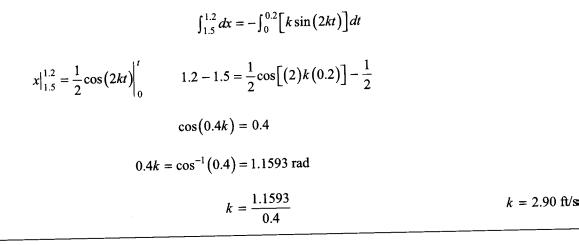


The acceleration of slider A is defined by the relation $a = -2k\sqrt{k^2 - v^2}$, where a and v are expressed in ft/s² and ft/s, respectively, and k is a constant. The system starts at time t = 0 with x = 1.5 ft and v = 0. Knowing that x = 1.2 ft when t = 0.2 s, determine the value of k.

SOLUTION

 $a = \frac{dv}{dt} = -2k\sqrt{k^2 - v^2}, \quad \text{and} \quad v = 0 \text{ when } t = 0$ $-2k \, dt = \frac{dv}{\sqrt{k^2 - v^2}} = d\left[\sin^{-1}\left(\frac{v}{k}\right)\right]$ $-2k \int_0^t dt = \left[\sin^{-1}\left(\frac{v}{k}\right)\right]_0^v$ $\sin^{-1}\left(\frac{v}{k}\right) = -2kt$ $v = k\sin(-2kt) = -k\sin(2kt)$ $dx = v \, dt = -k\sin(2kt) \, dt$

Integrating, using x = 1.5 ft at t = 0, and x = 1.2 ft at t = 0.2 s,



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SOLUTION

x as a function of v.

 $\frac{v}{154} = \sqrt{1 - e^{-0.00057x}}$

$$e^{-0.00057x} = 1 - \left(\frac{\nu}{154}\right)^2$$

$$0.00057x = \ln\left[1 - \left(\frac{\nu}{154}\right)^2\right]$$

$$x = -1754.4 \ln\left[1 - \left(\frac{\nu}{154}\right)^2\right]$$
 (1)

a as a function of x.

$$v^{2} = 23716 \left(1 - e^{-0.00057}\right)$$

$$a = v \frac{dv}{dx} = \frac{d}{dx} \left(\frac{v^{2}}{2}\right) = (11858)(0.00057)e^{-0.0005x}$$

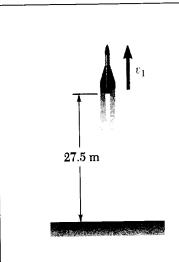
$$a = 6.75906 e^{-0.00057x} = 6.75906 \left[1 - \left(\frac{v}{154}\right)^{2}\right]$$
(2)
(a) $v = 20$ m/s.
From (1), $x = 29.843$ $x = 29.8$ m \blacktriangleleft
From (2), $a = 6.64506$ $a = 6.65$ m/s² \blacktriangleleft
(b) $v = 40$ m/s.

2

x = 122.5 m < x = 122.54From (1), $a = 6.30 \text{ m/s}^2$ a = 6.30306From (2),

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A group of students launches a model rocket in the vertical direction. Based on tracking data, they determine that the altitude of the rocket was 27.5 m at the end of the powered portion of the flight and that the rocket landed 16 s later. Knowing that the descent parachute failed to deploy so that the rocket fell freely to the ground after reaching its maximum altitude and assuming that $g = 9.81 \text{ m/s}^2$, determine (a) the speed v_1 of the rocket at the end of powered flight, (b) the maximum altitude reached by the rocket.

SOLUTION

Constant acceleration. Choose
$$t = 0$$
 at end of powered flight.

Then,

 $y_1 = 27.5 \text{ m}$ $a = -g = -9.81 \text{ m/s}^2$

PROBLEM 11.40

(a) When y reaches the ground, $y_f = 0$ and t = 16 s.

$$y_f = y_1 + v_1 t + \frac{1}{2}at^2 = y_1 + v_1 t - \frac{1}{2}gt^2$$
$$v_1 = \frac{y_f - y_1 + \frac{1}{2}gt^2}{t} = \frac{0 - 27.5 + \frac{1}{2}(9.81)(16)^2}{16} = 76.76 \text{ m/s}$$

 $v_1 = 76.8 \text{ m/s} \blacktriangleleft$

(b) When the rocket reaches its maximum altitude y_{max} ,

$$v = 0$$

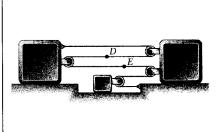
$$v^{2} = v_{1}^{2} + 2a(y - y_{1}) = v_{1}^{2} - 2g(y - y_{1})$$

$$y = y_{1} - \frac{v^{2} - v_{1}^{2}}{2g}$$

$$y_{\text{max}} = 27.5 - \frac{0 - (76.76)^{2}}{(2)(9.81)}$$

$$y_{\text{max}} = 328 \text{ m} \blacktriangleleft$$

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Slider block B moves to the left with a constant velocity of 50 mm/s. At t = 0, slider block A is moving to the right with a constant acceleration and a velocity of 100 mm/s. Knowing that at t = 2 s slider block C has moved 40 mm to the right, determine (a) the velocity of slider block C at t = 0, (b) the velocity of portion D of the cable at t = 0, (c) the accelerations of A and C.

SOLUTION

Let x be position relative to the anchor, positive to the right.

Constraint of cable.

Constraint of cable:

$$-x_{B} + (x_{C} - x_{B}) + 3(x_{C} - x_{A}) = \text{constant}$$

$$4v_{C} - 2v_{B} - 3v_{A} = 0 \qquad (1, 2)$$
When $t = 0$, $v_{B} = -50 \text{ mm/s}$ and $(v_{a})_{0} = 100 \text{ mm/s}$
(a) $(v_{C})_{0} = \frac{1}{4} [2v_{B} + 3(v_{A})_{0}] = \frac{1}{4} [(2)(-50) + (3)(100)] \qquad (v_{C})_{0} = 50 \text{ mm/s} \rightarrow \blacktriangleleft$
Constraint of point D: $(x_{D} - x_{A}) + (x_{C} - x_{A}) + (x_{C} - x_{B}) - x_{B} = \text{constant}$

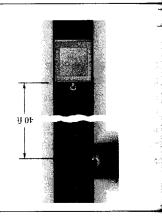
$$v_{D} + 2v_{C} - 2v_{A} - 2v_{B} = 0$$
(b) $(v_{D})_{0} = 2(v_{A})_{0} + 2v_{B} - 2(v_{C})_{0} = (2)(100) + (2)(-50) - (2)(50) \qquad (v_{D})_{0} = 0 \blacktriangleleft$

$$x_{C} - (x_{C})_{0} = (v_{C})_{0}t + \frac{1}{2}a_{C}t^{2}$$
(c) $a_{C} = \frac{2[x_{C} - (x_{C})_{0} - (v_{C})_{0}t]}{t^{2}} = \frac{2[40 - (50)(2)]}{(2)^{2}} = -30 \text{ mm/s}^{2}$

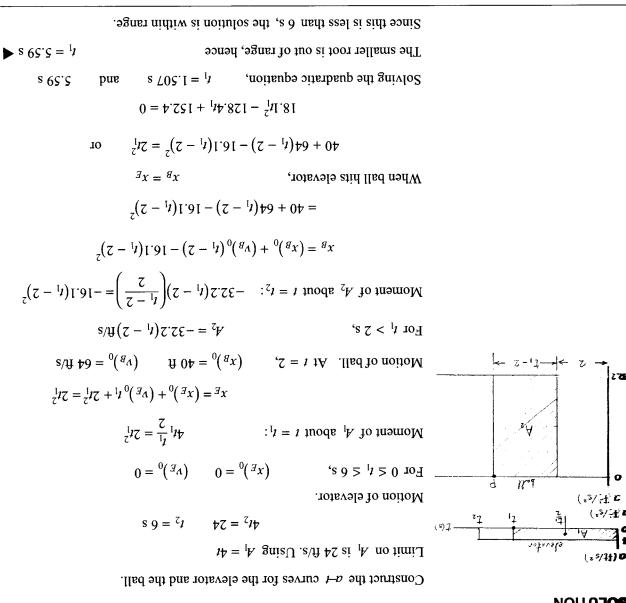
$$a_{C} = 30 \text{ mm/s}^{2} \leftarrow \blacktriangleleft$$
Solving (2) for a_{A}

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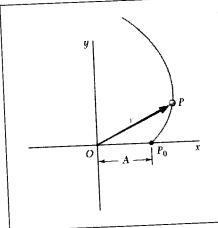
initial velocity of 64 ft/s. Determine when the ball will hit the elevator. initial position of the top of the elevator throws a ball upward with an seconds after the elevator begins to move, a man standing 40 ft above the 4 ft/s² until it reaches a speed of 24 ft/s, which it then maintains. Two An elevator starts from rest and moves upward, accelerating at a rate of



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The motion of a particle is defined by the position vector $\mathbf{r} = A(\cos t + t \sin t)\mathbf{i} + A(\sin t - t \cos t)\mathbf{j}$, where t is expressed in seconds. Determine the values of t for which the position vector and the acceleration vector are (a) perpendicular, (b) parallel.

SOLUTION

Give

Given:

$$\mathbf{r} = A(\cos t + t\sin t)\mathbf{i} + A(\sin t - t\cos t)\mathbf{j}$$

$$\mathbf{v} = \frac{d\mathbf{r}}{dt} = A(-\sin t + \sin t + t\cos t)\mathbf{i} + A(\cos t - \cos t + t\sin t)\mathbf{j}$$

$$= A(t\cos t)\mathbf{i} + A(t\sin t)\mathbf{j}$$

$$\mathbf{a} = \frac{d\mathbf{v}}{dt} = A(\cos t - t\sin t)\mathbf{i} + A(\sin t + t\cos t)\mathbf{j}$$
(a) When **r** and **a** are perpendicular, $\mathbf{r} \cdot \mathbf{a} = 0$

$$A[(\cos t + t\sin t)\mathbf{i} + (\sin t - t\cos t)\mathbf{j}] \cdot A[(\cos t - t\sin t)\mathbf{i} + (\sin t + t\cos t)\mathbf{j}] = 0$$

$$A^{2}[(\cos t + t\sin t)(\cos t - t\sin t) + (\sin t - t\cos t)(\sin t + t\cos t)] = 0$$

$$(\cos^{2} t - t^{2}\sin^{2} t) + (\sin^{2} t - t^{2}\cos^{2} t) = 0$$

$$1 - t^{2} = 0$$

$$t = 1s$$

(b) When **r** and **a** are parallel,
$$\mathbf{r} \times \mathbf{a} = 0$$

$$A[(\cos t + t\sin t)\mathbf{i} + (\sin t - t\cos t)\mathbf{j}] \times A[(\cos t - t\sin t)\mathbf{i} + (\sin t + t\cos t)\mathbf{j}] = 0$$

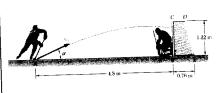
$$A^{2}[(\cos t + t\sin t)(\sin t + t\cos t) - (\sin t - t\cos t)(\cos t - t\sin t)]\mathbf{k} = 0$$

$$\left(\sin t \cos t + t \sin^2 t + t \cos^2 t + t^2 \sin t \cos t\right) - \left(\sin t \cos t - t \cos^2 t - t \sin^2 t + t^2 \sin t \cos t\right) = 0$$

$$t = 0$$

$$t = 0$$

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The initial velocity \mathbf{v}_0 of a hockey puck is 170 km/h. Determine (a) the largest value (less than 45°) of the angle α for which the puck will enter the net, (b) the corresponding time required for the puck to reach the net.

SOLUTION $x = (v_0 \cos \alpha)t$ or $t = \frac{x}{v_0 \cos \alpha}$ Horizontal motion: $y = (v_0 \sin \alpha)t - \frac{1}{2}gt^2$ Vertical motion: $=x\tan\alpha-\frac{gx^2}{2v_0^2\cos^2\alpha}$ $=x\tan\alpha-\frac{gx^2}{2v_0^2}\left(1+\tan^2\alpha\right)$ $\tan^2 \alpha - \frac{2v_0^2}{gx} \tan \alpha + \left(1 + \frac{2v_0^2 y}{gx^2}\right) = 0$ $v_0 = 170 \text{ km/h} = 47.222 \text{ m/s}, \quad x = 4.8 \text{ m} \text{ at point } C,$ Data: v = 1.22 m at point C. $\frac{2v_0^2}{gx} = \frac{(2)(47.222)^2}{(9.81)(4.8)} = 94.712$ $\frac{2v_0^2 y}{m^2} = \frac{(94.712)(1.22)}{4.8} = 24.073$ $\tan^2 \alpha - 94.712\alpha + 25.073 = 0$ (a) 94.45 $\tan\alpha=0.26547$ and $\alpha = 14.9^{\circ}$ 89.4° $\alpha = 14.869^{\circ}$ or $t = \frac{x}{v_0 \cos \alpha} = \frac{4.8}{(47.222) \cos 14.869^\circ}$ t = 0.1052 s(b)

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A motorist is traveling on a curved portion of highway of radius 350 m at a speed of 72 km/h. The brakes are suddenly applied, causing the speed to decrease at a constant rate of 1.25 m/s^2 . Determine the magnitude of the total acceleration of the automobile (*a*) immediately after the brakes have been applied, (*b*) 4 s later.

SOLUTION

Initial speed.

Tangential acceleration.

(a) Total acceleration at t = 0.

$$a_n = \frac{v_0^2}{\rho} = \frac{(20)^2}{350} = 1.14286 \text{ m/s}^2$$
$$a = \sqrt{a_t^2 + a_n^2} = \sqrt{(-1.25)^2 + (1.14286)^2} \qquad a = 1.694 \text{ m/s}^2 \blacktriangleleft$$

 $v_0 = 72 \text{ km/h} = 20 \text{ m/s}$

 $a_t = -1.25 \text{ m/s}^2$

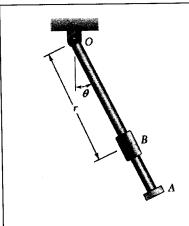
(b) Total acceleration at t = 4 s.

$$v = v_0 + a_t t = 20 + (-1.25)(4) = 15 \text{ m/s}$$

$$a_n = \frac{v^2}{\rho} = \frac{(15)^2}{350} = 0.6426 \text{ m/s}^2$$

$$a = \sqrt{a_t^2 + a_n^2} = \sqrt{(-1.25)^2 + (0.6426)^2} \qquad a = 1.406 \text{ m/s}^2 \blacktriangleleft$$

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The oscillation of rod OA about O is defined by the relation $\theta = (4/\pi)(\sin \pi t)$, where θ and t are expressed in radians and seconds, respectively. Collar B slides along the rod so that its distance from O is r = 10/(t+6), where r and t are expressed in mm and seconds, respectively. When t = 1 s, determine (a) the velocity of the collar, (b) the total acceleration of the collar, (c) the acceleration of the collar relative to the rod.

SOLUTION

Differentiate the expressions for r and θ with respect to time.

$$r = \frac{10}{t+6} \text{ mm}, \qquad \dot{r} = -\frac{10}{\left(t+6\right)^2} \text{ mm/s}, \qquad \ddot{r} = \frac{20}{\left(t+6\right)^3} \text{ mm/s}^2$$
$$\theta = \frac{4}{\pi} \sin \pi t \text{ rad}, \qquad \dot{\theta} = 4 \cos \pi t \text{ rad/s} \qquad \ddot{\theta} = 4\pi \sin \pi t \text{ rad/s}^2$$

At t = 1 s, $r = \frac{10}{7}$ mm; $\dot{r} = -\frac{10}{49}$ mm/s, $\ddot{r} = \frac{20}{343}$ mm/s² $\theta = 0$, $\dot{\theta} = -4$ rad/s, $\ddot{\theta} = 0$

(a) Velocity of the collar.

$$v_r = \dot{r} = 0.204 \text{ mm/s}, \quad v_\theta = r\theta = -5.71 \text{ mm/s}$$

$$\mathbf{v}_B = (0.204 \text{ mm/s})\mathbf{e}_r - (5.71 \text{ mm/s})\mathbf{e}_{\theta} \blacktriangleleft$$

(b) Acceleration of the collar.

$$a_{r} = \ddot{r} - r\dot{\theta}^{2} = \frac{20}{343} - \left(\frac{10}{7}\right)(-4)^{2} = -22.8 \text{ mm/s}^{2}$$

$$a_{\theta} = r\ddot{\theta} + 2\dot{r}\dot{\theta} = \left(\frac{10}{7}\right)(0) + (2)\left(-\frac{10}{49}\right)(-4) = 1.633 \text{ mm/s}^{2}$$

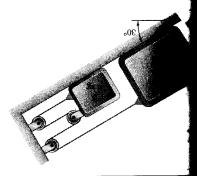
$$\mathbf{a}_{B} = -\left(22.8 \text{ mm/s}^{2}\right)\mathbf{e}_{r} + \left(1.633 \text{ mm/s}^{2}\right)\mathbf{e}_{\theta} \blacktriangleleft$$

(c) Acceleration of the collar relative to the rod.

$$\mathbf{a}_{B/OA} = \ddot{r}\mathbf{e}_r = \frac{20}{343}\mathbf{e}_r \qquad \qquad \mathbf{a}_{B/OA} = (0.0583 \text{ mm/s}^2)\mathbf{e}_r \blacktriangleleft$$

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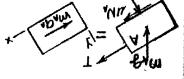
The two blocks shown are originally at rest. Neglecting the masses of the pulleys and the effect of friction in the pulleys and assuming that the coefficients of friction between both blocks and the incline are $\mu_s = 0.25$, and $\mu_k = 0.20$, determine (a) the acceleration of each block, (b) the tension in the cable.

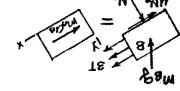


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Let the positive directions of
$$x_A$$
 and x_B be down the incide.
Constraint of the cable: $x_A + 3x_B = \text{constant}$
Block $A_1 + \sum E_y = 0$: $A_A + 3a_B = 0$ $a_B = -\frac{1}{3}a_A$
 $A_A + \sum F_y = 0$: $N_A - m_A g \cos 30^\circ - \mu N_A - T = m_A a_A$
Fliminate N_A .
Eliminate N_A .

Block B: $+ \sum E^{\lambda} = 0$: $N^{B} - w^{B} \cos 30^{\circ} = 0$





Eliminate N_{B} .

$$m_B g \left(\sin 30^\circ + \mu \cos 30^\circ \right) - 3T = -\left(\cos 30^\circ \right) - 3T = -\frac{1}{5} \left(\sin 30^\circ - 1 \right) = -\frac{1}{5} \left(\sin 30^\circ - 1 \right)$$

 $\frac{\varepsilon}{v p^{g} w} = g p^{g} w = I \varepsilon - g N \eta + 0 \varepsilon \operatorname{uis} g g w : p w = A \Sigma / +$

Eliminate T.

$${}_{k} \mathcal{D} \left(\frac{\mathbb{A}^{m}}{\mathbb{E}} + \mathbb{A}^{m} \mathbb{E} \right) = {}^{\circ} 0 \mathbb{E} \operatorname{sos} (\mathbb{B}^{m} + \mathbb{B}^{k} \mathbb{m} \mathbb{E}) \mathbb{U} - {}^{\circ} 0 \mathbb{E} \operatorname{nis} (\mathbb{B}^{m} - \mathbb{B}^{k} \mathbb{m} \mathbb{E})$$

Check the value of μ_s required for static equilibrium. Set $a_A = 0$ and solve for μ .

$$\mu = \frac{(3m_A - m_B)\sin 30^\circ}{(3m_A - m_B)\cos 30^\circ} = \frac{(75 - 20)}{(75 - 20)}\tan 30^\circ = 0.334.$$

Since $\mu_s = 0.25 < 0.334$, sliding occurs.

Calculate
$$\frac{\alpha_A}{8}$$
 for sliding. Use $\mu = \mu_k = 0.20$.

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PROBLEM 12.13 CONTINUED

$$\frac{a_A}{g} = \frac{(3m_A - m_B)\sin 30^\circ - \mu (3m_A + m_B)\cos 30^\circ}{3m_A + m_B/3}$$

= $\frac{(30 - 8)\sin 30^\circ - (0.20)(30 + 8)\cos 30^\circ}{30 + 2.667} = 0.13525$
(a) $a_A = (0.13525)(9.81) = 1.327 \text{ m/s}^2$ $\mathbf{a}_A = 1.327 \text{ m/s}^2 \neq 30^\circ$
 $a_B = -\left(\frac{1}{3}\right)(1.327) = -0.442 \text{ m/s}^2$ $\mathbf{a}_B = 0.442 \text{ m/s}^2 \neq 30^\circ$
(b) $T = m_A g (\sin 30^\circ - \mu \cos 30^\circ) - m_A a_A$
 $= (10)(9.81)(\sin 30^\circ - 0.20\cos 30^\circ) - (10)(1.327)$
 $T = 18.79 \text{ N}$

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tension in the cable. effect of friction, determine (a) the acceleration of each block, (b) the The weights of blocks A, B, and C are $W_A = W_C = 20$ lb, and $W_B = 10$ lb. Knowing that P = 50 lb and neglecting the masses of the pulleys and the

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e at the fixed anchor. at the positive direction for position coordinates, velocities, and accelerations be to the right. Let the origin

Constraint of cable:
$$3(x_{c} - x_{A}) + (x_{c} - x_{B}) + (-x_{B}) = \text{constant}}$$

 $4a_{c} - 2a_{B} - 3a_{A} = 0$
(1)

 $4a_{c} - 2a_{B} - 3a_{A} = 0$
(1)

 $4a_{c} - 2a_{B} - 3a_{A} = 0$
(1)

 $4a_{c} - 2a_{B} - 3a_{A} = 0$
(2)

 $4a_{c} - 2a_{B} - 3a_{A} = 0$
(2)

 $4a_{c} - 2a_{B} - 3a_{A} = 0$
(3)

 $4a_{c} - 2a_{B} - 3a_{A} = 0$
(3)

 $4a_{c} - 2a_{B} - 3a_{A} = 0$
(4)

 $a_{c} = \frac{2a_{c}}{m_{a}} = \frac{2a_{c}}{m_{b}} = \frac{2a_{c}}{20} =$

07 07

(1) oini (4) hns (2), (3) and (4) into (1),

(b) As determined above,

$$\mathbf{A}_{A} = \frac{20}{20} = 29.3 \text{ ft/s}^{2} \longrightarrow \mathbf{A}_{A} = \frac{20}{(3)(6.0605)(32.2)} = 29.3 \text{ ft/s}^{2} \longrightarrow \mathbf{A}_{A} = 29.3 \text{ ft/s}^{2} \longrightarrow \mathbf{A}_{A} = \frac{20}{20} \text{ ft/s}^{2} \longrightarrow \mathbf{A}$$

$$\mathbf{F}_{s}^{2} = \frac{1}{300} \int_{0}^{2} \int_{0}^{2}$$

From (4),
$$a_{\rm C} = \frac{20}{20} = 41.5 \, \text{ft/s}^2$$

 $\mathbf{a}_{\rm C} = 41.5 \, \text{ft/s}^2$

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dI = 6.06 Ib

During a hammer thrower's practice swings, the 16-lb head A of the hammer revolves at a constant speed v in a horizontal circle as shown. If $\rho = 3$ ft and $\theta = 60^{\circ}$, determine (a) the tension in wire BC, (b) the speed of the hammer's head.

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$$0 = W - \theta \operatorname{nis} T : 0 = \sqrt[q]{AZ} \qquad (a)$$

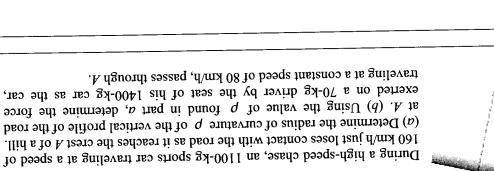
$$0 = W - \theta \operatorname{nis} T : 0 = \sqrt[q]{AZ} \qquad (b)$$

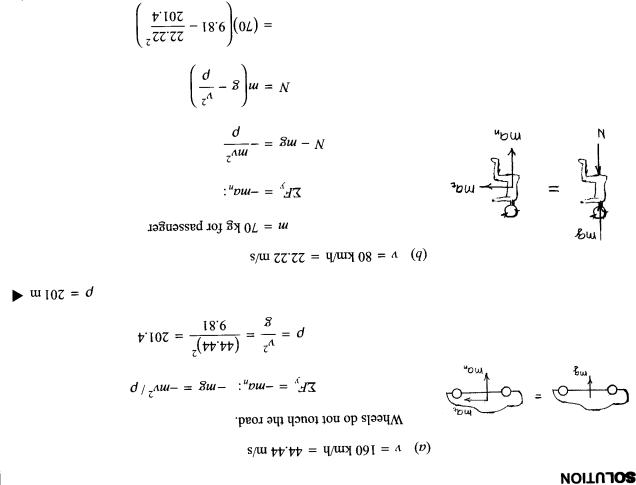
$$\frac{\partial \Omega \operatorname{nis}}{\partial \theta \operatorname{nis}} = \frac{\partial \Pi}{\partial \operatorname{nis}} = T$$

$$\frac{\zeta_{V}}{\partial \eta} m = \theta \operatorname{soo} T : \pi_{n} m = \sqrt[q]{AZ} + \qquad (d)$$

$$\frac{\zeta_{V}}{\partial \eta} m = \theta \operatorname{soo} T = \frac{\partial \Omega \operatorname{nis}}{\partial \eta} = \frac{\partial \Omega \operatorname{nis}$$

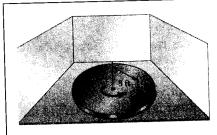
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 $\square N SIS = N$

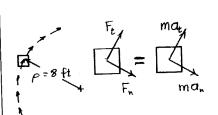
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A turntable A is built into a stage for use in a theatrical production. It is observed during a rehearsal that a trunk B starts to slide on the turntable 12 s after the turntable begins to rotate. Knowing that the trunk undergoes a constant tangential acceleration of 0.75 ft/s², determine the coefficient of static friction between the trunk and the turntable.

SOLUTION

Uniformly accelerated motion on a circular path. $\rho = 8$ ft



 $v = v_0 + a_t t$ = 0 + (0.75)(12) = 9 ft/s $F_t = ma_t = \frac{W}{g}a_t: \quad F_t = W\frac{a_t}{g} = \frac{0.75}{32.2}W = 0.0233 W$ $F_n = ma_n = \frac{Wv^2}{g\rho}: \quad F_n = \frac{W(9)^2}{(32.2)(8)} = 0.3144 W$

$$F = \sqrt{F_t^2 + F_n^2} = 0.315 \ W$$

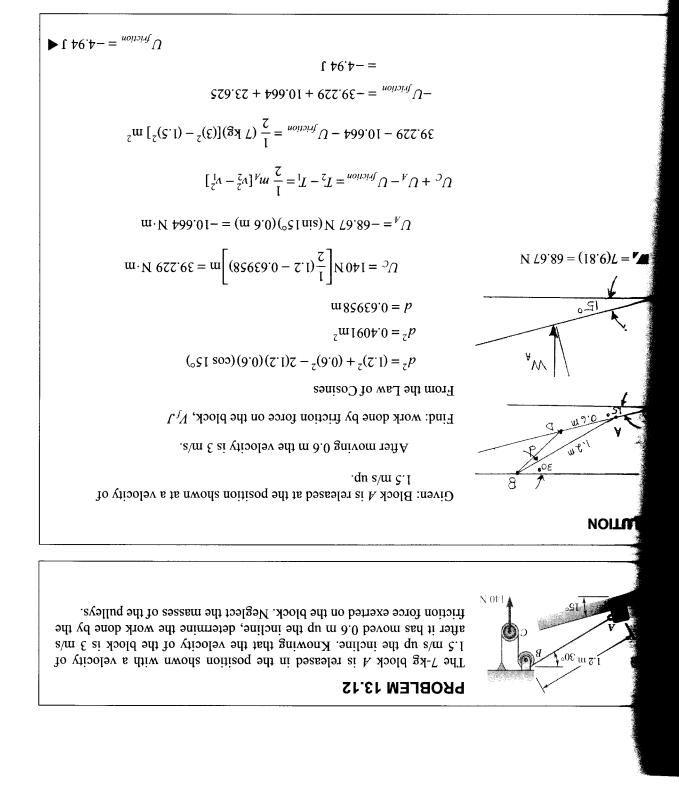
This is the friction force available to cause the trunk to slide.

The normal force N is calculated from equilibrium of forces in the vertical direction.

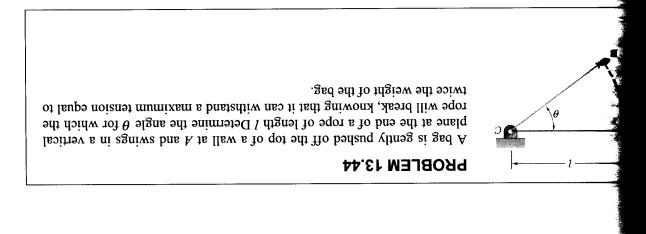
$$\Sigma F_y = 0: \quad N - W = 0 \qquad N = W$$

Since sliding is impending, $\mu_s = \frac{F}{W} = 0.315$ $\mu_s = 0.315$

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NOL

Use work - energy : position 1 is at A, position 2 is at B.

$$T_1 + U_{1 \to 2} = T_2$$

Where
$$T_1 = 0$$
; $U_{1 \to 2} = mg \ln \theta$; $T_2 = \frac{1}{2} mg^2$

Substitute

$$\frac{\zeta}{d} \sqrt{n} \frac{1}{\zeta} = \theta \operatorname{nis} 1 gm + 0$$

 θ uis $l_{SZ} = \frac{2}{3}v$

For T = 2 W use Newtons 2nd law.

(E)
$$\frac{1}{l} \frac{W}{l} = \theta \operatorname{nis} W - W = ma_n = X \qquad \sum_n W = \frac{1}{l} \frac{W}{l} \qquad (3)$$

$$\frac{1}{l} \frac{W}{l} = \theta \operatorname{nis} W - W = ma_n = 2W \qquad W = \frac{1}{l} \frac{W}{l} \qquad (3)$$

$$\frac{1}{l} \frac{\theta \operatorname{nis} l}{\theta \operatorname{nis} l} \frac{1}{2} \frac{\theta \operatorname{nis} l}{\theta \operatorname{nis} l} = \theta \operatorname{nis} \frac{1}{2} \qquad (3)$$

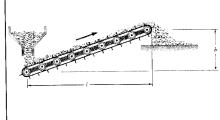
$$\frac{1}{l} \frac{\theta \operatorname{nis} l}{\theta \operatorname{nis} l} \frac{1}{2} = \theta \operatorname{nis} \frac{1}{2} = \theta \operatorname{nis} 10$$

 \bullet °8.14 = θ

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A power specification formula is to be derived for electric motors which drive conveyor belts moving solid material at different rates to different heights and distances. Denoting the efficiency of the motors by η and neglecting the power needed to drive the belt itself, derive a formula (a) in the SI system of units, for the power P in kW, in terms of the man flow rate m in kg/h, the height b, and the horizontal distance l in meters and (b) in U.S. customary units, for the power in hp, in terms of the man flow rate m in tons/h, and the height b and horizontal distance l in feet.

SOLUTION

(a) Material is lifted to a height b at a rate,

$$(m \text{ kg/h})(g \text{ m/s}^2) = [mg(N/h)]$$

Thus,

(b)

$$\frac{\Delta U}{\Delta t} = \frac{\left[mg(N/h)\right]\left[b(m)\right]}{(3600 \text{ s/h})} = \left(\frac{mgb}{3600}\right)N \cdot \text{m/s}$$

 $1000 \text{ N} \cdot \text{m/s} = 1 \text{ kw}$

Thus, including motor efficiency, η

$$P(kw) = \frac{mgb(N \cdot m/s)}{(3600) \left(\frac{1000 \text{ N} \cdot m/s}{\text{kw}}\right)(\eta)}$$

 $P(kw) = 0.278 \times 10^{-6} \frac{m_{\odot}}{m_{\odot}}$

$$\frac{\Delta U}{\Delta t} = \frac{\left[W(\text{tons/h})(2000 \text{ lb/ton})\right]\left[b(\text{ft})\right]}{3600 \text{ s/h}}$$

$$=\frac{Wb}{1.8}$$
 ft·lb/s; 1hp = 550 ft·lb/s

With
$$\eta$$
, $hp = \left[\frac{Wb}{1.8}(\mathbf{ft}\cdot\mathbf{lb/s})\right] \left[\frac{1\mathrm{hp}}{550\,\mathbf{ft}\cdot\mathbf{lb/s}}\right] \left[\frac{1}{\eta}\right]$

 $hp = \frac{1.010 \times 10^{-3}}{n}$

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A spring is used to stop a 200-lb package which is moving down a 20° incline. The spring has a constant k = 125 lb/in. and is held by cables so that it is initially compressed 6 in. Knowing that the velocity of the package is 8 ft/s when it is 25 ft from the spring and neglecting friction, determine the maximum additional deformation of the spring in bringing the package to rest.

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Квләиә fo иоңролләсиот

to is at the top of the incline; position (2) is when the spring has maximum deformation

$$\psi = 1200 \text{ JP}/\text{IP}$$

$$\mathbf{CLC} \qquad T_1 + V_1 = T_2 + V_2$$

At (1) A

$$V_{1} = V_{21} + V_{21} = \frac{1}{2} \left(\frac{200}{32.2} \right)^{2} (8)^{2} = \frac{1}{2} (32.2)^{2} = \frac{1}{2} V_{12} + V_{12} + V_{12} = \frac{1}{2} V_{12} + V_{12} + V_{12} = \frac{1}$$

$$= 200(0021)\frac{1}{2} + 002 \operatorname{nis}(x - 22)002 =$$

$$x = Deformation of the spring$$

$$\delta.781 + x^{4}0^{4}.88 + 1.0171 = \sqrt{10}$$

$$\Psi(5) \qquad L^{5} = 0; \qquad \Lambda^{5} = \frac{1}{\sqrt{2}} + \Lambda^{55} = \frac{1}{\sqrt{2}} + \chi^{5}_{5} = \frac{1}{\sqrt{2}} (1200) (0.5 + \chi)_{5}$$

$$^{2}(x + 2.0)087 = 2.781 + x404.88 + 1.0171 + 87.891$$
 (1) of a grit minuting using the second statement of the second statem

$$0 = 6.8061 - x362.183 + z_{x}027 \qquad \text{avior}$$

$$ff = -2.11$$
 or $+1.2044 ff$

● \hat{H} \$204 \hat{H} = 1.204 \hat{H} = 14.45 in. <

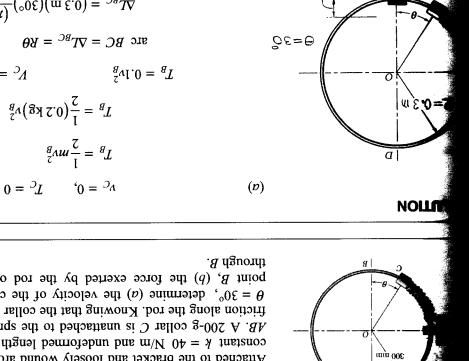
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point B, (b) the force exerted by the rod on the collar as is passes $\theta = 30^{\circ}$, determine (a) the velocity of the collar as it passes through friction along the rod. Knowing that the collar is released from rest when AB. A 200-g collar C is unattached to the spring and can slide without constant k = 40 N/m and undeformed length equal to the arc of circle Attached to the bracket and loosely wound around the rod is a spring of A thin circular rod is supported in a vertical plane by a bracket at A.

 $T_{g} = \frac{1}{2} \left(0.2 \, \mathrm{kg} \right) v_{2}^{2}$

 $L^{B} = \frac{5}{1} m m_{5}^{B}$



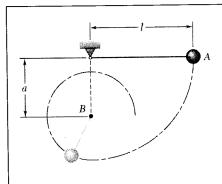
 \overline{a}

$$E_{R} = I.962 \, M + (0.2 \, R_{0}^{2}) \frac{(0.3 \, m)}{(2.7234 \, m_{2}/s_{2})} = 0.1 \, k_{2}^{B} = 0.1 \,$$

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 $F_R = 1.962 \text{ N} + 3.8156 \text{ N} = 5.7776 \text{ N}$

 $I = S^{-1} S = S^{-1} S$



The pendulum shown is released from rest at A and swings through 90° before the cord touches the fixed peg B. Determine the smallest value of a for which the pendulum bob will describe a circle about the peg.

1

(2

 $a = \frac{3}{5}l$

SOLUTION

Use conservation of energy from the point of release (A) and the top of the circle.

 $T_1 + V_1 = T_2 + V_2$ (1) (datum at lowest point)

where

$$T_1 = 0; \quad V_1 = mg\,\ell$$

At 2
$$T_2 = \frac{1}{2}mv^2; V_2 = mgz = mg(2)(\ell - a)$$

Substituting into (1)

$$0 + mg\ell = \frac{1}{2}mv^2 + 2mg(\ell - a)$$

We need another equation – use Newton's 2^{nd} law at the top. (Tension, $T_0 = 0$ at top)

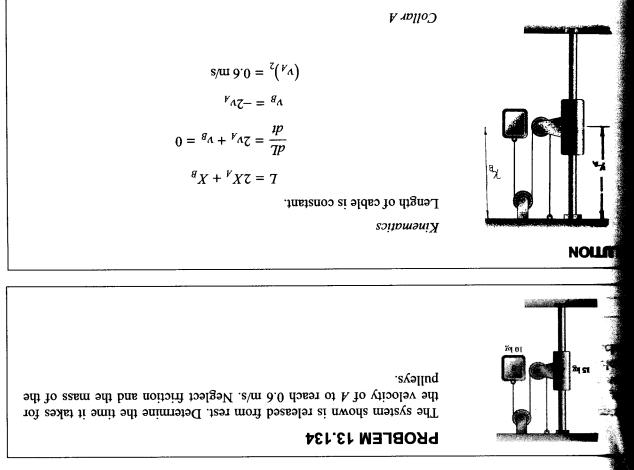
$$= \bigoplus_{n \neq 1} \sum_{l=0}^{\infty} f_{n} = ma_{n} \Rightarrow mg = \frac{mv^{2}}{\rho}$$

$$v^{2} = g\rho = g(\ell - a)$$

Substituting into (2)

$$mg \ell = \frac{1}{2}mg(\ell - a) + 2mg(\ell - a)$$
$$2\ell = \ell - a + 4\ell - 4a$$
$$5a = 3\ell$$

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(1)

$$gx^{I} SI = {}_{k}m$$

$$gx^{I} (I) = {}_{2-1}n^{k}W - ({}_{2-1}n)(TS) + {}_{1}({}_{k}v_{k}m)$$

$$gx^{I} SI = {}_{2-1}n^{k}(IS + 2I) - TS] + 0$$

$$gx^{I} SI = {}_{2-1}n^{k}(STS + 2I) - TS]$$

$$(1)$$

 $s/m \ \Omega.I = {}_{2}({}_{N}v) \ \Omega = {}_{2}({}_{N}v)$

 $m^{B} = 10 \text{ kg}$

▶ s £73.0 = 1

(7)

Collar B

$$\mathbf{e}_{\mathbf{r}}^{\mathbf{r}} + \mathbf{f}_{\mathbf{r}}^{\mathbf{r}} + \mathbf{f}_{\mathbf$$

A (2, T) 2 (-2

z(MM)

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 $t_{1-2} = \frac{24.52}{16.5} = 0.673 s$

(I) anitanimile) (2) noiteup E and (1) noiteup E bbA

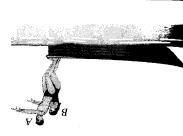
21 + 2.4 = 2 - 13(272.57 - 1.89)

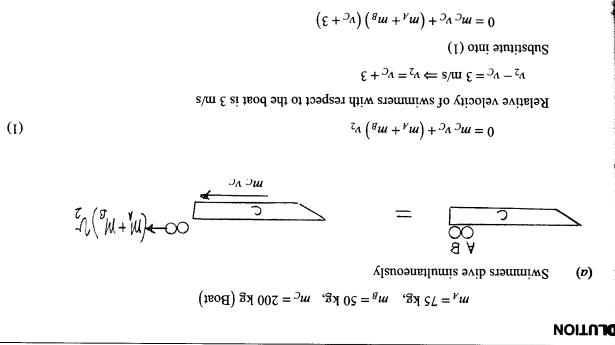
 ${}^{2}({}^{g}{}^{\Lambda}{}^{g}{}^{M}) = ({}^{2-1}{}^{j}){}^{g}{}^{M} + ({}^{2-1}{}^{j}){}^{J} - {}^{i}({}^{\mu}{}^{\Lambda}{}^{g}{}^{M}) +$

 $(2.1)0I = (_{2-1}i)[T - (18.9 \times 0I)] + 0$



Two swimmers A and B, of mass 75 kg and 50 kg, respectively, dive off the end of a 200-kg boat. Each swimmer has a relative horizontal velocity of 3 m/s when leaving the boat. If the boat is initially at rest, determine its final velocity, assuming that (a) the two swimmers dive simultaneously, (b) swimmer A dives first, (c) swimmer B dives first.





Solve

лp-x

$$\frac{(007 + 05 + 5L)}{(05 + 5L) \epsilon^{-}} = \frac{J_{u} + {}^{g} u + {}^{V} u}{({}^{g} u + {}^{V} u) \epsilon^{-}} = J_{A}$$

 $0 = \left(m_C + m_B\right) v_{C_2} - m_{A_1} v_{C_2}$

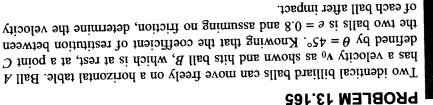
(a) A dives first and then B

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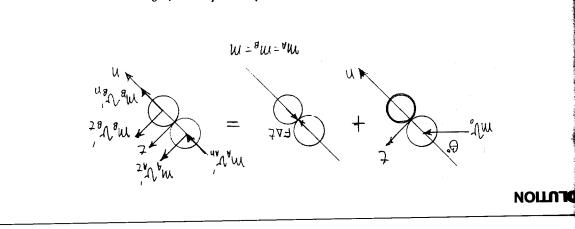
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 \rightarrow s/m $2 \leq 1.1 \leq \sqrt{2}$

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$$\theta$$
 us $_{0}^{\Lambda} = {}^{V} \Lambda \Leftarrow {}^{V} \Lambda \mu = \theta$ us $_{0}^{\Lambda} \mu \mu$

Ball A t-dir

$$0 = {}^{ig} \Lambda \rightleftharpoons {}^{ig} \Lambda \rightleftharpoons {}^{gu} = 0$$

$$^{ug}\Lambda \mu t + {}^{uv}\Lambda \mu t = 0 + \theta \cos^0 \eta \mu t$$

Coefficient of restitution

$$(u^{g} \Lambda - {}^{uv} \Lambda) \partial = {}^{uv} \Lambda - {}^{ug} \Lambda$$
$$(u^{g} \Lambda - {}^{uv} \Lambda) \partial = {}^{uv} \Lambda - {}^{ug} \Lambda$$

(2) bas (1) svlo2

$$\theta \operatorname{sool}\left(\frac{\zeta}{\partial+1}\right)_{0} = \overset{ug}{}_{n} \left(\theta \operatorname{sool}\frac{\zeta}{\partial-1}\right)_{0} = \overset{uv}{}_{n}$$

With numbers

$$e = 0.8; \quad \theta = 45^{\circ}$$

$$v_{Ah} = v_0 \sin 45^{\circ} = 0.707 v_0$$

$$v_{Ah} = v_0 \left(\frac{1 - 0.8}{2}\cos 45^{\circ}\right) = 0.0707 v_0$$

$$v_{Bh} = 0$$

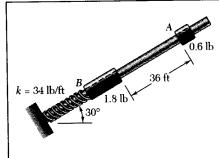
$$v_{Bh} = v_0 \left(\frac{1 + 0.8}{2}\cos 45^{\circ} = 0.6364 v_0$$

pənuituos

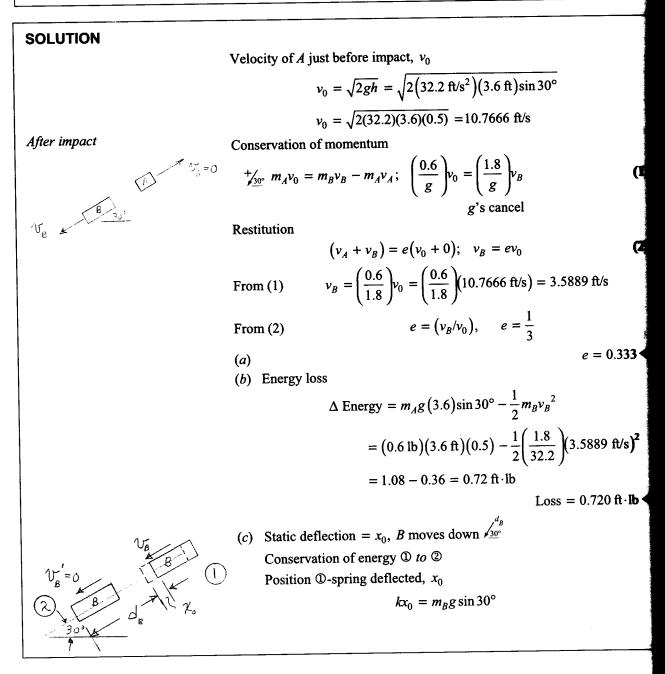
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A 0.6-lb collar A is released from rest, slides down a frictionless rod, and strikes a 1.8-lb collar B which is at rest and supported by a spring of constant 34 lb/ft. Knowing that the velocity of collar A is zero immediately after impact, determine (a) the coefficient of restitution between the two collars, (b) the energy lost in the impact, (c) the maximum distance collar B moves down the rod after impact.



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PROBLEM 13.183 CONTINUED

$$T_{1} + V_{1} = T_{2} + V_{2}; T_{1} = \frac{1}{2} m_{B} v_{B}^{2}, T_{2} = 0$$

$$V_{2} = V_{e}^{*} + V_{e}^{*} = \int_{0}^{x_{0} + d_{B}} kx dx = \frac{1}{2} k \left(d_{B}^{2} + 2d_{B} x_{0} + x_{0}^{2} \right) + 0 + 0$$

$$V_{2} = V_{e}^{*} + V_{e}^{*} = \int_{0}^{x_{0} + d_{B}} kx dx = \frac{1}{2} k \left(d_{B}^{2} + 2d_{B} x_{0} + x_{0}^{2} \right) + 0 + 0$$

$$V_{2} = V_{e}^{*} + V_{e}^{*} = \int_{0}^{x_{0} + d_{B}} kx dx = \frac{1}{2} k \left(d_{B}^{2} + 2d_{B} x_{0} + x_{0}^{2} \right) + 0 + 0$$

$$U_{1} = V_{e}^{*} + V_{e}^{*} = \int_{0}^{x_{0} + d_{B}} kx dx = \frac{1}{2} k \left(d_{B}^{2} + 2d_{B} x_{0} + x_{0}^{2} \right) + 0 + 0$$

$$U_{1} = V_{e}^{*} + V_{e}^{*} = \int_{0}^{x_{0} + d_{B}} kx dx = \frac{1}{2} k \left(\frac{1}{8} + 2d_{B} x_{0} + x_{0}^{2} \right) + 0 + 0$$

$$U_{1} = V_{e}^{*} + V_{e}^{*} = \int_{0}^{x_{0} + d_{B}} kx dx = \frac{1}{2} k \left(\frac{1}{8} + 2d_{B} x_{0} + x_{0}^{2} \right) + 0 + 0$$

$$U_{1} = V_{e}^{*} + V_{e}^{*} = \frac{1}{2} h x_{0}^{2} + \frac{1}{2} h x_{0$$

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