## Engineering Formula Sheet

## Statistics

## Mean

$\mu=\frac{\sum x_{i}}{n}$
$\mu=$ mean value
$\Sigma \mathrm{x}_{\mathrm{i}}=$ sum of all data values ( $\mathrm{x}_{1}, \mathrm{x}_{2}, \mathrm{x}_{3}, \ldots$ )
$\mathrm{n}=$ number of data values

## Standard Deviation

$$
\sigma=\sqrt{\frac{\sum\left(x_{i}-\mu\right)^{2}}{n}}
$$

$\sigma=$ standard deviation
$x_{i}=$ individual data value $\left(x_{1}, x_{2}, x_{3}, \ldots\right)$
$\mu=$ mean value
$\mathrm{n}=$ number of data values

## Probability

## Frequency

$\mathrm{f}_{\mathrm{x}}=\frac{\mathrm{n}_{\mathrm{x}}}{\mathrm{n}}$
$P_{x}=\frac{f_{x}}{f_{a}}$
$\mathrm{f}_{\mathrm{x}}=$ relative frequency of outcome x
$\mathrm{n}_{\mathrm{x}}=$ number of events with outcome x
$\mathrm{n}=$ total number of events
$\mathrm{P}_{\mathrm{x}}=$ probability of outcome x
$f_{a}=$ frequency of all events

## Binomial Probability (order doesn't matter)

## Independent Events

$\mathrm{P}(\mathrm{A}$ and B and C$)=\mathrm{P}_{\mathrm{A}} \mathrm{P}_{\mathrm{B}} \mathrm{P}_{\mathrm{C}}$
$P(A$ and $B$ and $C)=$ probability of independent events $A$ and $B$ and $C$ occurring in sequence $P_{A}=$ probability of event $A$

## Mutually Exclusive Events

$P(A$ or $B)=P_{A}+P_{B}$
$P(A$ or $B)=$ probability of either mutually exclusive
event A or B occurring in a trial
$\mathrm{P}_{\mathrm{A}}=$ probability of event A
$\Sigma x_{i}=$ sum of all data values ( $x_{1}, x_{2}, x_{3}, \ldots$ )
$P_{k}=\frac{n!\left(p^{k}\right)\left(q^{n-k}\right)}{k!(n-k)!}$
$\mathrm{P}_{\mathrm{k}}=$ binomial probability of k successes in n trials
$\mathrm{p}=$ probability of a success
$q=1-p=$ probability of failure
$k=$ number of successes
$\mathrm{n}=$ number of trials

## Mode

Place data in ascending order.
Mode = most frequently occurring value
If two values occur at the maximum frequency the data set is bimodal.
If three or more values occur at the maximum frequency the data set is multi-modal.

## Median

Place data in ascending order.
If n is odd, median = central value
If n is even, median = mean of two central values
$\mathrm{n}=$ number of data values

## Range

Range $=x_{\text {max }}-x_{\text {min }}$
$x_{\text {max }}=$ maximum data value
$x_{\text {min }}=$ minimum data value
$\mathrm{n}=$ number of data values

## Conditional Probability

$P(A \mid D)=\frac{P(A) \cdot P(D \mid A)}{P(A) \cdot P(D \mid A)+P(\sim A) \cdot P(D \mid \sim A)}$
$P(A \mid D)=$ probability of event $A$ given event $D$
$P(A)=$ probability of event $A$ occurring
$P(\sim A)=$ probability of event $A$ not occurring
$P(D \nmid \sim A)=$ probability of event $D$ given event $A$ did not occur

## Plane Geometry



## Triangle

Area $=1 / 2$ bh
$a^{2}=b^{2}+c^{2}-2 b c \cdot \cos \angle A$
$\mathrm{b}^{2}=\mathrm{a}^{2}+\mathrm{c}^{2}-2 \mathrm{ac} \cdot \cos \angle \mathrm{B}$
$c^{2}=a^{2}+b^{2}-2 a b \cdot \cos \angle C$


## Rectangle

Perimeter $=2 a+2 b$
Area $=a b$



## Regular Polygons

Area $=\mathrm{n} \frac{\mathrm{s}\left(\frac{1}{2} \mathrm{f}\right)}{2}$

$\mathrm{n}=$ number of sides

## Trapezoid

Area $=1 / 2(a+b) h$


## Sphere

Volume $=\frac{4}{3} \pi r^{3}$
Surface Area $=4 \pi r^{2}$


## Rectangular Prism

Volume = wdh
Surface Area $=2(w d+w h+d h)$


## Right Circular Cone

Volume $=\frac{\pi r^{2} h}{3}$
Surface Area $=\pi r \sqrt{r^{2}+h^{2}}$


## Pyramid

Volume $=\frac{A h}{3}$
$\mathrm{A}=$ area of base


| Cylinder |  |  |
| :--- | :--- | :--- |
| Volume $=\pi r^{2} h$ <br> Surface Area $=2 \pi r h+2 \pi r^{2}$ |  |  |

## Irregular Prism

Volume $=\mathrm{Ah}$
$A=$ area of base

## Constants

$$
\begin{aligned}
\mathrm{g} & =9.8 \mathrm{~m} / \mathrm{s}^{2}=32.27 \mathrm{ft} / \mathrm{s}^{2} \\
\mathrm{G} & =6.67 \times 10^{-11} \mathrm{~m}^{3} / \mathrm{kg} \cdot \mathrm{~s}^{2} \\
\pi & =3.14159
\end{aligned}
$$

## Conversions



## SI Prefixes

| Numbers Less Than One |  |  | Numbers Greater Than One |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Power of 10 | Prefix | Abbreviation | Power of 10 | Prefix | Abbreviation |
| $10^{-1}$ | deci- | d | $10^{1}$ | deca- | da |
| $10^{-2}$ | centi- | C | $10^{2}$ | hecto- | h |
| $10^{-3}$ | milli- | m | $10^{3}$ | kilo- | k |
| $10^{-6}$ | micro- | $\mu$ | $10^{6}$ | Mega- | M |
| $10^{-9}$ | nano- | n | $10^{9}$ | Giga- | G |
| $10^{-12}$ | pico- | p | $10^{12}$ | Tera- | T |
| $10^{-15}$ | femto- | f | $10^{15}$ | Peta- | P |
| $10^{-18}$ | atto- | a | $10^{18}$ | Exa- | E |
| $10^{-21}$ | zepto- | z | $10^{21}$ | Zetta- | Z |
| $10^{-24}$ | yocto- | y | $10^{24}$ | Yotta- | Y |

Equations

| Mass and Weight |
| :--- |
| $M=V D_{m}$ |
| $W=m g$ |
| $W=V D_{w}$ |
| $V=$ volume |
| $D_{m}=$ mass density |
| $m=$ mass |
| $D_{w}=$ weight density |
| $g=$ acceleration due to gravity |

## Temperature

$\mathrm{T}_{\mathrm{K}}=\mathrm{T}_{\mathrm{C}}+273$
$\mathrm{T}_{\mathrm{R}}=\mathrm{T}_{\mathrm{F}}+460$
$T_{F}=\frac{5}{9} T_{C}+32$
$\mathrm{T}_{\mathrm{K}}$ = temperature in Kelvin
$\mathrm{T}_{\mathrm{C}}=$ temperature in Celsius
$\mathrm{T}_{\mathrm{R}}$ = temperature in Rankin
$\mathrm{T}_{\mathrm{F}}=$ temperature in Fahrenheit

## Force

$\mathrm{F}=\mathrm{ma}$
F = force
$\mathrm{m}=$ mass
$\mathrm{a}=$ acceleration

## Equations of Static Equilibrium

$\Sigma \mathrm{F}_{\mathrm{x}}=0 \quad \Sigma \mathrm{~F}_{\mathrm{y}}=0 \quad \Sigma \mathrm{M}_{\mathrm{p}}=0$
$\mathrm{F}_{\mathrm{x}}=$ force in the x -direction
$F_{y}=$ force in the $y$-direction
$\mathrm{M}_{\mathrm{P}}=$ moment about point P

## Equations (Continued)

| Energy: Work |
| :--- |
| $\mathrm{W}=\mathrm{F}_{\\| 1} \cdot \mathbf{d}$ |
| $\mathrm{~W}=$ work |
| $\mathrm{F}_{\\| l}=$ force parallel to direction of |
| displacement |
| $\mathrm{d}=$ displacement |


| Power |
| :--- |
| $P=\frac{E}{t}=\frac{W}{t}$ |
| $P=\frac{\tau \cdot r p m}{5252}$ |
| $P=$ power |
| $E=$ energy |
| $W=$ work |
| $t=$ time |
| $\tau=$ torque |
| $r p m=$ revolutions per minute |


| Efficiency <br> Efficiency $(\%)=\frac{P_{\text {out }}}{P_{\text {in }}} \cdot 100 \%$ |
| :--- |
| $P_{\text {out }}=$ useful power output <br> $P_{\text {in }}=$ total power input |
| Energy: Potential <br> $U=$ mgh |
| m = potential energy <br> $g=$ acceleration due to gravity <br> $h=$ height |


| Energy: Kinetic |
| :--- |
| $\mathrm{K}=\frac{1}{2} \mathrm{mv}^{2}$ |
| $\mathrm{~K}=$ kinetic energy |
| $\mathrm{m}=$ mass |
| $\mathrm{V}=$ velocity |


| Energy: Thermal |
| :--- |
| $Q=m c \Delta T$ |
| $Q=$ thermal energy |
| $m=$ mass |
| $c=$ specific heat |
| $\Delta T=$ change in temperature |

## Fluid Mechanics

$p=\frac{F}{A}$
$\frac{V_{1}}{T_{1}}=\frac{V_{2}}{T_{2}} \quad$ (Charles' Law)
$\frac{p_{1}}{T_{1}}=\frac{\mathrm{p}_{2}}{\mathrm{~T}_{2}}$ (Gay-Lussanc's Law)
$\mathrm{p}_{1} \mathrm{~V}_{1}=\mathrm{p}_{2} \mathrm{~V}_{2} \quad$ (Boyle's Law)
$\mathrm{Q}=\mathrm{A} \mathrm{v}$
$\mathrm{A}_{1} \mathrm{~V}_{1}=\mathrm{A}_{2} \mathrm{~V}_{2}$
Horsepower $=\frac{\mathrm{Qp}}{1714}$
absolute pressure = gauge pressure

+ atmospheric pressure
$\mathrm{p}=$ absolute pressure
$\mathrm{F}=$ Force
A = Area
$\mathrm{V}=$ volume
$\mathrm{T}=$ absolute temperature
$\mathrm{Q}=$ flow rate
$\mathrm{v}=$ flow velocity

$$
\begin{array}{l|l}
\hline \text { Mechanics } \\
\overline{\mathrm{s}}=\frac{\mathrm{d}}{\mathrm{t}} \\
\overline{\mathrm{v}}=\frac{\Delta \mathrm{d}}{\Delta \mathrm{t}} \\
\mathrm{a}=\frac{v_{\mathrm{f}}-v_{i}}{\mathrm{t}} \\
\mathrm{X}=\frac{v_{i}^{2} \sin (2 \theta)}{-g}
\end{array} \quad\left\{\begin{array}{l}
\mathrm{P}=\mathrm{Q}^{\prime}=\mathrm{AU} \Delta \mathrm{~T} \\
\mathrm{P}=\frac{\mathrm{Q}}{\Delta \mathrm{t}} \\
\mathrm{U}=\frac{1}{\mathrm{R}}=\frac{\mathrm{k}}{\mathrm{~L}} \\
\mathrm{P}=\frac{\mathrm{kA} \Delta \mathrm{~T}}{\mathrm{~L}} \\
\mathrm{~A}_{1} \mathrm{v}_{1}=\mathrm{A}_{2} \mathrm{v}_{2} \\
\mathrm{P}_{\mathrm{net}}=\sigma \mathrm{Ae}\left(\mathrm{~T}_{2}^{4}-\mathrm{T}_{1}^{4}\right)
\end{array}\right.
$$

$$
\mathrm{v}=\mathrm{v}_{0}+\mathrm{at}
$$

$$
d=d_{0}+v_{0} t+1 / 2 a t^{2}
$$

$$
v^{2}=v_{0}^{2}+2 a\left(d-d_{0}\right)
$$

$$
\tau=\mathrm{dF} \sin \theta
$$

$\overline{\mathrm{s}}=$ average speed
$\overline{\mathbf{v}}=$ average velocity
$\mathrm{v}=$ velocity
$\mathrm{a}=$ acceleration
X = range
$\mathrm{t}=\mathrm{time}$
$\Delta \mathbf{d}=$ change in displacement
d = distance
$g=$ acceleration due to gravity
$\theta=$ angle
$\tau=$ torque

## Electricity

Ohm's Law
$\mathrm{V}=\mathrm{IR}$
$P=I V$
$R_{T}$ (series) $=R_{1}+R_{2}+\cdots+R_{n}$
$\mathrm{R}_{\mathrm{T}}($ parallel $)=\frac{1}{\frac{1}{R_{1}}+\frac{1}{R_{2}}+\cdots+\frac{1}{R_{n}}}$

## Kirchhoff's Current Law

$\mathrm{I}_{\mathrm{T}}=\mathrm{I}_{1}+\mathrm{I}_{2}+\cdots+\mathrm{I}_{\mathrm{n}}$

$$
\text { or } \mathrm{I}_{\mathrm{T}}=\sum_{\mathrm{k}=1}^{\mathrm{n}} \mathrm{I}_{\mathrm{k}}
$$

## Kirchhoff's Voltage Law

$\mathrm{V}_{\mathrm{T}}=\mathrm{V}_{1}+\mathrm{V}_{2}+\cdots+\mathrm{V}_{\mathrm{n}}$

$$
\text { or } \quad V_{T}=\sum_{k=1}^{n} V_{k}
$$

$\mathrm{V}=$ voltage
$\mathrm{V}_{\mathrm{T}}=$ total voltage
I = current
$\mathrm{I}_{\mathrm{T}}=$ total current
$R=$ resistance
$\mathrm{R}_{\mathrm{T}}=$ total resistance
$\mathrm{P}=$ power

## Thermodynamics

$\mathrm{P}=$ rate of heat transfer
$Q=$ thermal energy
A = Area of thermal conductivity
$U=$ coefficient of heat conductivity
(U-factor)
$\Delta T=$ change in temperature
$\Delta t=$ change in time
$\mathrm{R}=$ resistance to heat flow ( R-value)
$\mathrm{k}=$ thermal conductivity
$\mathrm{v}=$ velocity
$P_{\text {net }}=$ net power radiated
$\sigma=5.6696 \times 10^{-8} \frac{\mathrm{~W}}{\mathrm{~m}^{2} \cdot \mathrm{k}^{4}}$
e = emissivity constant
$\mathrm{L}=$ thickness

## Section Properties

## Moment of Inertia <br> $I_{x x}=\frac{b h^{3}}{12}$ <br> 

$\mathrm{I}_{\mathrm{xx}}=$ moment of inertia of a rectangular section about $x$ - $x$ axis

## Complex Shapes Centroid

$\bar{x}=\frac{\sum x_{i} A_{i}}{\sum A_{i}}$ and $\bar{y}=\frac{\sum y_{i} A_{i}}{\sum A_{i}}$
$\bar{x}=x$-distance to the centroid
$\bar{y}=y$-distance to the centroid
$x_{i}=x$ distance to centroid of shape $i$
$y_{i}=y$ distance to centroid of shape $i$
$\mathrm{A}_{\mathrm{i}}=$ Area of shape i

| Material Properties |
| :--- |
| Stress (axial) <br> $\sigma=\frac{\mathrm{F}}{\mathrm{A}}$ <br> $\sigma=$ stress <br> $\mathrm{F}=$ axial force <br> $\mathrm{A}=$ cross-sectional area |

## Strain (axial)

$\epsilon=\frac{\delta}{L_{0}}$
$\epsilon=$ strain
$\mathrm{L}_{0}=$ original length
$\delta=$ change in length

## Modulus of Elasticity

$E=\frac{\sigma}{\varepsilon}$
$\mathrm{E}=\frac{\left(\mathrm{F}_{2}-\mathrm{F}_{1}\right) \mathrm{L}_{0}}{\left(\delta_{2}-\delta_{1}\right) \mathrm{A}}$
$\mathrm{E}=$ modulus of elasticity
$\sigma=$ stress
$\varepsilon=$ strain
$\mathrm{A}=$ cross-sectional area
$\mathrm{F}=$ axial force
$\delta=$ deformation

## Deformation: Axial

$\delta=\frac{\mathrm{FL}_{0}}{\mathrm{AE}}$
$\delta=$ deformation
F = axial force
$\mathrm{L}_{0}=$ original length

## Rectangle Centroid

$\bar{x}=\frac{b}{2}$ and $\bar{y}=\frac{h}{2}$


Right Triangle Centroid
$\bar{x}=\frac{b}{3}$ and $\bar{y}=\frac{h}{3}$


## Semi-circle Centroid

$\bar{x}=r$ and $\bar{y}=\frac{4 r}{3 \pi}$

$\bar{x}=x$-distance to the centroid
$\bar{y}=y$-distance to the centroid

## Structural Analysis

| Beam Formulas |  |  |
| :---: | :---: | :---: |
|  | Reaction <br> Moment <br> Deflection | $\begin{gathered} R_{A}=R_{B}=\frac{P}{2} \\ \mathrm{M}_{\max }=\frac{\mathrm{PL}}{4} \text { (at point of load) } \\ \Delta_{\max }=\frac{\mathrm{PL}}{48 \mathrm{E}} \text { (at point of load) } \end{gathered}$ |
|  | Reaction <br> Moment <br> Deflection | $\begin{aligned} & R_{A}=R_{B}=\frac{\omega L}{2} \\ & M_{\max }=\frac{\omega L^{2}}{8} \quad \text { (at center) } \\ & \Delta_{\max }=\frac{5 \omega L^{4}}{384 L I} \quad \text { (at center) } \end{aligned}$ |
|  | Reaction Moment <br> Deflection | $\begin{aligned} & R_{A}=R_{B}=P \\ & M_{\max }=P a \\ & \left.\Delta_{\max }=\frac{P a}{24 E I}\left(3 L^{2}-4 a^{2}\right) \quad \text { (at center }\right) \end{aligned}$ |
|  | Reaction <br> Moment <br> Deflection | $\begin{aligned} & R_{A}=\frac{P b}{L} \text { and } R_{B}=\frac{P a}{L} \\ & M_{\max }=\frac{\mathrm{Pab}}{L} \quad(a t \text { Point of Load) } \\ & \Delta_{\max }=\frac{\operatorname{Pab}(a+2 b) \sqrt{3 a(a+2 b)}}{27 \mathrm{EL}} \\ & \left(\text { at } x=\sqrt{\frac{a(a+2 b)}{3}} \text { when } a>b\right) \end{aligned}$ |

A = cross-sectional area
$\mathrm{E}=$ modulus of elasticity

## Truss Analysis

$2 J=M+R$
$J=$ number of joints
M =number of members
$R=$ number of reaction forces

## Simple Machines

## Mechanical Advantage (MA)




Wheel and Axle


Effort at Wheel


## Pulley Systems

IMA = Total number of strands of a single string supporting the resistance

IMA $=\frac{D_{E}(\text { string pulled })}{D_{R}(\text { resistance lifted })}$


## Wedge

IMA $=\frac{L(\perp \text { to height })}{H}$


## Screw



$$
\text { Pitch }=\frac{1}{\mathrm{TPI}}
$$

Pitch

$C=$ Circumference
$r=$ radius
Pitch = distance between threads
TPI = Threads Per Inch

## Compound Machines

$M A_{\text {TOTAL }}=\left(\mathrm{MA}_{1}\right)\left(\mathrm{MA}_{2}\right)\left(\mathrm{MA}_{3}\right) \ldots$

Gears; Sprockets with Chains; and Pulleys with Belts Ratios

$$
\begin{aligned}
& G R=\frac{N_{\text {out }}}{N_{\text {in }}}=\frac{d_{\text {out }}}{d_{\text {in }}}=\frac{\omega_{\text {in }}}{\omega_{\text {out }}}=\frac{T_{\text {out }}}{T_{\text {in }}} \\
& \frac{d_{\text {out }}}{d_{\text {in }}}=\frac{\omega_{\text {in }}}{\omega_{\text {out }}}=\frac{T_{\text {out }}}{T_{\text {in }}} \text { (pulleys) }
\end{aligned}
$$

## Compound Gears

$\mathrm{GR}_{\text {TOTAL }}=\left(\frac{\mathrm{B}}{\mathrm{A}}\right)\left(\frac{\mathrm{D}}{\mathrm{C}}\right)$


[^0]
[^0]:    GR = Gear Ratio
    $\omega_{\text {in }}=$ Angular Velocity - driver
    $\omega_{\text {out }}=$ Angular Velocity - driven
    $\mathrm{N}_{\text {in }}=$ Number of Teeth - driver
    $\mathrm{N}_{\text {out }}=$ Number of Teeth - driven
    $\mathrm{d}_{\text {in }}=$ Diameter - driver
    $\mathrm{d}_{\text {out }}=$ Diameter - driven
    $\mathrm{T}_{\text {in }}=$ Torque - driver
    $\mathrm{T}_{\text {out }}=$ Torque - driven

