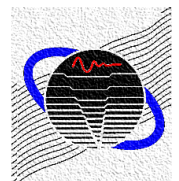
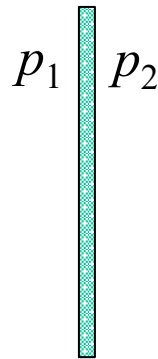
# Transfer Impedance

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A transfer impedance is commonly used to model perforates, covers and source impedance. Particle velocity is assumed to be continuous across the layer.

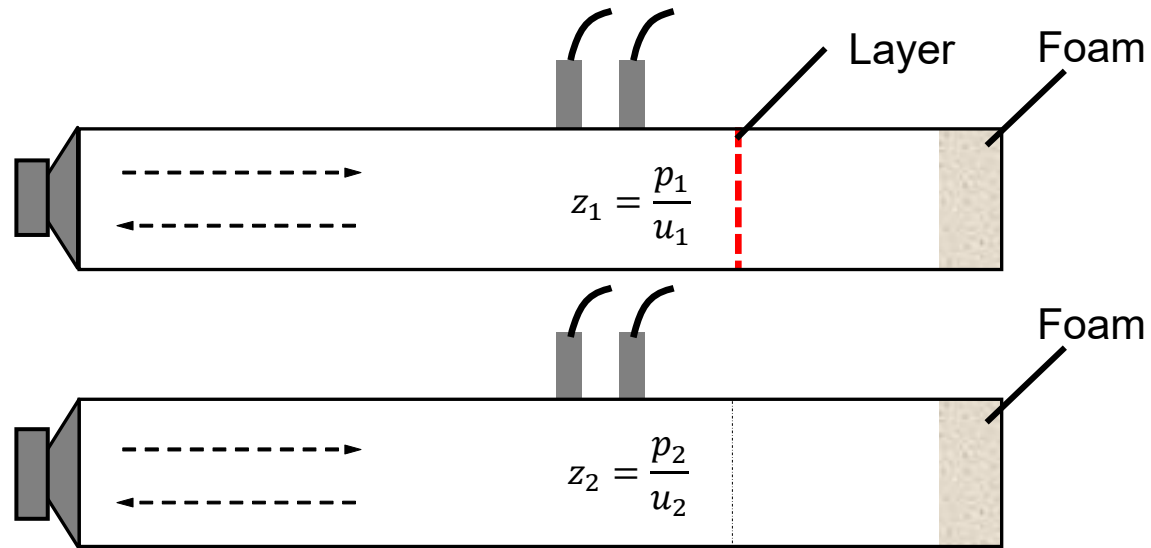
$$z_{tr} = \frac{p_1 - p_2}{u}$$

$$u_1 = u_2 = u$$

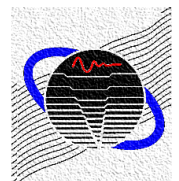


# Transfer Impedance Measurement

Measurement of perforates or fabrics

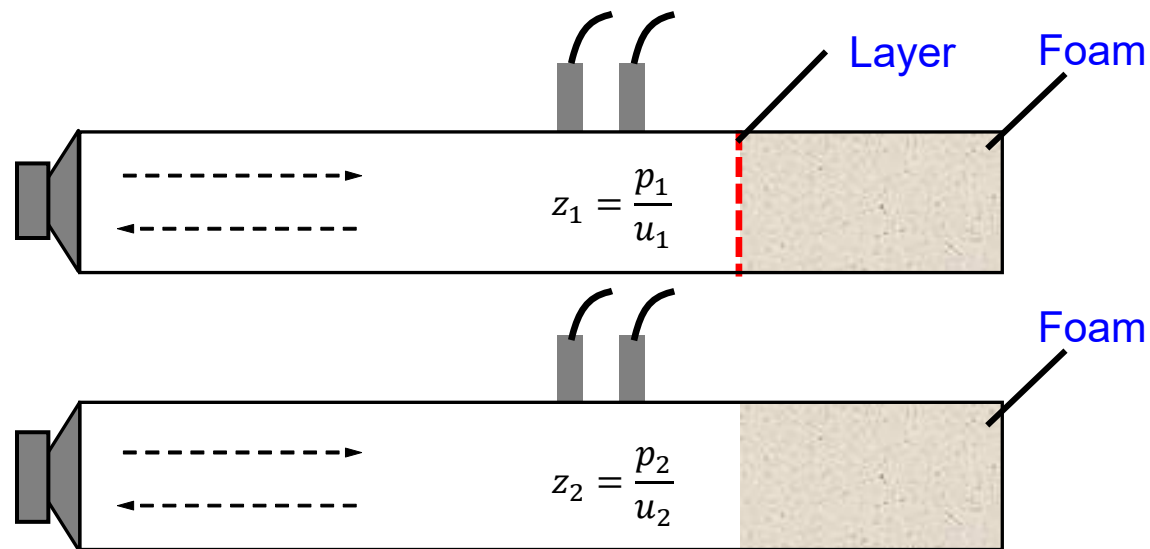


$$Z_{tr} = z_1 - z_2$$

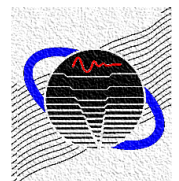


# Transfer Impedance Measurement

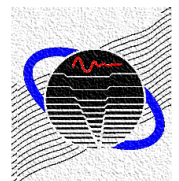
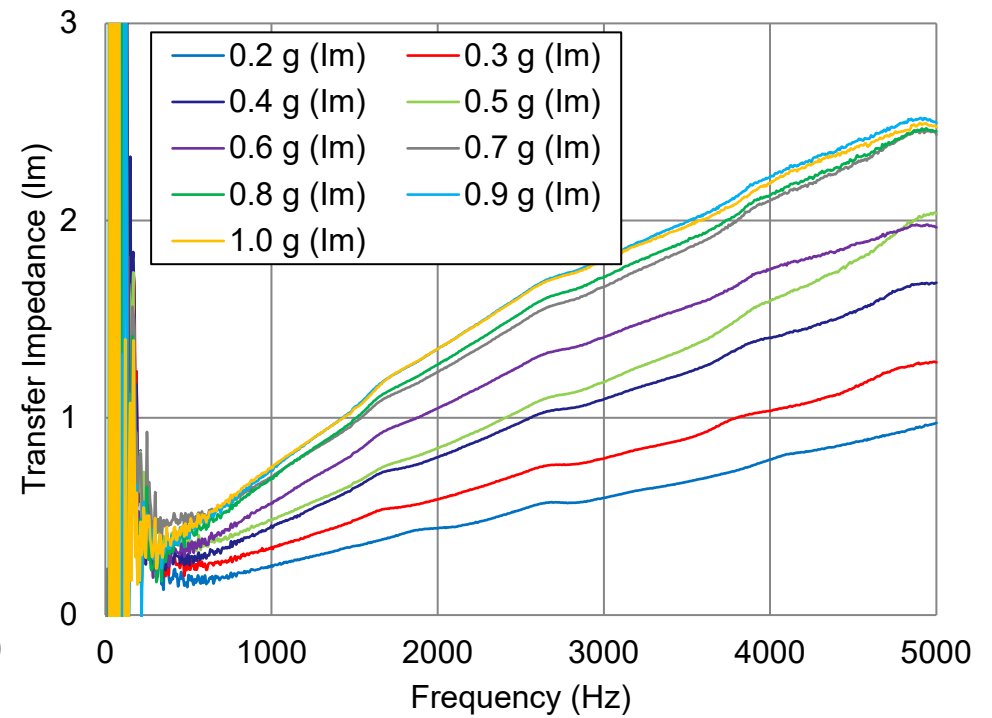
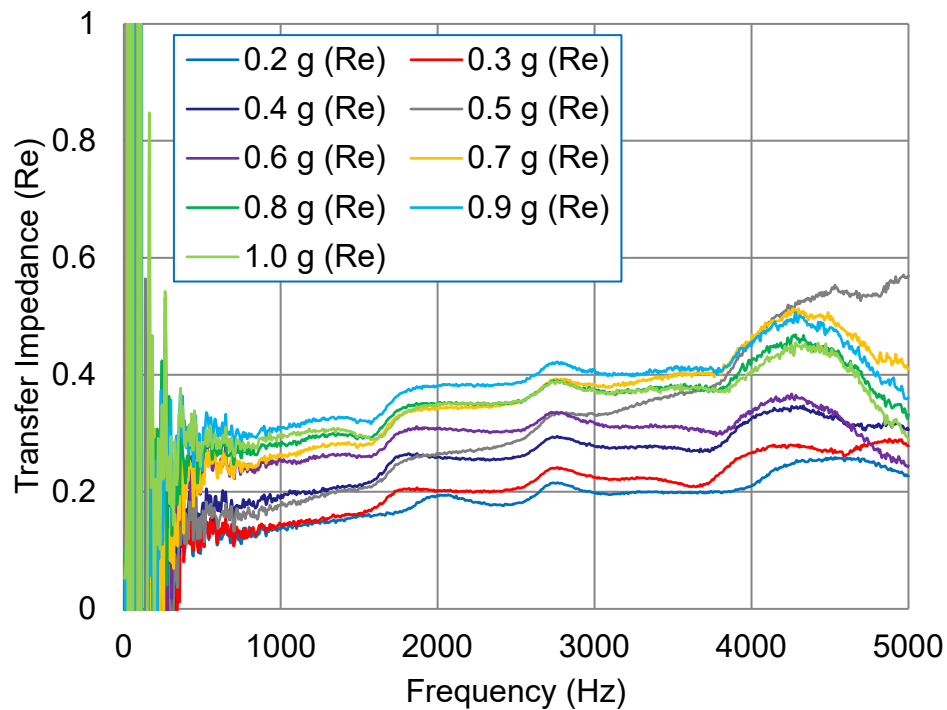
Measurement of adhesive layer or bonded cover



$$Z_{tr} = Z_1 - Z_2$$

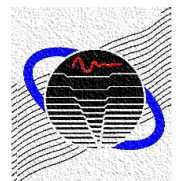
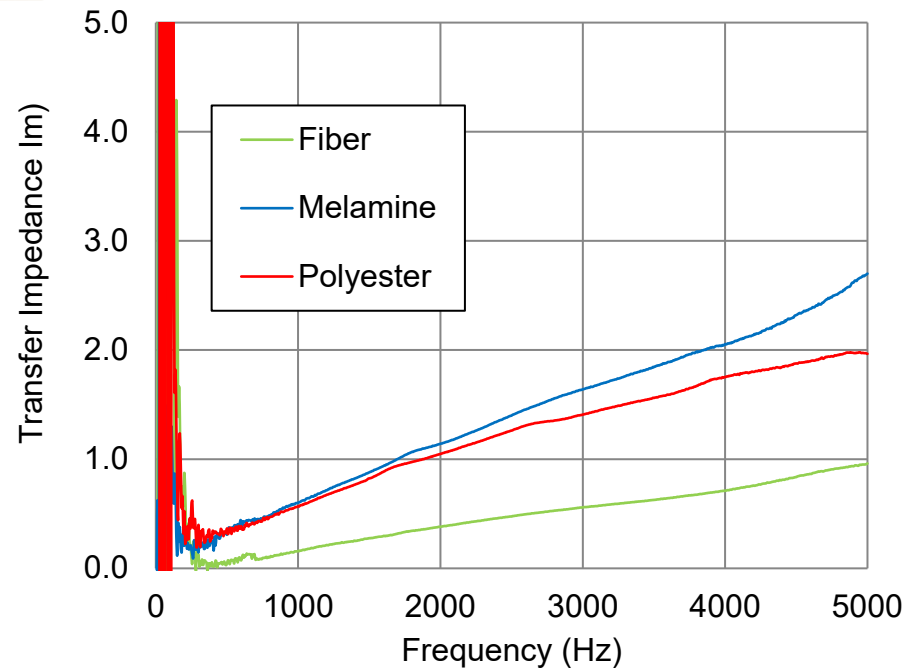
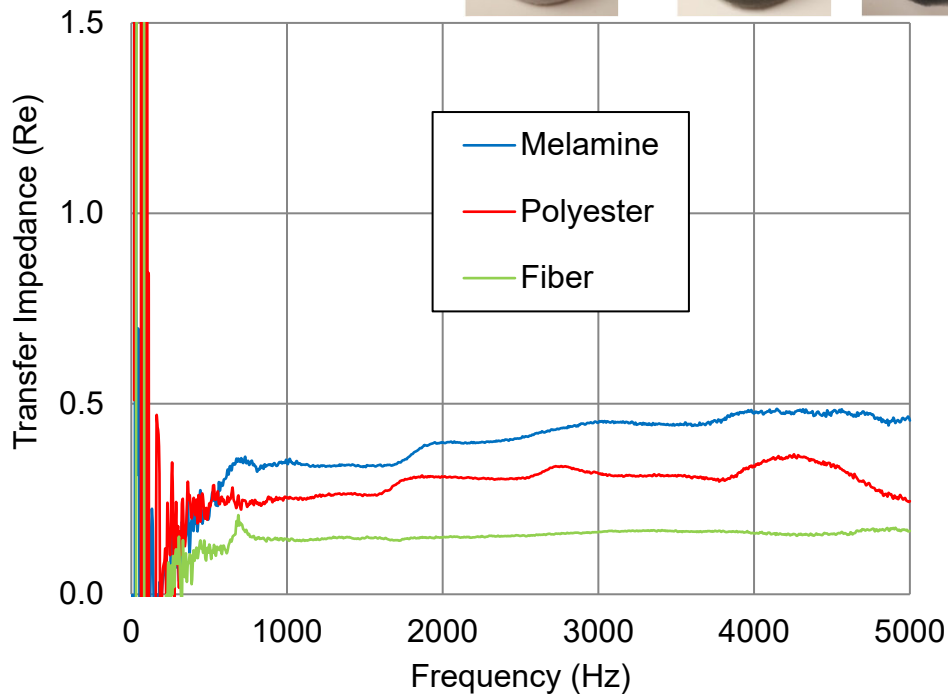
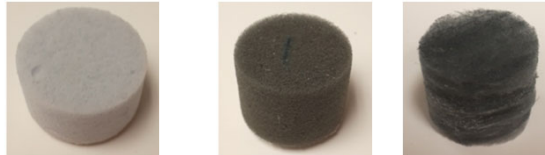


# Transfer Impedance **Glue**

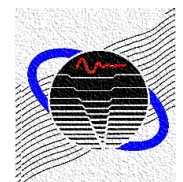
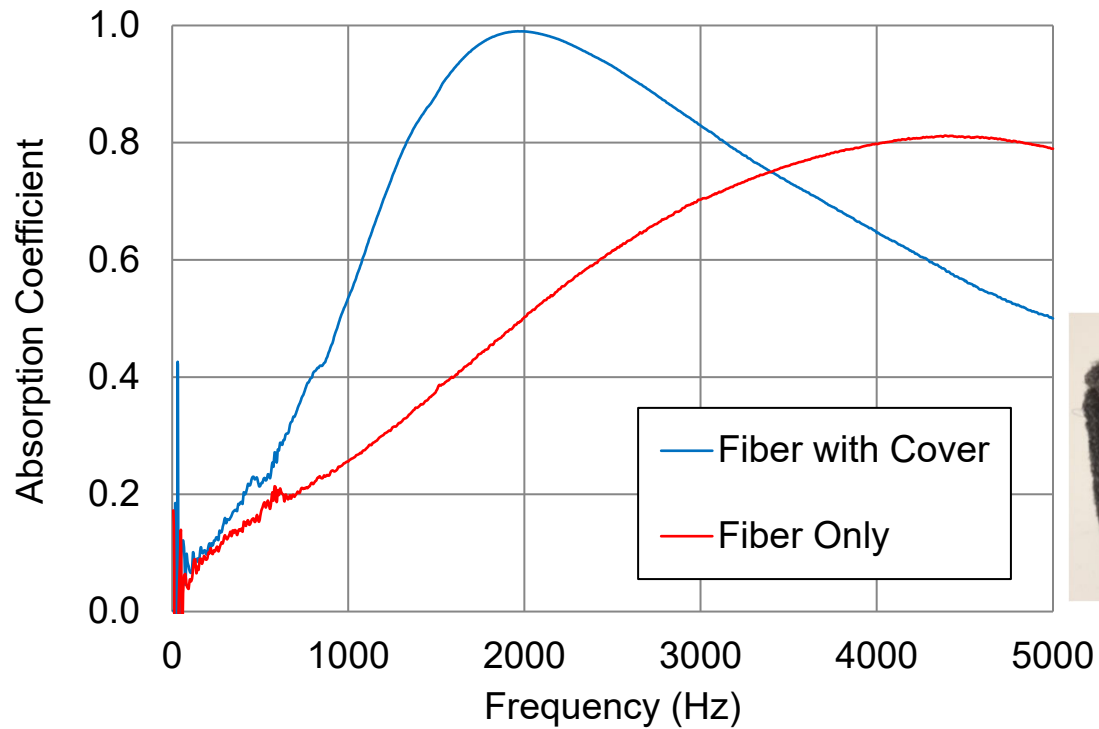




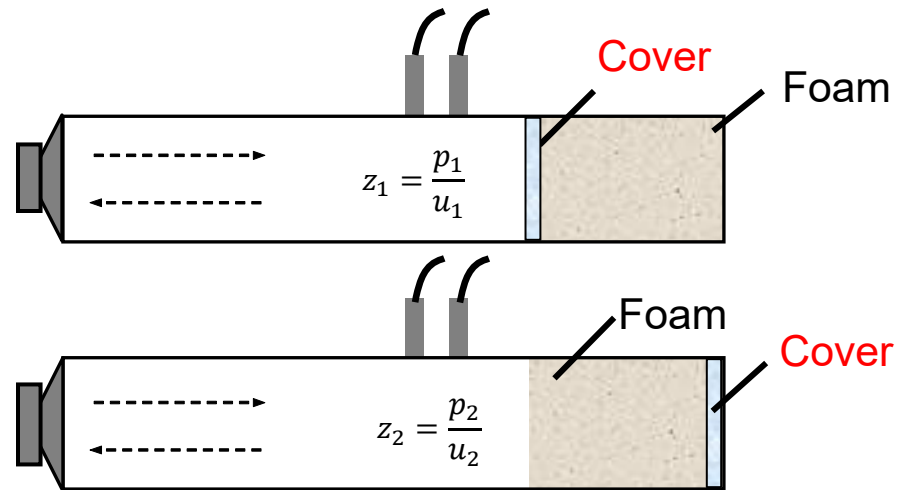
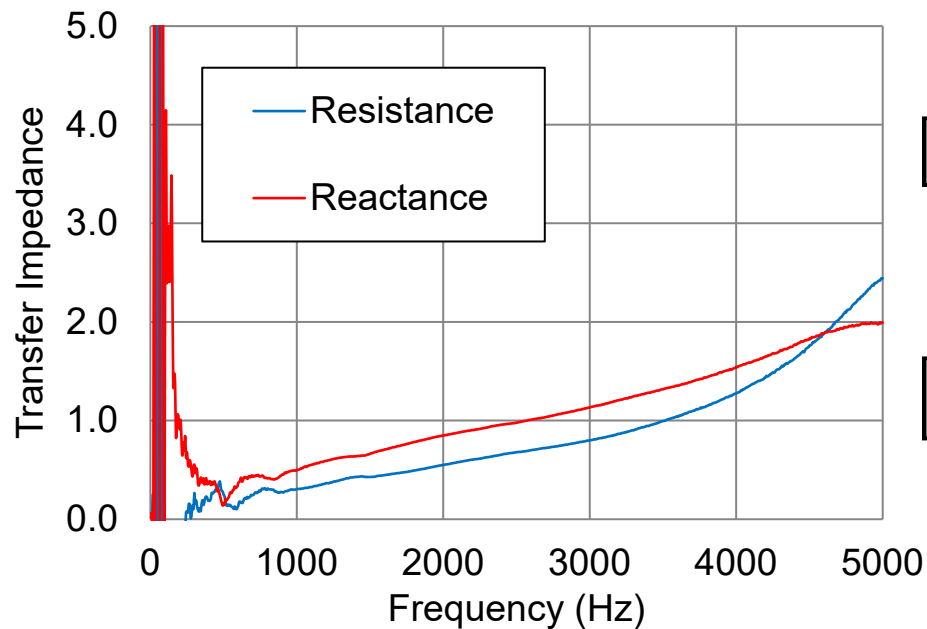
# 0.6 g of Glue Effect of Different Substrate



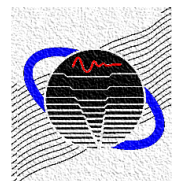
# Effect of Covers



# Measurement Transfer Impedance

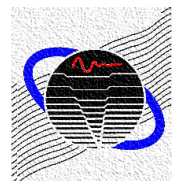
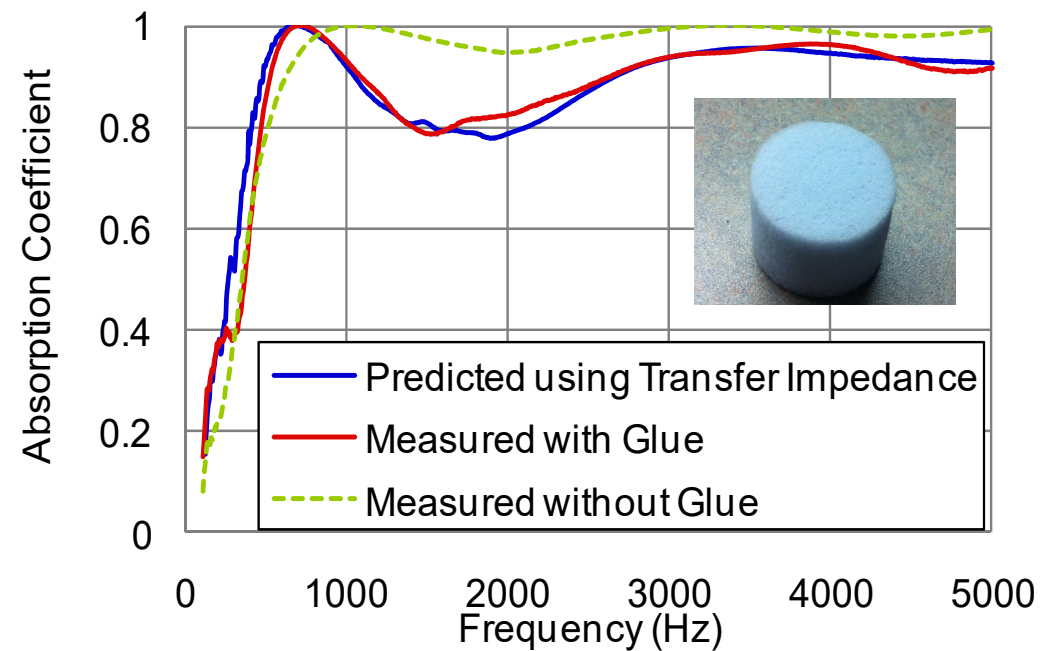
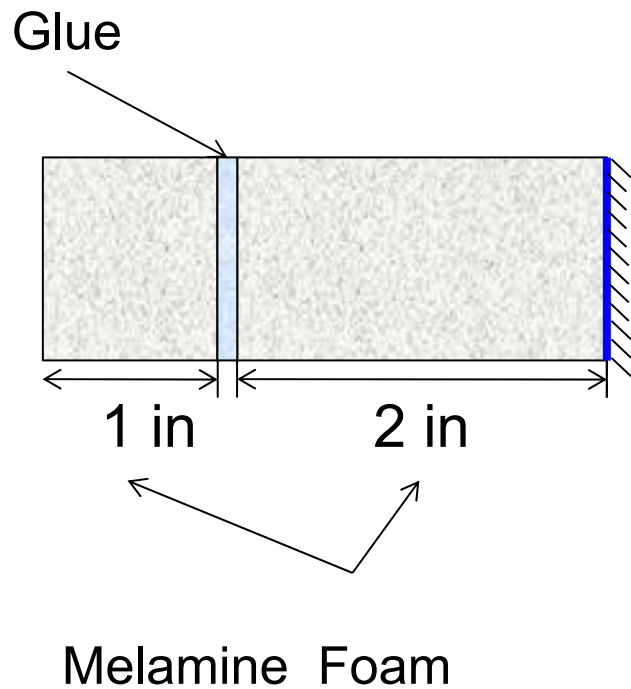


$$Z_{tr} = Z_1 - Z_2$$

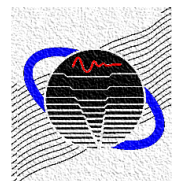
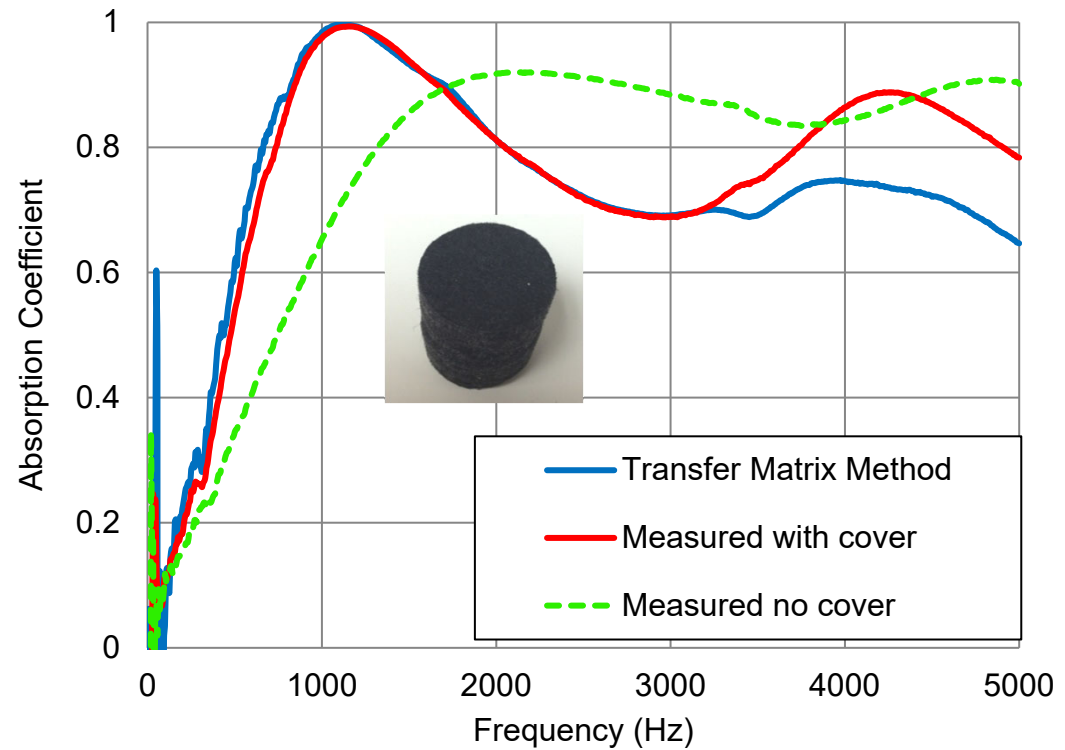
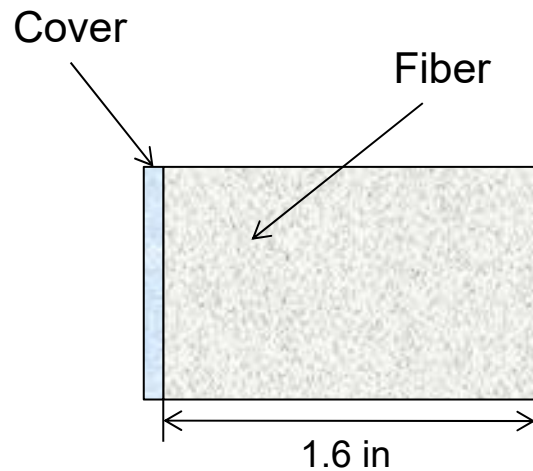


# Test Case 1

## Two Layers of Foam Bonded with Adhesive

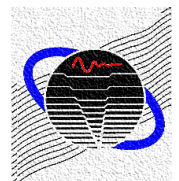
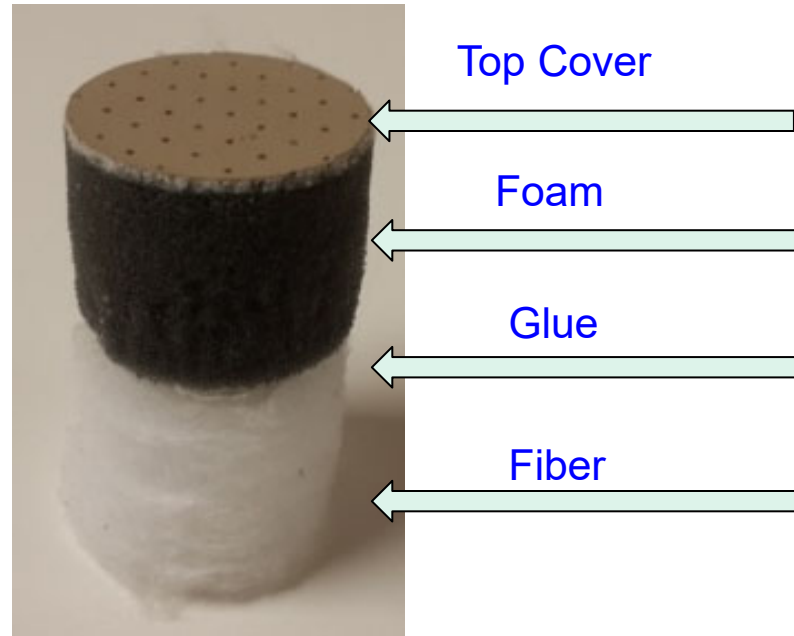
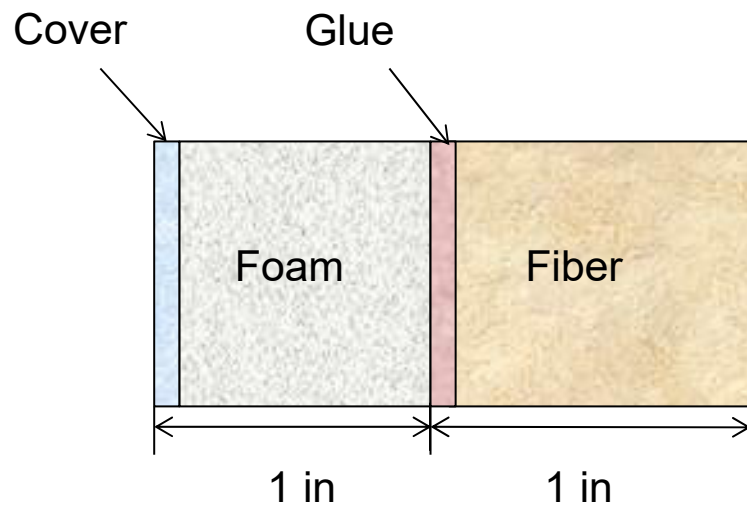


# Test Case 2



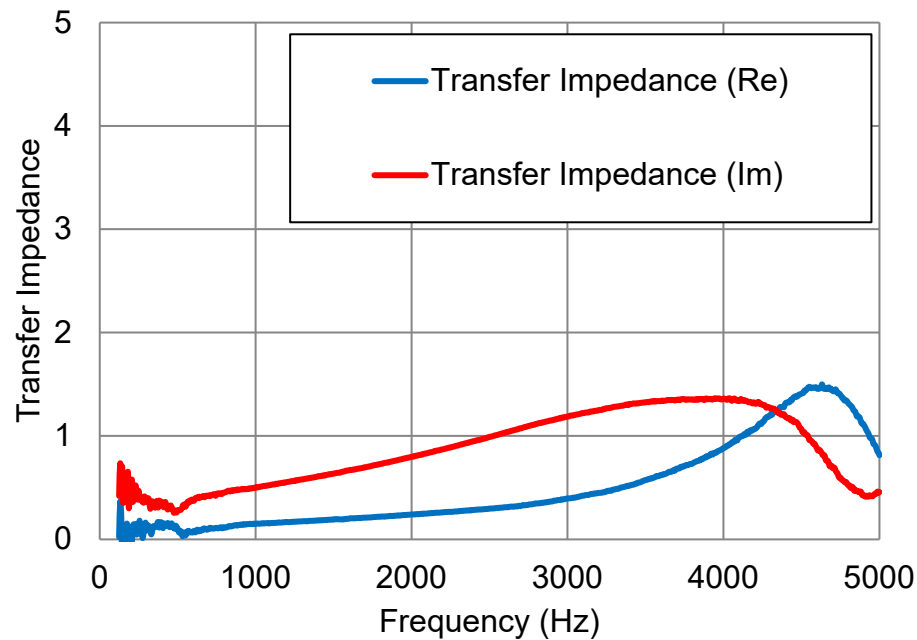
# Test Case 3

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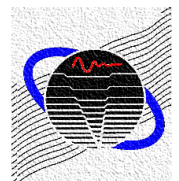
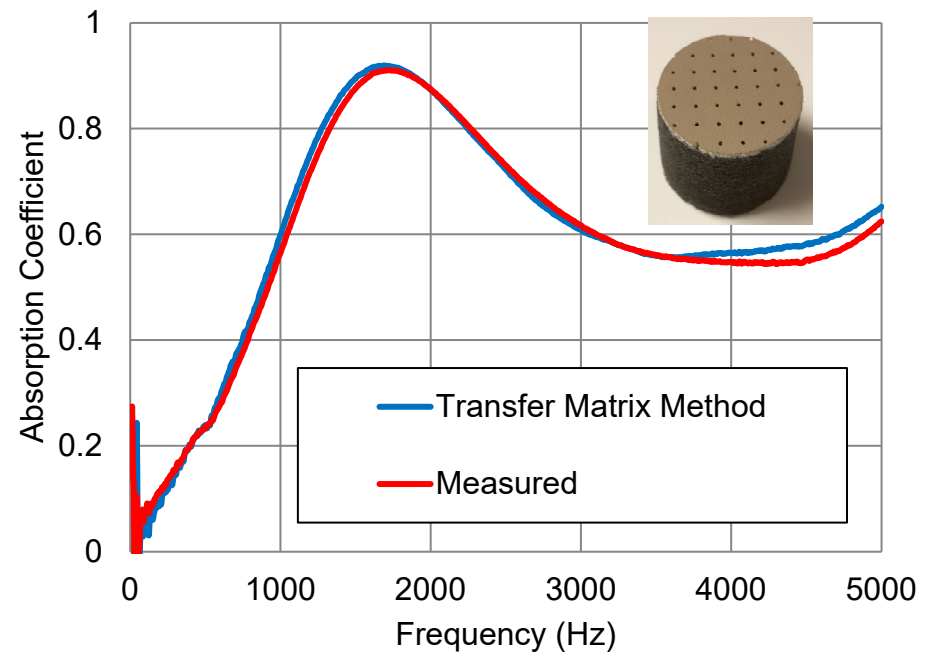


# Test Case 3

## Top Cover Transfer Impedance

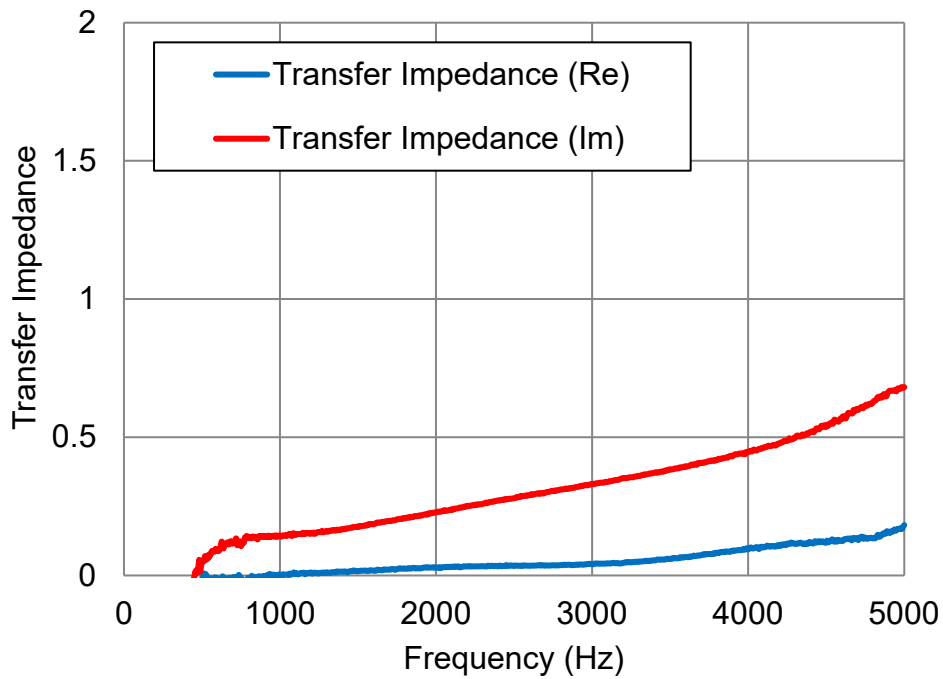


## Top Cover + Foam Layer

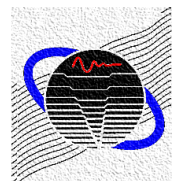
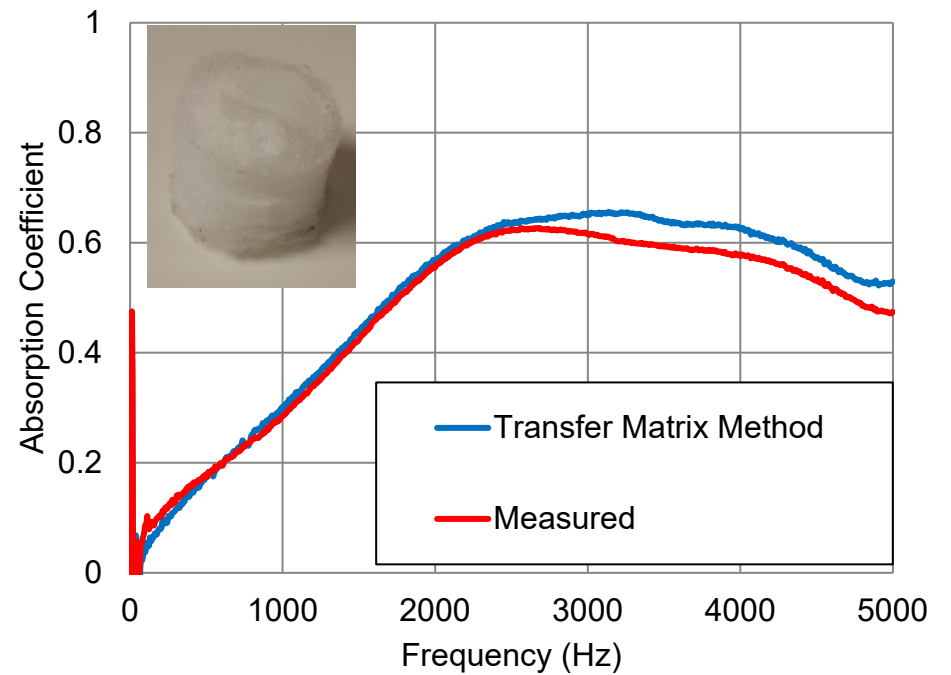


# Test Case 3

## Glue Transfer Impedance

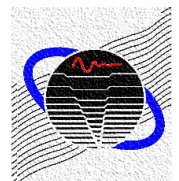
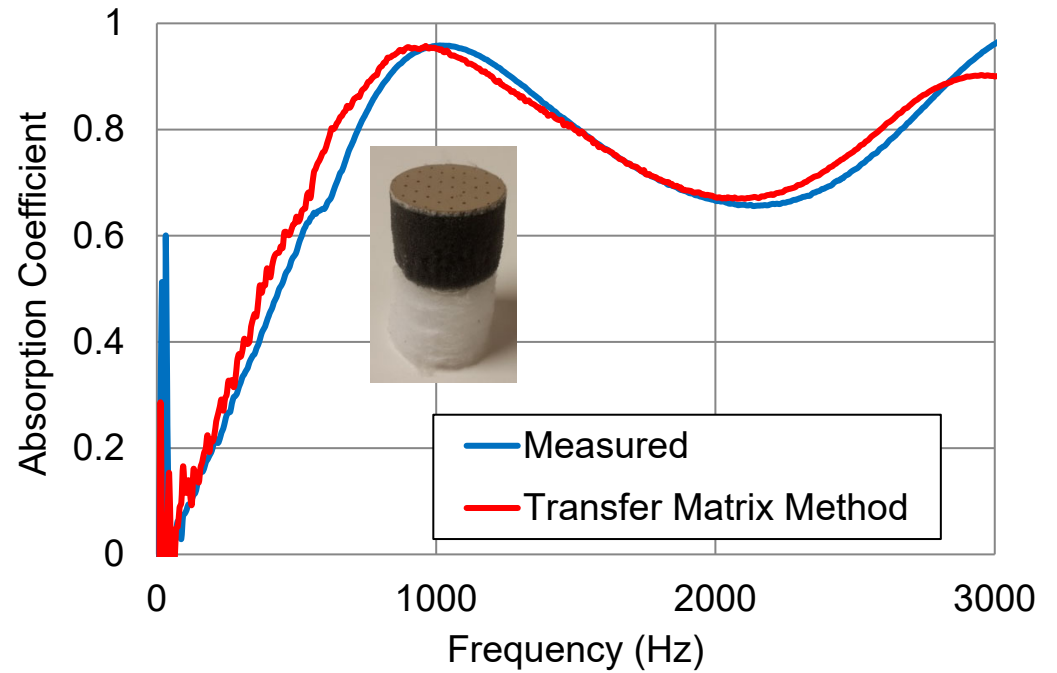
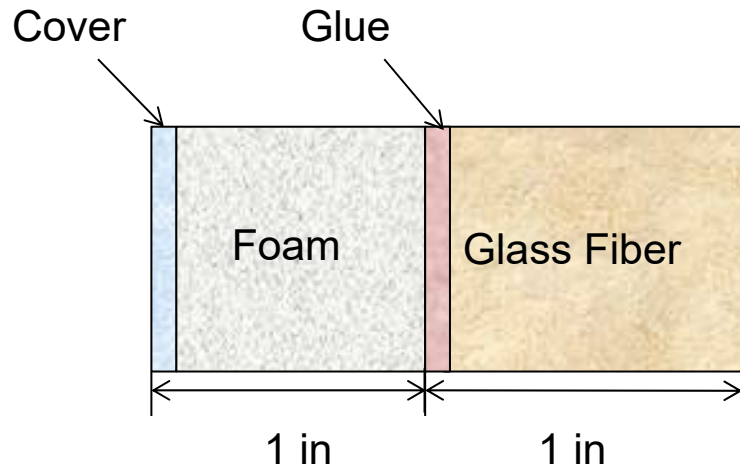


## Glue + Fiber Layer





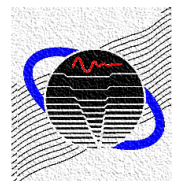
# Test Case 3



# Summary

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- Transfer impedance is determined using an impedance difference approach.
- Demonstrated capability to predict properties of layered absorbers.



# Transmission Loss Standards

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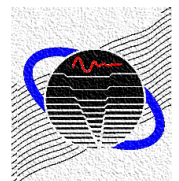
ASTM E2611-09, “Standard Test Method for Measurement of Normal Incidence Sound Transmission of Acoustical Materials Based on the Transfer Matrix Method,” American Society of Testing and Materials, Philadelphia, 2009.

This standard

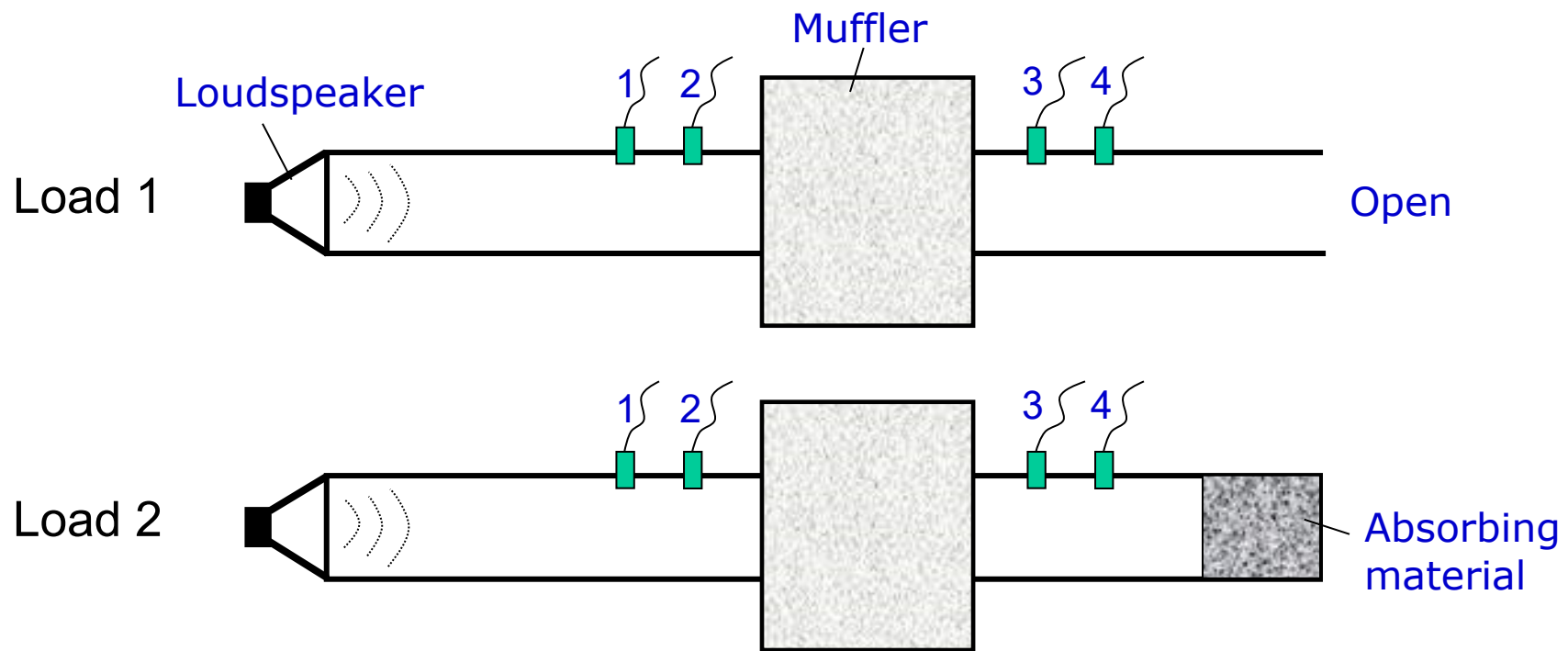
- Is designed to measure TL of ‘soft’ barrier materials
- Is not specifically aimed at mufflers
- Only mentions the two-load method\*

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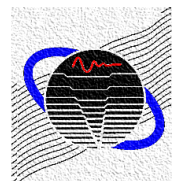
\* Also one-load method for uniform materials



# Two-Load Method

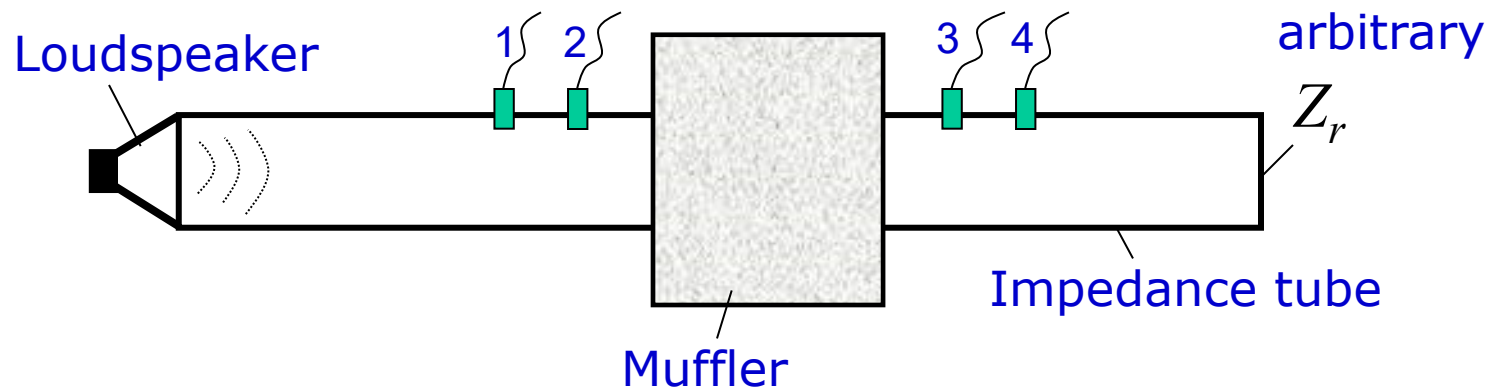


To and Doige, 1979

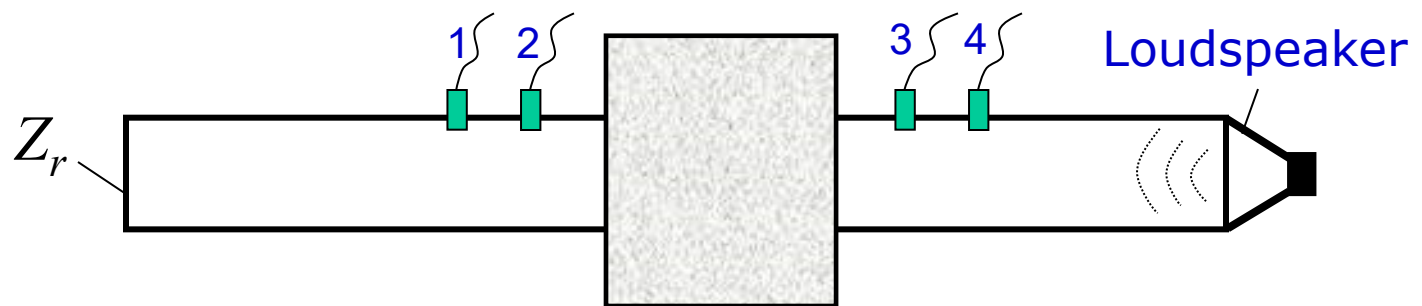


# Two-Source Method

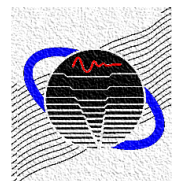
Configuration A



Configuration B

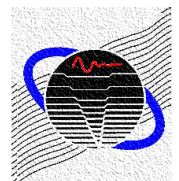
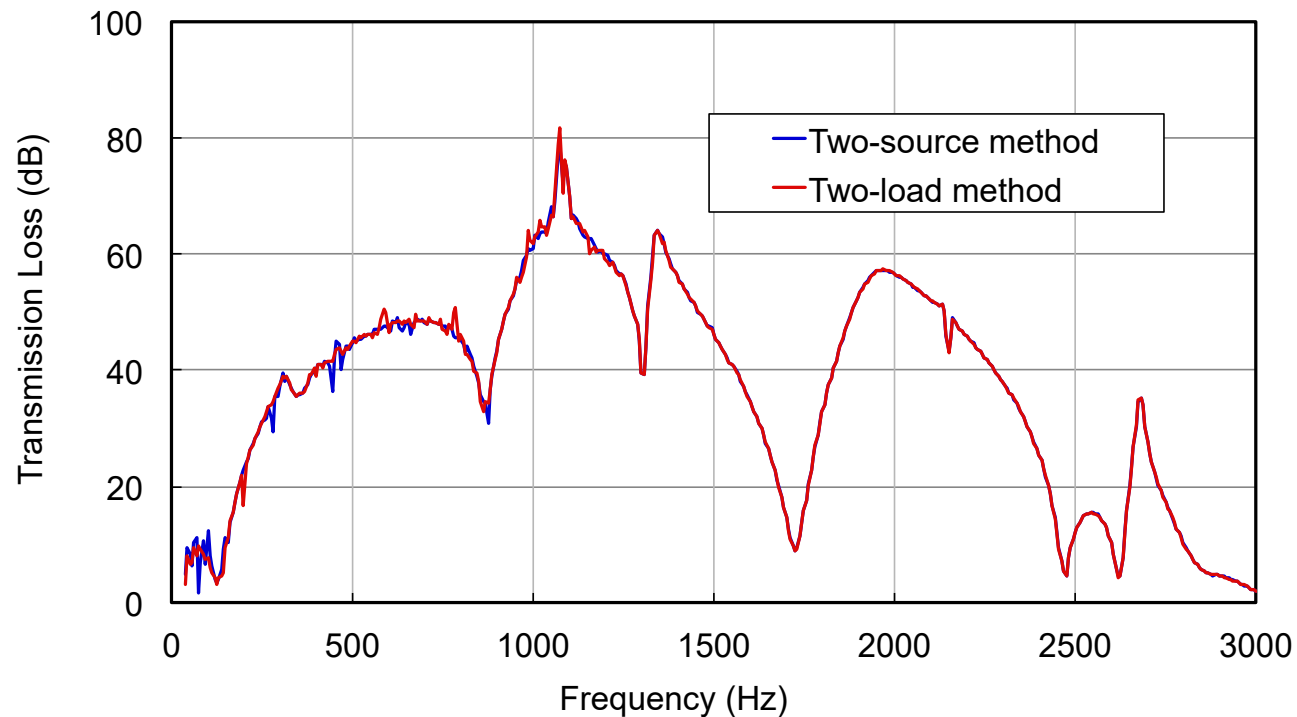


Munjal and Doige, 1990



# Transmission Loss Calculation

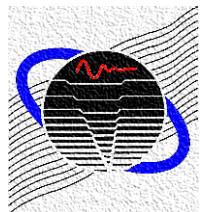
$$TL = 20 \log_{10} \left( \frac{1}{2} \left| T_{11} + \frac{T_{12}}{\rho c} + \rho c T_{21} + T_{22} \right| \right)$$



# Future Directions

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- Develop a better method (or refine the approach) for determining the transmission loss of hard samples in an impedance tube.
- Explore best practices for cutting samples using the new water jet cutter at the University of Kentucky.



# References

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- ASTM E1050-19, “Standard Test Method for Impedance and Absorption of Acoustical Materials Using a Tube, Two Microphones, and a Digital Frequency Analysis System”, Philadelphia, 2019.
- ASTM C522-03, “Standard Test Method for Air Flow Resistance of Acoustical Materials,” Philadelphia, 2003.
- ASTM E2611-19, “Standard Test Method for Measurement of Normal Incidence Sound Transmission of Acoustical Materials Based on the Transfer Matrix Method,” Philadelphia, 2019.
- T. J. Cox and P. D’Antonio, Acoustic Absorbers and Diffusers: Theory, Design and Application, 3rd Edition, CRC Press, Boca Raton, FL (2017).
- W. L. Li, X. Hua, and D. W. Herrin, “A Survey of Methods for Determining the Bulk Properties of Sound Absorbing Materials,” Noise-Con 2014, Fort Lauderdale, FL, September 8-10 (2014).
- W. L. Li, D. W. Herrin, and J. Haylett, “Measurement of the Transfer Impedance of Covers and Adhesives with Application to Multi-Layer Design,” Noise-Con 2014, Fort Lauderdale, FL, September 8-10 (2014).
- Y. Salissou and R. Panneton, “Wideband Characterization of the Complex Wave Number and Characteristic Impedance of Sound Absorbers,” Journal of the Acoustical Society of America, 128(5), 2868-2876 (2010).
- T. W. Wu, C. Y. R. Cheng and Z. Tao, “Boundary Element Analysis of Packed Silencers with Protective Cloth and Embedded Thin Surfaces,” Journal of Sound and Vibration, 261, 1-15 (2003).

