

Integrally Geared Centrifugal Air Compressor Vibration Analysis Case Studies 15-Feb-2008

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How do we monitor these types of machines?

- What kind of components are we trying to monitor in these machines?
 - High speed gearing
 - High speed shafts / impellers
 - ➤ Low speed shafts
 - > Geared couplings
 - Disc or shim-pack couplings
 - Sleeve bearings
 - Multi-segment tilt pad bearings

This does not include the driver which could be a motor or turbine of which either one can add more complexities to the setup of a monitoring route

What parameters do we have available for monitoring the overall condition?

- Bearing temperatures
- Stage and drive gear vibration
- Stage inlet and outlet temperatures
- Stage inlet and outlet pressures
- Intercooler inlet and outlet temperatures
- Intercooler inlet and outlet pressures
- Motor amperage or kW

There are a lot of parameters that can be used including the above items and:

- > Performance testing
- Oil Analysis
- > And visual inspections

What do we use to monitor the vibration of the machine?

- > Is there a permanent probe of some kind on each stage?
- ➤ Is there a permanent monitoring / vibration protection system installed?
- Or are our only options temporary accelerometers?

Many permanent monitoring systems will have proximity probes on each stage, are they?

- A single probe on the stage?
- ➤ Dual probes at 90 degrees?
- > Any there permanent accelerometers?
- Can you 'plug-in' to obtain the raw signal in your portable data collector?

Vibration Analysis:

- > Proximity probe use
- Magnetically mounted accelerometers
- High Frequency accelerometers

To use the above probes we need to figure out the expected fault frequencies So that we can set up the appropriate frequency spans and resolutions

The following segments of the case studies will be on several Joy (Cameron Compression) MSG 4-stage machines

What are typical Fault Frequencies:

- Shaft Running Speeds
- ➤ Journal Bearing Issues
- Gear Mesh (and other gear fault frequencies)
- Vane Pass (diffuser vs. impeller)
- Coupling/Alignment

Motor

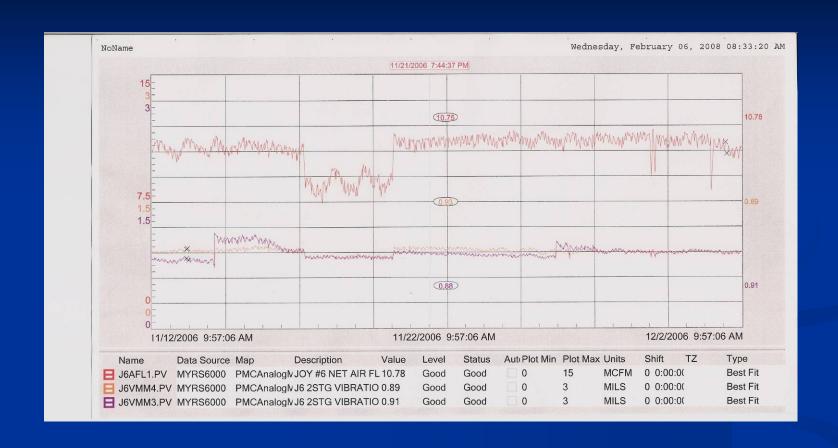
- ➤ Shaft Running Speeds
- > Journal Bearing Issues
- Electrical Frequencies

Example Fault Frequencies:

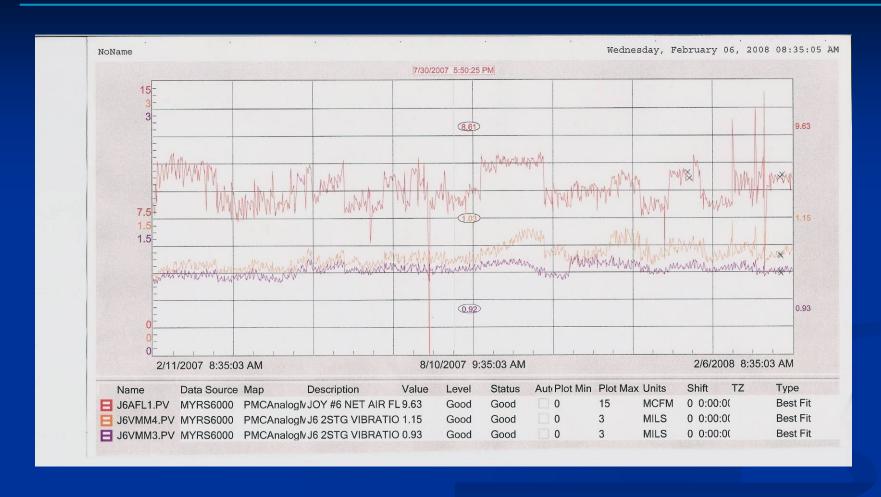
- ➤ Shaft Speeds:
 - ➤ Motor, Couplings, & Gear 1781 rpm
 - ➢ Pinion Speeds 14,385 & 21,372 rpm
- > Gears (420 teeth, 52 teeth, 35 teeth):
 - Gear Mesh 748,020 (12,467 Hz)
 - ➤ Assembly phase 187,000 & 21,372 cpm
 - ➤ Hunting Tooth 1198, 203 cpm
- ➤ Vane Pass 362,100 cpm (for bladed diffuser)

Displacement – Velocity – Acceleration What is an acceptable measurementfrequency range:

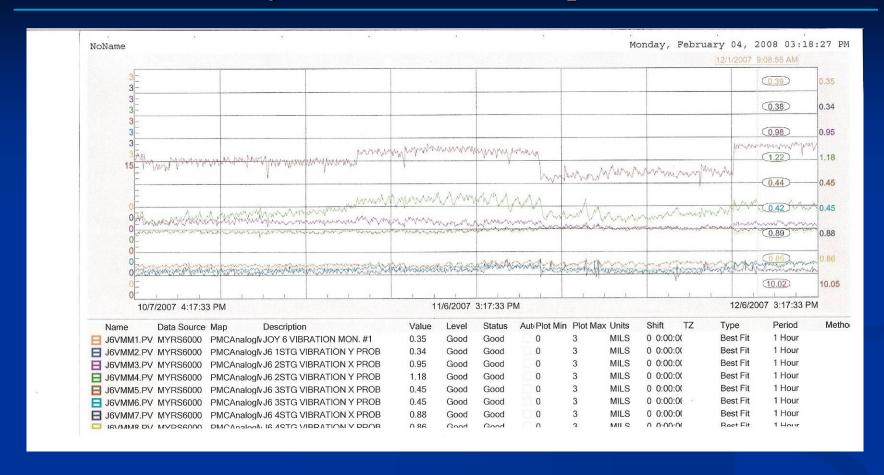
Displacement (mils)	3600 rpm	20,000 rpm	750,000 cpm
Velocity (in/s)	0.19	1.05	39.3
Acceleration (g)	0.18	5.68	7988
Displacement (mils) Velocity (in/s) Acceleration (g)	3600 rpm 27 5.1 5	20,000 rpm 0.88 0.921 5	750,000 cpm 0.0006 0.025 5



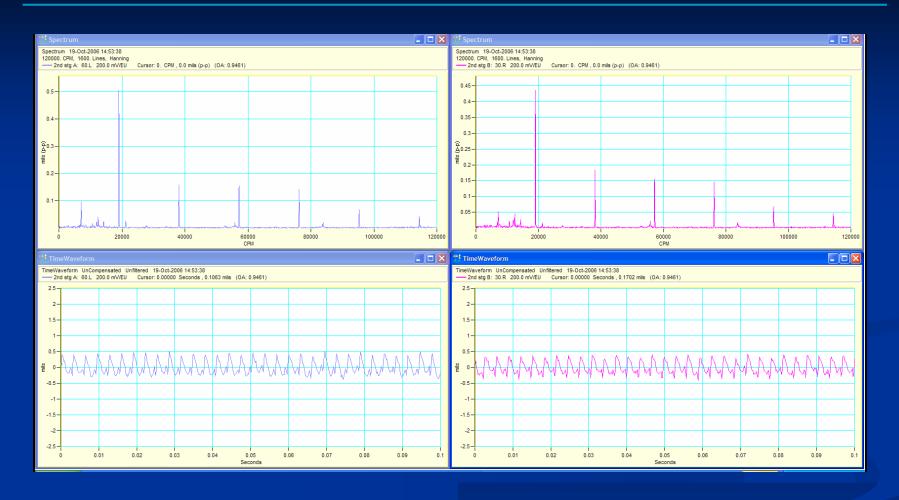
Note the step changes in the trend plot



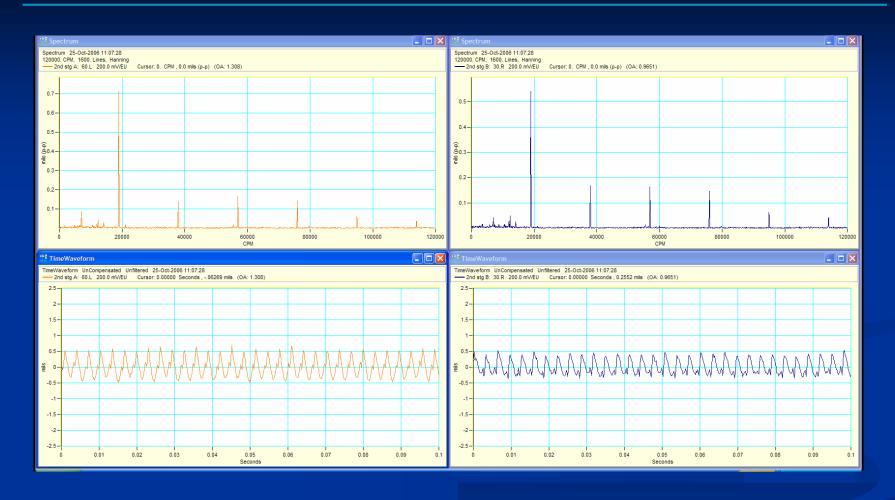
Note the variations in amplitude over time



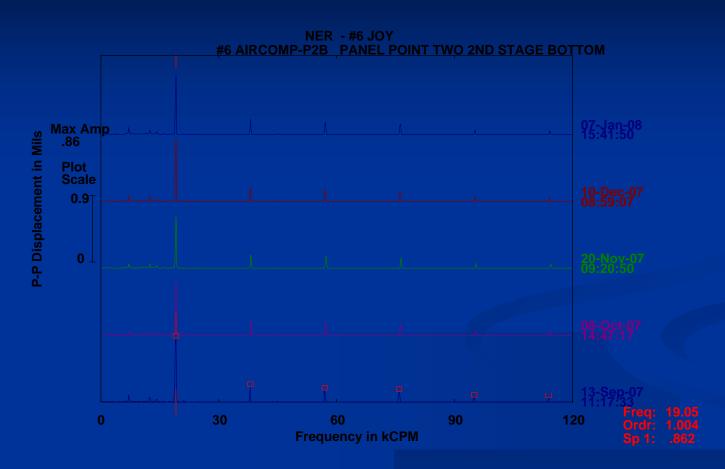
Here is a recent comparison to the other stages on the same machine



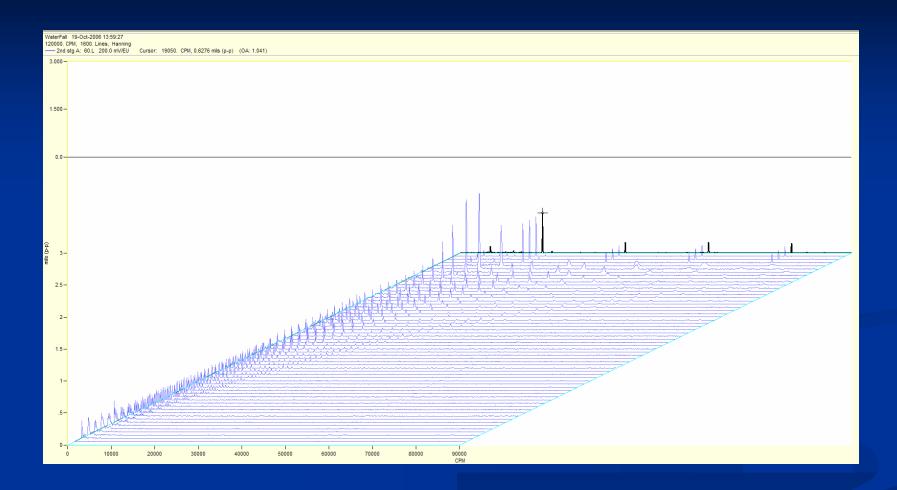
Just after the startup



This is only 6 days later



1X Amplitudes at different samples: 0.793; 0.851; 0.709; 0.713; 0.862





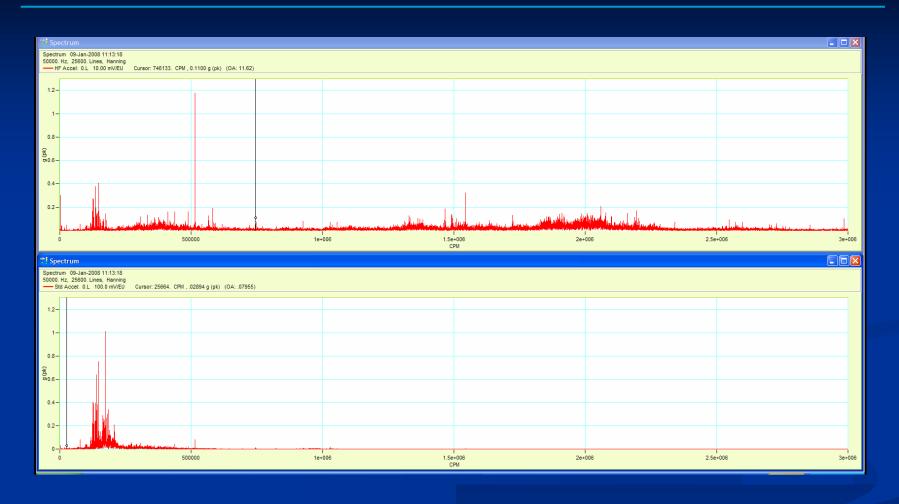
Note the difference in the noise floor of a 10 mV/g vs. 100 mV/g







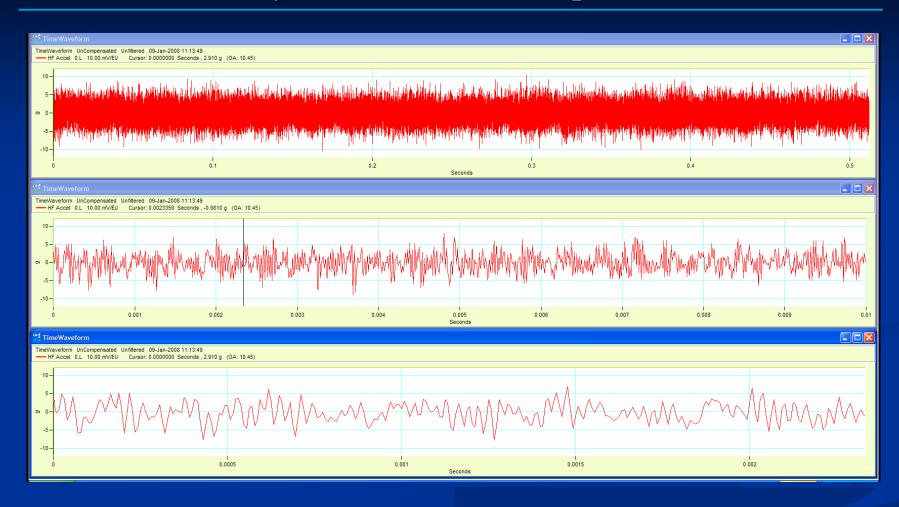
Now we are at 20 kHz Frequency Span



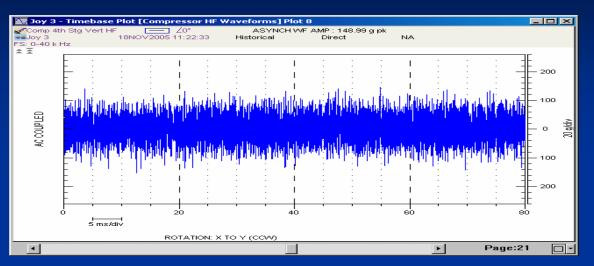
Now we are at 50 kHz



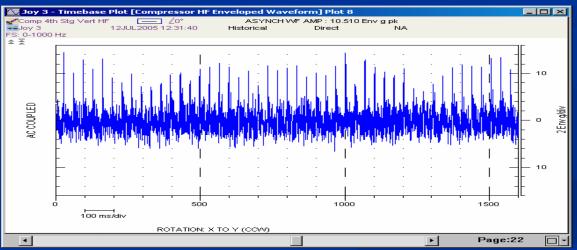
Are the overall amplitudes really higher than expected?



Differing time frames from the high frequency accel

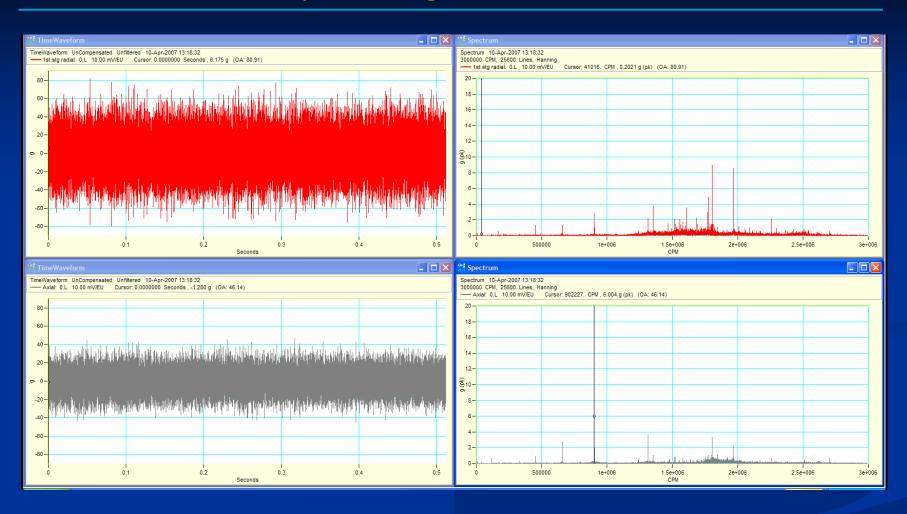


Raw Timebase or Waveform



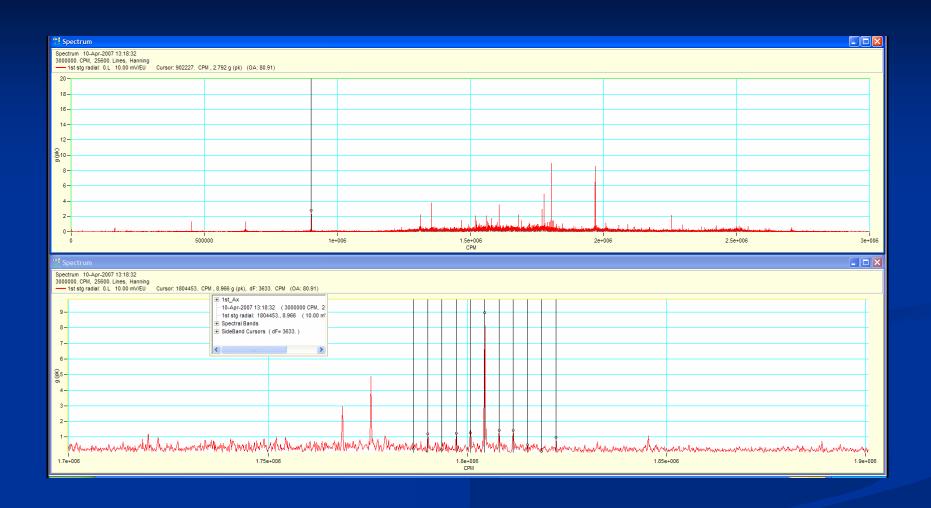
High Frequency Enveloping

Case Study #4 - Ingersoll-Rand Centac II

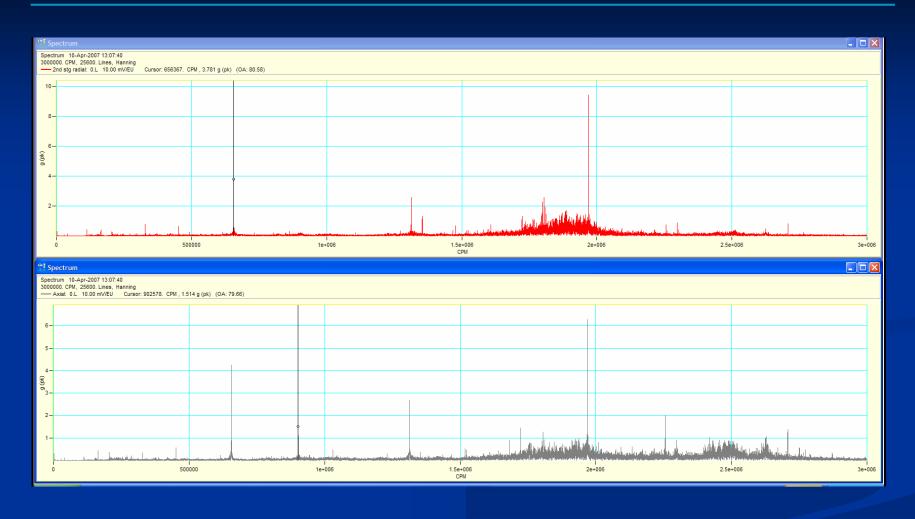


Note the amplitudes

Case Study #4 - Ingersoll-Rand Centac II



Case Study #4 - Ingersoll-Rand Centac II





Thank You – Any Questions?