## ECCENTRICALLY LOADED WELDED and Bolted Connections

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LECTURE XIV
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## ECCENTRIC LOADING

- Let's start with Simple Off

Axis Tension

- The simplified formula we used will not work in this case because the value of $\theta$ has changed!



## ECCENTRIC LOADING

- Calculating the capacity of the example in the figure:


$$
\begin{aligned}
\beta & =1.2-\frac{0.002 * L}{\text { Leg }}=0.912 \\
\varphi R_{n} & =\varphi * F_{n w} * A_{w e}=0.75\left[0.60 * F_{E X X} *\left(1+0.5 * \sin ^{1.5} \theta\right)\right] *\left[\frac{\sqrt{2}}{2} \beta * L * L e g\right] \\
\varphi R_{n} & =0.75 * 0.6 * 70^{k s i} *\left(1+0.5 * \sin ^{1.5} 32.30^{\circ}\right) * 0.707 * 0.912 * 36^{\text {inch }} * 0.25^{\text {inch }}=218.53^{k i p}
\end{aligned}
$$

## ECCENTRIC LOADING

- BUT, there is another $\quad \varphi R_{n}=\varphi *\left(0.85 * R_{\text {sidie.wede }}+1.5 * R_{\text {tumsusesese wed }}\right)$ equation...J2-9b

$$
\beta=1.2-\frac{0.002 * L}{L e g}=0.912
$$



$$
0.75 * 0.6 * 70^{k i} *\left(1+0.5 * \sin ^{1.5} 32.30^{\circ}\right) * 0.707 * 0.912 *\left[0.85 * 28^{\text {inch }}+1.5 * 8^{\text {inch }}\right] * 0.25^{\text {inch }}=217.31^{\text {kip }}
$$

- In which case, actually the largest governs, i.e. $218.53^{\text {kips }}$


## ECCENTRIC LOADING

- Ec Centric - 'Ек Kєvтрıкóv - Off Centered
- A load that is applied at a distance from the center of the element
- In case axial loading is anticipated...
$\square$ What is the effect?
- The effect is that of Force multiplied by Distance
- That results in a moment
- The effect is simple, but the way it is treated may be a bit more complicated
- Elements that are set to receive axial loading may end up being subjected to flexural stresses. How does this effect the connections of these elements?


## EFFECTS OF ECCENTRIC LOADING On Welded Connections

- Shear
- Occurs due to translational movement between surfaces
- Torsion
- Occurs due to the establishment of a pivotal point (the center of gravity of the specific design in the case of the illustration) and a load that passes at a distance
- The distance is the eccentricity
- Shear can take place in both linear and curvilinear manner.

- Torsion will cause shear in a curvilinear manner


## ELASTIC METHOD

- Analysis
- The elastic method is rather conservative as friction, or slip resistance, between the parts is neglected.
- The parts are considered perfectly rigid, hence deformation takes place exclusively in the weld
- The welds are assumed to be perfectly elastic



## ELASTIC METHOD

- The horizontal and the vertical components of the force exerted due to the torsion can be calculated by the following two expressions:
$r_{m u n}=\frac{T * v}{I_{p}}$

$$
r_{m u v}=\frac{T * h}{I_{p}}
$$

Eqs. 8-9a, 8-10a

- Where $r_{m u}$ is the resultant shear due to moment, $T$ is torsion (can consider as the product of $\mathrm{P}_{\mathrm{u}}$ and eccentricity), v \& h are the vertical and horizontal distances, and $\mathrm{I}_{\mathrm{p}}$ is the polar Moment of Inertia
(slightly modified characters from AISC manual)
- For the overall shear effect we use $\quad r_{m u s}=\frac{P_{u}}{L_{\text {tot }}}$


## Ultimate Strength Method

$\square$ A more realistic strength analysis of eccentrically loaded welded connections.

- The load causes a relative rotation and translation among the parts of the assembly
- Rotation will take place around the "instantaneous center of rotation" which is dependent upon the geometric forms, the location of the eccentric load, and the deformations that will take place
- Some parts will yield but some parts will remain and the less stress fibers will contribute more toward the resistance to the load.
- Failure will occur when all fibers yield


## Ultimate Strength Method

- A simple way to visualize this is by considering the least complicated scenario of a weld symmetrically set about the horizontal axis.
- Each differential element of an infinitesimally small width "ds" in the weld shall provide a resisting force "R."
- That force " $R$ " will act perpendicularly to a ray that passes through the instantaneous center.
- The element that is located the greatest distance from the instantaneous center, can be assumed to reach its limit first.



## Ultimate Strength Method

- Deformation:
- When maximum stress is attained, the deformation $\Delta m a x$ is determined by the following formula:

$$
\begin{equation*}
\Delta_{\max }=1.087 \mathrm{w}(\theta+6)^{-0.65} \leq 0.17 \mathrm{w} \tag{8-4}
\end{equation*}
$$

where $w$ is the weld leg

- Nominal Strength:
- The nominal strength can be given by:
$\Phi R_{n}=0.75 * C * C_{1} * D * l$
- $\mathrm{C}=$ tabular value
- C1=electrode coeff (Table 8-3)
- $D=$ Leg in sixteenths of inch
- I=length
ecc 10



## Ultimate Strength Method

 EXAMPLE- Using the same example that was applied for the Elastic method:
- Reversing the formula for "D":

$$
D=\frac{\Phi R_{n}}{0.75 * C * C_{1} * l}
$$

- Take $\Phi R_{n}$ as $P_{u}=24^{\text {kip }}$
- $e_{x}=12.611^{\text {in }}$
- $a=e_{x} / l=12.61 / 8=1.58$
- $k=5 / 8=0.625$

- $\quad C=1.3$ (by interpolation from table 8-8)
- $\quad \mathrm{C} 1=1$ (from table $8-3$ )

$$
D=\frac{\Phi R_{n}}{0.75 * C * C_{1} * l}=\frac{24^{k i p}}{0.75 * 1.3 * 1 * 8}=3.077 \approx 3^{+} \rightarrow 4 / 16^{\text {inch }} \text { or } 1 / 4^{\text {inch }}
$$

## Balanced Fillet Welds

E Eccentricity can be generated by designing a weld that has a center of gravity that does not coincide with the centroidal axis of the member:

- Welds that do not satisfy this criterion are called unbalanced fillet weld connections
- Eccentricity introduces a moment to the weld group in addition to the axial force



## Balanced Fillet Welds

- Calculating the Center of Gravity of a weld group
- If all the welds are of the same size the factor " $w$ " can be disregarded

$$
\bar{x}=\frac{\sum \bar{x}_{i}\left(l_{i} * w_{i}\right)}{\sum\left(l_{i} * w_{i}\right)}
$$

- Where
- $\quad l_{i}=$ length of weld
- $w_{i}=$ weld leg
- $x_{i}=$ distance to c.g. of weld


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## Balanced Fillet Welds In Class Example

$\square \quad$ Determine the center of gravity of the weld group and compare to the c.g. of the element to determine if the welds are balanced:

- The c.g. of the member is at $1.67^{\prime \prime}$ (pg. 1-43)
- Solving:

$$
\begin{aligned}
& \bar{x}=\frac{\sum \bar{x}_{i}\left(l_{i} * w_{i}\right)}{\sum\left(l_{i} * w_{i}\right)} \\
& \bar{x}=\frac{13 * 0+6 * 3+3 * 6}{13+6+3}=1.64^{\text {inch }}
\end{aligned}
$$

- Close enough to consider balanced



## BALANCED Fillet Welds

- Per AISC Section J1.7, it is not necessary to balance welds for a statically loaded single angle, double angle, or similar type of element.
$\square \quad$ Not using a balanced weld will result in a more compact connection.
$\square \quad$ For connections that are subject to cyclic loading a balanced weld design is beneficial.


# Bolts Subjected to ECCENTRIC SHEAR 

- Bolted eccentrically loaded connections can be analyzed in a manner similar to the methods of analysis of eccentrically loaded welds.
- In certain ways, the analysis is more evident given the clarity of how forces act upon each of the bolts.



## Bolts Subjected to ECCENTRIC SHEAR

- There are forces that act directly, and forces that are generated from the moment.
- The critical cases are the bolts on the very edge that is closest to the load
- They are subjected to the direct load, but they also have the longest momentarm and the resultant of the moment adds to the direct load




## ECCENTRIC LOAD ON BOLTS, Elastic Method

- Resultant forces caused on each bolt are proportional to the distance from the $\mathrm{cg}, \ldots \rightarrow \quad \frac{r_{1}}{d_{1}}=\frac{r_{2}}{d_{2}}=\frac{r_{3}}{d_{3}}$ etc
- That can lead to solve for moment by saying $r_{1}=\frac{r_{1} * d_{1}}{d_{1}} \quad$ redundant as it may seem but... this leads to $\quad M=\frac{r_{1} * d_{1}^{2}}{d_{1}}+\frac{r_{1} * d_{2}^{2}}{d_{1}}+\frac{r_{1} * d_{3}^{2}}{d_{1}}$ etc
$\square$ Thus $M=\frac{r_{1} * \Sigma d^{2}}{d_{1}}$

- and by reversing the equation for the force $\quad r_{1}=\frac{M * d_{1}}{\Sigma d^{2}}$


## ECCENTRIC LOAD ON BOLTS, Elastic Method

$\square$ As we see the Vertical and Horizontal components of force "r" represented by H and V in the diagram $\rightarrow$

- $\frac{r_{1}}{d_{1}}=\frac{H}{v}$ Which leads to $H=\frac{r_{1} * v}{d_{1}}=\frac{M * v}{\Sigma d^{2}}$
- And similarly,

$$
V=\frac{r_{1} * h}{d_{1}}=\frac{M * h}{\Sigma d^{2}}
$$



## ECCENTRIC LOAD ON BOLTS, Elastic Method Example

- Determine the force in the most stressed bolt

$$
\begin{aligned}
& e c c=9.5^{\text {inch }}+1.5^{\text {inch }}=11^{\text {inch }} \\
& M=P_{u} * e c c=24^{\text {kip }} * 11^{\text {inch }}=22^{k} \\
& \Sigma d^{2}=\Sigma h^{2}+\Sigma v^{2} \\
& \quad h=1.5^{\text {inch }} \\
& v=1.5^{\text {inch }} \text { or } 4.5^{\text {inch }} \text { accordingly } \\
& \Sigma d^{2}=\Sigma h^{2}+\Sigma v^{2}
\end{aligned}
$$



## ECCENTRIC LOAD ON BOLTS, Elastic Method Example

$$
\Sigma d^{2}=8 *(1.5)^{2}+4 *\left(1.5^{2}+4.5^{2}\right)=108
$$

- Taking lower left bolt...

$$
\begin{aligned}
& H=\frac{M * v}{\Sigma d^{2}}=\frac{264^{k^{k n *}} * 4.5^{\text {inch }}}{108^{\text {inch }}}=11^{\text {kip }} \leftarrow \\
& V=\frac{M * h}{\Sigma d^{2}}=\frac{264^{4^{k n *}} * 1.5^{\text {inch }}}{108^{\text {inch }}}=3.67^{\text {kip }} \downarrow \\
& \frac{P_{u}}{n}=\frac{24^{k^{k i}}}{8}=3^{k_{i p}} \downarrow \\
& R=\sqrt{11^{2}+(3.67+3)^{2}}=12.86^{\text {kip }} \quad \Phi R_{n}=17.146^{k i p}
\end{aligned}
$$



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## Instantaneous Center of Rotation Method

- Just like seen in welds, the elastic method is based on the assumption that the fasteners are behaving elastically and that there are no other factors besides the stiffness of the connectors.
- The Instantaneous center of rotation method gives less conservative and more realistic result.
- Using Table 7-7 (for 3") ...the process is much quicker:
- $e_{x}=11^{\text {in }}$
- $s=3^{\text {in }}$
- $\mathrm{n}=4$
$\Phi R_{n}=\frac{P_{u}}{C}=\frac{24^{\text {kip }}}{2.24}=10.71^{\text {kip }}$
- By interpolation C=2.24
- Using Table 7-1 a bolt type can be selected.
- Take Group A $5 / \mathrm{s}^{\prime \prime}$ diameter!

