

# ISS-CREAM Thermal and Fluid System Design and Analysis

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## Topics

- Introduction
- ISS-CREAM Overview
- Analysis and Design Overview
- Active Thermal Control System (ATCS) Analysis and Design
  - Coldplates
  - SCD
  - System Level
- Midsection, Structure, and Instrument Level Model
- Planned Verification / Testing
- Primary focus for TFAWS is thermal fluid analysis and design, not ISS-CREAM description



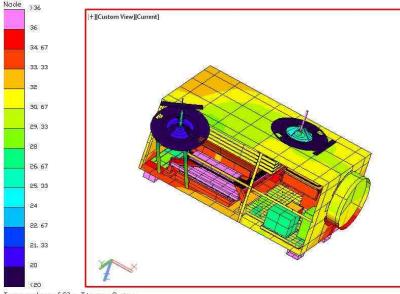
ISS-CREAM (Cosmic Ray Energetics And Mass for the International Space Station)

#### **Physical Characteristics**

Payload total mass: 1392 kg Payload Dimensions: 1.85 m x 0.95 m x 1 m (size of a refrigerator) Launch scheduled in 2016 Nominal data collection of 3 years on International Space Station (ISS) Kibo/Japanese Experiment Module (JEM) Exposed Facility (EF)

#### **ISS JEM EF Provided Services**

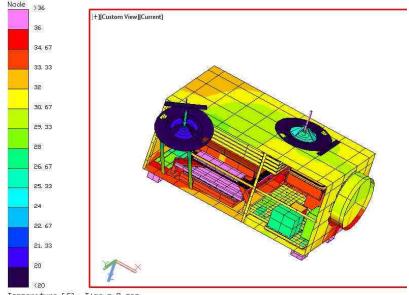
580 W operational power120 W survival heater power200 kg/hr Fluorinert supply at 16-24CData



Temperature [C], Time = 0 sec cream-eol-july/cream-jun15-for-iss-eol-hot-flightmode, sav

### **ISS-CREAM** Payload

- Primarily cooled by JEM EF fluid
- ISS payload on JEM EF location
  - Complex ISS models and ISS I/F
- Complex payload in small package
  - 5 Instruments (Calorimeter System, SCD, TCD, BCD, BSD)
  - 7 electronics boxes
- Previously flown balloon hardware + enhanced sensor suite
- Detectors and electronics built by variety of organizations: University of Maryland, NASA Goddard, South Korean universities (see final slide)
- Low cost project (\$20M)
- NPR 7120.8 / Do No Harm payload = Technical risk is high
- Only Calorimeter and HPDs and SCD are considered primary instruments
- Intent is to provide years of cosmic ray science



Temperature [C], Time = 0 sec cream-eol-july\cream-jun15-for-iss-eol-hot-flightmode.sav



## **ISS-CREAM Description / Schedule**

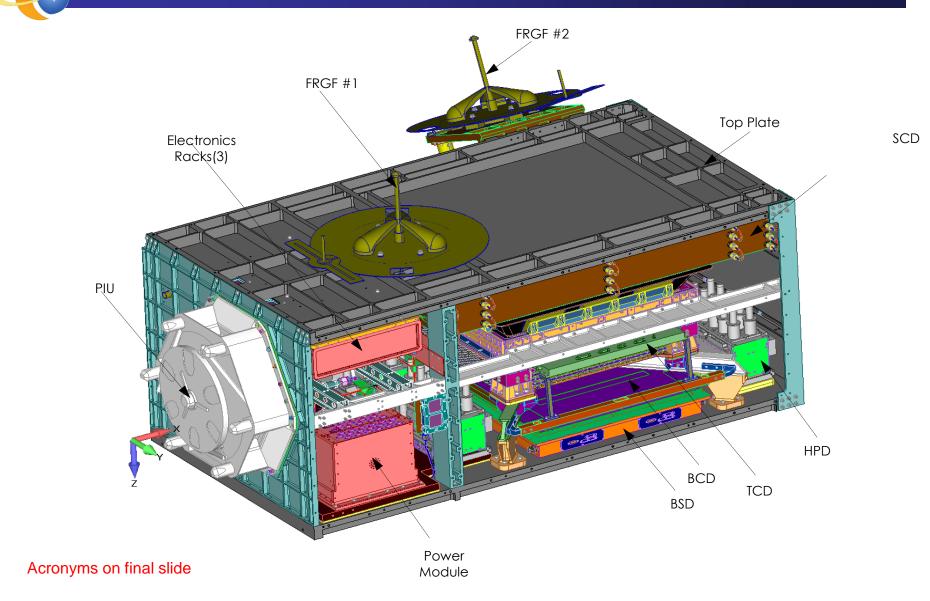
- ISS-CREAM, science objectives, and ISS JEM EF are documented on several websites available on the Internet including:
  - <u>http://cosmicray.umd.edu/iss-cream/</u>
  - <u>https://directory.eoportal.org/web/eoportal/satellite-missions/i/iss-cream</u>
  - <u>https://directory.eoportal.org/web/eoportal/satellite-missions/i/iss-jemef</u>

as well as many papers on both ISS-CREAM and prior CREAM balloon flights (Many graphics and ISS-CREAM descriptions were taken from these sites)

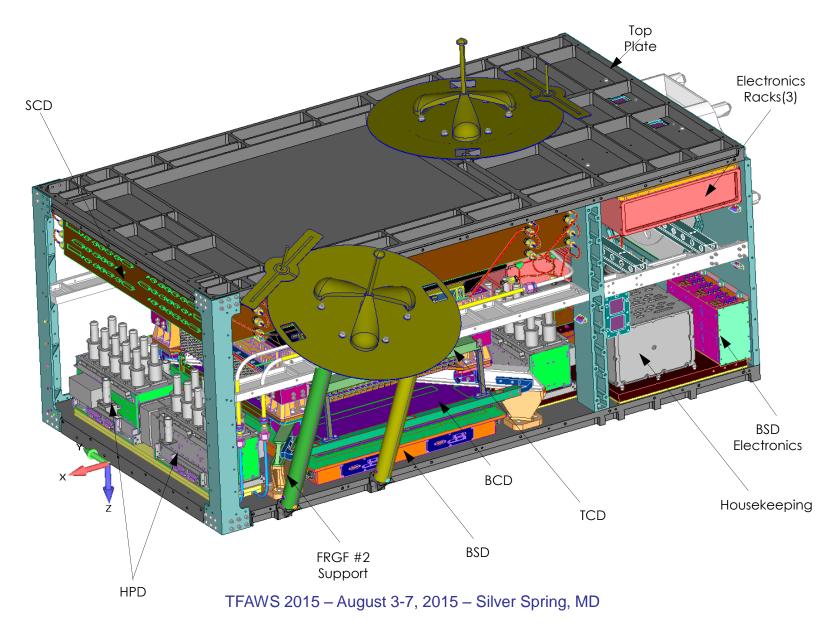
#### Current Schedule:

- ISS-CREAM was selected in response to a ROSES-10 proposal to repackage a balloon-borne instrument for accommodation on the ISS/JEMEF around 2012.
- CDR September 2013
- Extensive thermal redesign by SGT / Bastion team under GSFC MSES contract
- Delta-CDR for thermal design in June 2014
- Flow testing and ambient testing of payload from January 2015 to June 2015
- Tvac scheduled for July 2015
- Launch in 2016

### **ISS-CREAM** Payload (+Y view)

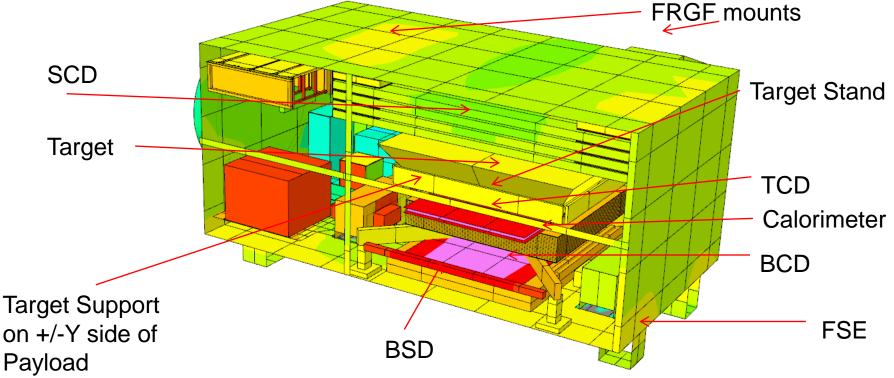


### **ISS-CREAM Payload (-Y view)**

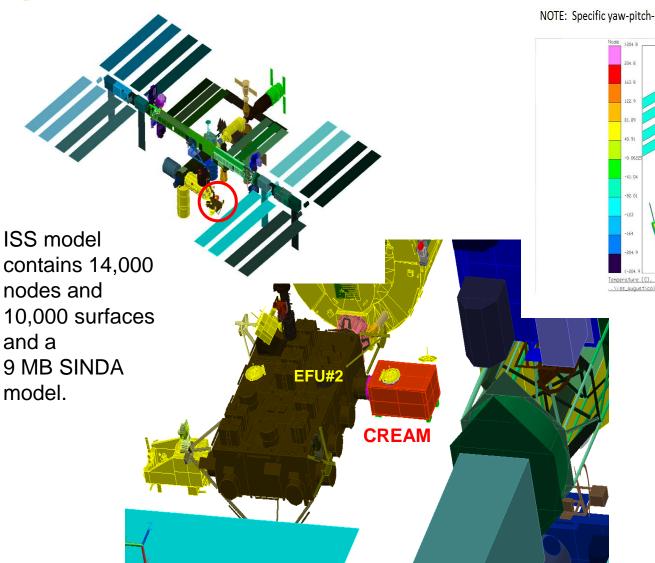


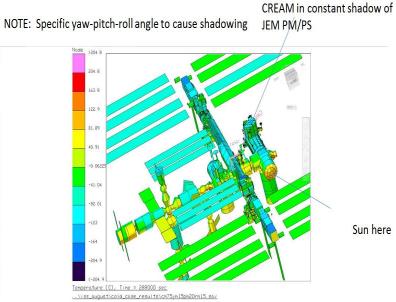
### **Passive Thermal Design**

- ISS-CREAM exists in ISS JEM EF environment, so despite full blanketing, environment does impact both hot and cold cases
  - 1392 kg mass limits orbital swings
  - PIU connects CREAM to JEM EF, some conduction losses
  - FSEs (SpaceX feet) have high a/e coating, some heat input/output
  - FRGFs (grapple fixtures, one on diving board structure) provide another heat leak
  - Midsection is passively cooled through connection to bottom plate
  - SCD has 8 large bolt connections to top plate
  - · Coldplates isolated from structure with MLI



#### ISS Thermal Model with CREAM Connected to JEM EF Module



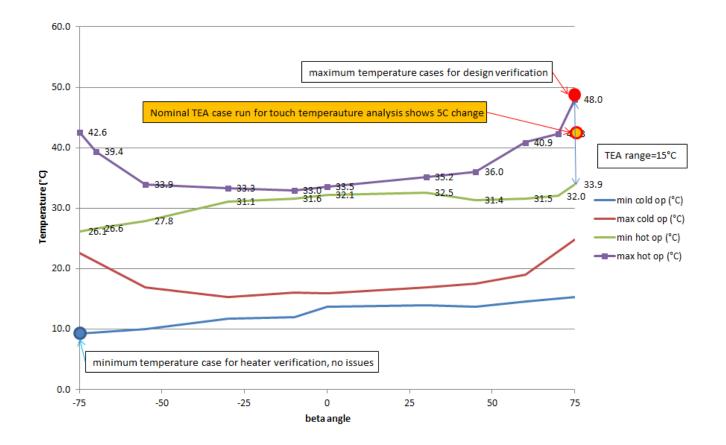


Coldest environments have shadowing by JAXA PM/PS module

Hottest environments are due to direct solar for extended periods

### **Beta / TEA Effect on Bottom Plate**

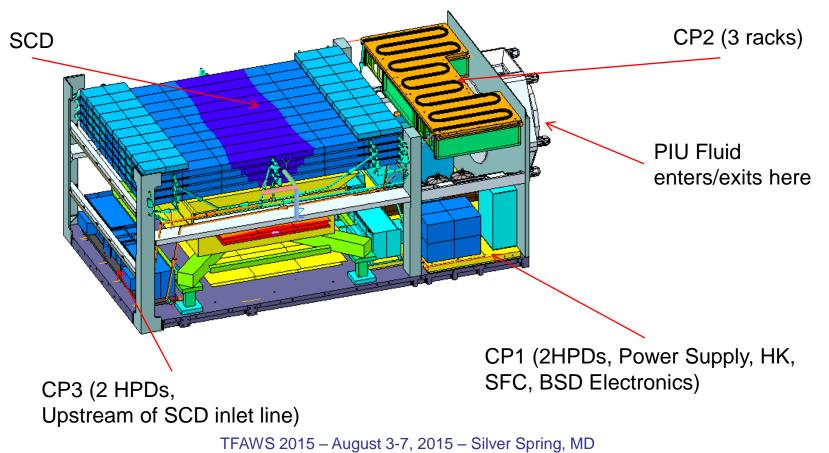
**Bottom Plate Temperature from JAXA analysis** 



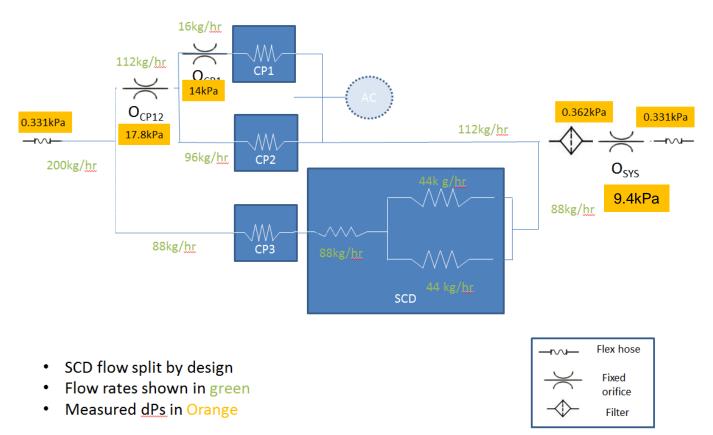
Surveyed 100s of Beta and Torque Equilibrium Attitude (TEA) combinations to determine worst cases

## **Active Thermal Design**

- ATCS provides cooling for most CREAM components and mitigation of some extreme hot and cold environments
- ATCS contains 3 coldplates in parallel. Coldplate 3 outlet flows into SCD assembly (20 bores in SCD plates). Orifices are used to distribute flow as required thermally.
- SCD was most critical component during ATCS design, complete redesign with optimum plumbing layout allowed flow distribution to be changed to maximize flow in CP1



### **ATCS Schematic**



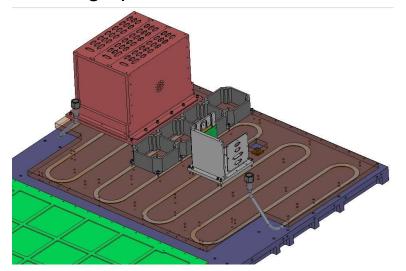
- ATCS Requirements include overall pressure drop (52-58 kPa), volume and some thermal isolation (fluid not to be used as a heater)
- Thermal requirement to keep all cooled elements in operational range
- Flow split adjustment through orifices is required to get adequate cooling on all three paths
- Packaging

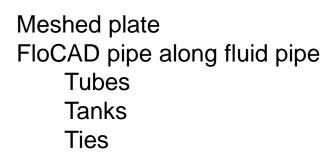


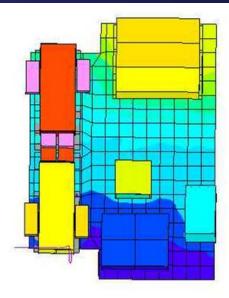
- FLUINT representation of entire CREAM fluid system / ATCS
- Coldplate 1 model with 6 reduced electronics box models
- Coldplate 2 models with 3 reduced electronics box models
- Coldplate 3 model with 2 reduced electronics box models
- Reduced SCD model
- FloCAD used to create a FLUINT model on full CREAM CAD model including pipe runs
  - Pipe runs designed with mechanical designer in CAD, pressure drop, volume constraints considered along with packaging and mounting options
- All plumbing runs include filter, flex hose, Gamah fittings, bends modeled using loss coefficient (FK) method
- Flow split
  - design optimized with MFRSET,
  - replace with orifice FKs based on sensitivity analysis,
  - replaced with as-built tested FKs

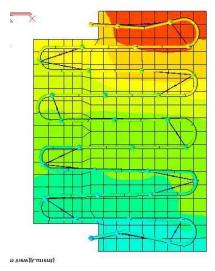
### **Coldplate 1**

Traditional cold plate Several mounted electronics boxes High power ~190W





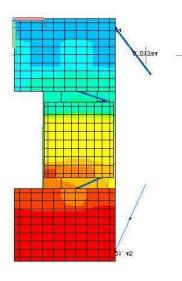


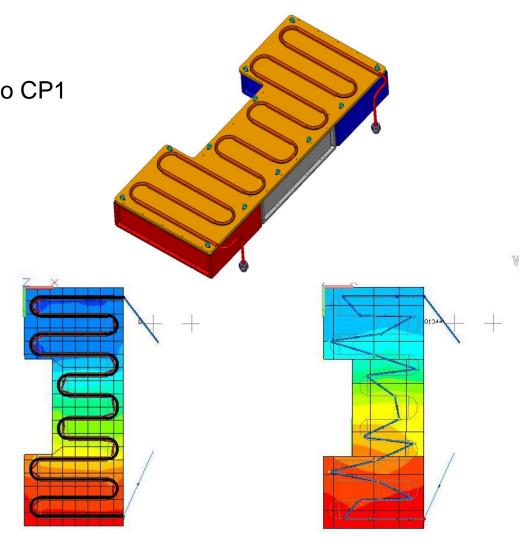




### **Coldplate 2**

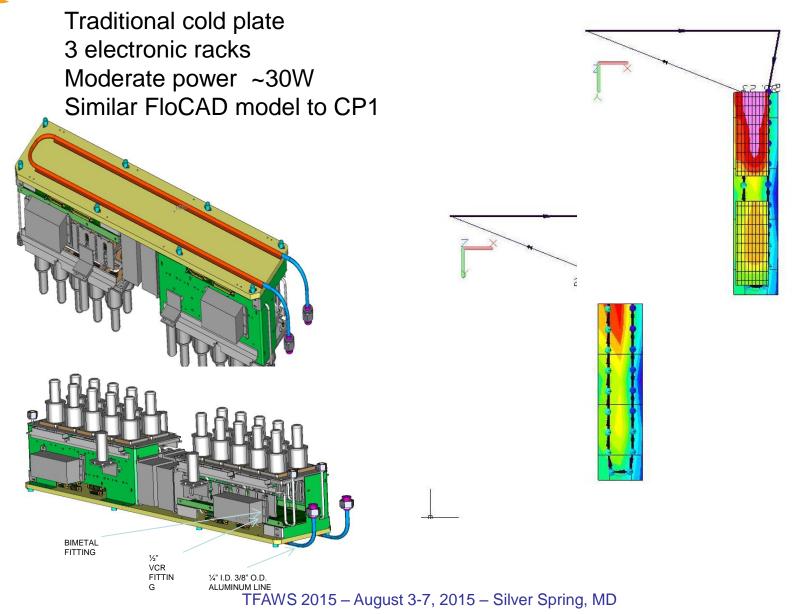
Traditional cold plate 3 electronic racks Low power ~20W Similar FloCAD model to CP1







### **Coldplate 3**



### **CP** models with integrated box models



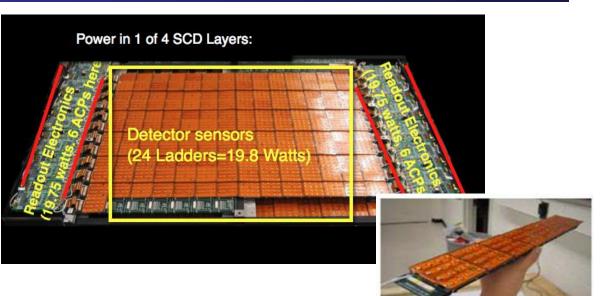
Temperature [C], Time = 0 sec

cream-eol-july/cream-jun15-for-iss-eol-hot-flightmode.sav

### Coldplate 1 and 3

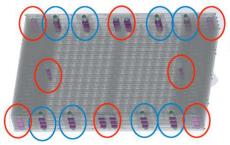
### **SCD Assembly**

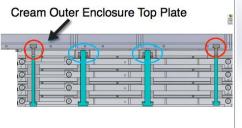
- Four major plate assemblies and cover
- Extensively 3-D machined aluminum plates
- 672 silicon sensors, low power, complex attachment, <40C for noise</li>
- 48 electronics cards
- Instrument bolts tie plates together
- Fluid system has 20 bores, 5 on each major plate assembly

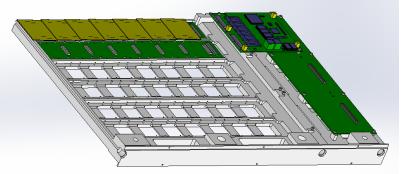


#### Ladder with 7 detectors

#### Attachment to CREAM top plate



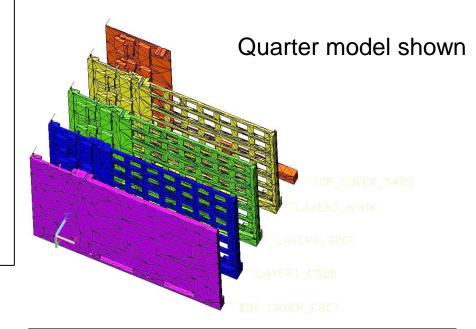


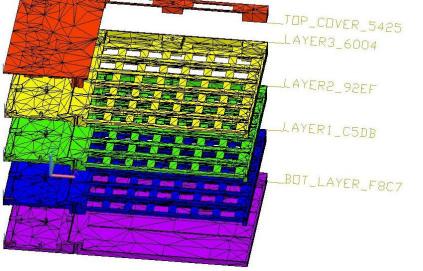


Machined plate with ladders and cards

### **SCD Plate and Cover Thermal Models**

- Unusual thermal design, hard predict heat flows
  - Light-weighting in prior design
- Finite Elements based on CAD solid
  - CAD model greatly simplified in Inventor
  - TDmesher used for meshing
  - eliminate details (reduce material= conservative) until mesh works
- 3000+ nodes per plate and cover





Resulting Model Includes:

Complex conduction in plate due to ladder details

Complex conduction in ACP area due to the pads to pick up heat from lower layers Bolt contact areas

Cover details

### **Bore locations**

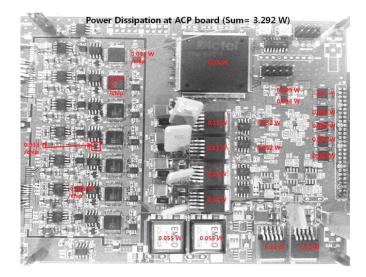
### **SCD Sensor / Ladders Thermal Model**

chips **Complex Plate** supports FEM model analog board Created modular model Duplicated as needed with submodel or numbering changes Use AutoCAD **External References** (Xref) to bring in various layers Xref-ed files are equivalent to INSERT files, but contain full

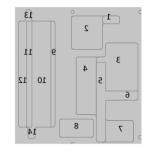
TD models that can be

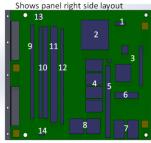
setup to run independently

### **SCD Electronics**



#### SUB & ACP board

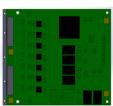


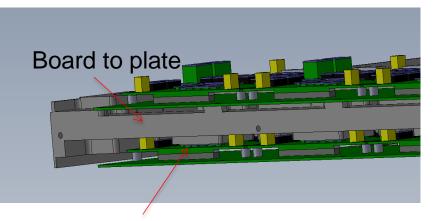


There are intrusions on structure to make contact to electronics chips.

Blue areas are <u>Therm</u>-A-Gap 579 that will be placed between intrusions and electronics chips.

The intrusions and the chips with same number will be contacted.

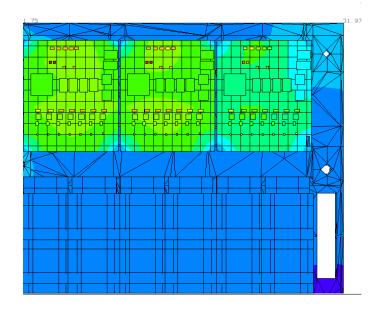


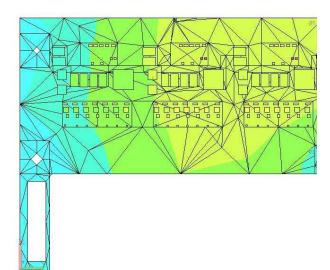


### Components to Plate highs

### **SCD ACP Thermal Model**

- Each chip is represented by a TD rectangle with applied heat load (size from photo or CAD)
- Chip is tied to card either dry or with solder
- Chip is tied to highs on cover (or plate above) with Therm-a-gap (chip area) 50% of published interface conductance = 1.4W/in2-C
- Cards tied to plate with 4 screws into solid at corners

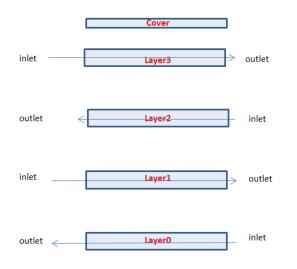




Top View ACP chips on L3 cards and plate Bottom View ACP chips on cover highs (card removed for visibility)

## Fluid Cooling of SCD

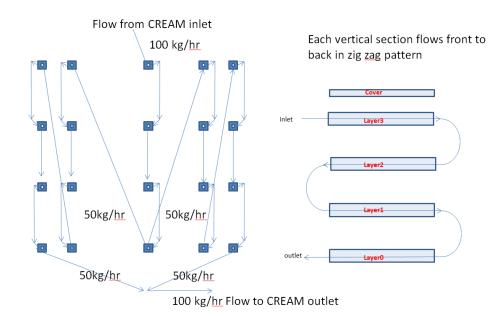
Original design had 20 parallel paths in 5 stacks like this



#### Laminar Flow

- Reynolds number < 500
- H = 54 W/m2-K
- Manifolds on both sides
- Flow balancing of 20 legs near impossible, backflow
- Very low pressure drop (use orifices to meet dP spec)

### Final design uses 2 parallel paths



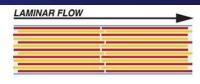
#### Turbulent Flow

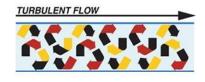
- Center Channel Re≈12,000
- H > 1000 W/m2-K
- Side Channel Re  $\approx$  6,000
- H> 600 W/m2-K
- Flow balance 2 legs (not critical)
- Pressure drop within range of requirement (35kPa vs.. 50kPa for system)

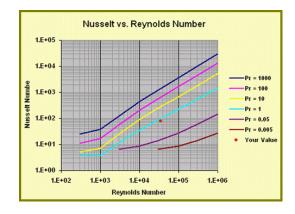
Temperature of hottest sensors dropped 10C and met specification TFAWS 2015 – August 3-7, 2015 – Silver Spring, MD

### **Fluorinert Cooling and flow regime**

- Laminar flow (CDR design) for 0.18" ID pipe
  - Fluorinert low conductivity (k) yields low laminar convection coefficient
  - H=3.66 k/D (Nu=3.66)
  - H(Fluorinert) = 54 W/m2-K (k=0.057 W/m-K)
  - H(Water) = 550 W/m2-K (k=0.58 W/m-K)
  - H(liquid ammonia) = 450 W/m2-K (k=0.48 W/m-K)
- Turbulent flow (dCDR design)
  - Re = 6000 (side channels), 12000 (center channel)
  - Pr=12.35 (water=7, liquid ammonia=1.24)
  - Cooling n=0.3
  - Nu (Re=6000) = 51.5
  - H(Fluorinert) = 690 W/m2-K
  - H(Water) = 5900 W/m2-K
  - H(liquid ammonia) =2900 W/m2-K







#### Dittus-Boelter equation [edit]

The Dittus-Boelter equation (for turbulent flow) is an explicit function for calculating commercial applications) is cautioned. The Dittus-Boelter equation is:

$$\mathrm{Nu}_D = 0.023 \operatorname{Re}_D^{4/5} \operatorname{Pr}^n$$

where:

 $\boldsymbol{D}$  is the inside diameter of the circular duct

#### $P_{\boldsymbol{r}}$ is the Prandtl number

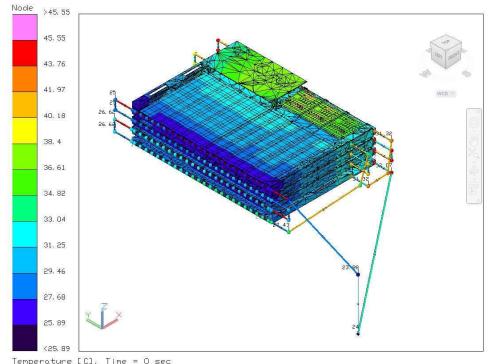
n=0.4 for heating of the fluid, and n=0.3 for cooling of the fluid.<sup>[1]</sup>

The Dittus-Boelter equation is valid for <sup>[3]</sup>

$$\begin{array}{l} 0.6 \leq \Pr \leq 160 \\ \operatorname{Re}_D \gtrsim 10\,000 \\ \frac{L}{D} \gtrsim 10 \end{array}$$

### **SCD Detailed Thermal Model Buildup**

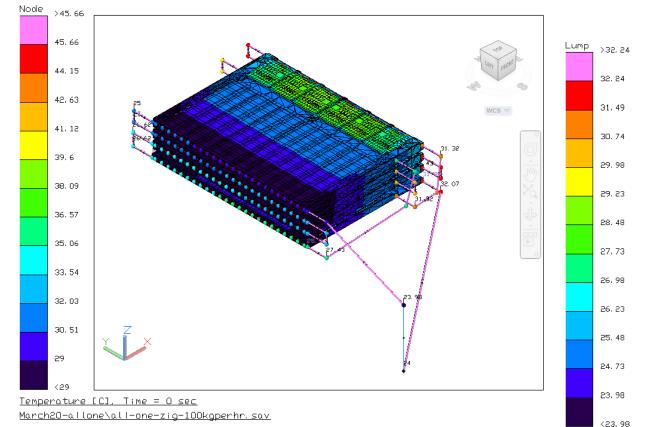
- Build up each layer quarter model with plate solid, ACP model, sensor ladder model. Add interface features for bolts, cards, fluid bore internal area (Tagsets).
- Top cover is included in Layer3 model
- Make clone of each layer quarter model
- Xref used to bring in these 8 models.
- FloCAD used to create fluid loop to plenum



March20-allone\all-one-zig-100kgperhr.sav

### **SCD Fluid Modeling**

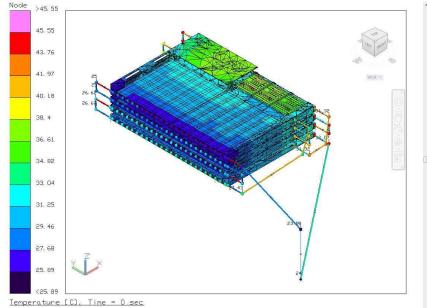
- FloCAD used to depict fluid network in Thermal Desktop model
- FLUINT Used for all flow calculations
- Use pipe representation inside each bore
- Model pipe bends with appropriate FKs
- Use MFRSET to determine flow imbalance pressure drop, design two legs to get good distribution
- Since this is a half model, use duplication factors



Temperature [C], ]



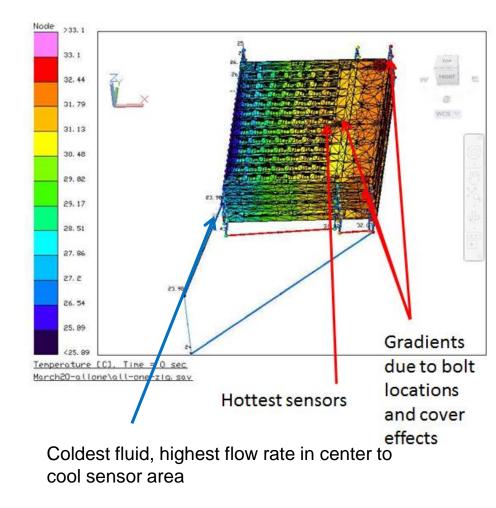
- Majority (67%) of power in ACP sections
  - Majority of power transferred through Therm-a-gap to highs of plate above or cover
  - Heat transfer to card, on card and to circuit board screws is tortuous
  - Majority of heat in this section transferred to fluid in two bores in each side
- Sensor power (33%) distributed on large section of plate through ladder and plate details, fairly low heat flux
  - Plate conduction and analog boards limit hot spots
  - Fluid transfers heat along fluid path
  - Majority of heat in this section transferred to center bore fluid
  - High flow rate, cold fluid (inlet supply) preferentially cools the sensor section



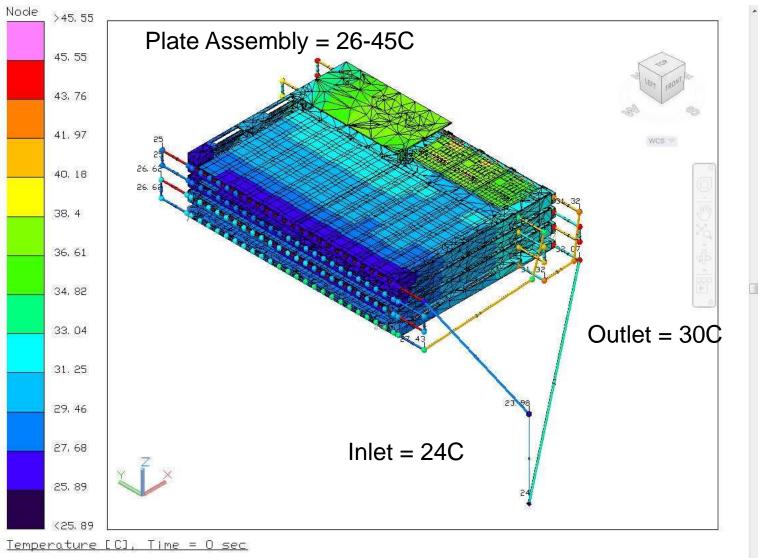
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## SCD Thermal Characteristics (2/2)

- Fluid flow provides heat transfer along width of plate
- Fluid direction and higher flow in center provides additional cooling and isothermalization of the plate in vertical dimension
- Bolts provide good vertical heat transfer
  - Some limitations on ACP section due to only one bolt in center
  - Sensor region only has bolts on two edges, OK for fluid cooled design
- Cover has gradients due to bolt locations and heat from ACPs on Layer3



### **SCD Temperatures**

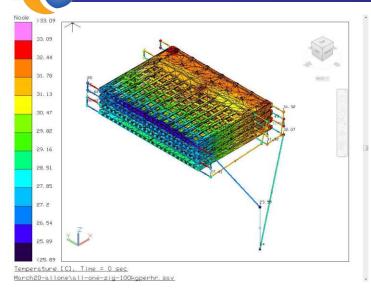


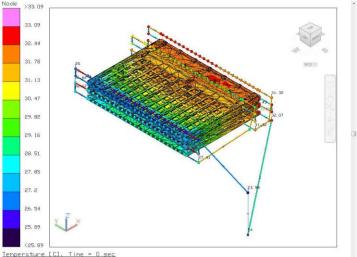
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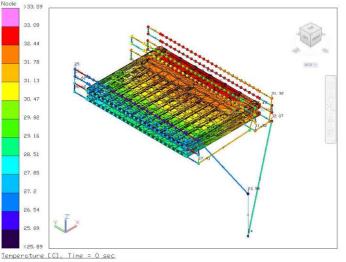
### **SCD Plate Temperatures**



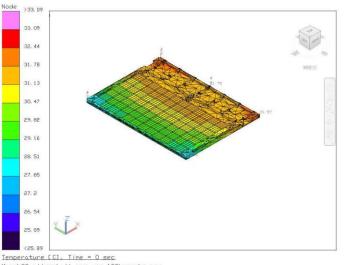


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Very isothermal due to fluid flow design

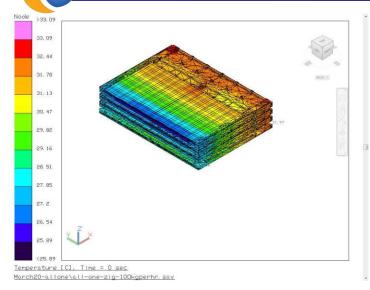


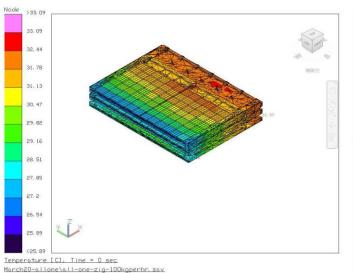
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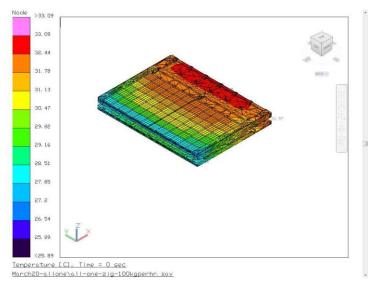
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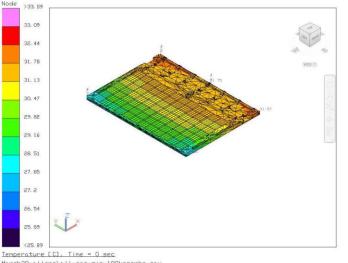
### **SCD Plate and Sensor Temperatures**





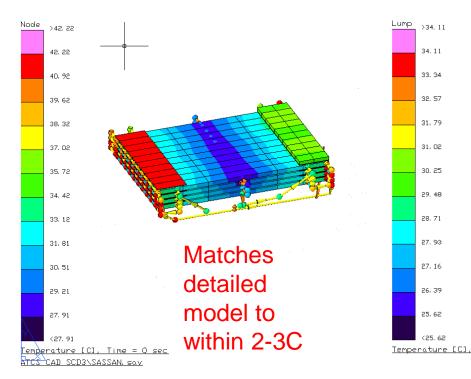
Very isothermal due to fluid flow design



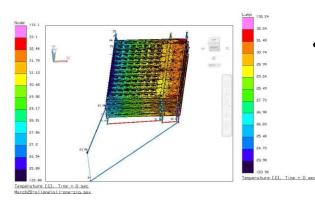


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### **SCD Reduced Model**



Half SCD Detailed model 30000 nodes total 14000 plate mesh nodes 550 cover mesh nodes 4000 ACP nodes 12000 sensor nodes 4000 other 340 FLUINT lumps



Reduced Model

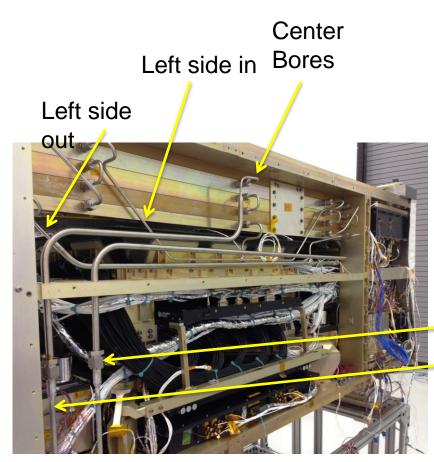
408 TD/RC Nodes 48 surfaces 10 fdsolids 240 FLUINT lumps 20 pipes 35 contactors

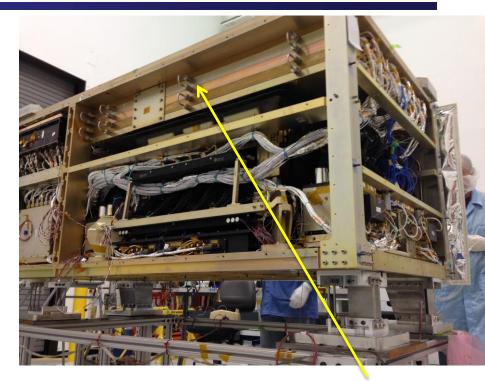
- Pipe lateral conductance ~
  10mm Aluminum pipe thickness
- Other conductances considered by contactors (tie each node to neighbor node on another pipe with equal conductance)
- Bolts conductances used to connect nodes
  - (total 60W/K layer to layer, 16 bolts)
- Plate conductances set to match detailed model



### **SCD Pictures**

- Shows final configuration of plumbing to SCD
- Tight packaging, but it works



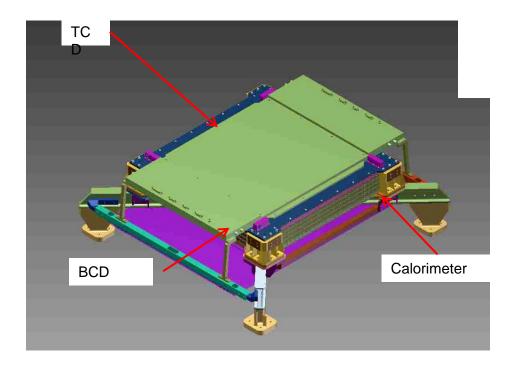


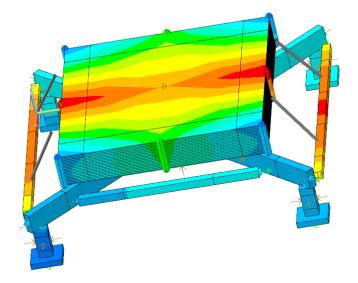
### Layer 1 to 2 loop

### Line to CP3 (from inlet) Lines from CP3 (supply)

### Stand with TCD, BCD, Calorimeter

Calorimeter stand provides structural support and heat paths from 400 kg calorimeter (tungsten plates layered with epoxy) to CREAM baseplate Also supports TCD and BCD, passively cooled instruments





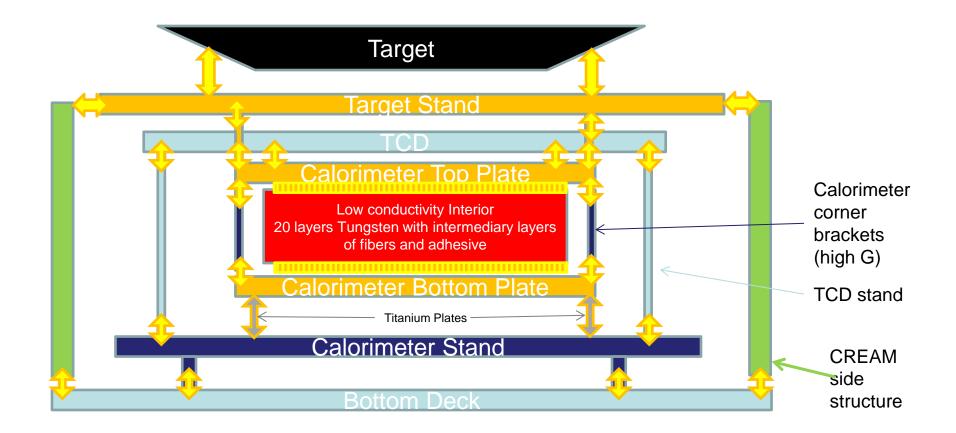
On-orbit, some temperature rise expected due to power in TCD and BCD

In ground testing, major factor in ability to cool instrument to hot and cold plateaus

### **Calorimeter Heat Paths**

Target = 175 kg of graphite

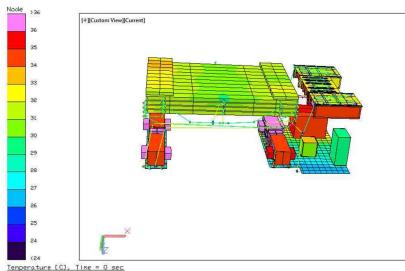
Calorimeter = 400 kg of Tungsten, fibers and adhesive



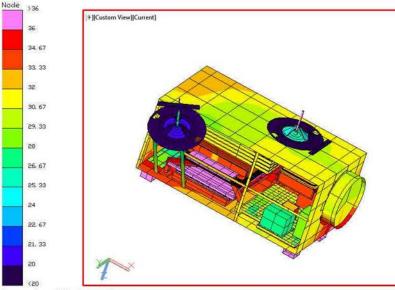
### **ISS-CREAM Integrated Model**

### **ATCS Components**

### Full ISS-CREAM model



cream-col-july/cream-jun15-for-iss-col-hot-flightmode. sav



Temperature [C], Time = 0 sec cream-eol-july/cream-jun15-for-iss-eol-hot-flightmode, sav

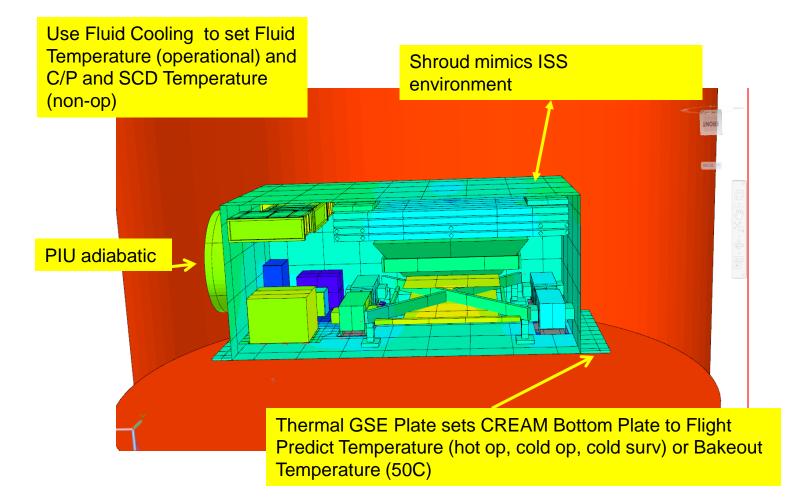
### **ISS-CREAM Pictures**



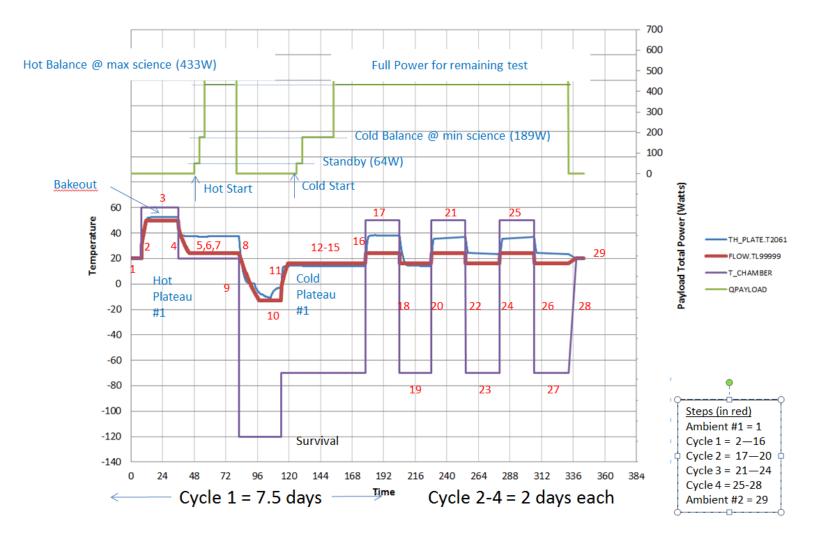
# **TESTING AND VERIFICATION**



Fluid cooling and conductive cooling of bottom plate Allows transitioning of 1300kg instrument within 1 day



### **Tvac Profile**



### **Thanks to the ISS-CREAM Team**

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### **Acronyms / Detectors**

- Cosmic Ray Energetics And Mass for the International Space Station payload (ISS-CREAM)
- ISS JEM EF International Space Station (ISS) Kibo/Japanese Experiment Module (JEM) Exposed Facility (EF)
- Active Thermal Control System (ATCS)
- Silicon Charge Detector (SCD) 4 layers of 2x2 cm silicon pixels used to determine incident particle charge Carbon Targets
- (C-targets) Layers of carbon plates to induce hadronic interactions for measurement in the calorimeter
- Top/Bottom Counting Detector (TCD/BCD) Plastic scintillator for electron/proton distinction
- Calorimeter (CAL) 20 layers of alternating tungsten plates and scintillating fibers used to measure incident particle energy and trajectory within the instrument. Four Hybrid Photodiode (HPD) boxes detect scintillation within the Calorimeter.
- Boronated Scintillator Detector (BSD) Boron-doped scintillator to capture thermal neutrons from hadronic interactions in the calorimeter providing additional e/p distinction
- Science Flight Computer (SFC) The onboard computer used to control detectors, assemble events and store science data
- FRGF Flight-Releasable Grapple Fixture
- Payload Interface Unit (PIU) Power, data, fluid interface to ISS JEM EF
- FSE Flight Support Interface SpaceX interface
- Detector info from http://cosmicray.umd.edu/iss-cream/files/ISSCREAM\_FactSheet.pdf