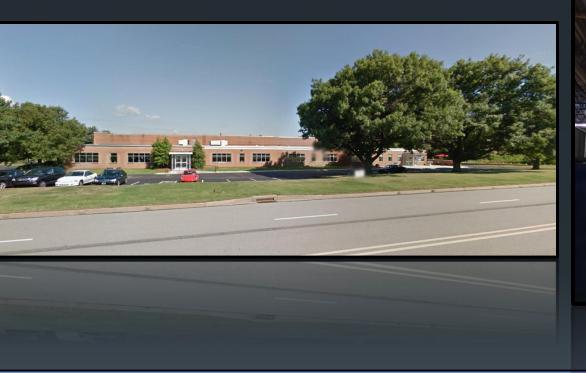


https://www.google.com/maps/place/150+Gamma+Dr/@40.4997521,-79.8681269,18z/data=!4m2!3m1!1s0x8834ecdbc348bd5f:0xd10a14f6b42509

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Introduction

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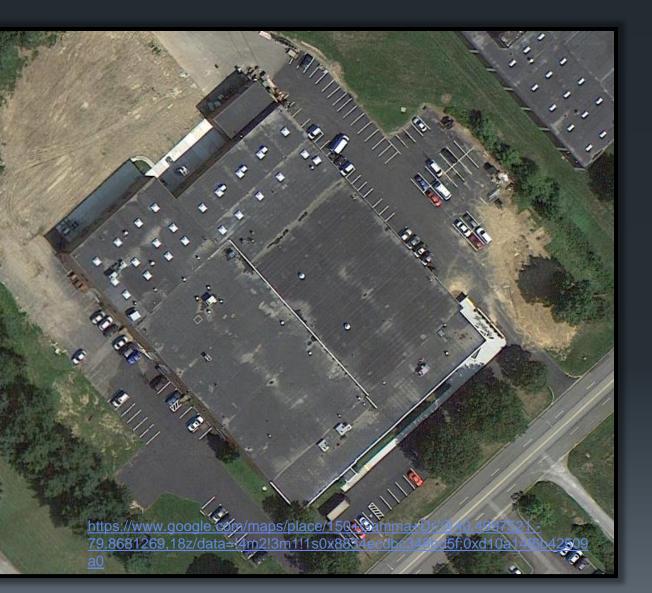
Acoustical Design

Conclusion

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Background



Location: Pittsburgh, Pennsylvania

Building Use:

- Warehouse Storage • Laboratory Space • Offices

Footprint: 74,900 square feet

Ceiling Height:

- Offices: 9' • Warehouse: 22' 6"

Renovated: 2012

Pittsburgh, <u>Pennsylv</u>ania

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OA Wet Bul

Design Conditions

Outdoor Design Conditions

Indoor Design Conditions

	Summer Design Cooling	Winter Design Heating
lb (°F)	89 °F	2.0 °F
lb (°F)	72 °F	.3 °F

	Offices & Lab	Warehouse & Packaging	Storage & Maintenance
Cooling Set Point	70 °F	85 °F	95 °F
Heating Set Point	55 °F	55 °F	60 °F
Relative Humidity	45%	-	-

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Existing Mechanical System

Office/Lab Space

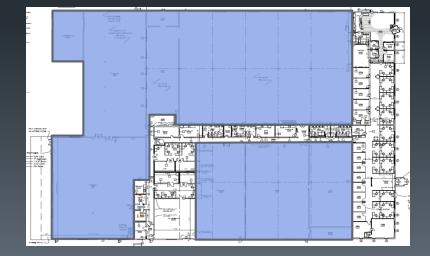
- 6 single zone CAV rooftop units (RTUs)
- CO2 preheat conditioning option available
- CO2 radiant floor cooling and heating



Warehouse and Storage

- Primarily electric resistance heat
- 8 air handling units (AHUs)
- Makeup air handling unit





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Annual Load Simulation

Monthly Energy Consumption

January March April May June July August emper october

Energy Consumption By Use

Lighting 46%

HVAC 18%

Electrical Equipment 36%

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Annual Energy Consumption By Use

<u>HVAC:</u> 595,045 kWh

Lighting: 2,657,011 kWh

Electrical Equipment: 3,252,057 kWh

Total Annual Energy Consumption:

Annual Load Simulation

6,5041,143 kWh

Lighting 46%

Energy Consumption By Use

HVAC 18%

Electrical Equipment 36%

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HVAC: \$0.96/ft²

Lighting: \$2.04/ft²

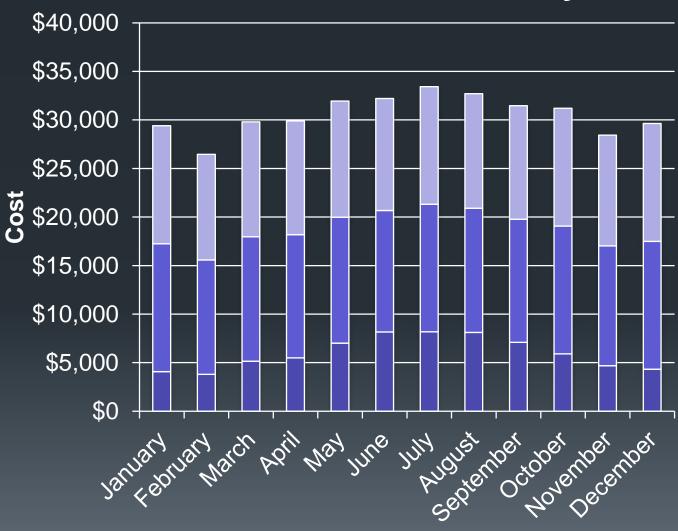
Electrical Equipment: \$1.89/ft²

Energy Cost and Consumption

Electrical Cost by Use

Annual Cost per Unit Floor Area

Total Annual Energy Cost: \$366,744



Electrical Equipment Lighting ■ HVAC

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- discharge gas)
- coil unit
- Indoor fan coil units

Variable Refrigerant Flow System

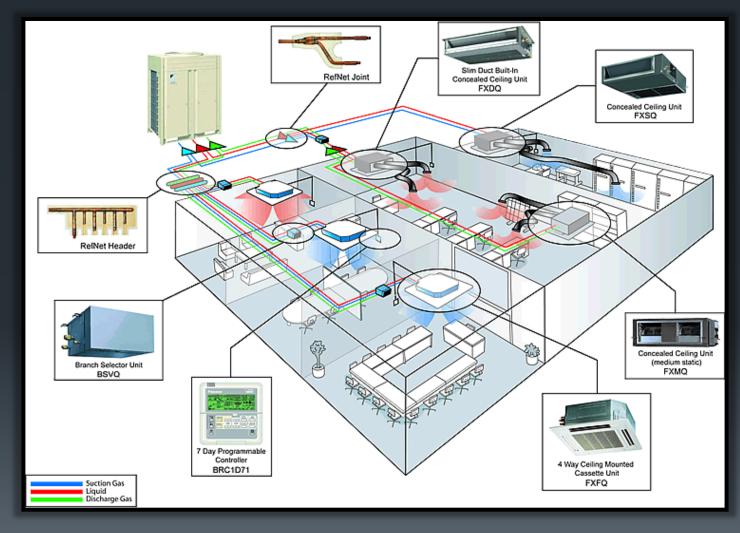
System Components

Outdoor/Indoor Condensing Unit

3 pipe system (suction, liquid,

Mode Change Unit (MCU)

2 pipe system from MCU to fan



Advantages

- Lower energy costs
- Lower CO2e emissions
- Simultaneous heating and cooling
- Precise and occupant control capabilities

Disadvantages

- Expensive first costs
- Requires DOAS system

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System Layout Restrictions

- Piping Length 656 feet maximum
- External Static Pressure 1" maximum
- Condensing Unit Elevation 360 feet
 maximum

VRF System Layout





VRF System 1

VRF System 2

VRF System 3

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Condensing Unit Quantity

Samsung DVM S Series Condensing

Unit Requirements

g ed	One	Two	Three
	6-12	14-24	26-36

VRF System	1	2	3
Size (tons)	11.5	17.2	12.5
Number of Condensing Units Needed	One	Two	Two

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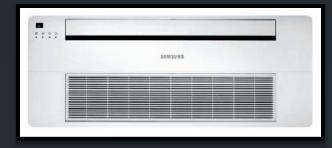
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1-Way Cassette Terminal Unit

- Located at edges of room ceiling •
- Diffuses in one direction •
- **4-Way Cassette Terminal Unit**
- Located in central area of ceiling •
- Distributes air in four directions •
- **High Static Pressure Duct Unit**
- Up to .99 in wg. external static pressure •
- Can condition multiple spaces at a time •

Indoor Fan Coil Unit Selection



1 Way Cassette Terminal Unit

Source: Samsung DVM S Series Catalog



Source: Samsung DVM S Series Catalog



4 Way Cassette Terminal Unit

Source: Samsung DVM S Series Catalog

High Static Pressure (HSP) Duct Unit

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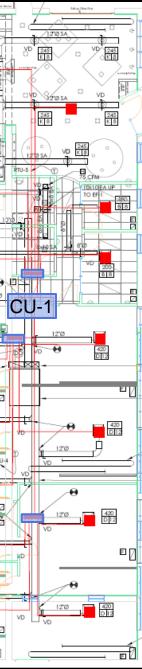
VRF System Selection

VRF System 1

12 ton Outdoor Condensing Unit (1) Mode Change Units (3) 4 Way Cassette FCU (6) 1 Way Cassette FCU (9)









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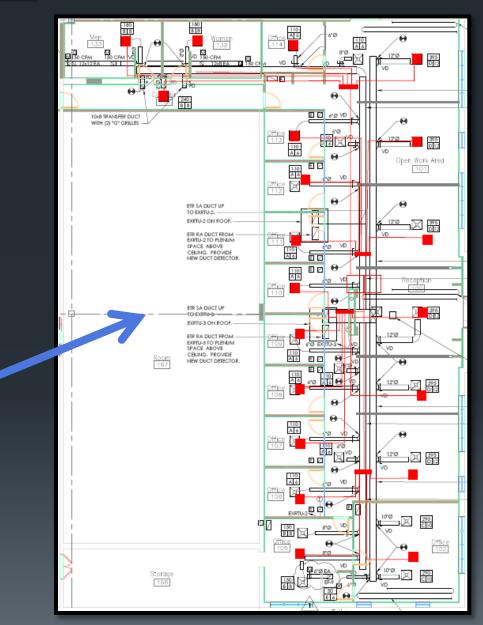
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VRF System Selection

VRF System 2

6 ton Outdoor Condensing Unit (1) 10 ton Outdoor Condensing Unit (1) Mode Change Units (4) 4 Way Cassette FCU (12) 1 Way Cassette FCU (7)





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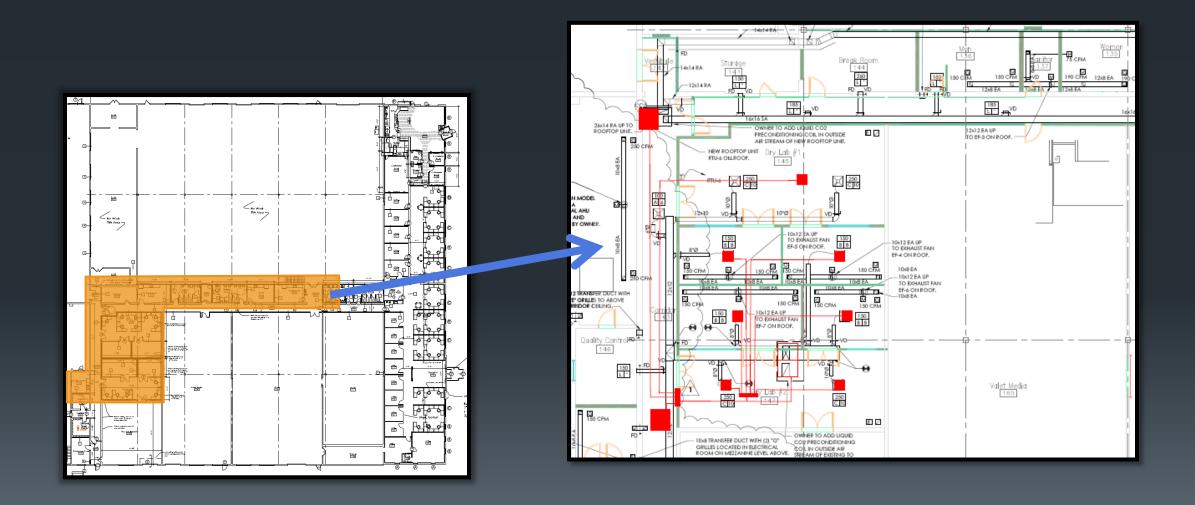
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VRF System Selection

VRF System 3

6 ton Outdoor Condensing Unit (1) 10 ton Outdoor Condensing Unit (1) Mode Change Units (3) 4 Way Cassette FCU (3) 1 Way Cassette FCU (4) High Static Pressure Unit (2)



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Semco FV-2000 Fresh Air Preconditioner

- 3 Angstrom Total Energy Wheel
- Parallel system to VRF system
- Manages humidification and de-humidification



Dedicated Outside Air System

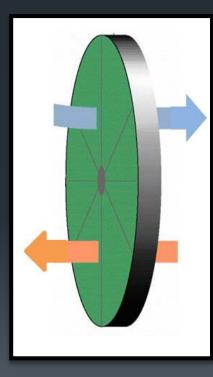
Airflow (cfm)	Ext. Static Pressure (in wg.)	Motor Brake Horsepower	Fan Speed (RPM)
1,250	1.50	1.00	1723
1,175	.75	.64	1413

Outdoor Airstream

Dry Bulb (°F) :	95.0
Wet Bulb (°F):	75.0
Enthalpy (BTU/lb):	38.4

Exhaust Airstream

Dry Bulb (°F) : 90.8 Wet Bulb (°F): 72.5 Enthalpy (BTU/lb): 36.1



Source: Semco FV 2000 Catalog

Design Conditions: Cooling Season

Supply Airstream

Dry Bulb (°F) :	80.2
Wet Bulb (°F):	65.7
Enthalpy (BTU/lb):	30.5
Airflow (CFM):	1,250

Return Airstream

Dry Bulb (°F) :	75.0
Wet Bulb (°F):	62.0
Enthalpy (BTU/lb):	27.8
Airflow (CFM):	1,175

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Dedicated Outside Air System

Efficiency and Static Pressure Loss

Design Conditions: Cooling Season

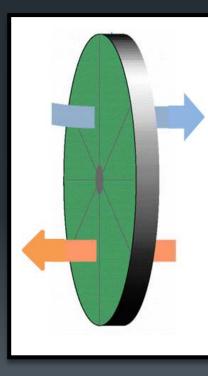
- Sensible Supply Efficiency: 74%
- Latent Supply Efficiency: 74%
- Supply Air Pressure Loss: .36
- Exhaust Air Pressure Loss: .33

Outdoor Airstream

Dry Bulb (°F) :	95.0
Wet Bulb (°F):	75.0
Enthalpy (BTU/lb):	38.4

Exhaust Airstream

Dry Bulb (°F) : 90.8 Wet Bulb (°F): 72.5 Enthalpy (BTU/lb): 36.1



Supply Airstream

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Airflow (CFM):	1,250

Return Airstream

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Enthalpy (BTU/lb):	27.8
Airflow (CFM):	1,175

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- Weekly/Daily Scheduling
- Temperature limitations
- ERV operation mode and fan speed
- Zone Management

Controls

BACnet Gateway Building Management System

• Can connect up to 256 indoor units/16 outdoor units

Occupant Controls

- MWR-WE10N Wired Remote Controller
- Can control up to 16 indoor units and ERV
- Operation mode
- Temperature setting (limited by BACnet)
- ERV operation and fan speed (limited by BACnet)



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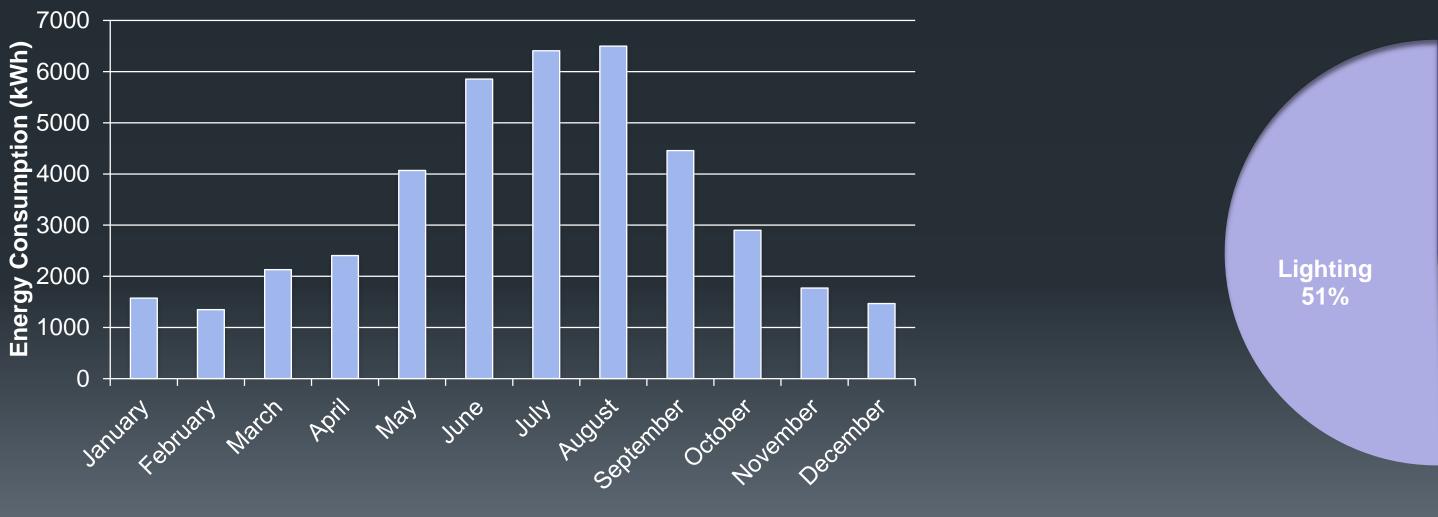
Emissions Analysis

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VRF System Monthly HVAC Energy Consumption

VRF Zone Annual Energy Consumption By Use

HVAC 9%

> **Electrical** Equipment 40%

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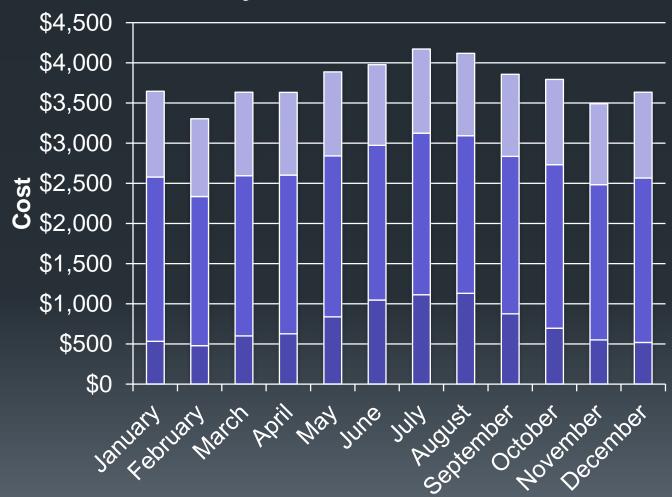
Cost Analysis

Annual HVAC Cost Per Unit Area:

\$0.71/ft²

Total Annual HVAC Energy Cost:

\$9,001



VRF System Zones Electrical Cost by Use

Electrical Equipment Lighting ■ HVAC

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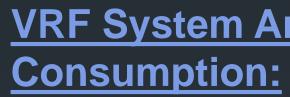
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Consumption:





Energy Consumption Comparison

Original Design Annual HVAC Energy

85,148 kWh

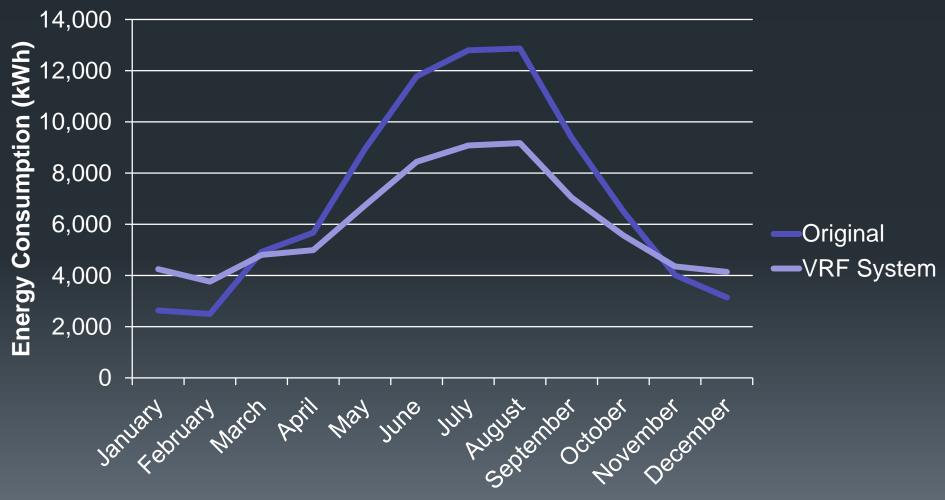
VRF System Annual HVAC Energy

72,316 kWh





Monthly HVAC Energy Consumption



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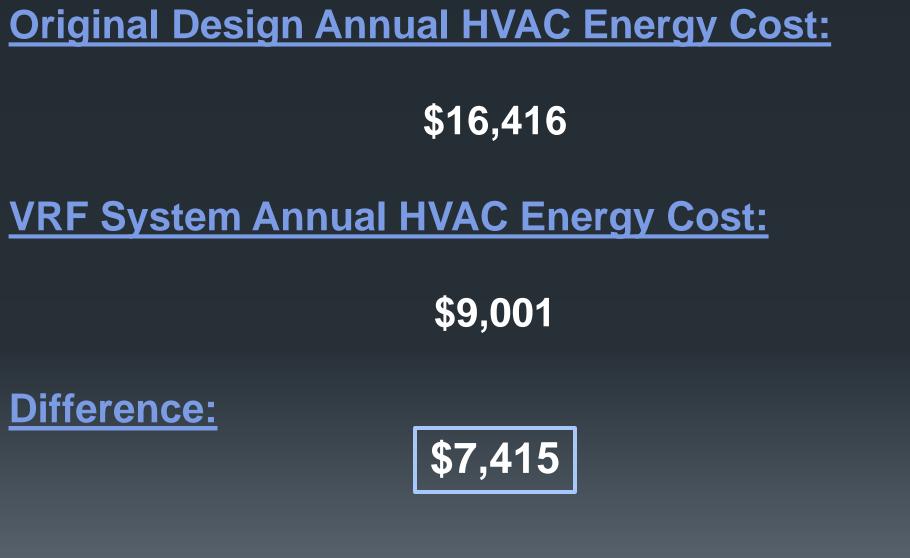
Conclusion

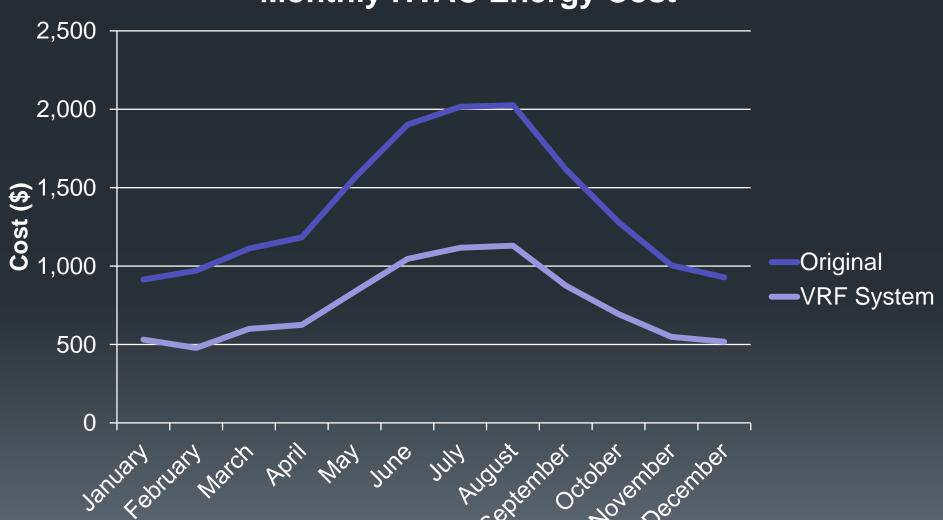
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Difference:

Energy Cost Comparison





Monthly HVAC Energy Cost

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Original HVAC System VRF System Difference

Mechanical First Costs Comparison

Original HVAC vs. VRF System First Costs

Materials	Labor	Total
\$144,663.88	\$79,733.00	\$222,396.88
\$327,702.26	\$20,690.81	\$348,393.07
-\$178,038.38	\$59,042.19	-\$118,996.19

Materials Difference:

Labor Difference:

Total First Cost Difference:

+118%

-74%



+51.8%

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Payback Period

Variables

- FC_{O} = First Cost of Original HVAC System
- AEC_O= Annual Energy Cost of Original HVAC System
- FC_{VRF}= First Cost of Original VRF System
- AEC_{VRF} = Annual Energy Cost of VRF HVAC System
- X= Payback Period in years







Equation

- $FC_{O} + (AEC_{O} * X) = FC_{VRF} + (AEC_{VRF} * X)$
- $229,396 + (16,416 \times X) = 348,393.07 + (9001 \times X)$
 - \$118996 = \$7<u>415 * X</u>

Payback Period

X= 16.04 years

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Emissions Comparison

Pollutant	Regional Grid Emission Factors 2007 (lb/kWh)	Calculated Emis	Emissions Reduction %	
		Original	VRF	
C02e	1.55	701,073	611,089	12.84%



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Open-Offic

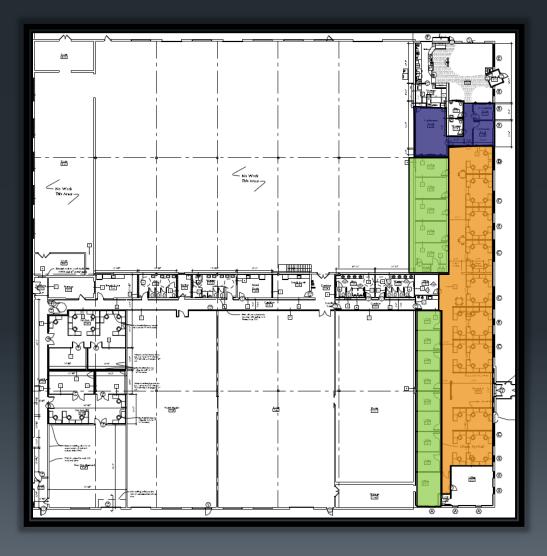
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Acoustical Breadth Investigation

Recommended Noise Criterion - NC

	Recommended NC Rating	Equivalent Sound Level dBA
Plan es	35-40	45-50
Offices	30-35	40-45
ence ns	25-30	35-40





Private Offices





Conference Rooms

Open Plan Offices

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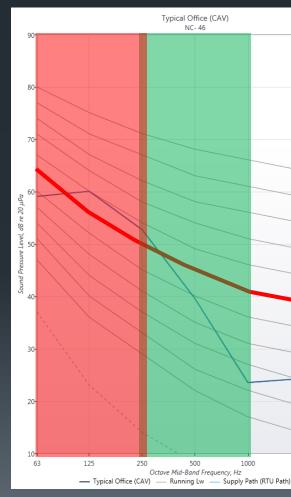
Acoustical Breadth Investigation

Existing Mechanical Conditions

- 5 Ton Carrier 50TCD06 Rooftop Unit
- 1" Fiberglass Insulation
- Room Dimensions : 34'x26'x8'
- Measure SPL to nearest diffuser

	Sound Power Level, dB (re 10^-12 W)						
Band cy, HZ	63	125	250	500	1000	2000	4000
irge	85.8	84.3	80.5	78.7	76.4	72.7	68.3

Existing Mechanical NC Rating





NC-46 ~ 48 dBA

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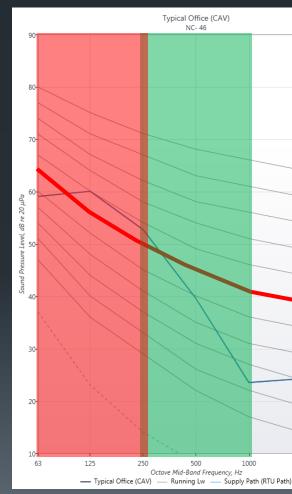
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Acoustical Breadth Investigation

Recommended Noise Criterion - NC

	Recommended NC Rating	Equivalent Sound Level dBA
Plan es	35-40	45-50
Offices	30-35	40-45
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Existing Mechanical NC Rating





NC-46 ~ 48 dBA

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Octave Band Frequency, HZ

Discharge

Acoustical Breadth Investigation

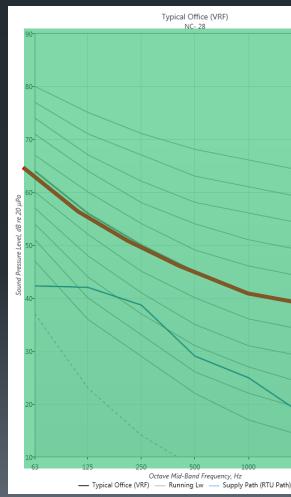
Proposed Mechanical Conditions

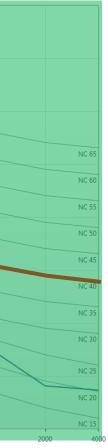
• Samsung DMV AM024FN4dCH/AA 4 Way Cassette Fan Coil Units

• Room Dimensions : 34'x26'x8'

Sound Power Level, dB (re 10^-12 W)						
63	125	250	500	1000	2000	4000
40.1	37.2	36.4	33.0	29.7	27.3	22.6

Recommended Noise Criterion - NC





NC-25 ~ 32 dBA

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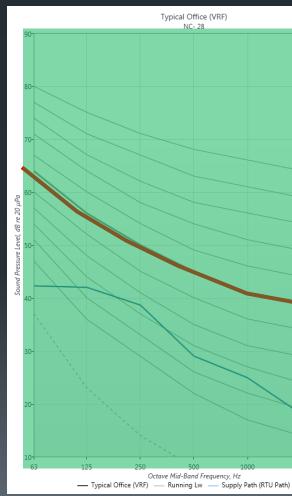
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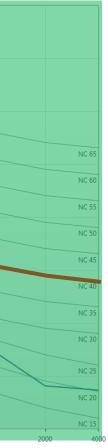
Acoustical Breadth Investigation

Recommended Noise Criterion - NC

	Recommended NC Rating	Equivalent Sound Level dBA
n-Plan fices	35-40	45-50
e Offices	30-35	40-45
erence oms	25-30	35-40

Recommended Noise Criterion - NC





NC-25 ~ 32 dBA

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- Emissions 13% less annually than original Improve occupant comfort

- First Costs 51% more expensive than original • Annual Costs – 45% cheaper than original system • Payback Period – 16 years (uneconomical)

Conclusion

Variable Refrigerant Flow System

- Ratings for office spaces
 - Potential for rumbly HVAC noise
 - and dBA values

Acoustical Invesitgation

Original Design – did <u>not</u> meet recommended NC

• VRF Design – met <u>all</u> recommended NC Rating

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<u>Acknowledgements</u>

AE Staff and Faculty Marc Portnoff Jonathan lams Joel Butler Dan Gardner My friends and family



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