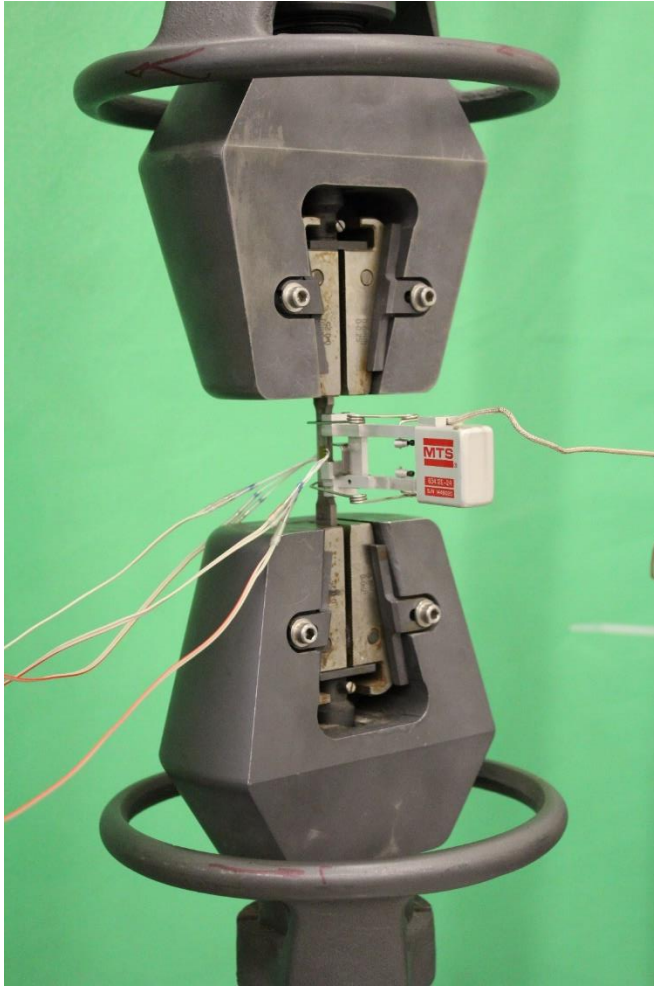


# PROJECT 2: EVALUATION OF THE NONLINEAR MECHANICAL RESPONSE IN THREADED FASTENERS

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**Mentors:** Gustavo Castelluccio (Cranfield), John Emery (SNL), Jeff Smith (SNL), and John Mersch (SNL)

# From Macroscopic Tensile Tests to Microscopic Mechanical Response of Components



## OBJECTIVES

- Calibrate constitutive models to uniaxial tension test data provided by SNL for Steel A286.
- Attempt to reproduce test results for A286 fastener tension data with FEMs of test.
- Evaluate reduced order fastener modelling approaches.
- Investigate more complicated load cases that expose differences in the constitutive models.

# Background: Modeling Fasteners

- In analysis of complex assemblies, fastened joints between components should be engineered.
- There might be many fasteners, and modeling them all in detail is not feasible.
- Reduced order fastener models must be used.
- In transient analyses involving extreme loading conditions, significant plastic strain can occur in the fasteners. This requires fastener models that still accurately capture the post-yield behavior of the actual fasteners.

With only a limited amount of tension testing data, how accurately can we produce simplified models for various fasteners without having test data on each one?

# Background: Yield Surface

- Von Mises Yield Criterion:

$$\sigma_{vm} = \sqrt{\frac{1}{2}[(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2]}$$

(Where  $\sigma_{1,2,3}$  are principal stresses)

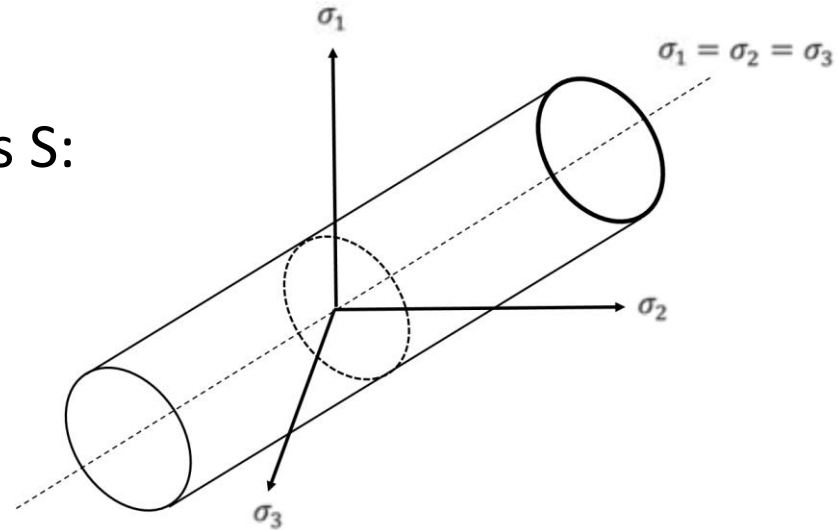
- This defines a cylindrical 3D yield surface in principal stress space.
  - Axis is along hydrostatic stress states

- $\sigma_{vm}$  comes from deviatoric stress S:

$$\sigma_{ij} = S_{ij} + \frac{1}{3}\sigma_{kk}\delta_{ij}$$

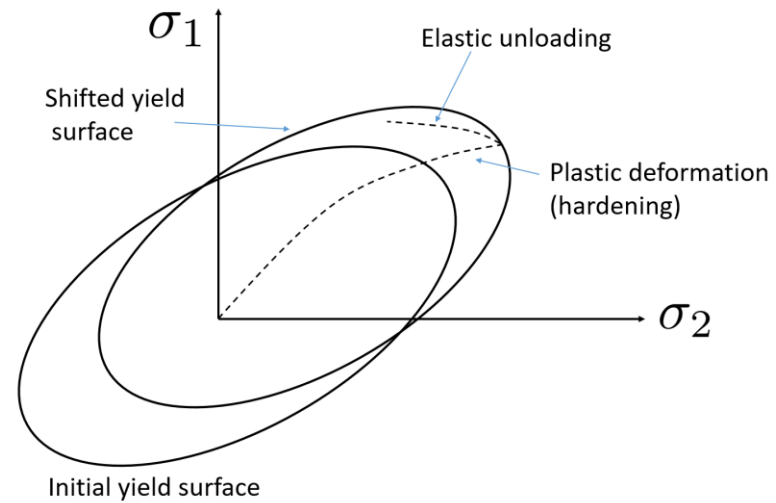
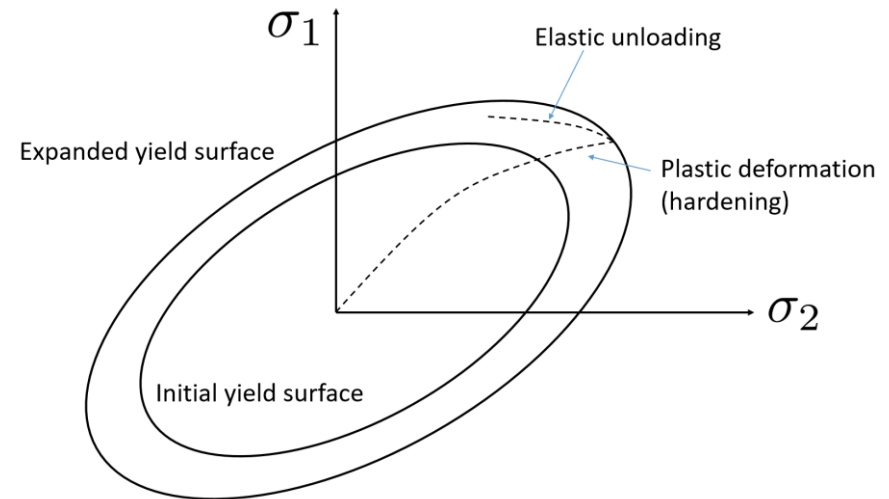
$$J_2 = \frac{1}{2}S_{ij}S_{ij}$$

$$\sigma_{vm} = \sqrt{3J_2}$$



# Background: Constitutive Models

- Isotropic Hardening
  - Yield Surface retains its shape and is symmetric about the origin
  - Increases uniformly as the material deforms plastically
- Kinematic Hardening
  - Yield Surface retains its shape and size
  - Shifts as the material deforms plastically



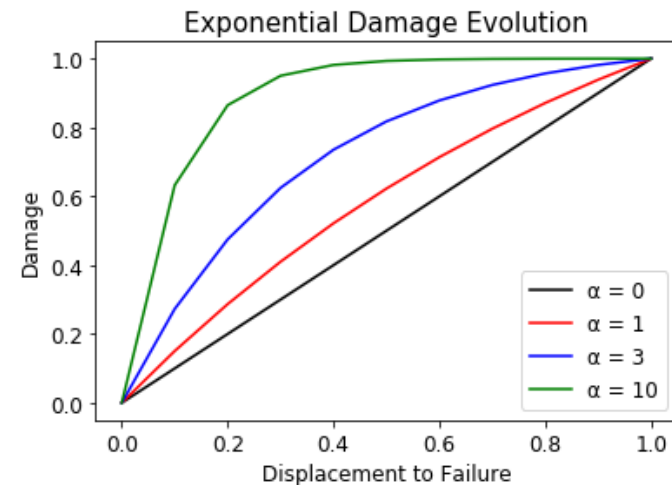
# Constitutive Models Background

- Hardening Curve Definition
  - Multi Linear Elastic-Plastic
    - Linear piecewise hardening curve defined with discrete pairs of equivalent plastic strain and yield stress
  - Johnson-Cook
    - Yield stress follows an analytical function of the equivalent plastic strain
    - Can take into account strain rate dependence and temperature effects
    - Ideal for high strain rate deformations

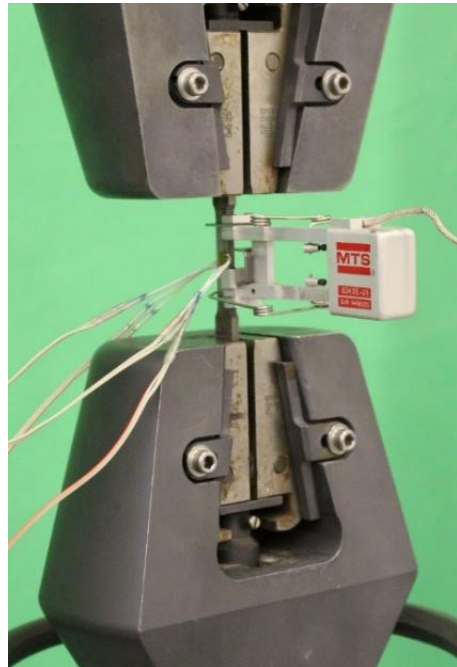
$$\sigma_e = [A + B(\epsilon_e^p)^n][1 + C \ln(\frac{\dot{\epsilon}_e^p}{\dot{\epsilon}_{e0}^p})][1 - \hat{T}^m]$$

# Damage Criterion

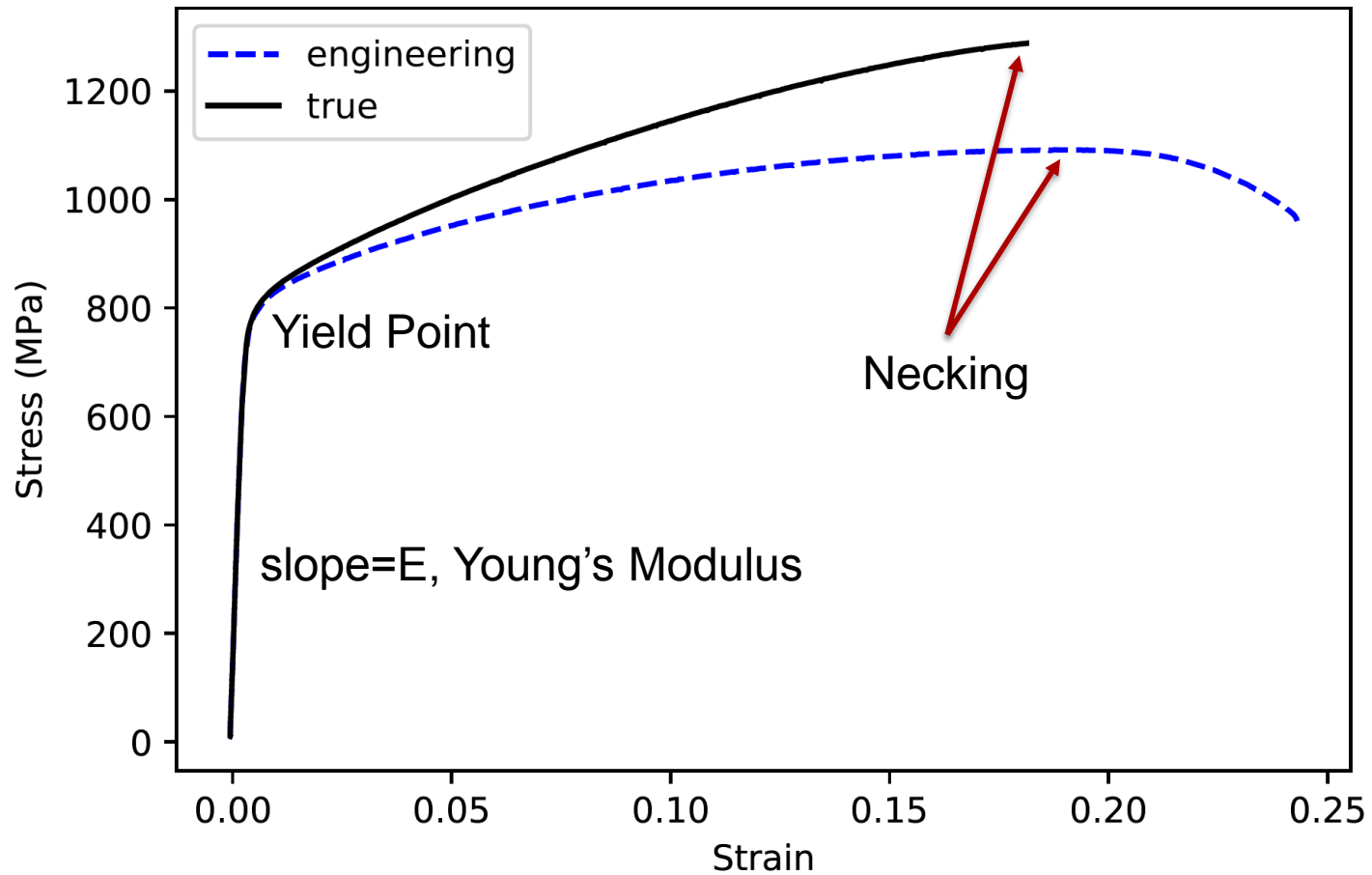
- Initiation: Ductile damage
  - Phenomenologically predicts the onset of damage due to void nucleation, growth, and coalescence
  - At a local equivalent plastic strain (fracture strain) damage “initiates”
- Evolution: Independent of damage initiation model
  - Progressive degradation of material stiffness, leading to material failure
  - Can be described exponentially



# A286 Tension Test (SNL)

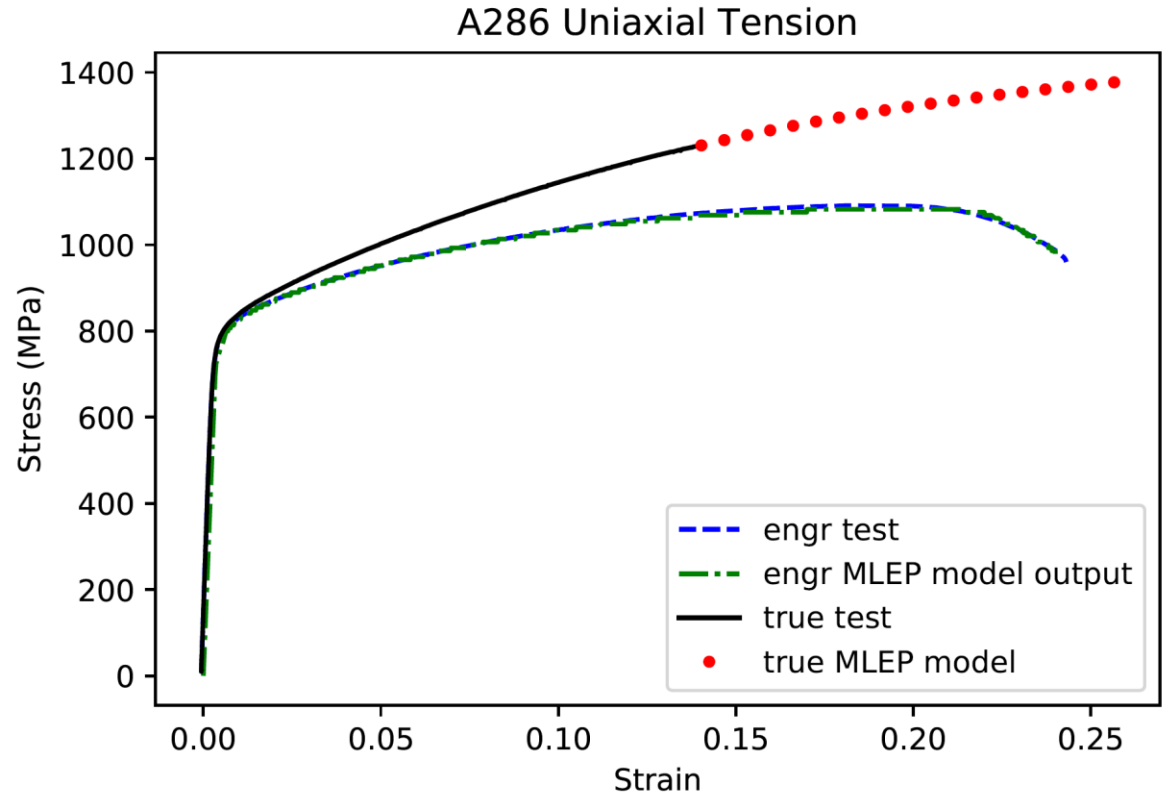
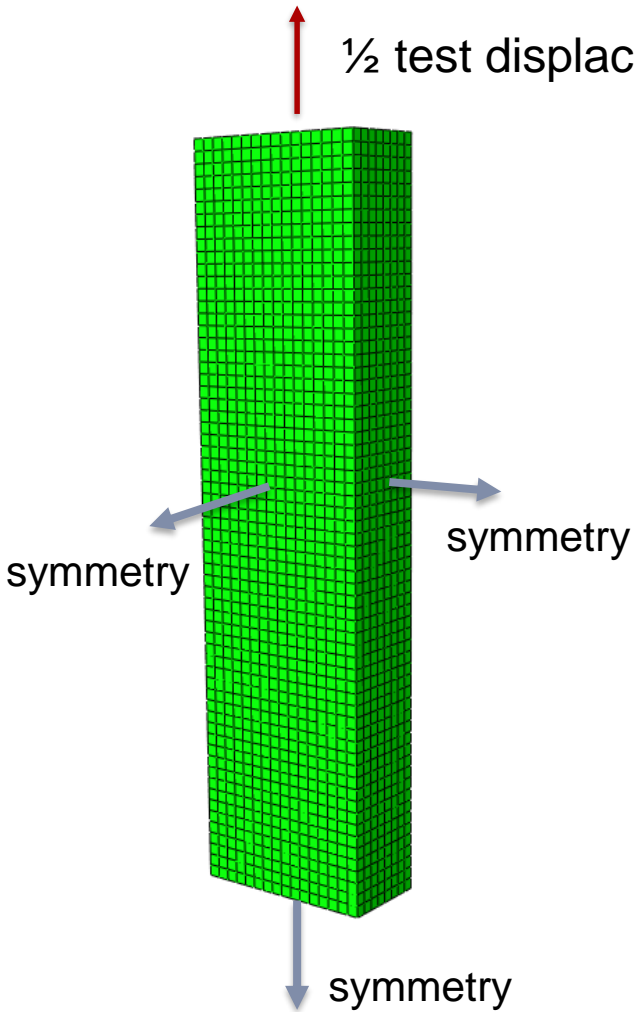


A286 Uniaxial Tension



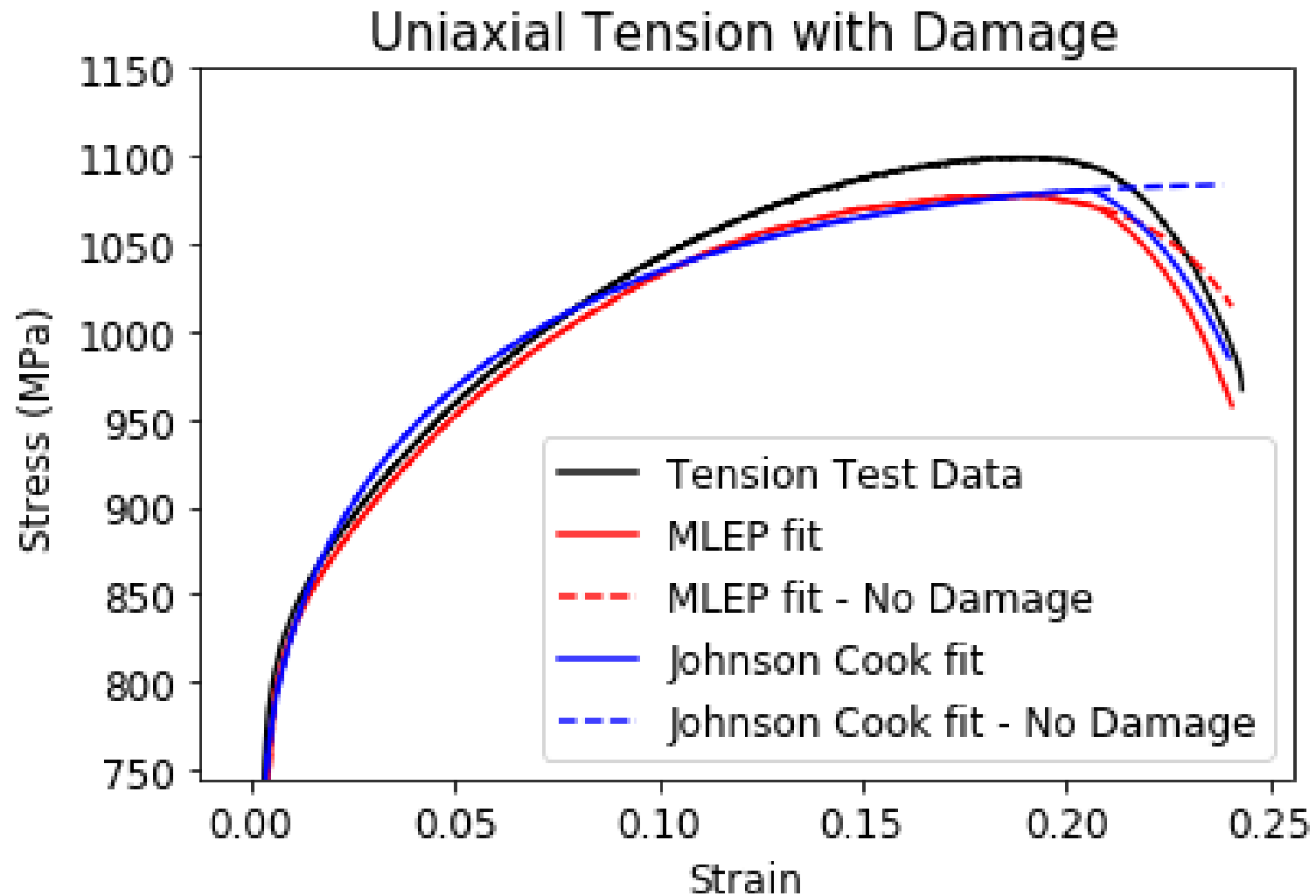


# Uniaxial Tension Calibration

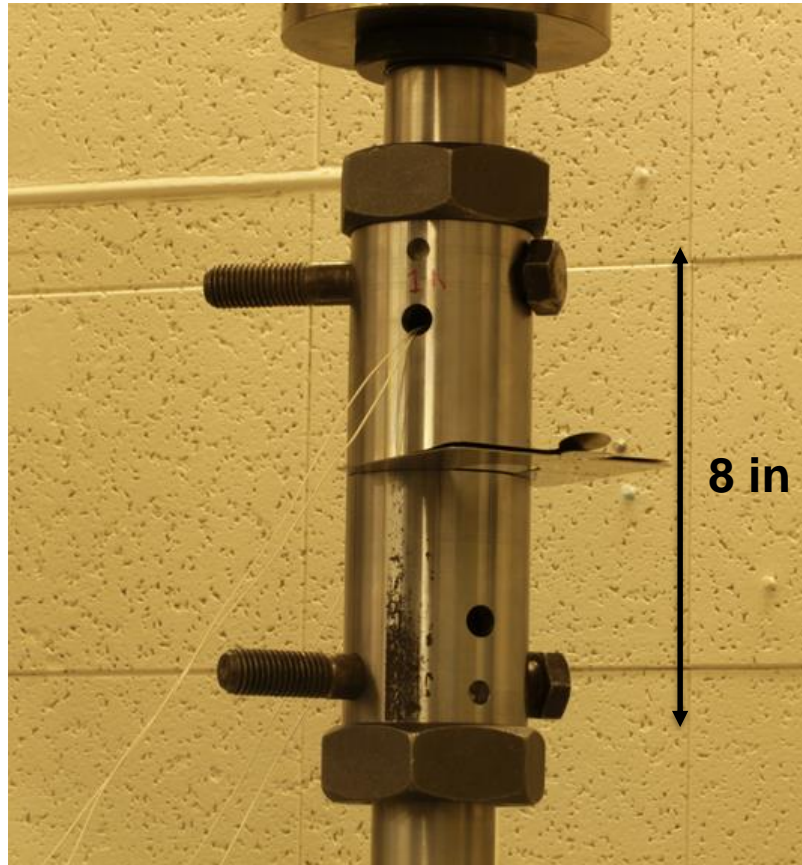


Uniaxial Tension FEM used for calibration

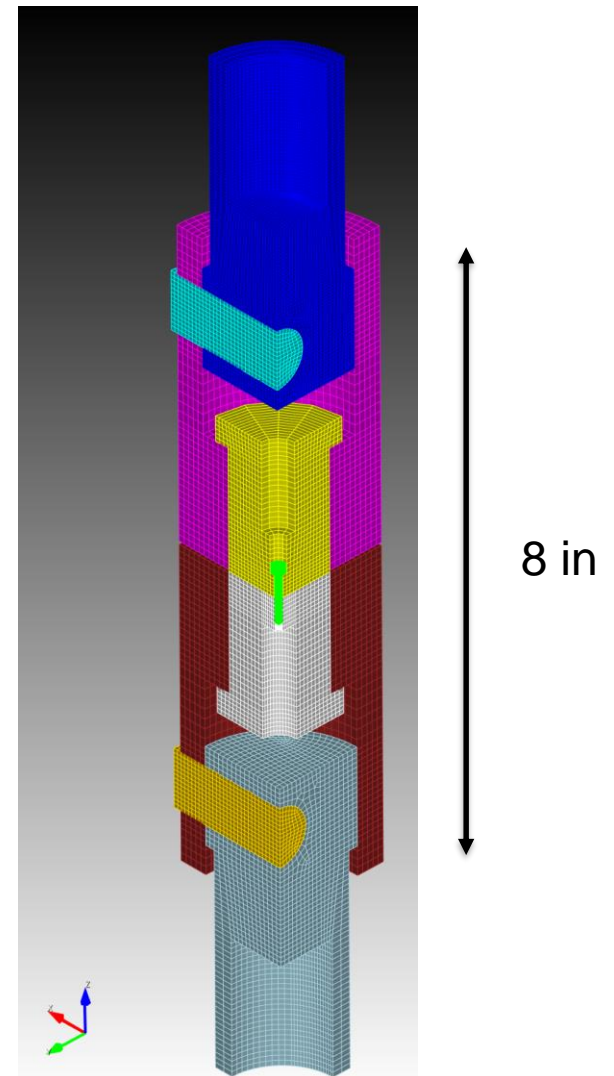
# Uniaxial Tension Calibration



# Fastener Test Setup



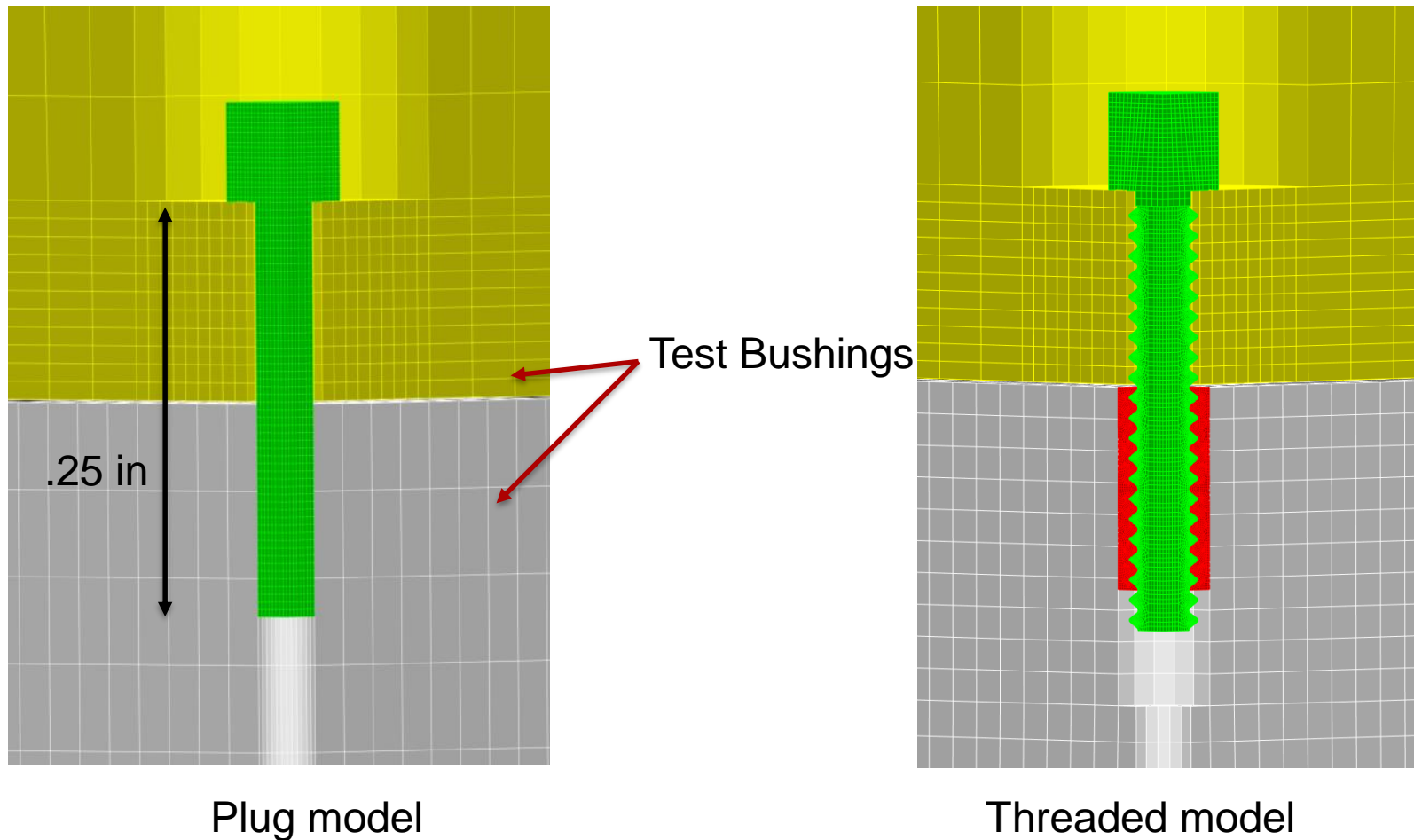
Real Test



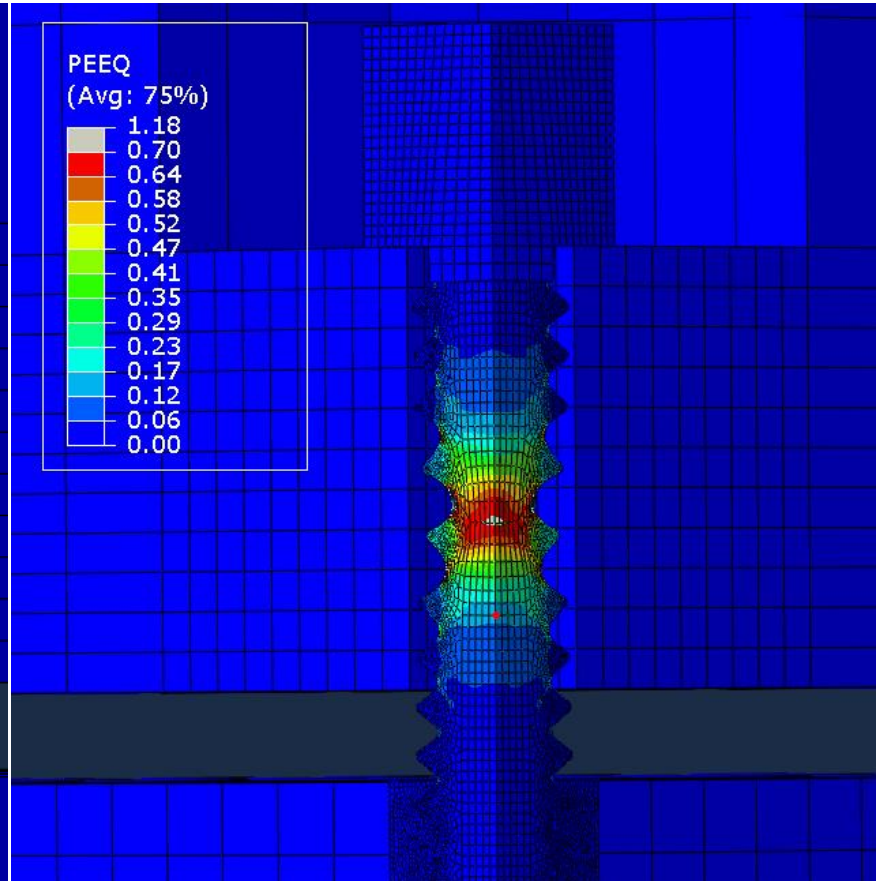
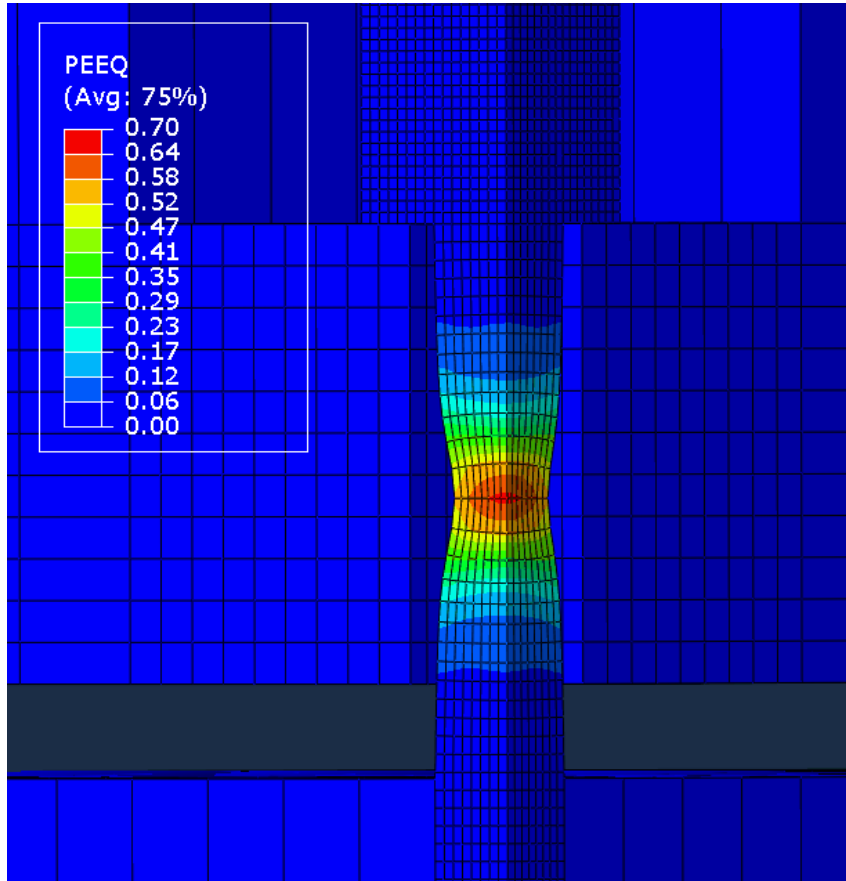
FEM of test (quarter symmetry)

# Reduced Order Fastener Models

- Two types of simplified fastener models (#0-80 pictured)

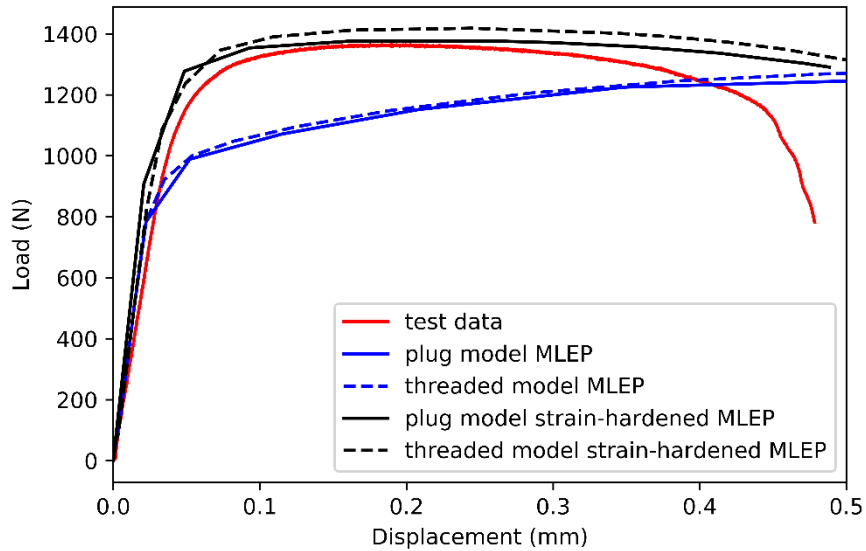


# Equivalent Plastic Strain in Models

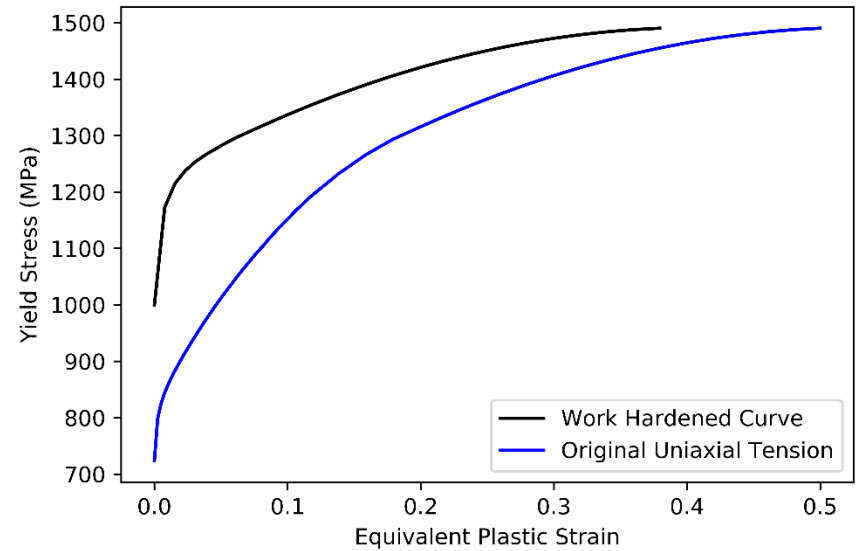


# Blind Predictions of Fastener Test Data

#0 Fastener Test vs Plug Models

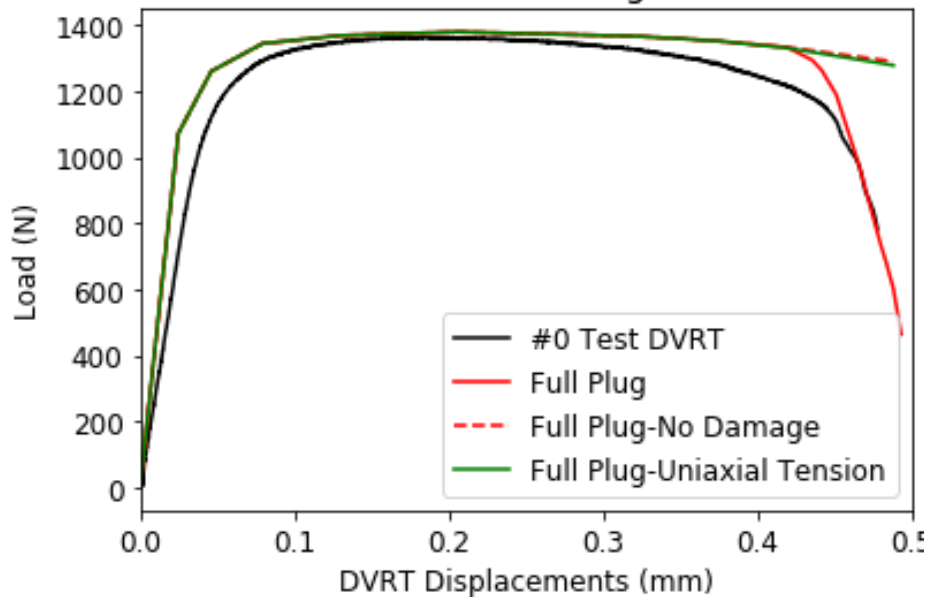


#0 Work-Hardened MLEP Curve

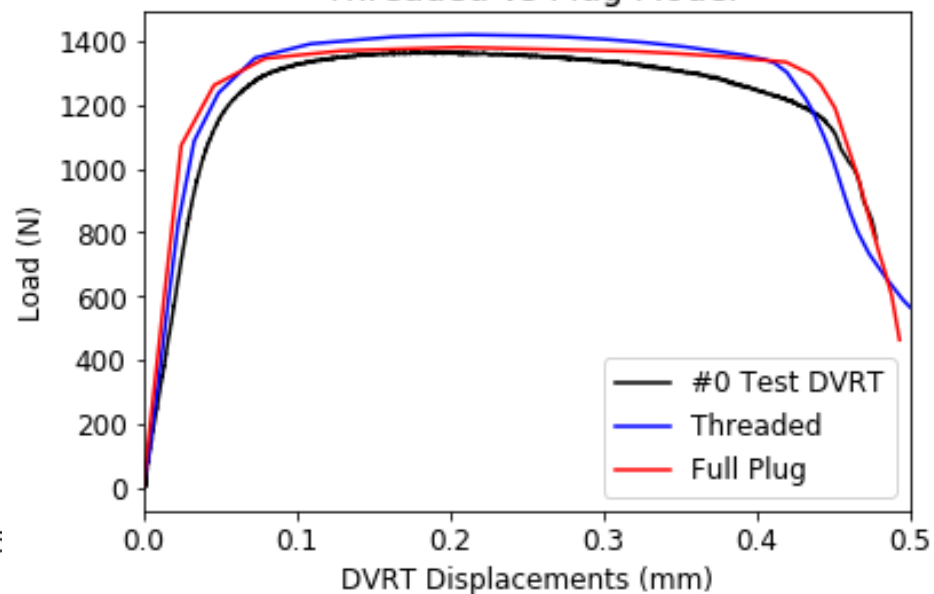


# Blind Predictions of Fastener Test Data

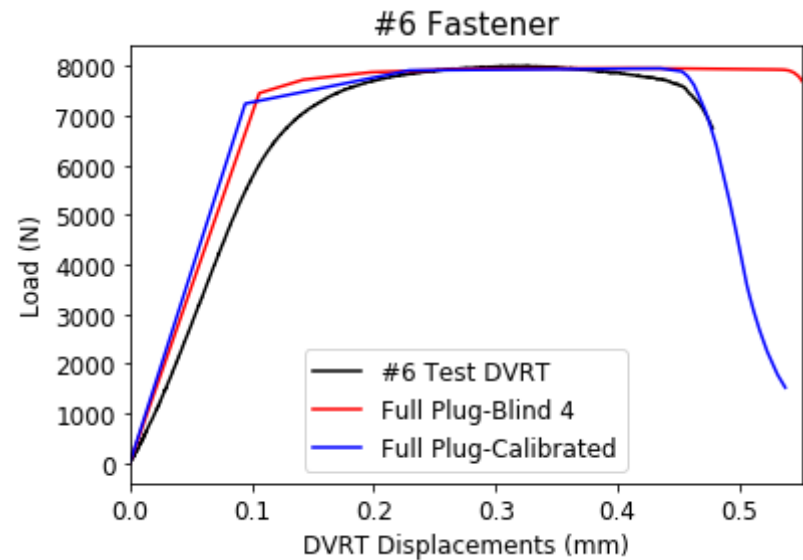
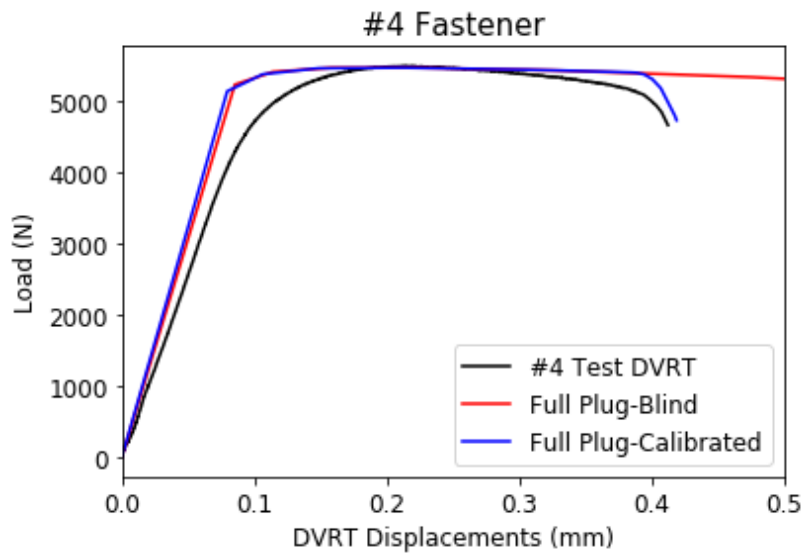
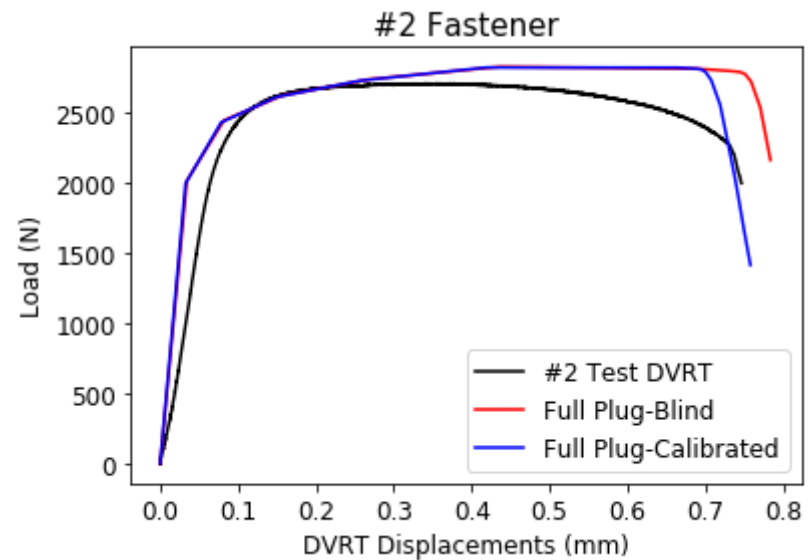
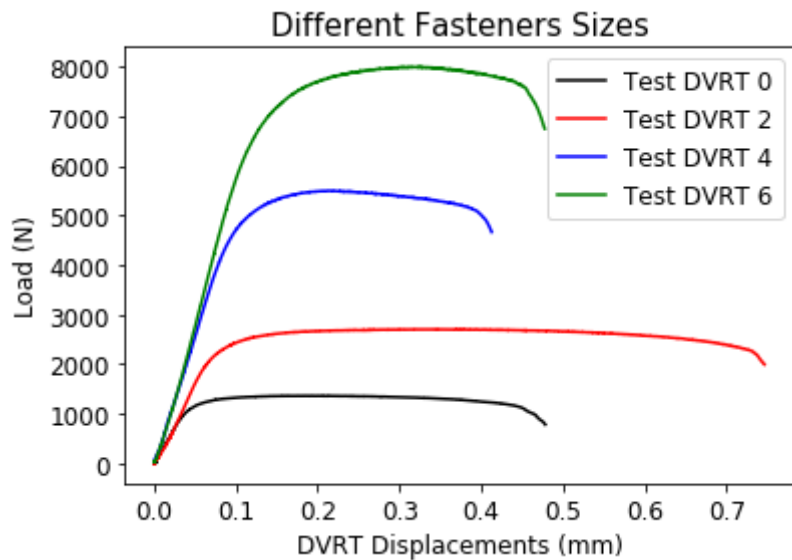
## #0 Fastener Blind Damage Prediction



## #0 Fastener Threaded vs Plug Model

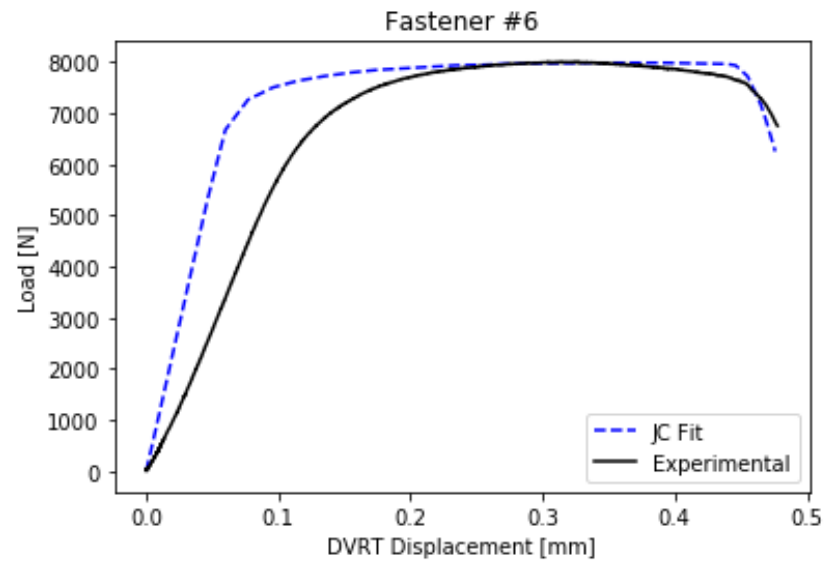
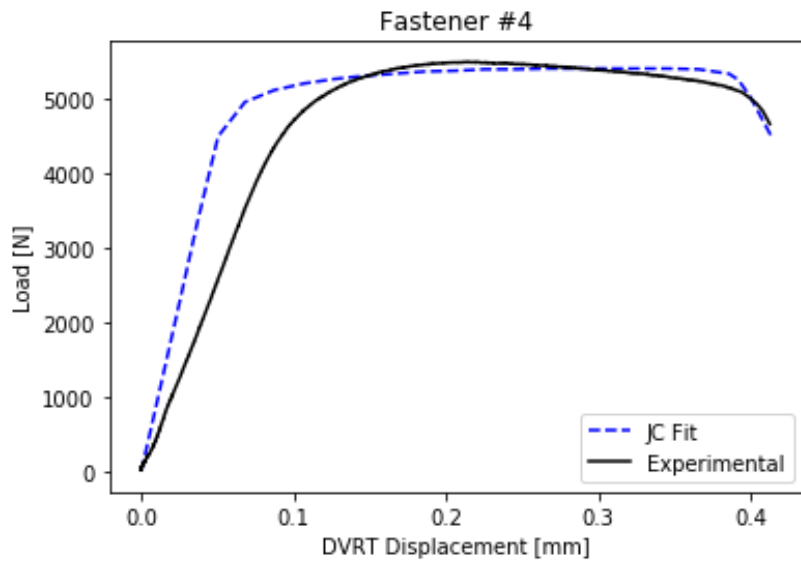
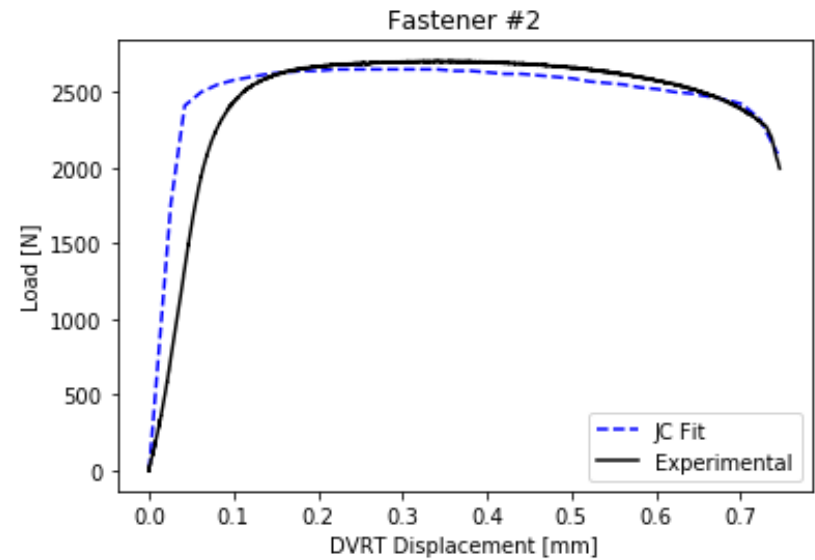
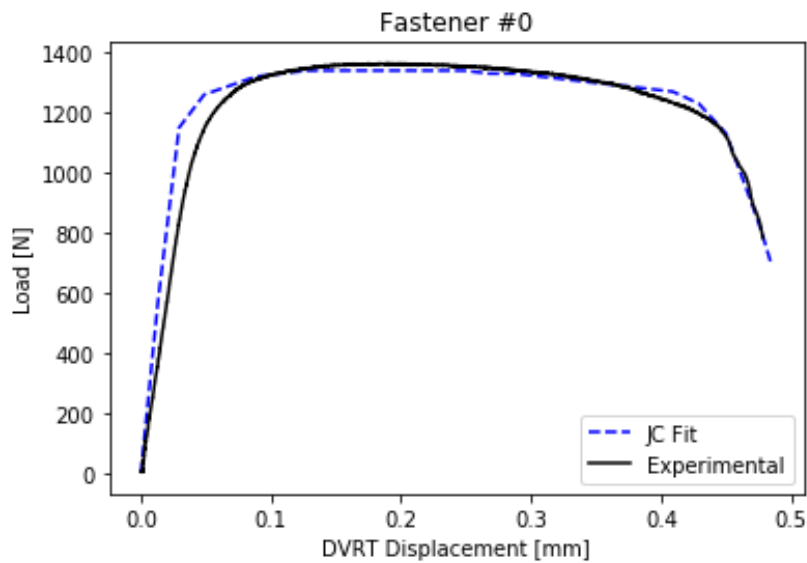


# Extrapolating MLEP to Other Sizes





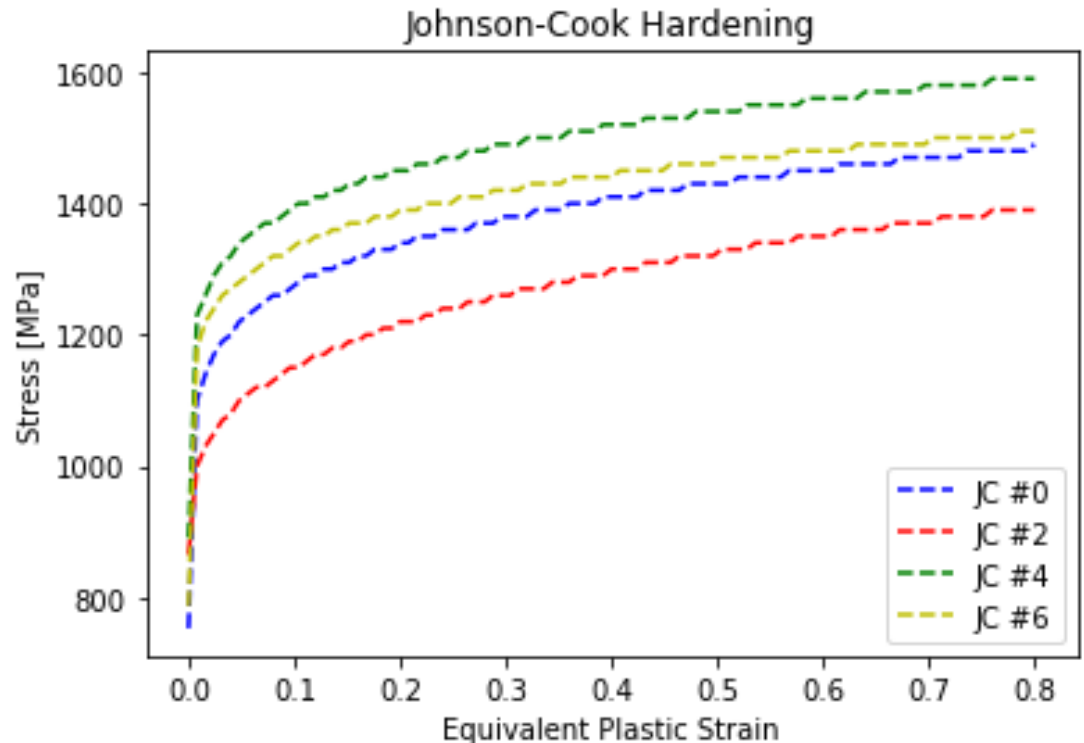
# Johnson-Cook Fastener Calibrations



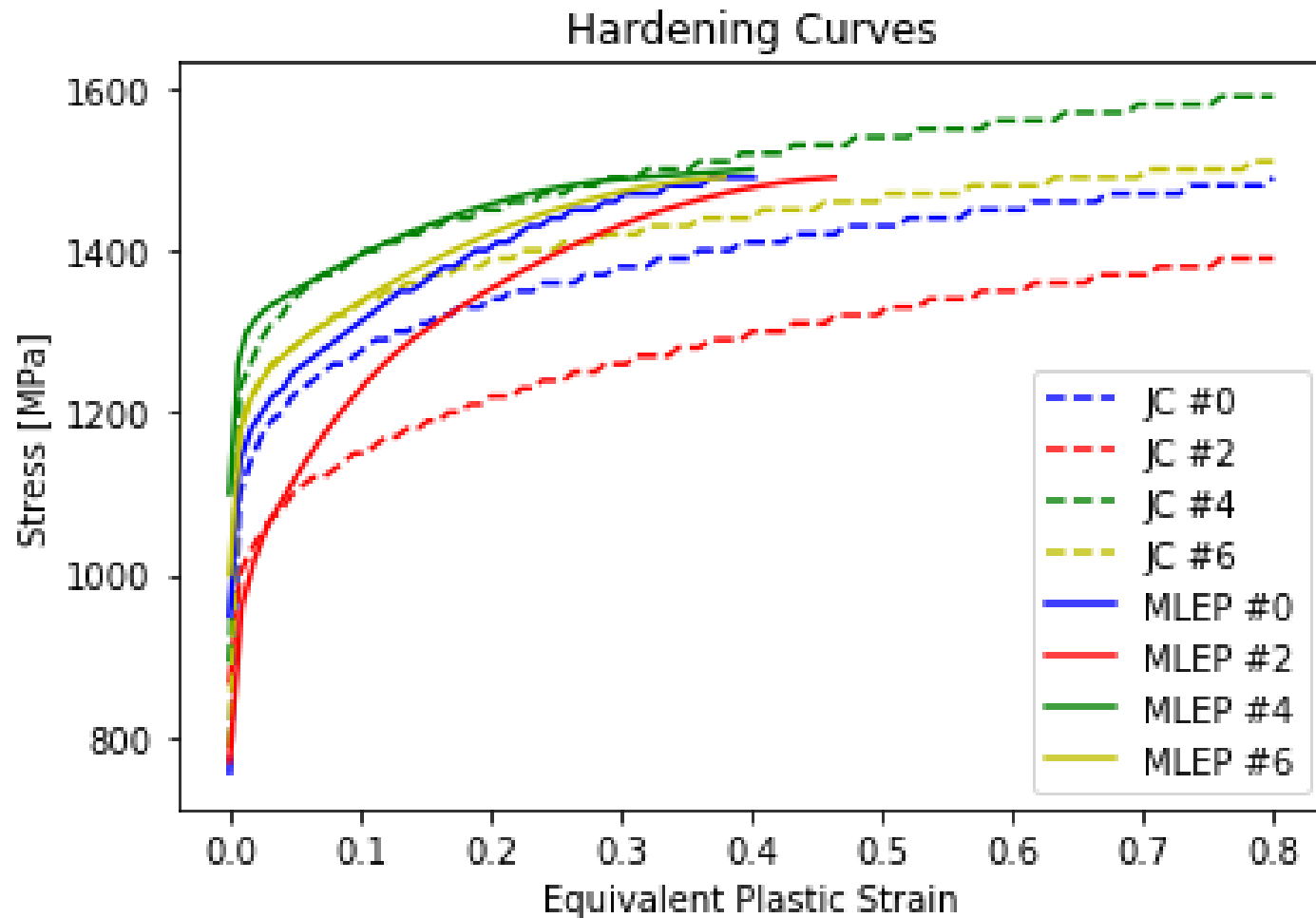
# Johnson-Cook Hardening Curves

Parameters	#0	#2	#4	#6
A	752.74	865.05	890.87	786.50
B	760.02	565.59	725.53	743.02
n	0.1627	0.2963	0.1603	0.1325

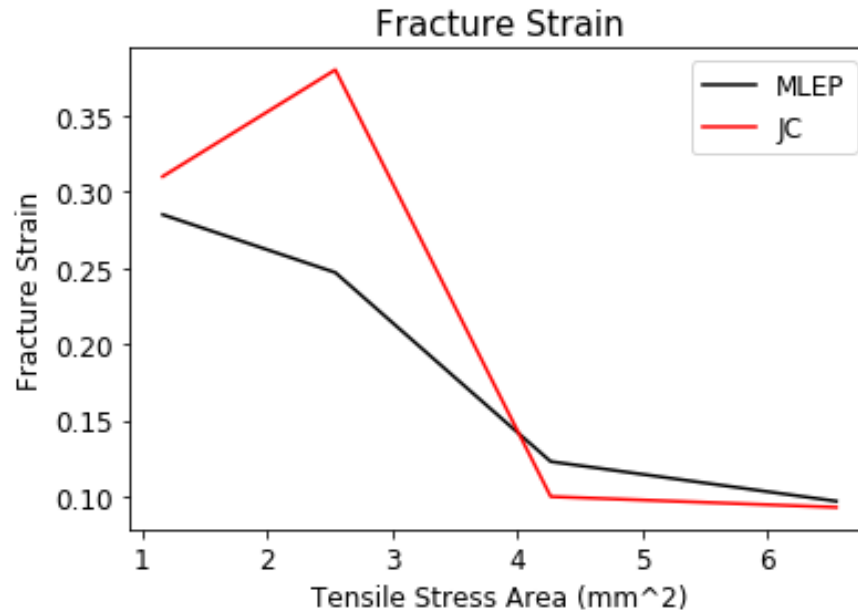
$$\sigma = A + B(\varepsilon_p)^n$$



# Hardening Curves



# Trend of Damage Parameters

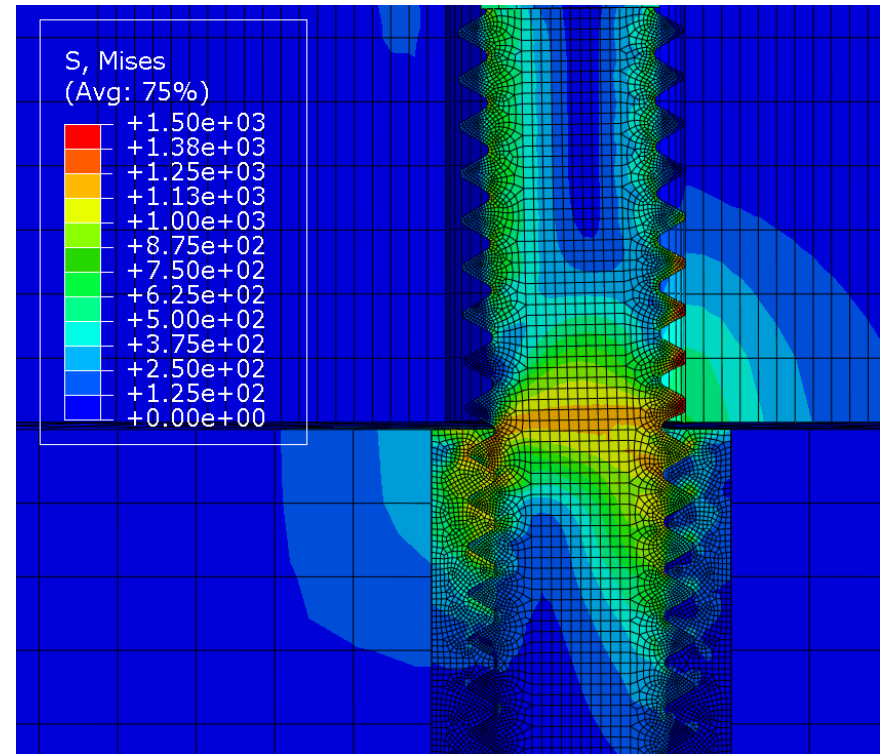
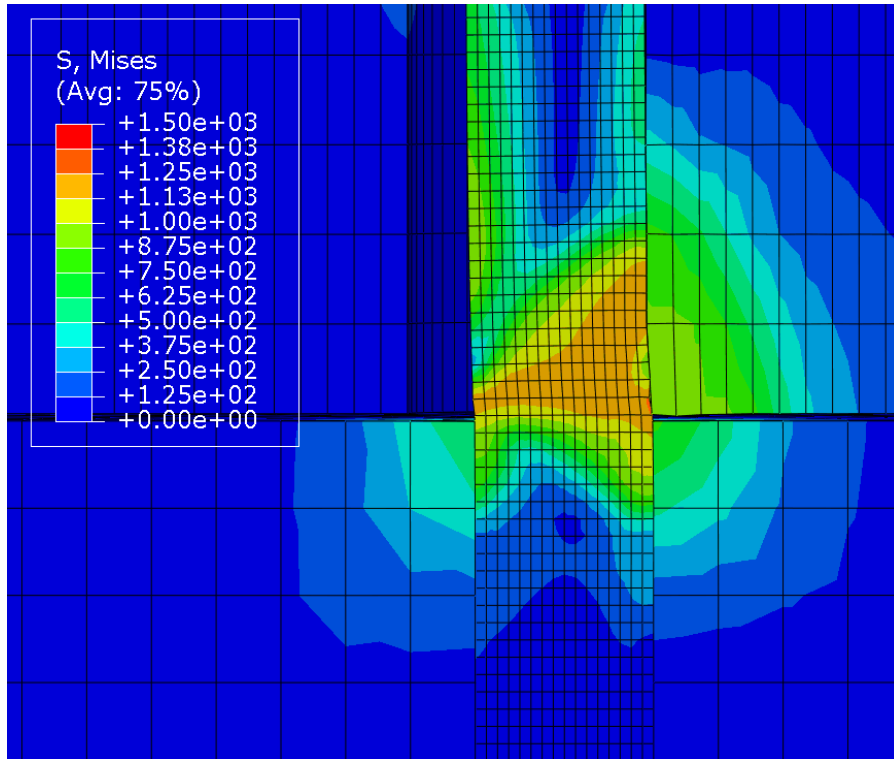


Model	Fracture Strain		Displacement at Failure (mm)		Alpha	
	MLEP	Johnson Cook	MLEP	Johnson Cook	MLEP	Johnson Cook
Smooth Specimen	<b>0.23</b>	0.19	<b>1.27</b>	0.508	<b>1</b>	1
#0 Plug	<b>0.285</b>	0.31	<b>0.1</b>	0.1	<b>8</b>	7
#2 Plug	<b>0.247</b>	0.38	<b>0.1</b>	0.1	<b>8</b>	7
#4 Plug	<b>0.123</b>	0.1	<b>0.1</b>	0.1	<b>7</b>	6
#6 Plug	<b>0.097</b>	0.093	<b>0.1</b>	0.1	<b>7</b>	7
<b>Bold: MLEP</b>			Regular: Johnson Cook			

# Combined Loading

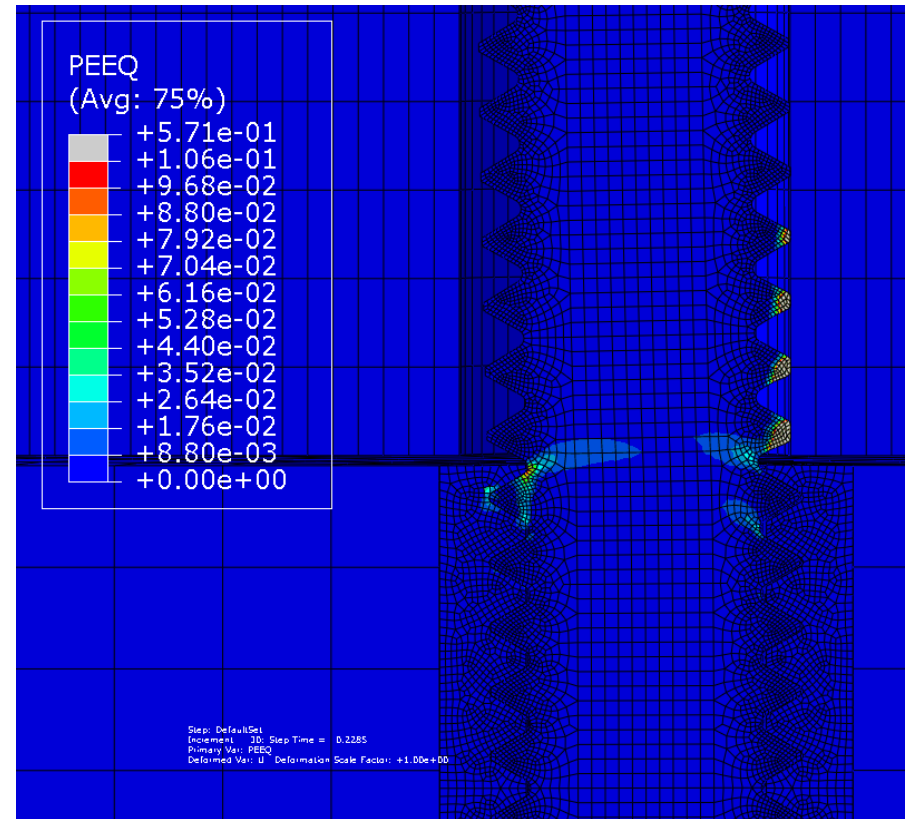
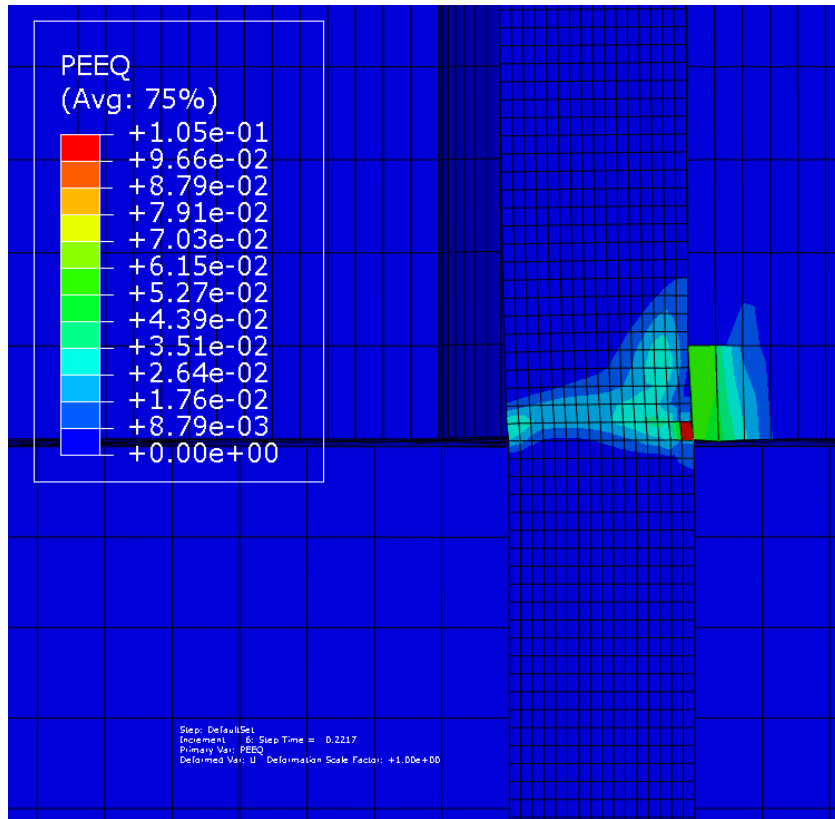
# Shear

- Shearing deformations applied with BC's similar to SNL tests.



# Shear

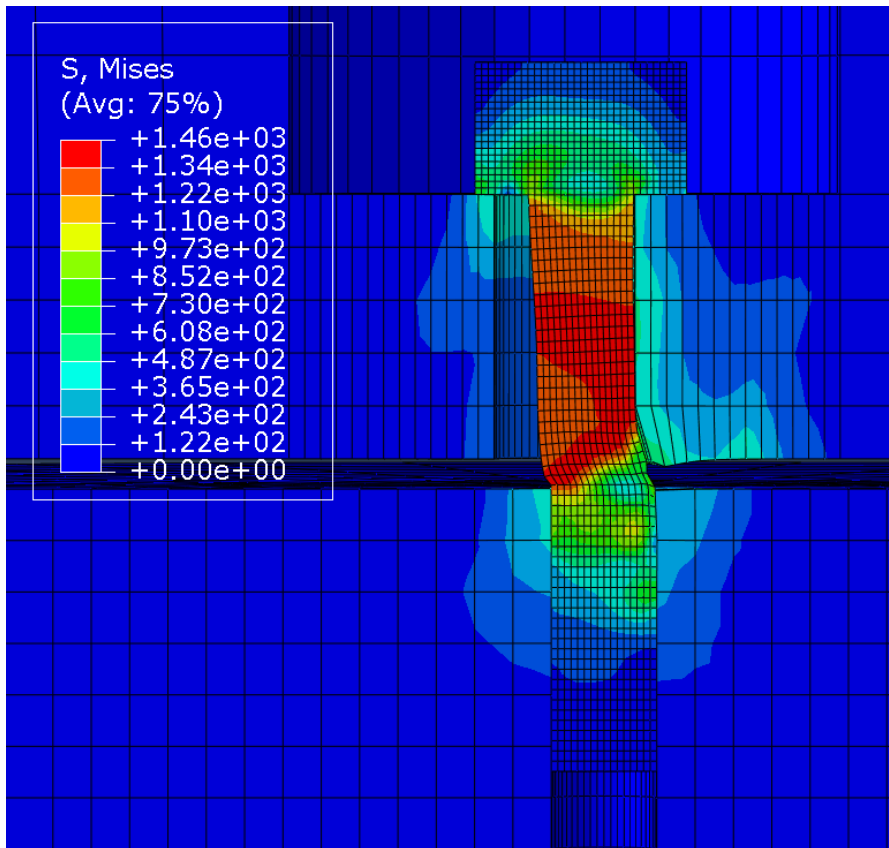
- Shearing deformations applied with BC's similar to SNL tests.



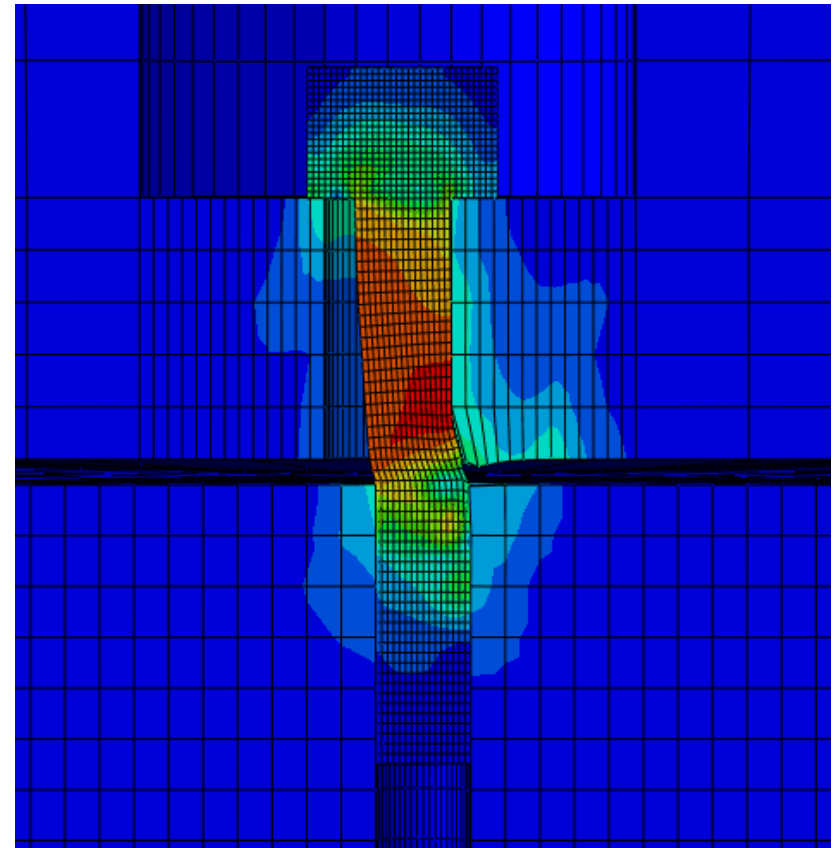
# Shear then Tension

- Non-proportional loading exposes differences between isotropic and kinematic hardening

Isotropic



Kinematic

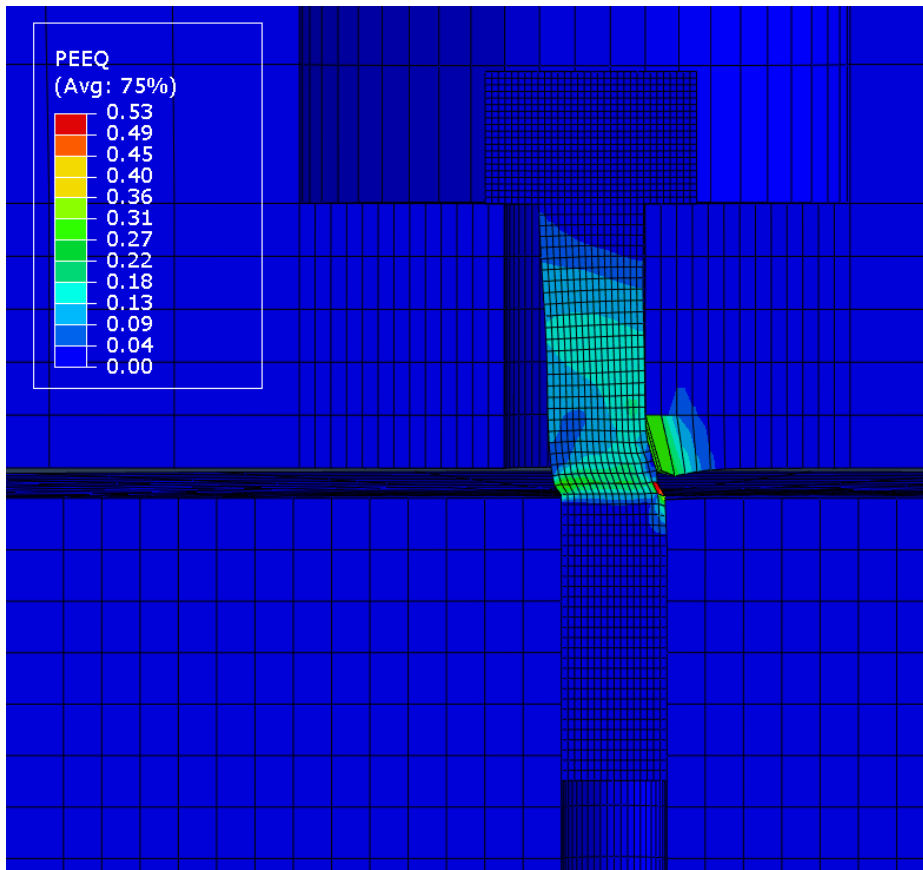




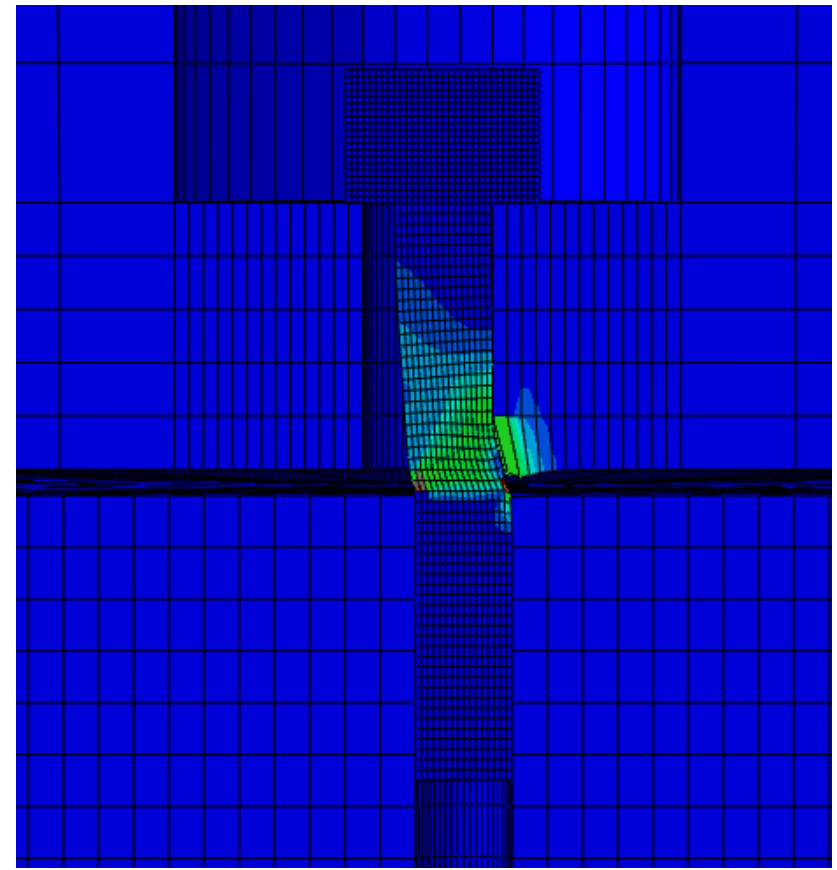
# Shear then Tension

- Non-proportional loading exposes differences between isotropic and kinematic hardening

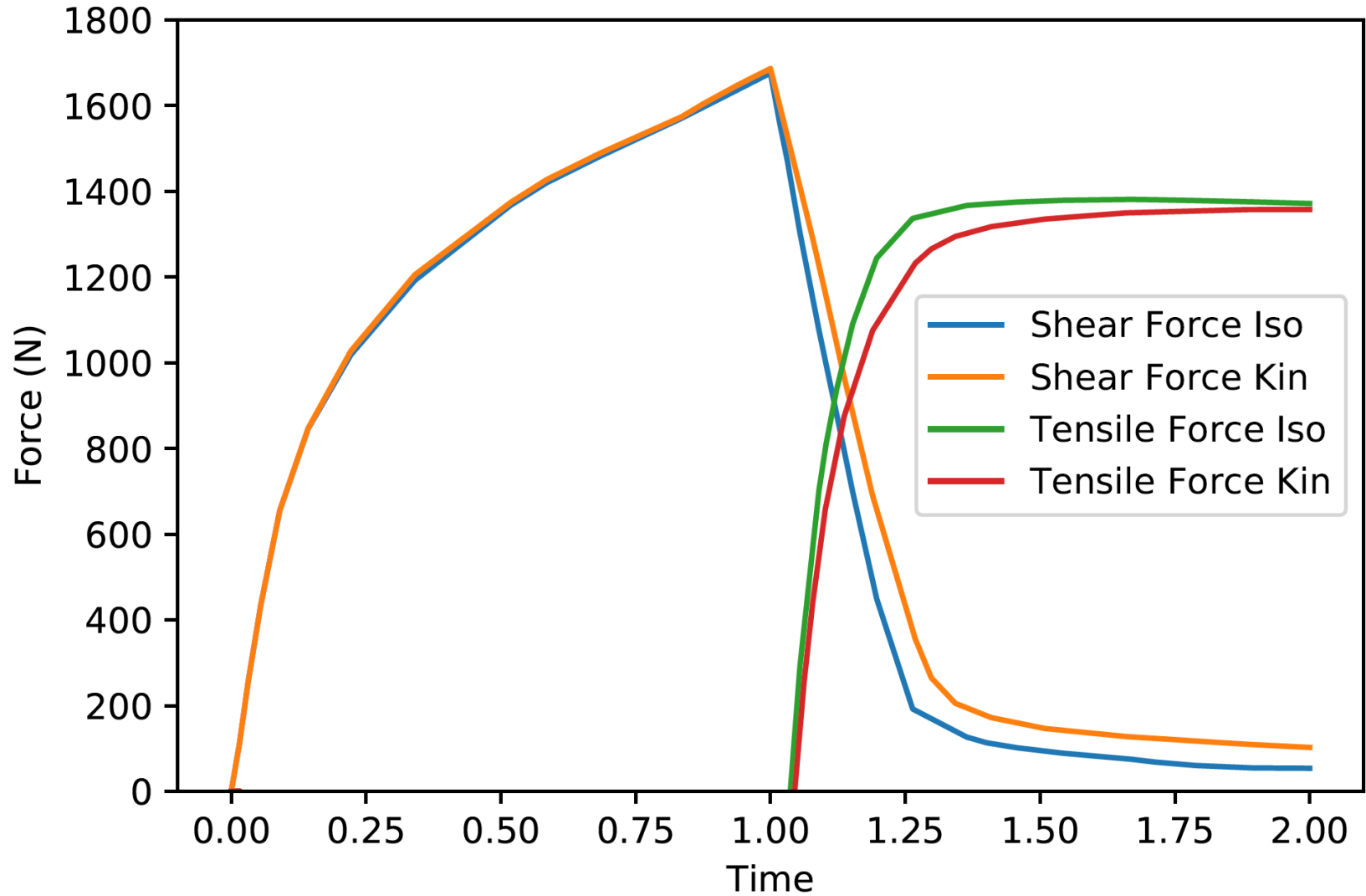
Isotropic



Kinematic



# Shear then Tension



# Conclusions

- The hardening curve of the original material can be shifted to approximate the hardening curve of a fastener.
  - Different fasteners have different yield stresses, but the Johnson-Cook hardening curves all have the same shape.
  - All the fasteners have yield stresses between 1000 and 1300 MPa.
- In uniaxial tension the material nonlinearities dominate, so the plug model can sufficiently describe the fastener.
- One set of damage criterion values cannot be universally applied for a given material. Generally, the fracture strain varies for different specimen sizes.
- There are small differences in stress and plastic strain fields between isotropic and kinematic hardening when plug models undergo non-proportional loading.

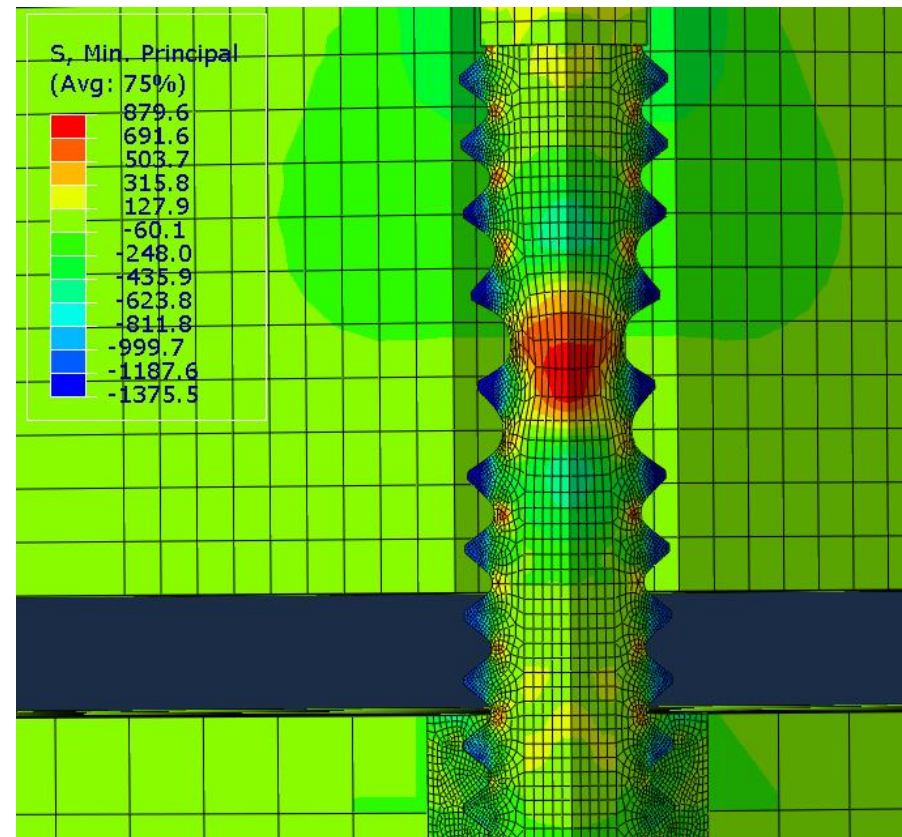
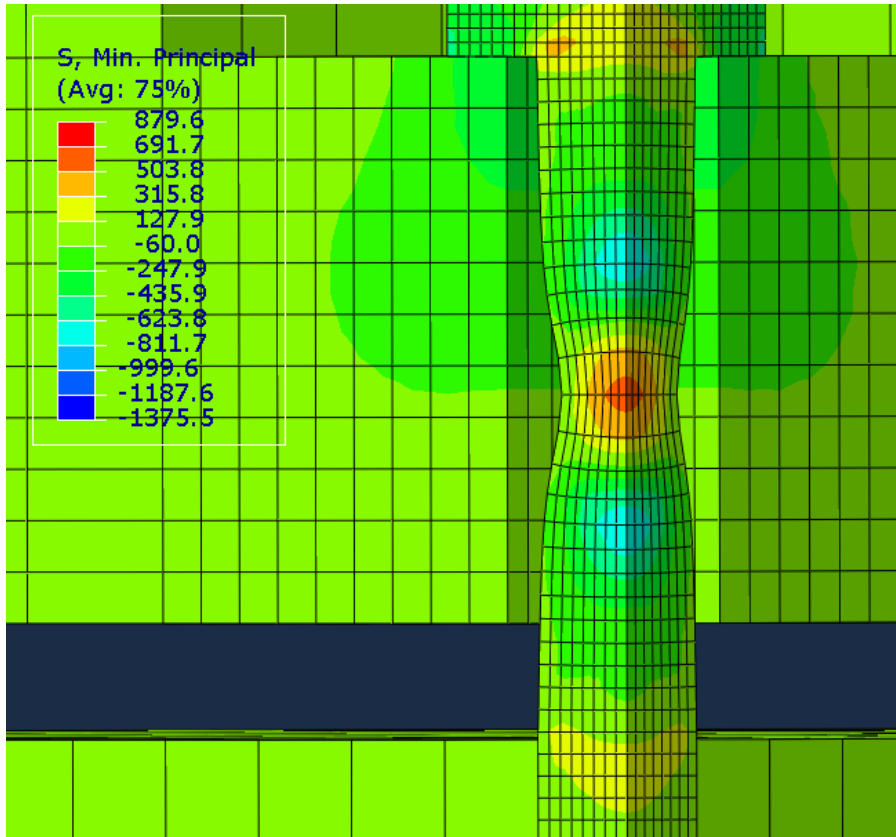
# Future Work

- Explore strain rate dependencies using Johnson-Cook calibrations to dynamic data.
- Quantify the error in the damage calibrations for different specimens of the same material.
- Investigate the influence of the angle at which shear loadings are applied.
- Explore the application of non-proportional loadings (e.g. torsion-tension) and their influences in each of the constitutive models used.

# Acknowledgments

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# Minimum Principal Stress in Models



# Background: Stainless Steel A286

- Stainless steel alloy, commonly used in high performance, high temperature applications.
- Several key advantages:
  - Retains strength and corrosion resistance at high temperatures
  - Can be precipitation hardened
  - Heat treatable
- SNL has provided tension test data for smooth specimens, and tension and shear test data for various sized threaded fasteners of A286 for our study.