



USER INSTRUCTIONS

MJ Slurry Pump

Installation Operation Maintenance

Vertical Wet Pit

PCN=71569294 – 02/08 (E)

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1 INTRODUCTION AND SAFETY

1.1 General



These instructions must always be kept close to the product's operating location or directly with the product.

Flowserve's products are designed, developed and manufactured with state-of-the-art technologies in modern facilities. The unit is produced with great care and commitment to continuous quality control, utilising sophisticated quality techniques, and safety requirements.

We are committed to continuous quality improvement and being at your service for any further information about the product in its installation and operation or about its support products, repair and diagnostic services.

These instructions are intended to facilitate familiarization with the product and its permitted use. Operating the product in compliance with these instructions is important to help ensure reliability in service and avoid risks. The instructions may not take into account local regulations; ensure such regulations are observed by all, including those installing the product. Always coordinate repair activity with operations personnel, and follow all plant safety requirements and applicable safety and health laws and regulations.



These instructions must be read prior to installing, operating, using and maintaining the equipment in any region worldwide. The equipment must not be put into service until all the conditions relating to safety noted in the instructions, have been met.

1.2 CE marking and approvals

It is a legal requirement that machinery and equipment put into service within certain regions of the world shall conform with the applicable CE Marking Directives covering Machinery and, where applicable, Low Voltage Equipment, Electromagnetic Compatibility (EMC), Pressure Equipment Directive (PED) and Equipment for Potentially Explosive Atmospheres (ATEX).

Where applicable, the Directives and any additional Approvals, cover important safety aspects relating to machinery and equipment and the satisfactory provision of technical documents and safety instructions. Where applicable this document incorporates information relevant to these Directives.

To establish approvals and if the product itself is CE marked, check the serial number plate and the Certification. (See section 9, *Certification*.)

1.3 Disclaimer

Information in these User Instructions is believed to be reliable. In spite of all the efforts of Flowserve Corporation to provide sound and all necessary information the content of this manual may appear insufficient and is not guaranteed by Flowserve as to its completeness or accuracy.

Flowserve manufactures products to exacting International Quality Management System Standards as certified and audited by external Quality Assurance organisations. Genuine parts and accessories have been designed, tested and incorporated into the products to help ensure their continued product quality and performance in use. As Flowserve cannot test parts and accessories sourced from other vendors the incorrect incorporation of such parts and accessories may adversely affect the performance and safety features of the products. The failure to properly select, install or use authorised Flowserve parts and accessories is considered to be misuse. Damage or failure caused by misuse is not covered by Flowserve's warranty. In addition, any modification of Flowserve products or removal of original components may impair the safety of these products in their use.

1.4 Copyright

All rights reserved. No part of these instructions may be reproduced, stored in a retrieval system or transmitted in any form or by any means without prior permission of Flowserve Pump Division.

1.5 Duty conditions

This product has been selected to meet the specifications of your purchaser order. The acknowledgement of these conditions has been sent separately to the Purchaser. A copy should be kept with these instructions.



The product must not be operated beyond the parameters specified for the application. If there is any doubt as to the suitability of the product for the application intended, contact Flowserve for advice, quoting the serial number.

If the conditions of service on your purchase order are going to be changed (for example liquid pumped, temperature or duty) it is requested that you/the user seek our written agreement before start up.

1.6 Safety

1.6.1 Summary of safety markings

These user instructions contain specific safety markings where non-observance of an instruction would cause hazards. The specific safety markings are:



DANGER This symbol indicates electrical safety instructions where non-compliance would affect personal safety.



This symbol indicates safety instructions where non-compliance would affect personal safety.



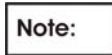
This symbol indicates safety instructions where non-compliance would affect protection of a safe life environment.



This symbol indicates safety instructions where non-compliance would affect the safe operation or protection of the pump or pump unit.



This symbol indicates explosive atmosphere zone marking according to ATEX. It is used in safety instructions where non-compliance in the hazardous area would cause the risk of an explosion.



This sign is not a safety symbol but indicates an important instruction in the assembly process.

1.6.2 Personnel qualification and training

All personnel involved in the operation, installation, inspection and maintenance of the unit must be qualified to carry out the work involved. If the personnel in question do not already possess the necessary knowledge and skill, appropriate training and instruction must be provided. If required the operator may commission the manufacturer/supplier to provide applicable training.

Always coordinate repair activity with operations and health and safety personnel, and follow all plant safety requirements and applicable safety and health laws and regulations.

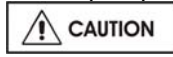
1.6.3 Safety action

This is a summary of conditions and actions to prevent injury to personnel and damage to the environment and to equipment. (For products used in potentially explosive atmospheres section 1.6.4 also applies.)



PREVENT EXCESSIVE EXTERNAL PIPE LOAD

Do not use pump as a support for piping. Do not mount expansion joints, unless allowed by Flowserve in writing, so that their force, due to internal pressure, acts on the pump flange.



ENSURE CORRECT LUBRICATION
(See section 5, *Commissioning, startup, operation and shutdown.*)



START THE PUMP WITH OUTLET VALVE PART OPENED

(Unless otherwise instructed at a specific point in the user instructions.)

This is recommended to minimize the risk of overloading and damaging the pump motor at full or zero flow. Pumps may be started with the valve further open only on installations where this situation cannot occur. The pump outlet control valve may need to be adjusted to comply with the duty following the run-up process. (See section 5, *Commissioning start-up, operation and shutdown.*)

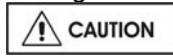


NEVER RUN THE PUMP DRY



INLET VALVES TO BE FULLY OPEN WHEN PUMP IS RUNNING

Running the pump at zero flow or below the recommended minimum flow continuously will cause damage to the seal.



DO NOT RUN THE PUMP AT ABNORMALLY HIGH OR LOW FLOW RATES

Operating at a flow rate higher than normal or at a flow rate with no backpressure on the pump may overload the motor and cause cavitation. Low flow rates may cause a reduction in pump/bearing life, overheating of the pump, instability and cavitation/ vibration.



DANGER NEVER DO MAINTENANCE WORK WHEN THE UNIT IS CONNECTED TO POWER



HAZARDOUS LIQUIDS

When the pump is handling hazardous liquids care must be taken to avoid exposure to the liquid by appropriate siting of the pump, limiting personnel access and by operator training. If the liquid is flammable and/or explosive, strict safety procedures must be applied.

Gland packing must not be used when pumping hazardous liquids.



DRAIN THE PUMP AND ISOLATE PIPEWORK BEFORE DISMANTLING THE PUMP

The appropriate safety precautions should be taken where the pumped liquids are hazardous.

FLUORO-ELASTOMERS (When fitted.)
 When a pump has experienced temperatures over 250 °C (482 °F), partial decomposition of fluoro-elastomers (e.g. Viton) will occur. In this condition these are extremely dangerous and skin contact must be avoided.

HANDLING COMPONENTS
 Many precision parts have sharp corners and the wearing of appropriate safety gloves and equipment is required when handling these components. To lift heavy pieces above 25 kg (55 lb) use a crane appropriate for the mass and in accordance with current local regulations.

GUARDS MUST NOT BE REMOVED WHILE THE PUMP IS OPERATIONAL

THERMAL SHOCK
 Rapid changes in the temperature of the liquid within the pump can cause thermal shock, which can result in damage or breakage of components and should be avoided.

NEVER APPLY HEAT TO REMOVE IMPELLER
 Trapped lubricant or vapour could cause an explosion.

HOT (and cold) PARTS
 If hot or freezing components or auxiliary heating supplies can present a danger to operators and persons entering the immediate area action must be taken to avoid accidental contact. If complete protection is not possible, the machine access must be limited to maintenance staff only, with clear visual warnings and indicators to those entering the immediate area. Note: bearing housings must not be insulated and drive motors and bearings may be hot.

If the temperature is greater than 68 °C (175 °F) or below 5 °C (20 °F) in a restricted zone, or exceeds local regulations, action as above shall be taken.

1.6.4 Products used in potentially explosive atmospheres

- Measures are required to:
- Avoid excess temperature
 - Prevent build up of explosive mixtures
 - Prevent the generation of sparks
 - Prevent leakages
 - Maintain the pump to avoid hazard

The following instructions for pumps and pump units when installed in potentially explosive atmospheres must be followed to help ensure explosion protection.

Both electrical and non-electrical equipment must meet the requirements of European Directive 94/9/EC.

1.6.4.1 Scope of compliance

Use equipment only in the zone for which it is appropriate. Always check that the driver, drive coupling assembly, seal and pump equipment are suitably rated and/or certified for the classification of the specific atmosphere in which they are to be installed.

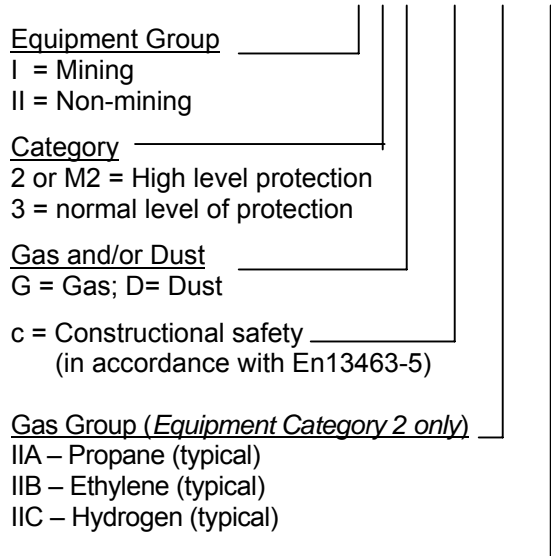
Where Flowserve has supplied only the bare shaft pump, the Ex rating applies only to the pump. The party responsible for assembling the pump set shall select the coupling, driver and any additional equipment, with the necessary CE Certificate/ Declaration of Conformity establishing it is suitable for the area in which it is to be installed.

The output from a variable frequency drive (VFD) can cause additional heating affects in the motor and so, for pumps sets with a VFD, the ATEX Certification for the motor must state that it covers the situation where electrical supply is from the VFD. This particular requirement still applies even if the VFD is in a safe area.

1.6.4.2 Marking

An example of ATEX equipment marking is shown below. The actual classification of the pump will be engraved on the nameplate.

II 2 GD c IIC 135 °C (T4)



Maximum surface temperature (Temperature Class)
 (See section 1.6.4.3.)

1.6.4.3 Avoiding excessive surface temperatures

ENSURE THE EQUIPMENT TEMPERATURE CLASS IS SUITABLE FOR THE HAZARD ZONE

Pumps have a temperature class as stated in the ATEX Ex rating on the nameplate. These are based on a maximum ambient of 40 °C (104 °F); refer to Flowserve for higher ambient temperatures.

The temperature of the liquid handled influences the surface temperature on the pump. The maximum permissible liquid temperature depends on the temperature class and must not exceed the values in the table that follows.

The temperature rise at the seals and bearings and due to the minimum permitted flow rate is taken into account in the temperatures stated.

Temperature class to prEN 13463-1	Maximum surface temperature permitted	Temperature limit of liquid handled (* depending on material and construction variant - check which is lower)
T6	85 °C (185 °F)	Consult Flowserve
T5	100 °C (212 °F)	Consult Flowserve
T4	135 °C (275 °F)	115 °C (239 °F) *
T3	200 °C (392 °F)	180 °C (356 °F) *
T2	300 °C (572 °F)	275 °C (527 °F) *
T1	450 °C (842 °F)	400 °C (752 °F) *

The responsibility for compliance with the specified maximum liquid temperature is with the plant operator.

Temperature classification “Tx” is used when the liquid temperature varies and when the pump is required to be used in differently classified potentially explosive atmospheres. In this case the user is responsible for ensuring that the pump surface temperature does not exceed that permitted in its actual installed location.

If an explosive atmosphere exists during the installation, do not attempt to check the direction of rotation by starting the pump unfilled. Even a short run time may give a high temperature resulting from contact between rotating and stationary components.

Where there is any risk of the pump being run against a closed valve generating high liquid and casing external surface temperatures it is recommended that users fit an external surface temperature protection device.

Avoid mechanical, hydraulic or electrical overload by using motor overload trips, temperature monitor or a

power monitor and make routine vibration monitoring checks.

In dirty or dusty environments, regular checks must be made and dirt removed from areas around close clearances, bearing housings and motors.

1.6.4.4 Preventing the build up of explosive mixtures

ENSURE THE PUMP IS PROPERLY FILLED AND VENTED AND DOES NOT RUN DRY

Ensure the pump and relevant suction and discharge pipeline system is totally filled with liquid at all times during the pump operation, so that an explosive atmosphere is prevented. In addition it is essential to make sure that seal chambers, auxiliary shaft seal systems and any heating and cooling systems are properly filled.

If the operation of the system cannot avoid this condition the fitting of an appropriate dry run protection device is recommended (eg liquid detection or a power monitor).

To avoid potential hazards from fugitive emissions of vapour or gas to atmosphere the surrounding area must be well ventilated.

1.6.4.5 Preventing sparks

To prevent a potential hazard from mechanical contact, the coupling guard must be non-sparking and anti-static for Category 2.

To avoid the potential hazard from random induced current generating a spark, the earth contact on the baseplate must be used.

Avoid electrostatic charge: do not rub non-metallic surfaces with a dry cloth; ensure cloth is damp.

The coupling must be selected to comply with 94/9/EC and correct alignment must be maintained.

1.6.4.6 Preventing leakage

The pump must only be used to handle liquids for which it has been approved to have the correct corrosion resistance.

Avoid entrapment of liquid in the pump and associated piping due to closing of suction and discharge valves, which could cause dangerous excessive pressures to

occur if there is heat input to the liquid. This can occur if the pump is stationary or running.

Bursting of liquid containing parts due to freezing must be avoided by draining or protecting the pump and ancillary systems.

Where there is the potential hazard of a loss of a seal barrier fluid or external flush, the fluid must be monitored.

If leakage of liquid to atmosphere can result in a hazard, the installation of a liquid detection device is recommended.

To avoid potential explosion hazards during maintenance, the tools, cleaning and painting materials used must not give rise to sparking or adversely affect the ambient conditions. Where there is a risk from such tools or materials, maintenance must be conducted in a safe area.

It is recommended that a maintenance plan and schedule is adopted. (See section 6, *Maintenance*.)

1.6.4.7 Maintenance to avoid the hazard



CORRECT MAINTENANCE IS REQUIRED TO AVOID POTENTIAL HAZARDS WHICH GIVE A RISK OF EXPLOSION

The responsibility for compliance with maintenance instructions is with the plant operator.

1.7 Nameplate and warning labels

1.7.1 Nameplate

For details of nameplate, see the *Declaration of Conformity*.

1.7.2 Warning labels

WARNING
MECHANICAL SEAL FITTED
DO NOT ADJUST PUMP CLEARANCE WITHOUT REFERENCE TO MANUFACTURER'S INSTRUCTION MANUAL

MECHANICAL SEAL WARNING
P/N 2113931-001

WARNING
BEFORE STARTING THE PUMP ON SERVICE, CHECK TO ENSURE CORRECT ROTATION OF MOTOR. FAILURE TO DO THIS COULD RESULT IN SERIOUS DAMAGE TO THE EQUIPMENT.

ROTATION WARNING
P/N 2113932-001

WARNING
BEFORE GROUTING, REALIGN THIS UNIT RECHECK ALIGNMENT BEFORE STARING, FAILURE TO DO THIS COULD RESULT IN SERIOUS DAMAGE TO THE EQUIPMENT. REFER TO INSTALLATION MANUAL AND, OR COUPLING INSTRUCTIONS FOR METHOD OF CHECKING ALIGNMENT.

GROUT WARNING
P/N 2113934-001

WARNING
THESE EYEBOLTS ARE TO BE USED FOR LIFTING BEARING CARTRIDGE AND SHAFT ONLY. DO NOT USE FOR LIFTING ENTIRE UNIT.

LIFTING WARNING
P/N 9901701-001

QF-440-R01

FLOWERVE

LUBRICATION WARNING TAG

LUBRICATION WARNING - CAUTION

The lubricating oil in this pump has been drained for shipping. The proper lubricating oil must be added before starting pump. Refer to Instruction Manual.

ADVERTISSEMENT PUR LE GRAISSAGE - ATTENTION

Avant expédition de la pompe, l'huile de graissage a été vidée. Il faut ajouter l'huile appropriée avant d'utiliser la pompe. Voir le Mode D'emploi.

ADVERTENCIA RESPECTO A LA LUBRICACIÓN - CUIDADO

El aceite lubricante de este bombeador ha sido vaciado por razones de envío. Antes de comenzar a bombear, deberá agregarse el aceite lubricante correcto. Consultar el Manual de Instrucciones.

LUBRICATION WARNING – QF-440-R01 (2124841)

Oil lubricated units only:

1.8 Specific machine performance

For performance parameters see section 1.5, *Duty conditions*. When the contract requirement specifies these to be incorporated into User Instructions these are included here. Where performance data has been supplied separately to the purchaser these should be obtained and retained with these User Instructions if required.

1.9 Noise level

When pump noise level exceeds 85 dBA attention must be given to prevailing Health and Safety Legislation, to limit the exposure of plant operating personnel to the noise. The usual approach is to control exposure time to the noise or to enclose the machine to reduce emitted sound. You may have already specified a limiting noise level when the equipment was ordered, however if no noise requirements were defined then machines above a certain power level will exceed 85 dBA. In such situations consideration must be given to the fitting of an acoustic enclosure to meet local regulations.

Pump noise level is dependent on a number of factors - the type of motor fitted, the operating capacity, pipework design and acoustic characteristics of the building. The levels specified in the table below are estimated and not guaranteed.

The dBA values are based on the noisiest ungeared electric motors that are likely to be encountered. They are Sound Pressure levels at 1 m (3.3 ft) from the directly driven pump, for "free field over a reflecting plane". For estimating L_{WA} sound power level (re 1 pW) add 14dBA to the sound pressure value.

If a pump unit only has been purchased, for fitting with your own driver, then the "pump only" noise levels from the table should be combined with the level for the driver obtained from the supplier. If the motor is driven by an inverter, it may show an increase in noise level at some speeds. Consult a Noise Specialist for the combined calculation.



For units driven by equipment other than electric motors or units contained within enclosures, see the accompanying information sheets and manuals.

M SLURRY pump size	Noise Level dbA (at 1 m (3 ft) reference 20 µPa)	Pump speed rpm	Sound Power Level dbA
2.5 MJ091	75	2250	89
3 MJ091	75	2250	89
3 MJ111	77	2000	91
4 MJ121	77	1300	91
5 MJ141	78	1180	92
4 MJ122	80	1780	94
5 MJ142	80	1600	94
6 MJ162	77	1180	91
8 MJ192	78	1000	92
6 MJ163	81	1350	95
8 MJ193	82	1180	96

2 TRANSPORT AND STORAGE

2.1 Consignment receipt and unpacking

Immediately after receipt of the equipment it must be checked against the delivery/shipping documents for its completeness and that there has been no damage in transportation. Any shortage and/or damage must be reported immediately to Flowserve Pump Division and must be received in writing within one month of receipt of the equipment. Later claims cannot be accepted.

Check any crate, boxes or wrappings for any accessories or spare parts that may be packed separately with the equipment or attached to side walls of the box or equipment.

Each product has a unique serial number. Check that this number corresponds with that advised and always quote this number in correspondence as well as when ordering spare parts or further accessories.

2.2 Handling

Boxes, crates, pallets or cartons may be unloaded using forklift vehicles or slings dependent on their size and construction.

The pump should be lifted with suitably sized and located slings. Do not use the shaft for lifting and take special care to prevent the pump from rotating in the slings due to unbalanced weight distribution.

2.3 Lifting



A crane must be used for all pump sets in excess of 25 kg (55 lb). Fully trained personnel must carry out lifting, in accordance with local regulations. The driver and pump weights are recorded on their respective nameplates or massplates.

2.4 Storage



2.4.1 Short-Term Storage

When it is necessary to store a pump for a short time before it can be installed, place it in a dry, cool location. Protect it thoroughly from moisture and condensation. Protective flange covers should not be removed until the pump is being installed.

Wrap the exposed portions of the shaft and coupling to protect against sand, grit or other foreign matter. Oil lubricated units should be lubricated (refer to Section III) to protect the bearings. Grease lubricated units are lubricated at the factory during assembly. Turn the

rotor over by hand at least once a week to maintain a protective film on the bearing components.

2.4.2 LONG-TERM STORAGE

More than precautions are required if long-term storage in excess of 90 days from factory shipment is unavoidable.

The internal surfaces of the pump should be sprayed with a rust preventative such as water-soluble oil or other suitable alternative. Particular attention should be given to the impeller, wear plate and stuffing box.

An optional method of protection is to suspend bags of desiccant material inside casing and completely seal all openings from the surrounding atmosphere. The stuffing box should be packed with clean, dry rags. Use of this method requires that the casing be initially free of liquid. The desiccant material should be checked at regular intervals to ensure that it has not absorbed excessive water vapour. A warning instruction, advising that the desiccant must be removed prior to installation should be wired to the pump.

A rust inhibitor should be added to the lubricating oil of oil lubricated units to give additional protection without destroying the lubricating properties of the oil. For specific recommendations, consult your lubrication dealer. Grease lubricated units, which can be identified by the grease fitting at each bearing location, should be well lubricated prior to placing in storage. Small amounts of additional grease should be added at regular intervals during storage. Refer to Section III for additional information related to grease lubrication.

Storage of pumps in areas of high ambient vibration should be avoided to prevent bearing damage due to brinelling. The risk of such damage can be reduced by frequent rotation of the shaft.

The pump half coupling and key should be removed from the shaft, coated with rust preventative and wrapped to prevent metal-to-metal contact. Exposed surfaces of the pump shaft should be protected with a rust preventative. All dismantled parts should be wrapped and tagged according to pump serial number and a record kept of their location.



Pumps covered with plastic should not be stored in a cool environment because resulting condensation can cause rusting.

2.5 Recycling and end of product life

At the end of the service life of the product or its parts, the relevant materials and parts should be recycled or disposed of using an environmentally acceptable method and in accordance with local regulations. If the product contains substances that are harmful to the environment, these should be removed and disposed of in accordance with current local regulations. This also includes the liquids and/or gases that may be used in the "seal system" or other utilities.



Make sure that hazardous substances are disposed of safely and that the correct personal protective equipment is used. The safety specifications must be in accordance with the current local regulations at all times.

3 PUMP DESCRIPTION

3.1 Configurations

Flowserve "MJ Slurry" pumps are single stage, vertical bottom suction centrifugal pumps specifically designed for handling abrasive slurries typical of mining and mineral process. A semi-concentric volute type casing is pedestal mounted with tangential discharge nozzle. The closed impeller with rear pump-out vanes is capable of passing solids of various sizes. The rigid three point thrust bearing housing support permits precision bearing alignment. The casing is supported from the top plate by column pipe(s).

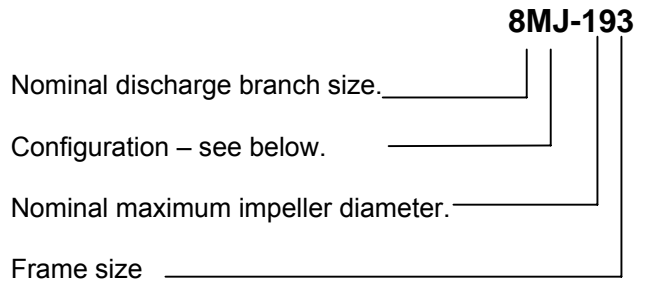
The pump has a water lubricated steady bearing located in a lower housing located above the impeller. Intermediate steady bearings will be supplied to maintain shaft critical speeds are above the operating speeds.

Pumps can be v-belt driven or direct driven by an electric motor. The bearing frame design varies according to the drive method.

All pumps are carefully inspected and prepared for shipment. All exterior machined surfaces are coated with a rust preventative compound and openings are provided with covers or plugs. The axial impeller running clearance is preset at the factory but should be checked prior to final alignment in case of tampering.

3.2 Name nomenclature

The pump size will be engraved on the nameplate. The following example explains how the pump name identifies the construction features and options.



MJ is added for vertical bottom-bearing configuration
MSJ is added for recessed impeller

3.3 Design of major parts

3.3.1 Pump casing

The pump casing is a semi-concentric volute type casing and tangential discharge nozzle. The casing is pedestal mounted therefore the discharge can be rotated to meet a number of optional orientations. On standard duty casing the suction nozzle is integral to the casing. On severe duty pumps the pump uses a removable suction cover. The casing joints are gasketed connections.

3.3.2 Impeller

The impeller is closed design as standard but may be changed to open designs for mud, froth and viscous applications. The impeller is thread mounted to the shaft. All impellers are supplied with pump out vanes to minimise wear and reduce pressure at the stuffing box. The vanes are curved to optimize efficiency however, the "S" configuration utilizes straight radial vanes to reduce blockage. The impeller of the MS pump has larger front clearances to reduce shear and improve froth handling capabilities.

3.3.3 Shaft

The large diameter stiff shaft, mounted on bearings, has a keyed drive end.

3.3.4 Pump bearings and lubrication

Ball bearings are fitted as standard above the top plate and grease lubricated. Pumps that are direct driven have a set of angular contact ball bearings mounted back-to-back. Pumps that are MVD driven have 2 sets the bearings. The lower bearing, located just above the mounting plate is a deep groove ball bearing. A set of angular contact ball bearings is located towards the sheave. All bearings located below the top plate are water lubricated. These are generally cutlass rubber

bearings contained in a stainless sleeve and mounted in a bearing housing. Intermediate bearings are located to ensure shaft critical speeds are above the operating speed.

3.3.5 Bearing housing

Grease nipples enable grease-lubricated bearings to be replenished between major service intervals.

3.3.6 Lower and Intermediate bearing housing

The bearing housings below the top plate have a spigot (rabbet) fit between the pump casing and column pipe or between column pipes for optimum concentricity.

3.3.7 Driver

The driver is normally an electric motor. Due to the hardness of the impeller the norm is to use multi-v-belts.

3.3.8 Accessories

Accessories may be fitted when specified by the customer.

3.4 Performance and operating limits

This product has been selected to meet the specifications of your purchase order see section 1.5. The following data is included as additional information to help with your installation. It is typical, and factors such as temperature, materials, and seal type may influence this data. If required, a definitive statement for your particular application can be obtained from Flowserve.

3.4.1 Operating limits

Pumped liquid temperature limits	up to +66 °C (150 °F)
Maximum ambient temperature	up to +50 °C (122 °F)
Maximum soft solids in suspension	up to 7 % by volume
Maximum pump speed	Refer to the nameplate
Maximum Water Flush temperature	up to +38 °C (100 °F)

3.4.2 Speed torque curves

To bring a centrifugal pump up to rated speed, the driver must be capable of providing more torque at each speed than required by the pump. The margin between the available and required torque affects the time it takes the unit to reach full speed. If the torque required by the pump exceeds the torque capability of the drive at any run-up speed, the unit will not accelerate to full speed. Normally, this is not a problem with standard induction or synchronous motors provided the proper voltage is supplied at the motor.

For pumps started at shut valve conditions, 100 percent full speed torque can be calculated by using the formula:

$$\text{Torque (Nm)} = 9545 \frac{\text{Shutoff Power (kW)}}{\text{r/min}}$$

$$\text{Torque (lbfx ft)} = 5250 \frac{\text{Shutoff Power (hp)}}{\text{r/min}}$$

Torque required by the pump at any other speed during start-up can be determined from the curve above. Note that the driver manufacturer usually bases 100 percent torque on the design power of the driver and consequently the speed-torque curves should be plotted in torque units (e.g. Nm) instead of percentage torque to avoid confusion.

3.4.3 MAXIMUM WORKING PRESSURES -bar (psi).

MAXIMUM WORKING PRESSURE		
TEMPERATURE		STANDARD DUTY
(°F)	(°C.)	Bar (PSIG.)
-20 TO 100	-7 TO 38	7.6 (110)
150	65	7.6 (110)

High Chrome Iron is standard for all pumps, special materials are available upon request. Consult a Flowserve Sales Office or a Distributor for material selection and compatibility with the slurry product.

3.5 Engineering Data

TABLE OF ENGINEERING DATA
(FRAMES 1 to 3 - LIQUID END)

ENGINEERING DATA	2½ MJ	3 MJ	3 MJ	4 MJ	5 MJ	4 MJ	5 MJ	6 MJ	8 MJ	6 MJ	8 MJ		
	-	-	-	-	-	-	-	-	-	-	-		
	0	0	1	1	1	1	1	1	1	1	1		
	9	9	1	2	4	2	4	6	9	6	9		
	1	1	1	1	1	2	2	2	2	3	3		
PUMP DESIGN TYPE	STD	STD	STD	STD	STD	STD	STD	STD	STD	STD	STD		
SUCTION SIZE	mm (IN)	64 (2.5)	76 (3)	102 (4)	127 (5)	152 (6)	127 (5)	152 (6)	203 (8)	254 (10)	203 (8)	254 (10)	
DISCHARGE SIZE	mm (IN)	64 (2.5)	76 (3)	76 (3)	102 (4)	127 (5)	102 (4)	127 (5)	152 (6)	203 (8)	152 (6)	203 (8)	
MAX.SPHERE SIZE	mm (IN)	15 (0.6)	23 (0.9)	31 (1.2)	43 (1.7)	48 (1.9)	43 (1.7)	48 (1.9)	53 (2.1)	71 (2.8)	53 (2.1)	71 (2.8)	
WK ² (wet)	kg x m ² lb-ft ²	.12 (2.8)	0.11 (2.6)	0.21 (4.9)	0.44 (11)	0.82 (19)	2.9 (36)	2.9 (37)	2.96 (70)	5.75 (136)	1.81 (43)	3.62 (86)	
PUMP WT.	Kg (Lbs)	241 (530)	264 (580)	286 (630)	391 (860)	486 (1070)	714 (1570)	1045 (2300)	1114 (2450)	5830 (2650)	832 (1830)	1009 (2220)	
IMPELLER CLEARANCE	mm (IN)	0.38 (.015)			0.76 (.030)			1.14 (.045)					
DIA. AT IMPELLER	mm (in.)	28.58 (1.125)			34.93 (1.375)			50.80 (2.000)					
O.D. LOWER SLEEVE	mm (IN)	63.5 (2.50)				88.9 (3.50)				114.3 (4.50)			
I.D. LOWER HOUSING	mm (IN)	85.9 (3.38)				114.3 (4.50)				139.7 (5.50)			
O.D. SLEEVE INTERMEDIATE	mm (IN)	76.2 (3.00)				101.6 (4.00)				127.0 (5.00)			
I.D. INTERM. HOUSING	mm (IN)	95.3 (3.75)				127 (5.00)				152.4 (6.00)			
DIA. AT COUPLING	mm (in.)	41.27 (1.625)				63.50 (2.500)				85.72 (3.375)			
LINE BEARING		6311Z				6317Z				6321Z			
THRUST BEARING		7309 BUA				7314 BUA				7318 BUA			
THRUST BEARING LIP SEAL-outer		C/R 16245				C/R 24910				C/R 33735			
THRUST BEARING LIP SEAL-inner - Direct Drive only		C/R 21172				C/R 32396				C/R41170			
LINE BEARING LIP SEAL-mvd only		C/R 29871				C/R 39933				C/R 45110			

MATERIALS OF CONSTRUCTION
(LIQUID END)

BASIC CONSTRUCTION	MATERIAL CLASS	
CASING IMPELLER WEAR PLATE	ASTM A532 CL III TYPE A	
LOWER BEARING HOUSING	ASTM A48 CL35	
IMPELLER SPACER	HARD STEEL -360 BHN	
SHAFT SLEEVE	ASTM A743 CG8M	
SHAFT	AISI 1045	
WATER LUBRICATED BEARINGS	316 SS W/ BUNA-N LINER	
WEAR PLATE STUDS & NUTS	AISI 316	
GASKETS-CASING	SYNTHETIC FIBRE	
MISC. FASTNERS, PARTS	STEEL	

4 INSTALLATION



Equipment operated in hazardous locations must comply with the relevant explosion protection regulations. See section 1.6.4, *Products used in potentially explosive atmospheres*.

4.1 Location

The pump should be located to allow room for access, ventilation, maintenance and inspection with ample headroom for lifting and should be as close as practicable to the supply of liquid to be pumped. Allow sufficient room to facilitate the back pull-out feature on V-belt driven units.

Refer to the general arrangement drawing for the pump set.

4.2 Part assemblies

Motors may be supplied loose on M Slurry pumps, typically on frame sizes 400 and above. It is the responsibility of the installer to ensure that the motor is assembled to the pump and lined up as detailed in section 4.5.2.

4.3 Foundation



The foundation may consist of any material that will afford permanent, rigid support to the top plate. It should be of sufficient size and mass to absorb expected strains and shocks that may be encountered in service. The sump opening must be

large enough to pass the pump liquid end and discharge piping. The top surface must be flat and level for future pump alignment.

In the cases where maximum support is desired or where levelling problems are anticipated a sole plate (as illustrated in Figure 1) may be used as the bearing surface for the top plate. A sole plate is sometimes referred to as a curb ring. When a sole plate is used it should be bolted to the concrete foundation and grouted into position.

Foundation bolts should be high strength steel (SAE GR 5 or equal). Each bolt should be surrounded by a pipe sleeve two or three times the diameter of the bolt. The sleeve should be securely anchored and designed to allow the bolts to be moved to align with holes in the top plate or sole plate as required. The bolts should be sufficiently long to allow for leveling shims or wedges. Extra long bolts can be shortened after the installation is complete.

4.3.1 Sole Plate Installation.

Position the sole plate next to the foundation and clean the foundation surface thoroughly. Remove the rag packing from the pipe sleeve and place wedges or shims as close to the foundation bolts as possible. Leveling nuts may be used on the bolts instead of wedges.

Note: Sole plates are normally not supplied by Flowserve.

4.4 Baseplate installation

Position the pump next to the foundation and clean the foundation surface thoroughly. Remove the rag packing from the pipe sleeves and place wedges or shims as close to the foundation bolts as possible. These may be omitted if a jacking nut on the foundation anchor bolts is preferred for levelling. Initial levelling should be within 0.75 mm (.030 inches).

Remove the flange covers and check inside the pump nozzles for cleanliness. Kerosene is recommended as the best solvent for removing factory applied rust preventative. Ensure that all traces of rust preventative are removed from the discharge and suction flange faces, the exposed shafting and all coupling surfaces. Flush the pump internals of any rust preventative applied for long-term storage.

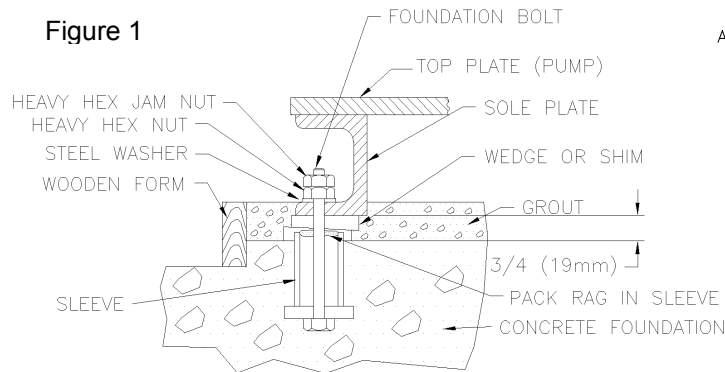


Figure 1

Lift the baseplate assembly, remove the shipping skids and clean the underside of the baseplate. Position the baseplate over the foundation and lower the unit over the foundation bolts and onto the wedges, shims or jacking nuts.

With the aid of a machinist's level, adjust the wedges, shims or jacking nuts to level the pump and driver mounting pads in each direction. Check to ensure that the suction and discharge flanges are plumb, level, and at the correct elevation. It is normal practice to set the mounting pads slightly low in order to permit lowering of units which may be required to suit future piping or minor changes. Place washers over the foundation bolts and install nuts. Tighten finger tight only.

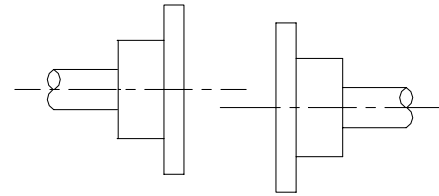
Check the impeller axial clearance and that the rotor turns freely by hand.

Note: Grout is not poured until an initial alignment of the pump and driver has been performed.

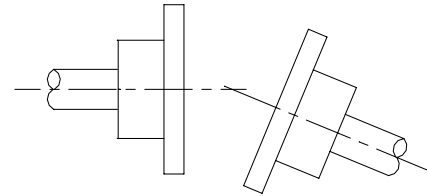
4.5 Initial alignment

4.5.1 Thermal expansion

The pump and motor will normally have to be aligned at ambient temperature and



PARALLEL MISALIGNMENT— Shafts with axis parallel but not concentric



ANGULAR MISALIGNMENT — Shafts with axis concentric but not parallel.

should be corrected to allow for thermal expansion at operating temperature. In pump installations involving high liquid temperatures, the unit should be run at the actual operating temperature, shut down and the alignment checked immediately.

4.5.2 Alignment methods

DANGER Ensure pump and driver are isolated electrically and the half couplings are disconnected.

CAUTION The alignment **MUST** be checked. Although the pump will have been aligned at the factory it is most likely that this alignment will have been disturbed during transportation or handling. If necessary, align the motor to the pump, not the pump to the motor.

4.5.2.1 Direct Driven Units:

The importance of accurate alignment of pump and driver shafts cannot be overemphasized.

IMPROPER ALIGNMENT IS THE PRIMARY CAUSE OF VIBRATION PROBLEMS AND REDUCED BEARING LIFE.

A flexible coupling is used to compensate for slight changes in alignment that occur during normal operation and is not used to correct for installation errors. Install the pump and driver half couplings in accordance with the coupling manufacturer's instructions. Note that the coupling hub faces are not always mounted flush with the ends of the shafts. Place the driver on the topplate such and ensure that motor shaft and pump shaft are spaced apart by at least 3mm (0.12").

The purpose of the alignment procedure is to ensure

that the pump and driver shafts are in parallel and angular alignment under the normal operating conditions of load and temperature.

When the pump coupling and driver are assembled at the factory, the units are aligned prior to shipment. For pumps and drivers that operate at different temperatures compensation must be made at the initial alignment stage (when the units are at the same temperature) to allow for thermal expansion during operation. Consult the instruction manual supplied with the driver for the manufacturer's recommendations.

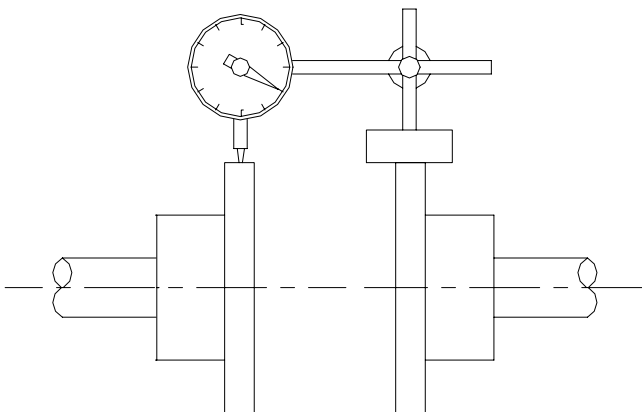
Shaft alignment is greatly simplified by the use of a dial indicator with extension rods and a magnetic base. Before taking readings, ensure that the pump and driver mounting bolts are secure, and that the thrust bearing housing is properly aligned in the bearing frame or cartridge.

Parallel Alignment:

Mount the magnetic base on the pump half coupling hub, either the face or O.D. as shown in the sketch. Place the dial indicator button on the outside diameter of the driver half coupling hub.

Note: The length of extension rods should be kept at a minimum to reduce deflection.

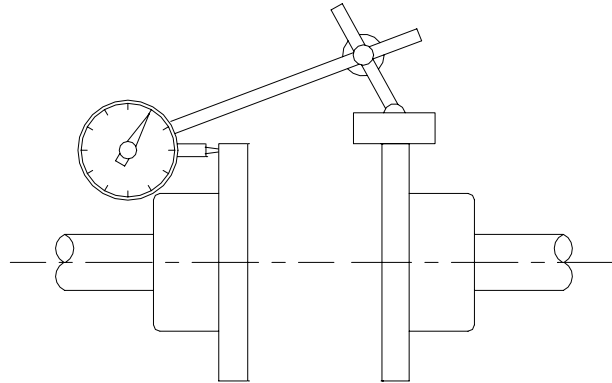
Rotate the pump shaft and record the dial reading at the top, bottom and each side. Correct the parallel alignment by adding or removing shims under the driver and/or moving the driver horizontally. Repeat this procedure until the maximum total indicator reading (T.I.R.) is within 0.076 mm (0.003 inch.)



CHECKING PARALLEL MISALIGNMENT

Angular Alignment:

Mount the magnetic base on the pump half coupling hub, either the face or O.D. as shown in the sketch. Move the dial indicator button to indicate on the face of the driver half coupling hub as close to the outside diameter as possible. When convenient the indicator



CHECKING ANGULAR MISALIGNMENT

can be placed on the inside face to keep spans short. Turn both shafts 360° and record the dial readings at 90° intervals. Adjust the shims under the motor as required and repeat the procedure until the angular alignment is within 0.0005 mm (T.I.R.) per mm (0.0005 inch per inch) of maximum hub diameter.

Repeat the checks on parallel and angular alignment, ensuring the mounting bolts are secure, until the unit is properly aligned. Note that correction in one direction may affect the alignment in another direction. Re-check the gap between the coupling hubs.

If any difficulty is encountered in achieving the recommended alignment tolerances, the runout of the pump and driver shafts and each coupling hub diameter and face should be checked. Occasionally, due to practical and unavoidable manufacturing tolerance build-up associate with the pump, coupling and driver, it may be necessary to match up the two coupling hubs in the most advantageous relative angular position in order to achieve an acceptable alignment.

Do not connect the couplings alignment is re-checked.

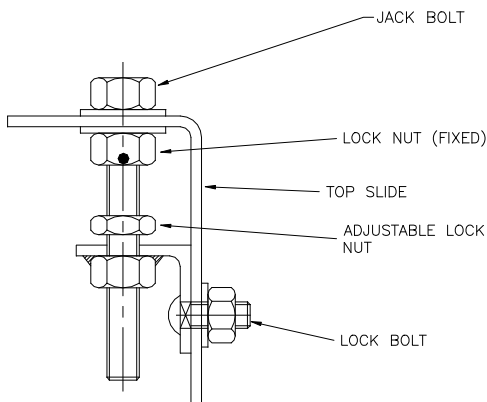
Complete piping as below and see sections 4.7, *Final shaft alignment check* up to and including section 5, *Commissioning, startup, operation and shutdown* before connecting driver and checking actual rotation.

4.5.2.2 V-belt Drive Units:

Check that both sheaves are free of grease, rust, nicks or burrs. Install the correct size sheave on the pump shaft and locate the sheave axially to minimize overhang. Re-check the impeller axial clearance and ensure that the pump is properly secured to the baseplate. Install the driver on the adjustable base provided and install the driver sheave in line with the pump sheave. Ensure that the sheaves are tight on the shafts. With a dial indicator, check the runout on the periphery and face of each sheave to ensure that each is running true. Tighten the adjustable base and check that the driver rotation is in the correct direction and that vibration levels are not unacceptable.

CAUTION Before starting the driver, refer to the manufacturer's instruction manual. The correct rotation of the pump shaft is marked on the pump casing or frame.

Check that all belts making up one drive set have matched code numbers. Loosen the adjustable base and install the belts in their proper grooves. Adjust the center distance between the sheaves to obtain proper belt tension. Check the alignment of the pump and driver sheaves with a taught string or straight edge. For proper alignment the sheave faces must be parallel to each other and in line. Adjustments are made by slackening the belts and moving the sheaves. Retighten the drive or driver sheave then recheck. The procedure may need to be repeated to get alignment as close as possible.



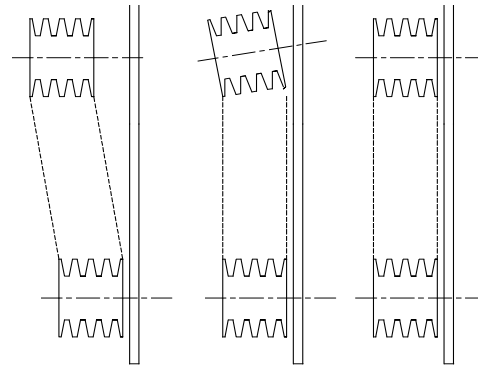
Motor Mount Jacking Bolt Assembly

When the sheaves are aligned that the shafts rotate freely by hand and install safety guard.

Note: Maintaining V-belt tension is the most important rule of v-belt care.

V-belts that are too loose will cause excessive belt wear or breakage.

V-belts are too tight will cause excessive loading on the pump and motor bearings and could cause overheating or reduced bearing lives.



WRONG WRONG RIGHT
CHECKING V-BELT SHEAVE ALIGNMENT



Additional information is available from the sheave and belt supplier.

The motor drive for sheave driven pumps is frequently bolted to a motor support which is constructed of fabricated steel. There are normally 4 adjustment bolts to alter the center distance between the sheaves.

The adjustment procedure is as follows:

- a) Release the lock bolts on each side of the motor stand.
- b) Release the locking nuts on the 4 jacking bolts. (2 on each bolt).
- c) Turn the jacking bolts to move the motor out or in.
- d) To install or remove belts it may be necessary to move the motor 25mm (1") towards the pump frame.
- e) Move the motor away from the pump frame to tighten the belt tension.
- f) Recheck the face of the sheaves and to ensure that the faces are flat and in-line.
- g) Lock the jacking bolts in place with the lock nuts.
- h) Tighten the lock bolts on the side of the motor mount.
- i) Rotate the sheaves by hand to ensure that the pump impeller is free to move.
- j) Install the sheave/belt guard.

4.6 Grouting

The purpose of grouting is to provide rigid support to the pump and driver by increasing the structural rigidity of the top plate and making it an integral mass with the foundation.

Clean the roughed foundation surface and build a wooden form around the soleplate (see Fig. 1). For initial grouting forms should be placed to isolate shims and levelling nuts. The foundation surface should be thoroughly saturated with water before grouting. A typical mixture for grouting-in a pump base is composed of one part pure Portland cement and two parts of clean building sand with sufficient water to provide the proper consistency. The grout should flow freely but not be so wet as to cause the sand and cement to separate.

Thoroughly puddle the grout while pouring to eliminate air pockets and low spots. Pour sufficient grouting to ensure that the bottom surface of the soleplate is completely submerged. Do not fill isolated areas around the shims or levelling nuts. Once the grout has set sufficiently, remove the wooden forms and finish off the sides and top as desired. At the same time, roughen the grout surface inside the soleplate. Cover with wet burlap and allow the grout to cure for at least 40 hours.

After grouting has cured, shims and levelling nuts should be removed or backed off. Tighten down soleplate to the new grout to put bolts in tension and ensure rigidity of structure. Install jam nuts and cut the bolts to the desired length. Finish grouting isolated areas.

4.7 Piping



Protective covers are fitted to the pipe connections to prevent foreign bodies entering during transportation and installation. Ensure that these covers are removed from the pump before connecting any pipes.

4.7.1 Suction and discharge pipework

In order to minimize friction losses and hydraulic noise in the pipework it is good practice to choose pipework that is one or two sizes larger than the pump suction and discharge. Typically main pipework velocities should not exceed 2 m/s (6 ft/sec) suction and 3 m/s (9 ft/sec) on the discharge.

Take into account the available NPSH, which must be higher than the required NPSH of the pump.



Never use the pump as a support for piping.

Maximum forces and moments allowed on the pump flanges vary with the pump size and type. To minimize these forces and moments that may, if excessive, cause misalignment, hot bearings, worn couplings, vibration and the possible failure of the pump casing, the following points should be strictly followed:

- Prevent excessive external pipe load
- Never draw piping into place by applying force to pump flange connections
- Do not mount expansion joints so that their force, due to internal pressure, acts on the pump flange

The table in 4.7.2 summarizes the maximum forces and moments allowed on M pump casings. Refer to Flowserve for other configurations.

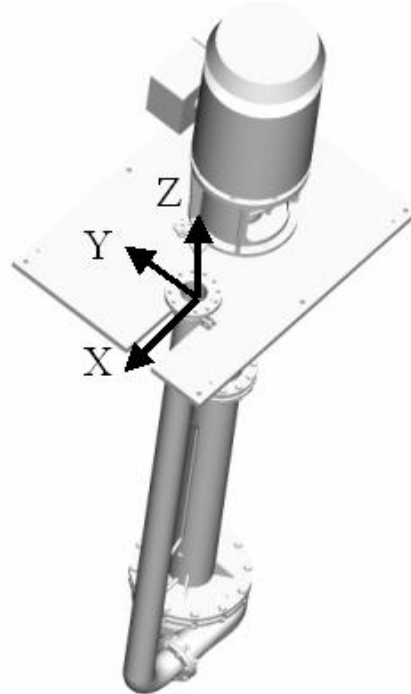


Ensure piping and fittings are flushed before use.



Ensure piping for hazardous liquids is arranged to allow pump flushing before removal of the pump.

4.7.2 Maximum forces and moments allowed on MJ Slurry pump flanges



Flange Size DN (in.)	Maximum forces (F) in N (lbf) and moments (M) in Nm (lbf•ft)					
	Discharge					
	Mx	My	Mz	Fx	Fy	Fz
50 (2)	460 (340)	230 (170)	350 (260)	710 (160)	580 (130)	890 (200)
65 (2-1/2)	710 (520)	350 (260)	540 (400)	890 (200)	760 (170)	1110 (250)
80 (3)	950 (700)	470 (350)	720 (530)	1070 (240)	890 (200)	1330 (300)
100 (4)	1330 (980)	680 (500)	1000 (740)	1420 (320)	1160 (260)	1780 (400)
125 (5)	1830 (1340)	940 (690)	1450 (1070)	1960 (440)	1600 (360)	2450 (550)
150 (6)	2300 (1700)	1180 (870)	1780 (1300)	2490 (560)	2050 (460)	3110 (700)
200 (8)	3530 (2600)	1760 (1300)	2580 (1900)	3780 (850)	3110 (700)	4890 (1100)
250 (10)	5020 (3700)	2440 (1800)	3800 (2800)	5340 (1200)	4450 (1000)	6670 (1500)
300 (12)	6100 (4500)	2980 (2200)	4610 (3400)	6670 (1500)	5340 (1200)	8000 (1800)

Notes:

- 1) F = External force (tension or compression)
M = External moment, clockwise or counter-clockwise
- 2) Forces and moments may be applied simultaneously in any direction.
- 3) Values apply to all materials.
- 4) Higher loads may be applicable, if direction and magnitude of individual loads are known, but these need written approval from Flowserve
- 5) Pumps must be on rigid foundations and baseplates must be fully grouted
- 6) Pump/baseplate should not be used as pipe anchor. Expansion joints must be properly tied.
- 7) The pump mounting bolt torques specified must be used to prevent relative movement between the pump casing and baseplate. (See section 6.6, *Fastener torques*.) The bolt material must have a minimum yield strength of 600 N/mm² (87 000 lb/in.²)

4.7.2 Discharge piping

A non-return valve should be located in the discharge pipework to protect the pump from excessive back pressure and hence reverse rotation when the unit is stopped.

Pipework reducers should have a maximum total angle of divergence of 15 degrees. Fitting an isolation valve will allow easier maintenance.

4.7.3 Auxiliary piping

The water lubricated bearings are fitted with lube lines that end above the top plate. Pipe ends terminate above the topplate and are normally left as NPT male pipe nipples. The ends are capped.

4.7.4 Final checks


Check the tightness of all bolts in the discharge pipework. Check also the tightness of all foundation bolts.


4.8 Final shaft alignment check


After connecting piping to the pump, rotate the shaft several times by hand to ensure there is no binding and all parts are free.

Recheck the coupling alignment, as previously described, to ensure no pipe strain. If pipe strain exists, correct piping.


4.9 Electrical connections


4.9.1  **DANGER** Electrical connections must be made by a qualified electrician in accordance with relevant local, national and international regulations.

4.9.2  It is important to be aware of the EUROPEAN DIRECTIVE on potentially explosive areas where compliance with IEC60079-14 is an additional requirement for making electrical connections.

4.9.3  It is important to be aware of the EUROPEAN DIRECTIVE on electromagnetic compatibility when wiring up and installing equipment on site. Attention must be paid to ensure that the techniques used during wiring/installation do not increase electromagnetic emissions or decrease the electromagnetic immunity of the equipment, wiring or


any connected devices. If in any doubt contact Flowserve for advice.

4.9.4  **DANGER** The motor must be wired up in accordance with the motor manufacturer's instructions (normally supplied within the terminal box) including any temperature, earth leakage, current and other protective devices as appropriate. The identification nameplate should be checked to ensure the power supply is appropriate.


4.9.5  A device to provide emergency stopping must be fitted.

4.9.6 If not supplied pre-wired to the pump unit, the controller/starter electrical details would also be supplied within the controller/starter.

4.8.7 For electrical details on pump sets with controllers see the separate wiring diagram.

4.8.8  **CAUTION** See section 5.3, *Direction of rotation* before connecting the motor to the electrical supply.

4.10 Protection systems

 The following protection systems are recommended particularly if the pump is installed in a potentially explosive area or is handling a hazardous liquid. If in doubt consult Flowserve.

If there is any possibility of the system allowing the pump to run against a closed valve or below minimum continuous safe flow a protection device should be installed to ensure the temperature of the liquid does not rise to an unsafe level.

If there are any circumstances in which the system can allow the pump to run dry, or start up empty, a power monitor should be fitted to stop the pump or prevent it from being started. This is particularly relevant if the pump is handling a flammable liquid.

If leakage of product from the pump or its associated sealing system can cause a hazard it is recommended that an appropriate leakage detection system be installed.

To prevent excessive surface temperatures at bearings it is recommended that temperature or vibration monitoring are carried out. See sections 5.7.4 and 5.7.5.

5 COMMISSIONING, START-UP, OPERATION AND SHUTDOWN



Fully qualified personnel must carry out these operations.

5.1 Pre-commissioning procedure

5.1.1 Lubrication

Determine the mode of lubrication of the pump set, e.g. grease, product lubrication etc.

Grease lubricated pumps and electric motors are supplied pre-greased. Other drivers and gearboxes, if appropriate, should be lubricated in accordance with their manuals.



In the case of product lubricated bearings the source of product supply should be checked against the order. The bearings below the top plate are water lubricated. These will require external clean supply. Commencement of lubrication supply is required before pump start-up.

5.2 Bearing lubricants

5.2.1 Recommended grease lubricants

Grease	Grease nipples	
	NLGI 2 *	NLGI 3 **
Temp. range °C (°F)	-20 to +100 (-4 to +212)	-20 to +100 (-4 to +212)
Designation according to DIN	K2K-20	K2K 30
BP	Energrease LS2	Energrease LS3
DEA	Glissando 20	Glissando 30
Elf	Elfmulti 2	Elfmulti 3
Esso	Beacon 2	Beacon 3
Mobil	Mobilux 2	Mobilux 3
Q8	Rembrandt 2	Rembrandt 3
Shell	Alvania Fett G2 Alvania Fett R2	Alvania R3
Texaco	Multilak 20 Multilak EP2	Multilak 30 Multilak EP3
Wintershall (BASF Group)	Wiolub LFK 2	-
SKF	LGMT 2	LGMT 3
Silkolene	G55/T	G56/T

* NLGI 2 is an alternative grease and is not to be mixed with other grades

** Factory packed bearings for the temperature range with grease nipples

5.2.1.1 Recommended fill quantities

Refer to section 3.4.2, *Pump and impeller data*.



5.2.1.2 Lubrication schedule

When grease nipples are fitted, one charge between grease changes is advisable for most operating conditions, i.e. 1 000 hours interval.

Normal intervals between grease changes are 4 000 hours or at least every 6 months.

The characteristics of the installation and severity of service will determine the frequency of lubrication. Lubricant and bearing temperature analysis can be useful in optimising lubricant change intervals.

The bearing temperature may be allowed to rise to 55 °C (131 °F) above ambient but should not exceed 95 °C (204 °F). For most operating conditions quality grease having a lithium soap base and NLGI consistency of No 2 or No 3 is recommended. The drop point should exceed 175 °C (350 °F).



Never mix greases containing different bases, thickeners or additives.

5.2.2 Water Lubricated Bearings

Bearings must be thoroughly wetted before operation. In most cases the lower bearing will be submerged but it is essential that a flush of clean liquid be supplied.

5.2.2.1 Water Supply Pressure - 30-40 psig

5.2.2.2 Temperature – Supply temperature should not exceed 38C (100F)

5.2.2.3 Flush Rates

	Lower Bearing	Intermediate Bearing (each)
Frame 1	0.5 m ³ /hr (2 USGPM)	0.7 m ³ /hr (3 USGPM)
Frame 2	0.7 m ³ /hr (3 USGPM)	0.9 m ³ /hr (4 USGPM)
Frame 3	0.9 m ³ /hr (4 USGPM)	1.1 m ³ /hr (5 USGPM)

5.3 Direction of rotation



Ensure the pump is given the same rotation as the pump direction arrow cast on the pump casing. Rotation is clockwise when the pump is viewed from the driver. The impeller may unscrew if run in reverse rotation.

On shut down, the impeller must be free to rotate. A racket type motor must not be used.

Minimum submergence is recommended to avoid vortexing and entrainment of air. Vortexing will result in increased vibration, loss of performance and reduced bearing lives.

If maintenance work has been carried out to the site's electricity supply, the direction of rotation should be re-checked as above in case the supply phasing has been altered.

5.4 Guarding

Guarding is supplied fitted to the pump set. If this has been removed or disturbed ensure that all the protective guards around the pump coupling and exposed parts of the shaft are securely fixed.

5.5 Priming and auxiliary supplies

Ensure all electrical, hydraulic, pneumatic, sealant and lubrication systems (as applicable) are connected and operational.

Ensure the inlet pipe and pump casing are completely full of liquid before starting continuous duty operation. Pump fluid must be above the pump casing.

5.6 Starting the pump

- a) Ensure flushing and/or cooling liquid supplies are turned ON before starting the pump.
- b) CLOSE the outlet valve.
- c) Ensure all vent connections are closed before starting.
- d) Start motor and check outlet pressure.
- e) If the pressure is satisfactory, slowly OPEN outlet control valve.
- f) Do not run the pump with the outlet valve closed for a period longer than 60 seconds.
- g) If NO pressure, or LOW pressure, STOP the pump. Refer to section 7, *Faults; causes and remedies*, for fault diagnosis.

5.7 Running the pump

5.7.1 Venting the pump

Vent the pump to enable all trapped air to escape taking due care with hot or hazardous liquids.

Under normal operating conditions, after the pump has been fully primed and vented, it should be unnecessary to re-vent the pump.

5.7.4 Bearings

If the pumps are working in a potentially explosive atmosphere temperature or vibration monitoring at the bearings is recommended

If bearing temperatures are to be monitored it is essential that a benchmark temperature is recorded at the commissioning stage and after the bearing temperature has stabilized. Record the bearing temperature (t) and the ambient temperature (ta). Estimate the likely maximum ambient temperature (tb). Set the alarm at $(t+tb-ta+5)^{\circ}\text{C}$ [$(t+tb-ta+10)^{\circ}\text{F}$] and the trip 105°C (220°F) for grease lubrication.

It is important, particularly with grease lubrication, to keep a check on bearing temperatures. After start up the temperature rise should be gradual, reaching a maximum after approximately 1.5 to 2 hours. This temperature rise should then remain constant or marginally reduce with time. Refer to section 6.2.3.1 for further information.

5.7.5 Normal vibration levels, alarm and trip

For guidance, pumps generally fall under a classification for rigid support machines within the International rotating machinery standards and the recommended maximum levels below are based on those standards.

Alarm and trip values for installed pumps should be based on the actual measurements (N) taken on the pump in the fully commissioned as new condition. Measuring vibration at regular intervals will then show any deterioration in pump or system operating conditions.

Vibration velocity – unfiltered mm/s (in./s) r.m.s.	MJ
Normal N	≤ 7.1 (0.28)
Alarm N x 1.25	≤ 9.0 (0.35)
Shutdown trip N x 2.0	≤ 14.2 (0.56)



5.7.6 Stop/start frequency

Pump sets are normally suitable for the number of equally spaced stop/starts per hour shown in the table below. Check actual capability of the driver and control/starting system before commissioning.

Motor rating kW (hp)	Maximum stop/starts per hour
Up to 15 (20)	15
Between 15 (20) and 90 (120)	10
Above 90 (120)	6

Where duty and standby pumps are installed it is recommended that they be run alternately every week.

5.8 Stopping and shutdown

- a)  Close the outlet valve, but ensure that the pump runs in this condition for no more than a few seconds.
- b) Stop the pump.
- c) Switch off flushing and/or cooling/heating liquid supplies at a time appropriate to the process.
- d)  For prolonged shut-downs and especially when ambient temperatures are likely to drop below freezing point, the pump and any cooling and flushing arrangements must be drained or otherwise protected.

5.9 Hydraulic, mechanical and electrical duty

This product has been supplied to meet the performance specifications of your purchase order, however it is understood that during the life of the product these may change. The following notes may help the user decide how to evaluate the implications of any change. If in doubt contact your nearest Flowserve office.

5.9.1 Specific gravity (SG)

Pump capacity and total head in metres (feet) do not change with SG, however pressure displayed on a pressure gauge is directly proportional to SG. Power absorbed is also directly proportional to SG. It is therefore important to check that any change in SG will not overload the pump driver or over-pressurize the pump.

5.9.2 Viscosity

For a given flow rate the total head reduces with increased viscosity and increases with reduced viscosity. Also for a given flow rate the power absorbed increases with increased viscosity, and reduces with reduced viscosity. It is important that checks are made with your nearest Flowserve office if changes in viscosity are planned.

5.9.3 Pump speed

Changing pump speed effects flow, total head, power absorbed, NPSH_R, noise and vibration. Flow varies in direct proportion to pump speed, head varies as speed ratio squared and power varies as speed ratio

cubed. The new duty, however, will also be dependent on the system curve. If increasing the speed, it is important therefore to ensure the maximum pump working pressure is not exceeded, the driver is not overloaded, NPSH_A > NPSH_R, and that noise and vibration are within local requirements and regulations.

5.9.4 Net positive suction head (NPSH_A)

NPSH available (NPSH_A) is a measure of the head available in the pumped liquid, above its vapour pressure, at the pump suction branch.

NPSH required (NPSH_R) is a measure of the head required in the pumped liquid, above its vapour pressure, to prevent the pump from cavitating. It is important that NPSH_A > NPSH_R. The margin between NPSH_A > NPSH_R should be as large as possible.


If any change in NPSH_A is proposed, ensure these margins are not significantly eroded. Refer to the pump performance curve to determine exact requirements particularly if flow has changed. If in doubt please consult your nearest Flowserve office for advice and details of the minimum allowable margin for your application.

5.9.5 Pumped flow

Flow must not fall outside the minimum and maximum continuous safe flow shown on the pump performance curve and/or data sheet.

6 MAINTENANCE

6.1 General

 It is the plant operator's responsibility to ensure that all maintenance, authorized and qualified personnel who have adequately familiarized themselves with the subject matter by studying this manual in detail carry out inspection and assembly work. (See also section 1.6.2.)

Any work on the machine must be performed when it is at a standstill. It is imperative that the procedure for shutting down the machine is followed, as described in section 5.8.

On completion of work all guards and safety devices must be re-installed and made operative again.

Before restarting the machine, the relevant instructions listed in section 5, *Commissioning, start up, operation and shut down* must be observed.

Oil and grease leaks may make the ground slippery. Machine maintenance must always begin and finish by cleaning the ground and the exterior of the machine.

If platforms, stairs and guard rails are required for maintenance, they must be placed for easy access to areas where maintenance and inspection are to be carried out. The positioning of these accessories must not limit access or hinder the lifting of the part to be serviced.

When air or compressed inert gas is used in the maintenance process, the operator and anyone in the vicinity must be careful and have the appropriate protection.

Do not spray air or compressed inert gas on skin.

Do not direct an air or gas jet towards other people.

Never use air or compressed inert gas to clean clothes.

Before working on the pump, take measures to prevent an uncontrolled start. Put a warning board on the starting device with the words:

"Machine under repair: do not start".

With electric drive equipment, lock the main switch open and withdraw any fuses. Put a warning board on the fuse box or main switch with the words:

"Machine under repair: do not connect".

Never clean equipment with inflammable solvents or carbon tetrachloride. Protect yourself against toxic fumes when using cleaning agents.

6.2 Maintenance schedule



It is recommended that a maintenance plan and schedule is adopted, in line with these User Instructions, to include the following:

- Any auxiliary systems installed must be monitored, if necessary, to ensure they function correctly.
- Gland packings must be adjusted correctly to give visible leakage and concentric alignment of the gland follower to prevent excessive temperature of the packing or follower.
- Check for any leaks from gaskets and seals. The correct functioning of the shaft seal must be checked regularly.
- Check bearing lubricant level, and if the hours run show a lubricant change is required.

- Check that the duty condition is in the safe operating range for the pump.
- Check vibration, noise level and surface temperature at the bearings to confirm satisfactory operation.
- Check dirt and dust is removed from areas around close clearances, bearing housings and motors.
- Check coupling alignment and re-align if necessary.

Our specialist service personnel can help with preventative maintenance records and provide condition monitoring for temperature and vibration to identify the onset of potential problems.

If any problems are found the following sequence of actions should take place:

- Refer to section 7, *Faults; causes and remedies*, for fault diagnosis.
- Ensure equipment complies with the recommendations in this manual.
- Contact Flowserve if the problem persists.

6.2.1 Routine inspection (daily/weekly)



The following checks should be made and the appropriate action taken to remedy any deviations:

- Check operating behaviour. Ensure noise, vibration and bearing temperatures are normal.
- Check that there are no abnormal fluid or lubricant leaks (static and dynamic seals) and that any sealant systems (if fitted) are full and operating normally.
- Check that shaft seal leaks are within acceptable limits.
- Check the level and condition of oil lubricant. On grease lubricated pumps, check running hours since last recharge of grease or complete grease change.
- Check any auxiliary supplies e.g. heating/cooling (if fitted) are functioning correctly.



Refer to the manuals of any associated equipment for routine checks needed.

6.2.2 Periodic inspection (six monthly)



- Check foundation bolts for security of attachment and corrosion.
- Check pump-running records for hourly usage to determine if bearing lubricant requires changing.
- The coupling should be checked for correct alignment and worn driving elements.



Refer to the manuals of any associated equipment for periodic checks needed.

6.2.3 Re-lubrication

Lubricant and bearing temperature analysis can be useful in optimizing lubricant change intervals. In general however, the following is recommended.

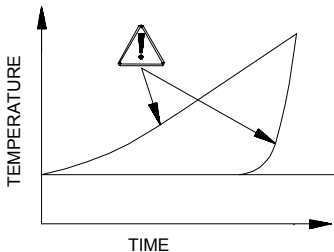


See section 5.2.2 for grease recommendations.

Regrease - via grease nipples every 1 000 hours or sooner depending on the severity of the application.

It is important not to under or over grease the bearings as this will lead to over heating and premature failure.

- Grease lubricated bearing housings have grease nipples fitted in the bearing covers.
- Move the axial seal ring back so the gap between the pump shaft and bearing cover can be seen.
- Connect grease gun to the nipple.
- Press grease into the bearing housing until the first signs of it appear in the gap between the housing and shaft, then stop greasing.
- The maximum allowable operating temperatures for anti friction bearings will vary from unit to unit, depending on ambient and fluid temperature. The rise above ambient should not normally exceed 55 °C (131 °F) or a combined maximum of 95 °C (204 °F).
- A continuously rising temperature or an abrupt temperature rise indicates a problem. If these symptoms occur, stop the pump immediately and investigate the cause.



Grease change - every 4 000 hours or sooner depending on the severity of the application.

- Remove the bearing housing from the rotor assembly.
- Brush the bearing housing with hot kerosene (100 to 115 °C/212 to 240 °F) or other non-toxic solvent.

- Clean and flush out the housing with a light mineral oil.
- Do not use waste oil to clean the housing.

To clean the bearings:

- Wipe off as much grease as possible with a clean lint-free cloth.
- Brush bearings with hot kerosene (80 to 90 °C/ 175 to 195 °F) while gently spinning the outer bearing ring.
- Spin each ball to ensure that it is clean.

To remove badly oxidized grease, which refuses to come off:

- Support the rotor in a vertical position and immerse the bearing in hot kerosene or a mixture of alcohol and light mineral solvent.
- Gently spin the bearing outer ring.
- Dry and reflush the bearing with clean light oil.
- It is important not to under or over grease the bearings as this will lead to over heating and premature failure. It is recommended that the bearings be filled with grease using a suitable spatula. In addition the housings should be no more than half filled.

6.3 Spare parts

6.3.1 Ordering of spares

Flowserve keep records of all pumps that have been supplied. When ordering spares the following information should be quoted:

- Pump serial number
- Pump size
- Part name – taken from section 8
- Part number – taken from section 8
- Number of parts required

The pump size and serial number are shown on the pump nameplate.

To ensure continued satisfactory operation, replacement parts to the original design specification should be obtained from Flowserve.

Any change to the original design specification (modification or use of a non-standard part) will invalidate the pump's safety certification.

6.3.2 Storage of spares

Spares should be stored in a clean dry area away from vibration. Inspection and re-treatment of metallic surfaces (if necessary) with preservative is recommended at 6 monthly intervals.

6.4 Recommended spares and consumables

For start up purposes:

- 1 - complete set of gland packing
- 2 - shaft sleeves
- 1 - set of gaskets and seals
- (optional: 2 - mechanical seals)

For 2 years operation:

- 1 - set of bearings (line and thrust)
- 2 - sets of gland packing
- 2 - shaft sleeves
- 2 - sets of gaskets and seals
- 2 - lantern rings
- 2 - casing wear rings
- (optional: 2 - mechanical seals
- 2 - impeller wear rings)

For 4 years operation:

- 1 - set of bearings (line and thrust)
- 2 - sets of gland packing
- 2 - shaft sleeves
- 2 - sets of gaskets and seals
- 2 - lantern rings
- 2 - casing wear rings
- 1 - impeller
- (optional: 2 - mechanical seals
- 2 - impeller wear rings)

6.5 Tools required

A typical range of tools that will be required to maintain these pumps is listed below.

Readily available in standard tool kits, and dependent on pump size:

- Open ended spanners (wrenches) to suit up to M 48 screws/nuts
- Socket spanners (wrenches), up to M 48 screws
- Allen keys, up to 10 mm (A/F)
- Range of screwdrivers
- Soft mallet

More specialized equipment:

- Bearing pullers
- Bearing induction heater
- Dial test indicator
- C-spanner (wrench) - for removing shaft nut. (If difficulties in sourcing are encountered, consult Flowserve.)

6.6 Fastener torques

Bolt size	Torque Nm (lb•ft)	
	Pump feet fasteners	All other fasteners
M 16 (5/8 in.)	170 (125)	84 (62)
M 20 (3/4 in.)	340 (250)	165 (120)
M 24 (7/8 in.)	590 (435)	285 (210)
M 27 (1 in.)	770 (570)	375 (275)
M 30 (1 1/8 in.)	1 100 (810)	540 (400)
M 36 (1 3/8 in.)	1 840 (1 350)	900 (660)
M 42 (1 7/8 in.)	2 000 (1 475)	1 410 (1 040)
M 48 (1 7/8 in.)	2 240 (1 650)	2 060 (1 500)

M 16 (5/8 in.)	170 (125)	84 (62)
M 20 (3/4 in.)	340 (250)	165 (120)
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M 48 (1 7/8 in.)	2 240 (1 650)	2 060 (1 500)

TIGHTENING TORQUE FOR STAINLESS STEEL STUDS WITH LUBRICATED THREADS		
THREAD SIZE	TIGHTENING TORQUE	
	Nm.	Ft. lbs.
M10x1.5 (3/8-16UNC)	13	10
M12x1.75 (1/2-13UNC)	27	20
M16x2 (5/8-11UNC)	60	45
M20x2.5 (3/4-10UNC)	100	75

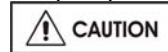
6.7 Renewal clearances

As wear takes place between the impeller and casing ring the overall efficiency of the pump set will decrease. To maintain optimum efficiency it is recommended that rings are replaced and the impeller renovated when the radial clearance detailed in section 3.4.2 has doubled to 0.6 to 0.8 mm (0.024 to 0.032 in.), depending on pump size. On FRBH it is recommended that the product lubricated bearing be renewed at a diametrical clearance of 0.5 mm (0.04 in.).

6.8 Disassembly



Refer to section 1.6, *Safety*, before dismantling the pump.



Before dismantling the pump for overhaul, ensure genuine Flowserve replacement parts are available.

To disassemble the pump consult the sectional drawings, see section 8, *Parts list and drawings*.

6.8.1 Rotating Element

The following procedure is recommended for removing and disassembling the rotating assembly.



Note that the replacement gaskets and similar consumable materials should be available since they will be required for reassembly.

- a) Isolate motor and lock off electrical supply in accordance with local regulations.
- b) Isolate suction and discharge valves.
- c) Remove guards

- d) Disconnect the coupling halves or remove belts from sheaves.
- e) Disconnect water lubrication lines for lower bearings.
- f) Disconnect pump from piping and remove spool pieces as necessary.
- g) Unbolt the top plate [6140] from the foundation and remove pump from well.

Note:

For most pumps it is recommended that the motor be removed.

The remaining steps assume that the pump has been removed.



NEVER APPLY HEAT TO REMOVE IMPELLER
Trapped lubricant or vapour could cause an explosion.

Heat could also cause damage to the impeller.

- a) Place the pump in a horizontal position. Support the casing and column pipe.
- b) Disconnect discharge pipe from the casing. The pipe clamp at the top plate may be removed to allow complete removal of the discharge pipe.
- c) Remove drop pipes or suction strainer from casing.
- d) Unbolt the casing from column pipe and remove.
- e) The impeller is threaded to the shaft and has been self tightened to the shaft sleeve. To unscrew the impeller the torque must be broken. It will be necessary to either block the impeller or shaft at the drive-end. To unscrew the impeller the shaft must be turned counter-clockwise.
- f) On small pumps the impeller can be removed and held by hand however, larger units will require that a hoist be used to lift the impeller. Place a sling through the impeller vanes. Remove the impeller [2200].
- g) Remove impeller spacer and gaskets the shaft.
- h) Remove flush piping from and pipe nipple from lower housing [3200].
- i) Remove lower bearing housing [3200] from the column pipe.

6.8.3 Lower bearing

- a) The cutlass rubber bearing is secured in place with 2 radial set screws. Locate and remove.
- b) Press the bearing from the housing.

6.8.4 Column Pipe and Shaft

- a) Disconnect the column pipe from the top plate or intermediate pipe support [1360] and remove.

6.8.4.1 2 Pc Shafts

- a) If the shaft is a 2 pc construction it will be necessary to uncouple the 2 shafts.

6.8.4.1.1 Frames 1& 2

- a) The coupling is threaded unto both shafts. Turn the lower shaft clockwise while holding the coupling.
- b) Remove the coupling from the upper shaft, hold the shaft in position and turn the coupling clockwise.

6.8.4.1.2 Frame 3

Note:

Many older frame 3 assemblies may use a coupling similar to frame 1 & 2. The current design for frame 3 uses a clamp type coupling. This design is distinguishable by the retaining rings on the shaft at each end of the coupling.

- a) Remove the retaining rings.
- b) Slide the coupling sleeve onto the lower shaft. Note that the coupling is keyed to both shafts.
- c) The shafts are locked together with a split collar that can be removed once the coupling sleeve is removed.
- d) Place the lower shaft on a set of v-blocks
- e) Remove coupling key [6700] from the upper shaft.

6.8.4.2 Intermediate bearing

- a) Remove the intermediate bearing housing from the upper column pipe, where applicable.
- b) Remove the radial setscrews from the housing.
- c) Press the bearing from the housing.

6.8.4.3 Upper column pipe

- c) Unbolt the upper column pipe from the motor stool (direct drive) or bearing frame (MVD drive).
Remove the upper column pipe.

6.8.4.4 Shaft removal

6.8.4.4.1 Direct Drive

- d) Remove adjusting hardware from the thrust bearing housing.
- e) Slide the thrust bearing housing and shaft assembly from the motor stool.
- f) Lift and place onto a v-block to support.

6.8.4.4.2 MVD Drive

- a) Unfasten the line bearing cover [3260.1] from the bearing frame [3130].
- b) Slide the cover over the shaft and remove.
- c) Remove adjusting hardware from the thrust bearing housing.
- d) Slide the thrust bearing housing and shaft assembly from the bearing frame.
- e) Lift and place horizontally onto a v-block to support.

6.8.5 Thrust bearings

- a) Remove the thrust bearing cover [3260.2] from the thrust bearing housing [3240].
- b) Remove the thrust bearing housing [3240] from bearings.

- c) Bend up the locking tab on the bearing lockwasher [6541] and remove the bearing locknut [3712] and lockwasher [6541].
- d) Only if necessary remove the thrust bearings [3031] from the shaft. Bearings removed and reused can easily be damaged and undetected until pump is put back in operation.
- e) Remove the thrust bearing housing [3240] from the shaft.

6.8.6 Line bearing – MVD units

- a) Remove the bearing retaining ring.
- b) Only if necessary remove the line bearing [3011] from the shaft. Bearings removed can easily be damaged and undetected until pump is put back in operation.
- c) The shaft sleeve is held in place with an anaerobic sealant. To remove heat bearing to break bond or cut loose from making an axial cut and prying the sleeve off.

6.8.7 Gland side Wearplates

- a) The wearplate [1915.2] can be removed from the lower housing [3200] if required. The wearplate is secured to the stuffing box head [3200] with studs, nuts and washers.

6.8.8 Suction side wearplates

- a) The wearplate [1915.1] can be removed from the casing [1110] if required. The wearplate is secured to the casing [1110] with studs, nuts and washers.
- b) To remove the wearplate it may be necessary to turn the casing upside down and lever the end of the mounting studs. To prevent damaging the studs nut can be threaded on and left loose.

6.9 Examination of parts



Used parts must be inspected before assembly to ensure the pump will subsequently run properly.

In particular, fault diagnosis is essential to enhance pump and plant reliability.

6.9.1 Casing, seal housing and impeller

- a) Inspect for excessive wear, pitting, corrosion, erosion or damage and any sealing surface irregularities.
- b) Replace as necessary.
- c) Inspect the impeller [2200] and the wearplates [1915.1] and [1915.2] for excessive wear or damage. Remove the wearplate from the casing [1110] if necessary.
- d) Inspect the casing [1110] and lower bearing housing [3200] for damage or excessive thinning of wall sections due to wear or corrosion. Clean the

- e) internal surfaces to maintain pump efficiency.
- e) Inspect the protector plate [4132] and impeller spacer [2460] and remove if damaged or worn. Clean the internal bore of the stuffing box.

6.9.2 Shaft and sleeve

- d) Replace sleeves if grooved, pitted or worn. The shaft sleeves are held in place with an anaerobic sealant or interference fit. To remove heat bearing to break bond or cut loose from making an axial cut and prying the sleeve off.
- a) Clean the shaft and inspect for evidence of corrosion, evidence of cracking, fatigue or mechanical damage. Remove all burrs or nicks paying particular attention to the areas under the lip seals. Check that the shaft is straight within 0.002 inch (0.050 mm).

6.9.3 Gaskets and O-rings

After dismantling, discard and replace.

6.9.4 Bearings

- a) It is recommended that bearings are not re-used after any removal from the shaft.
- b) The plain liquid lubricated bearings may be re-used if both the bearing bush and bearing sleeve show no sign of wear, grooving or corrosion attack. (It is recommended that both the bush and sleeve be replaced at the same time.)

6.9.5 Bearing isolators, labyrinths or lip seals (if fitted)

- a) The lubricant, bearings and bearing housing seals are to be inspected for contamination and damage. If oil bath lubrication is utilised, these provide useful information on operating conditions within the bearing housing.
- b) If bearing damage is not due to normal wear and the lubricant contains adverse contaminants, the cause should be corrected before the pump is returned to service.
- c) Bearing seals are not totally leak free devices. Oil from these may cause staining adjacent to the bearings. Inspect and replace if necessary.

6.10 Assembly

To assemble the pump consult the sectional drawings, see section 8, *Parts list and drawings*.

Ensure threads, gasket and O-ring mating faces are clean. Apply thread sealant to non-face sealing pipe thread fittings.

6.10.1 Line Bearing – MVD Units

- a) Install the shaft sleeve [3400] onto the, securing in place with Loctite A (anaerobic sealant). Ensure sleeve is set against the shaft shoulder.

- b) Lightly lubricate the shaft [2110] at the line bearing position. Install the line bearing [3011] on the shaft. Use an induction heated or hot oil bath to first heat the bearing [250° F recommended], press the bearing on shaft with the aid of a sleeve designed to push the inner race only. Note that the bearing must remain square to the shaft during assembly and that the inner race must seat on the sleeve [3400] shoulder.
- c) Install retaining ring [2530].
- d) Allow bearing to cool.
- e) Protect the bearing by wrapping with a clean, lint free cloth.
- f) Pack the line bearing with grease if the bearings are being grease lubricated.

6.10.2 Thrust Bearing

- a) Install lipseal in thrust bearing housing [3240] (Direct Drive units only).
- b) Slip the housing [3240] over the end of the shaft [2110] with the opening towards the drive end.
- c) Install the angular contact bearings [3031] on the shaft [2110] using the same procedure as described in step 6.10.1 . The bearings are mounted back to back as shown.
- d) Slide the bearing lockwashers [6541] on the shaft and fit the bearing locknut [3712]. Tighten the locknut snugly and allow the bearings to cool. Check the tightness and bend one tab on the lockwasher into a slot in the locknut.
- e) Protect the bearing by wrapping with a clean, lint free cloth.

6.10.3 Packing the bearings

- a) Pack thrust bearings with grease.
- b) Pack the line bearing with grease.
- c) Slide the thrust bearing housing [3240] over the bearings [3031].
- d) Carefully install the lip seal [4300.2] in the thrust bearing cover [3260] by pressing it squarely into the bore. The primary sealing lip [spring loaded] on seal should be installed facing towards the coupling.
- e) Install the grease fitting on the tapped hole in the thrust bearing cover[3260].
- f) Assemble the cover [3260] over the shaft and fit to the housing [3240]. Care must be taken to prevent damage of the seal on the shaft.
- g) Using capscrews and lockwashers, attach the thrust bearing cover [3260] to the thrust bearing housing [3240]. Lock the threads using Loctite 242 or equivalent.

Tighten the capscrews evenly ensuring that the clamp ring is not distorted and gap to the bearing housing is even all around. Tighten in accordance with Table 6.6.

6.10.4 Frame assembly

- a) Place the top plate on edge. Orientate the discharge pipe slot vertically. Secure in place with angle supports.
- b) Lift the bearing cartridge [3130] (MVD units) or the motor stool [3160] (Direct Drive units). Secure in position.
- c) Lift the shaft assembly and install into the bearing frame or motor stool.
- d) Orientate the thrust bearing housing adjusting lug holes with those in the bearing frame or motor stool.
- e) Install the adjusting hardware. Set the thrust bearing housing flange about 12mm (1/2in.) from the mating face of the bearing frame or motor stool.
- f) Pack the top of the bearing and inside of the bearing housing

Note:

The grease fitting in the thrust bearing cover [3260] must align with one of the openings in the motor stool.

- g) Install the coupling key [6700] and tape to the shaft.

6.10.5 Line Bearing Cover-MVD units

- h) Carefully install the lip seal [4300.1] in the line bearing cover [3260.1] by pressing it squarely into the bore. The primary sealing lip [spring loaded] on seal should be installed facing away from the bearing.
- i) Pack the lower part of the bearing cavity with grease.
- j) Assemble the line bearing cover [3260.1] over the shaft and squarely into the bearing frame bore. Fasten to the bearing frame with capscrews, washers and hex nuts. Tighten firmly, but not excessively.
- k) Lift and install the column pipe [1341] over the shaft and to the motor stool or bearing frame.

6.10.6 Intermediate Bearings

- a) Install the intermediate bearing (when appropriate) into the bearing housing. There may be a light press fit. Orientate the holes in the bearing 90 degrees to the flush hole in the bearing housing.
- b) Secure in place with 2 radial screws.
- c) Lift and install the intermediate bearing housing over the shaft and into the column pipe locating fit. Secure in place.

Note:

Orientate the shorter portion of the bearing housing towards the top plate. The flush port should be orientated offset to the left when looking towards the top plate.

6.10.6.1 Intermediate Shaft Sleeve

- a) The shaft sleeve has a loose fit to the shaft. Install the sleeve using Loctite 680 and orientate the end of

the sleeve such that it protrudes about 19 mm (0.75) below the bearing.

6.10.7 Multiple Intermediate Bearings

When there are multiple intermediate bearings there will be 2 pc shafts. These will be coupled together normally with a threaded type coupling. The direction of drive will prevent coupling from coming loose.

6.10.7.1 2 pc Shafts

- a) Inspect the end of the shafts and remove all dirt and debris. Ensure that there are no burrs that could prevent the shafts from seating against each other squarely.
- b) Thread the coupling onto the upper shaft.
- c) Lift the lower shaft and align with the coupling and screw into the coupling.

Note:

The end of the shafts must seat against each other squarely to ensure that bearings and impeller are properly aligned.

- d) Install the intermediate bearing [3044] into the bearing housing [3240], lock into place using stainless steel setscrews.
- e) Slide the housing onto shaft and position into the support pipe spigot. The housing should align to the top plate flush ports.
- f) Slide the next section of support pipe over the shaft and bolt into position. Secure bolts with Loctite A. Tighten in accordance with table 2 for steel bolts or table 3 for stainless steel bolts.

Repeat steps for addition intermediate bearings.

6.10.8 Lower Column Pipe

- a) The orientation of the lower pipe must be such that the access hole for the lower bearing [3000] flush water is in alignment with the top plate.
- b) Install the lower shaft sleeve onto the shaft. The sleeve must be seated with Loctite A or 680.
- c) Assemble the lower bearing [3000] into the lower housing [3240] Align the flush port in the seal 90 degrees to the ports in the housing. Secure into position using setscrews.
- d) Slide the bearing housing [3240] over the shaft and align the flushing port to the support pipe.
- e) Install the sleeve stop [2460] on to the shaft [2110] and seat to the shaft sleeve.

6.10.9 Lower Bearing Housing

- a) Install studs in the gland side wear plate [1915.2] and secure with loctite 242.
- b) With gland side wear plates positioned with the studs facing up; lift the lower seal housing [3200] and lower onto the wear plate. Place the seal

washers [2905] over the studs followed by the steel washers. Apply Loctite 242 or equivalent to nut threads and fasten wear plate in position.

Torque nuts per table.

Assemble the lower bearing housing [3200] over the shaft sleeve [2445] and orientate the flush port with the mating hole in the column pipe to suit auxiliary piping.

- d) Seat the housing against the column pipe. (Temporarily clamp the housing in place.)

6.10.10 Impeller Installation

- a) Install the impeller spacer [2460] on standard pumps only and install the impeller gasket [4590.2] on all pumps.
- b) Due to the method of assembling the impeller, it is recommended to smear the sleeve face and both sides of the impeller spacer [2460] with heavy grease or a silicon sealant before assembling onto the shaft, which will help to hold the impeller spacer in position during assembly.

- c) Apply anti-seize compound to the shaft threads and screw the impeller onto the shaft. For large sizes it is more practical to hold impeller using a sling and hoist and turn the shaft.



Failure to tighten the impeller [2200] spacer [2460] securely against the sleeve may cause serious damage to pump components.

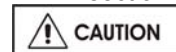
Note:

The thread is designed so that during operation the impeller will tend to tighten onto shaft. Therefore a clockwise shaft rotation (looking from coupling end) will screw impeller on.

- d) Release the thrust bearing housing hold down bolts and tighten the jacking screws to pull the impeller back to the stuffing box head [3200]. The stuffing box head must not be rigidly clamped.

6.10.11 Suction Side Wearplate

- a) Install the studs [6572] in the wearplate [1915.1] using Loctite grade A or equivalent.
- b) Lift wearplate and install into casing.
- c) Place the sealing washer [2905] over the studs [6572]. Secure wearplate [1915.1] in place using hex nuts and washers using Loctite 242 or equivalent. Tighten in accordance with Table tables in section 6.6.



Excessive or uneven tightening torque may distort wearplate affecting impeller running clearances or result in broken studs.

6.10.12 Casing

- a) Smear a small amount of grease or anti-seize compound over one face of the gasket [4590] and place it on the stuffing box head [3200] with coated face against flange.
- b) Lift the casing and assemble to the pedestal. The discharge may be orientated to various configurations. Check the installation or GA drawing for the proper position.
- c) Secure into position. Ensure that the impeller is free to rotate before tightening bolts.
- d) Set the impeller front clearance in accordance with instruction earlier in this section of the manual.

MINIMUM IMPELLER NUT TIGHTENING TORQUE		
FRAME	Ft. lbs.	Nm.
1	100	140
2	300	400
3	300	400
4	550	750

- a) Assemble the mechanical seal gland plate and gasket and fasten using gland studs [6572]. Secure with nuts [6521] and tighten each by hand. Further tighten the nuts in accordance with Table in 6.6. Rotate the shaft to ensure that it turns freely without rubbing or binding.
- b) Re-assemble the rotating element into the casing. Do not adjust the thrust bearing housing.
- c) Set the deflector [2540] at the line bearing cover [3260.1] so that they do not contact when the shaft is rotated. Lock in place with the setscrews provided.

6.11 Impeller axial clearance adjustment



DANGER NEVER ATTEMPT TO CHANGE THE CLEARANCE WHEN THE PUMP IS RUNNING.

If the coupling has limited axial adjustment capability, the pump and driver must be uncoupled prior to adjusting the clearance in order to permit free movement.

- a) Loosen the two setscrews retaining the deflector [2540] and check that the deflector is free to move axially on the shaft.
- b) Loosen the thrust bearing housing jam nuts and back off the three jacking screws at least 1.5 mm (0.060 inch).
- c) Move rotor towards the wear plate [1915.1] by

tightening the three hold-down capscrews evenly and uniformly until the impeller [2200] just touches the wear plate. Rotating the shaft and stopping the forward motion at the first sign of rubbing can best establish this. If the shaft cannot be rotated, back off the bearing housing with the jacking screws until a just detectable rub is obtained. Check that the gap between the two machined faces of thrust bearing housing [3240] and the bearing frame [3130] are parallel within 0.003 inch (0.076 mm). Adjust the jackscrews and hold down capscrews as required to achieve this parallelism. When impeller [2200] just touches wear plate [1915.1] and thrust bearing housing [3240] is parallel to the bearing frame [3130] the axial clearance between the impeller and wear plate is zero.

6.11.1 Option1

- a) Place a dial indicator, set to end of shaft [1110] or on housing [3240] face.
- b) Set indicator reading to zero (0).
- c) Note required impeller clearance.

6.11.2 Option 2

- a) Measure and record the axial gap between the thrust bearing housing flange and bearing frame end face. Determine the required impeller axial running clearance from Section I and add this to the above measurement to establish the required gap setting.

6.11.3

- a) Loosen the thrust bearing housing hold down capscrews slightly and tighten the jackscrews. Until the required dial indicator reading (6.11.1) or housing gap reading (6.11.2) is achieved.
- b) Alternately and gradually tighten the hold down capscrews and jackscrews until the required gap setting is achieved at each hold down capscrew location. Note that the gap at each jackscrew will be slightly larger as a result of minor elastic distortion of the thrust bearing housing flange caused by the high pre-load forces. The gap setting at any set of screws must be the same within 0.003 inch [0.076 mm]. Careful attention to this procedure will help ensure long thrust bearing life.
- c) While preventing the jackscrews from rotating, tighten the jam nuts to lock them in position.
- d) Adjust the axial position of the deflector [2540] so that it is clear of the line bearing cover [3260.1] by approximately 0.030 inch (0.75 mm) and tighten the setscrews firmly. Excessive tightening may mar the shaft.
- e) Manually rotate the shaft to ensure that there is no rubbing or binding.
- f) On belt driven units, adjust the pump or driver sheave to maintain belt alignment. (Refer to

- Section 4.5.2.2).
 g) Check the alignment on direct driven units (refer to Section 4) and reassemble the coupling components.

- h) Replace any safety guards, which may have been removed.

7 FAULTS; CAUSES AND REMEDIES

FAULT SYMPTOM

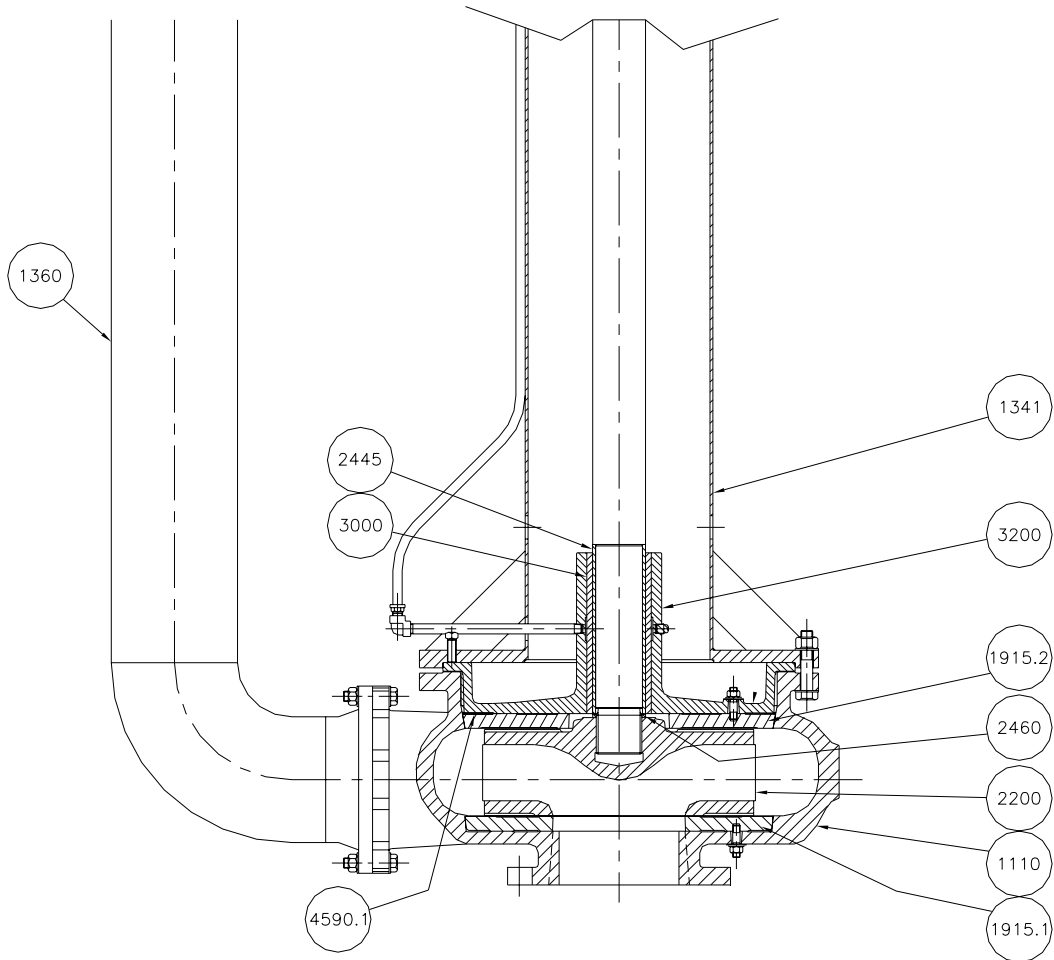
Pump overheats and seizes											
↓	Bearings have short life										
↓	Pump vibrates or is noisy										
↓	Mechanical seal has short life										
↓	Mechanical seal leaks excessively										
↓	Pump requires excessive power										
↓	Pump loses prime after starting										
↓	Insufficient pressure developed										
↓	Insufficient capacity delivered										
↓	Pump does not deliver liquid										
						PROBABLE CAUSES			POSSIBLE REMEDIES		
A. System troubles											
●						●	Pump not primed.			Check complete filling. Vent and/or prime.	
	●			●	●	●	Pump or suction pipe not completely filled with liquid.				
	●			●	●	●	Suction lift too high or level too low.			Check NPSHa>NPSHr, proper submergence, losses at strainers/fittings.	
●	●				●	●	Insufficient margin between suction pressure and vapour pressure.				
				●	●	●	Excessive amount of air or gas in liquid.			Check and purge pipes and system.	
				●	●	●	Air or vapour pocket in suction line.			Check suction line design for vapour pockets.	
				●	●		Air leaks into suction line.			Check suction pipe is airtight.	
				●	●		Air leaks into pump through mechanical seal, sleeve joints, casing joint or pipe plugs.			Check and replace faulty parts. CONSULT FLOWSERVE.	
	●					●	Foot valve too small.			Investigate replacing the foot valve.	
	●					●	Foot valve partially clogged.			Clean foot valve.	
	●			●	●	●	Inlet of suction pipe insufficiently submerged.			Check out system design.	
					●	●	Speed too low.			CONSULT FLOWSERVE.	
				●			Speed too high.			CONSULT FLOWSERVE.	
					●	●	Total head of system higher than differential head of pump.			Check system losses. Remedy or CONSULT FLOWSERVE.	
				●			Total head of system lower than pump design head.				
				●			Specific gravity of liquid different from design.			Check and CONSULT FLOWSERVE.	
				●	●	●	Viscosity of liquid differs from that for which designed.				
●	●						Operation at very low capacity.			Measure value and check minimum permitted. Remedy or CONSULT FLOWSERVE.	
	●	●		●			Operation at high capacity.			Measure value and check maximum permitted. Remedy or CONSULT FLOWSERVE.	
B. Mechanical troubles											
●	●	●	●	●	●		Misalignment due to pipe strain.			Check the flange connections and eliminate strains using elastic couplings or a method permitted.	

FAULT SYMPTOM

Pump overheats and seizes												
↓	Bearings have short life											
↓	Pump vibrates or is noisy											
↓	Mechanical seal has short life											
↓	Mechanical seal leaks excessively											
↓	Pump requires excessive power											
↓	Pump loses prime after starting											
↓	Insufficient pressure developed											
↓	Insufficient capacity delivered											
↓	Pump does not deliver liquid											
											PROBABLE CAUSES	POSSIBLE REMEDIES
											Improperly designed foundation.	Check setting of baseplate: tighten, adjust, grout base as required.
			●	●	●	●	●				Shaft bent.	Check shaft runouts are within acceptable values. CONSULT FLOWSERVE.
			●	●	●	●	●				Rotating part rubbing on stationary part internally.	Check and CONSULT FLOWSERVE, if necessary.
			●	●	●	●	●				Bearings worn	Replace bearings.
								●	●	●	Wearing ring surfaces worn.	Replace worn wear ring/surfaces.
									●	●	Impeller damaged or eroded.	Replace or CONSULT FLOWSERVE for improved material selection.
									●		Leakage under sleeve due to joint failure.	Replace joint and check for damage.
									●	●	Shaft sleeve worn or scored or running off centre.	Check and renew defective parts.
									●	●	Mechanical seal improperly installed.	Check alignment of faces or damaged parts and assembly method used.
									●	●	Incorrect type of mechanical seal for operating conditions.	CONSULT FLOWSERVE.
									●	●	Shaft running off centre because of worn bearings or misalignment.	Check misalignment and correct if necessary. If alignment satisfactory check bearings for excessive wear.
									●	●	Impeller out of balance resulting in vibration.	Check and CONSULT FLOWSERVE.
									●	●	Abrasive solids in liquid pumped.	
									●	●	Internal misalignment of parts preventing seal ring and seat from mating properly.	
									●	●	Mechanical seal was run dry.	Check mechanical seal condition and source of dry running and repair.
									●	●	Internal misalignment due to improper repairs causing impeller to rub.	Check method of assembly, possible damage or state of cleanliness during assembly. Remedy or CONSULT FLOWSERVE, if necessary.
									●	●	Excessive thrust caused by a mechanical failure inside the pump.	Check wear condition of impeller, its clearances and liquid passages.
									●	●	Excessive grease in ball bearings.	Check method of regreasing.
									●	●	Lack of lubrication for bearings.	Check hours run since last change of lubricant, the schedule and its basis.
									●	●	Improper installation of bearings (damage during assembly, incorrect assembly, wrong type of bearing etc).	Check method of assembly, possible damage or state of cleanliness during assembly and type of bearing used. Remedy or CONSULT FLOWSERVE, if necessary.
									●	●	Damaged bearings due to contamination.	Check contamination source and replace damaged bearings.
C. MOTOR ELECTRICAL PROBLEMS												

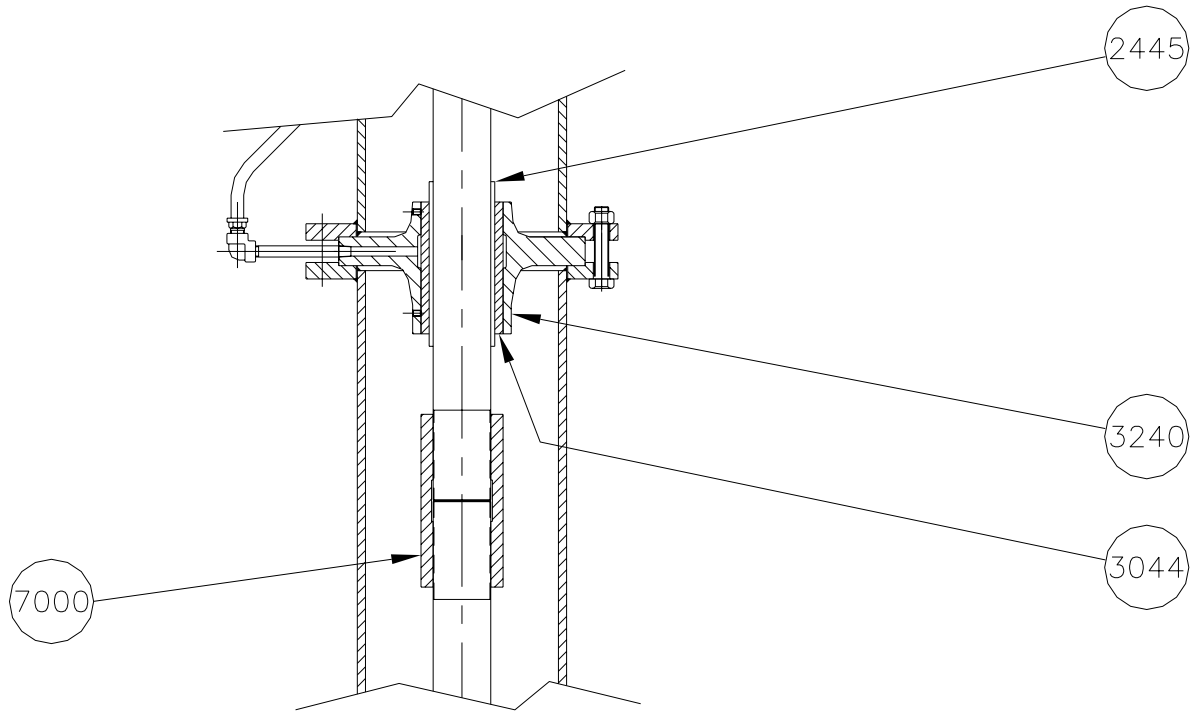
8 PARTS LIST AND DRAWINGS

8.1 Liquid-end



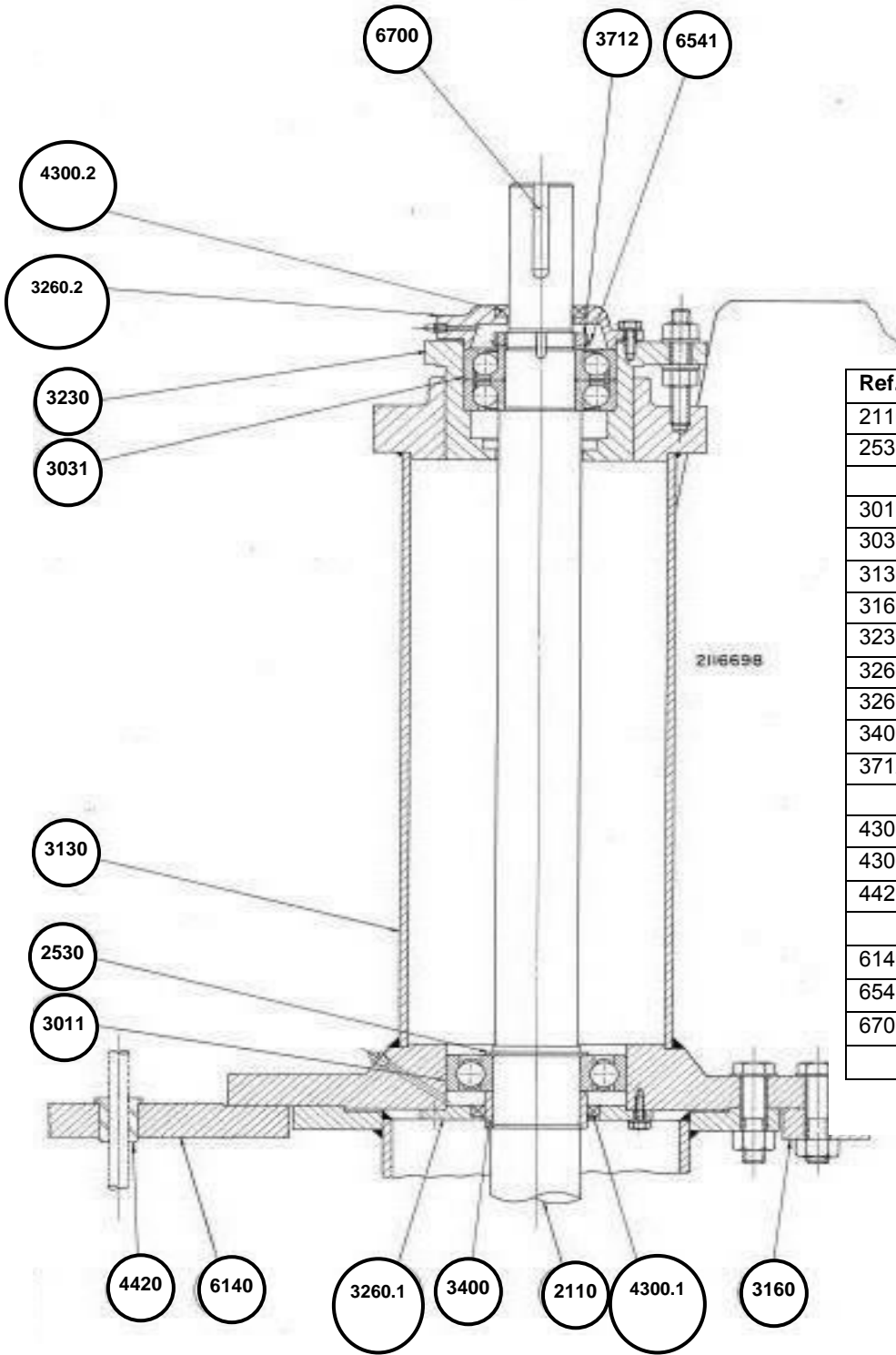
Ref.	Description	Ref.	Description
1110	Casing	3000	Bearing, Lower
1341	Column Pipe	3200	Lower Bearing Housing
1360	Discharge Pipe		
1915.1	Wear plate, Suction	4300.1	Line bearing seal
1915.2	Wear plate, Gland	4300.2	Thrust Bearing Seal
		4590.1	Gasket, Casing
2110	Shaft	4590.2	Gasket, wear plate
2200	Impeller		
2445	Shaft Sleeve	6521	Nut, wear plate
2460	Spacer, Sleeve Stop	6572	Stud, wear plate
2540	Deflector	6700	Key, coupling

8.2 Intermediate bearing



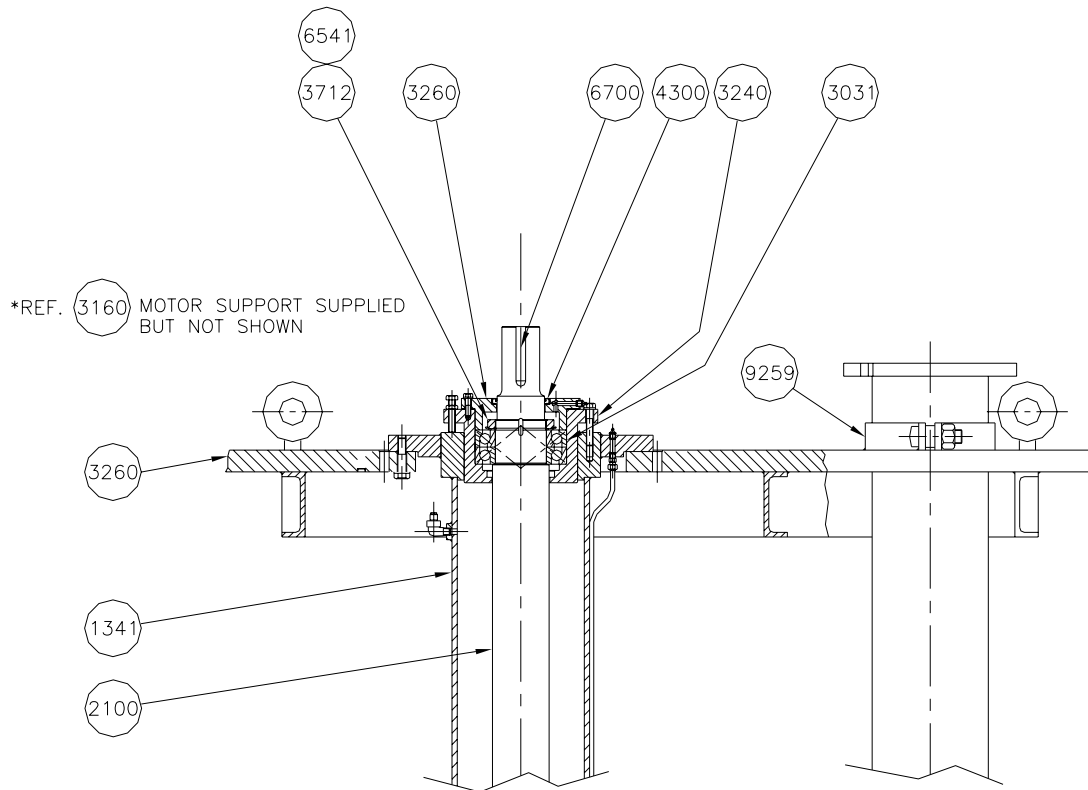
Ref.	Description
2445	Shaft Sleeve
2542	Clamp Ring
3044	Intermediate Bearing
3240	Bearing Housing, Intermediate
4300.1	Line Bearing Seal
4300.2	Thrust Bearing Seal
6700	Key, Coupling
7000	Coupling

8.3 MVD Drive-end



Ref.	Description
2110	Pump Shaft
2530	Retaining Ring
3011	Line bearing
3031	Thrust roller bearing
3130	Bearing cartridge
3160	Motor Support
3230	Thrust bearing housing
3260.1	Line bearing Cover
3260.2	Bearing Cover
3400	Bearing Sleeve
3712	Bearing Lock Nut
4300.1	Seal, Line bearing
4300.2	Seal, Thrust bearing
4420	Sealing pipe
6140	Sole plate
6541	Lockwasher
6700	Key

8.4 Direct Drive



Ref.	Description
1341	Column Pipe
2100	Shaft
3031	Thrust bearing
3130	Bearing Cartridge
3160	Motor Support
3240	Bearing Housing
3260	Bearing Cover
3712	Lock Nut, Bearing
4300	Thrust Bearing Seal
4420	Sealing Pipe
6140	Sole Plate
6541	Lockwasher
6700	Key
9259	Pipe Clamp

8.9 General arrangement drawing

The typical general arrangement drawing and any specific drawings required by the contract will be sent to the Purchaser separately unless the contract specifically calls for these to be included into the User Instructions. If required, copies of other drawings sent separately to the Purchaser should be obtained from the Purchaser and retained with these User Instructions.

9 CERTIFICATION

Certificates, determined from the contract requirements will be provided with this manual. Examples are certificates for CE marking and ATEX marking. If required, copies of other certificates sent separately to the Purchaser should be obtained from Purchaser for retention with the User Instructions. See section 1.9, *Noise level*, for details of typical noise certification.

10 OTHER RELEVANT DOCUMENTATION AND MANUALS

10.1 Supplementary User Instruction manuals

Supplementary instruction determined from the contract requirements for inclusion into User Instructions such as for a driver, instrumentation, controller, sub-driver, seals, sealant system, mounting component etc are included under this section. If further copies of these are required they should be obtained from the purchaser for retention with these User Instructions.

Where any pre-printed set of User Instructions are used, and satisfactory quality can be maintained only by avoiding copying these, they are included at the end of these User Instructions such as within a standard clear polymer software protection envelope.

10.2 Change notes

If any changes, agreed with Flowserve Pump Division, are made to the product after its supply, a record of the details should be maintained with these User Instructions.

10.3 Additional sources of information

Reference 1:

NPSH for Rotordynamic Pumps: a reference guide, Europump Guide No. 1, Europump & World Pumps, Elsevier Science, United Kingdom, 1999.

Reference 2:

Pumping Manual, 9th edition, T.C. Dickenson, Elsevier Advanced Technology, United Kingdom, 1995.

Reference 3:

Pump Handbook, 2nd edition, Igor J. Karassik et al, McGraw-Hill Inc., New York, 1993.

Reference 4:

ANSI/HI 1.1-1.5
Centrifugal Pumps - Nomenclature, Definitions, Application and Operation.

Reference 5:

ANSI B31.3 - Process Piping.

NOTES:

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