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Steam Turbine Upgrades: The Key to Optimizing Overall Plant Cycle Performance

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Steam turbine Applied Technology .

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Effective Upgrades Come in Many Forms

- ◆ **Component Upgrades** for :
 - Improved Turbine Efficiency
 - Unit / Component Reliability
 - Increased operating times between outages.
- ◆ **Turbine Upgrades** to Support Overall Plant Optimizations
 - Review overall plant thermal cycle
 - Identify any changes in Plant Process Requirements
 - Review Overall Turbine Capability to Accommodate Anticipated Process Changes;
 - Revised Section flows / Pressures / Stage loadings
 - Boiler / Condenser/ Generator Capability
 - Minimize off Peak operation / Optimize Section performance.

Effective overall cycle optimization often requires accommodating a wide range of OEM equipment

Impulse

- GE
- MHI
- W Industrial
- Delaval
- Toshiba
- Thermodyn
- Allen
- Shin Nippon
- ABB
- Ansaldo Industrial
- Elliot
- Franco Tosi
- Hitachi
- Metro Vickers
- Skoda
- BHEL
- Bellis
- DR
- Worthington
- IR
- TurboDyne
- Peter Brotherhood
- Terry
- Murray
- Coppus

Reaction

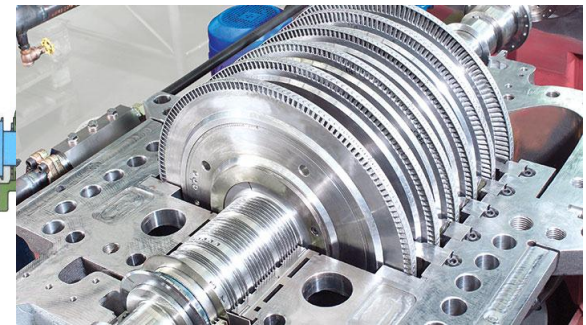
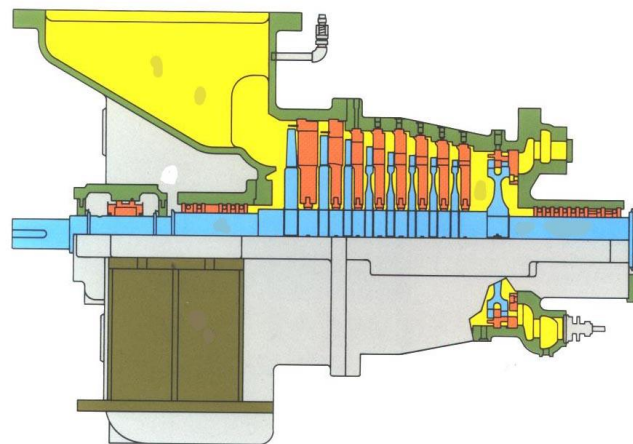
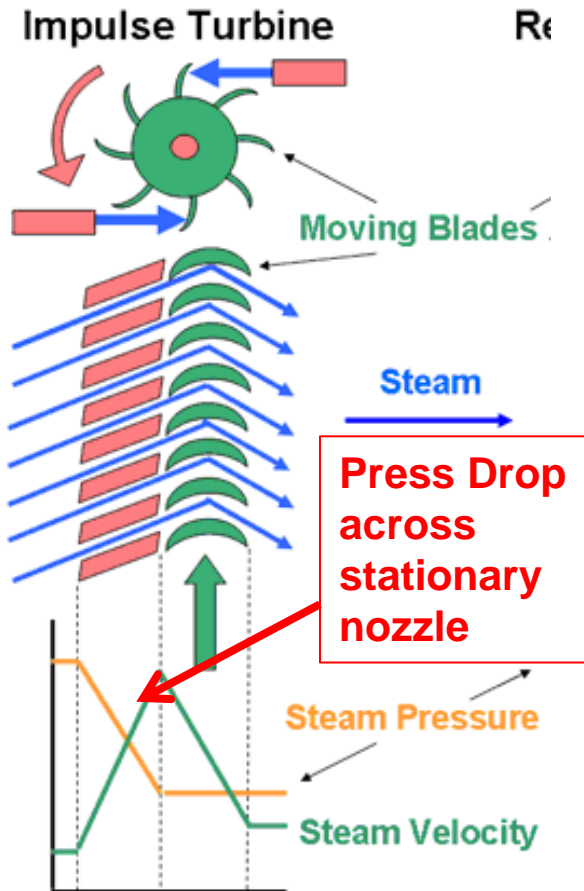
- Siemens
- Parsons
- W Utility
- Nuovo Pignone
- Fuji
- Allis Chalmers
- Alstom
- Ansaldo Utility
- Escher Weiss
- Man
- BHEL



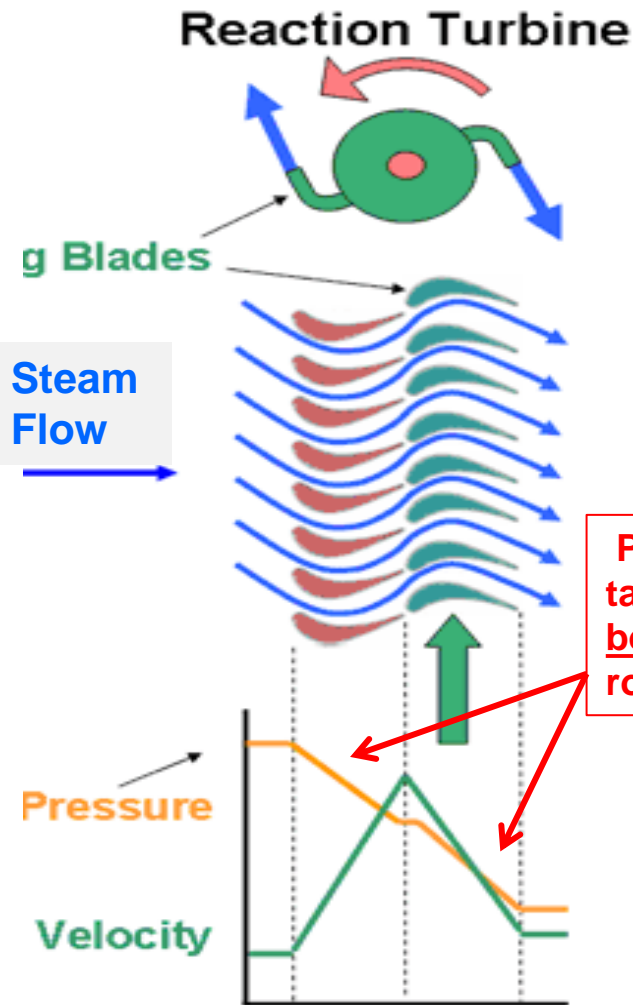
Impulse Turbine Design

◆ Impulse

- Steam accelerated by pressure drop across stationary components
- Pressure drop limited to stationary components.
- High velocity steam impinges on rotating blades creating torque
- Most common US design due to rugged construction and reliability



Reaction Turbine Design Methodology



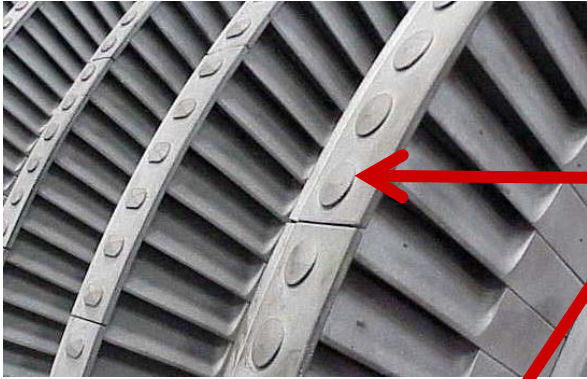
Reaction

- ◆ Stage pressure drop occurs equally across stationary and rotating components.
- ◆ Lower pressure ratios across each stage
- ◆ Results in twice as many stages for given energy range.
- ◆ Lower steam velocities, results in lower losses and higher theoretical performance
- ◆ Drum rotor construction / Less physically robust design.
- ◆ Requires balance piston to counteract stage thrust.



Typical Component / Reliability Upgrades

◆ Modernization of Obsolete / Unreliable Blade Designs



From

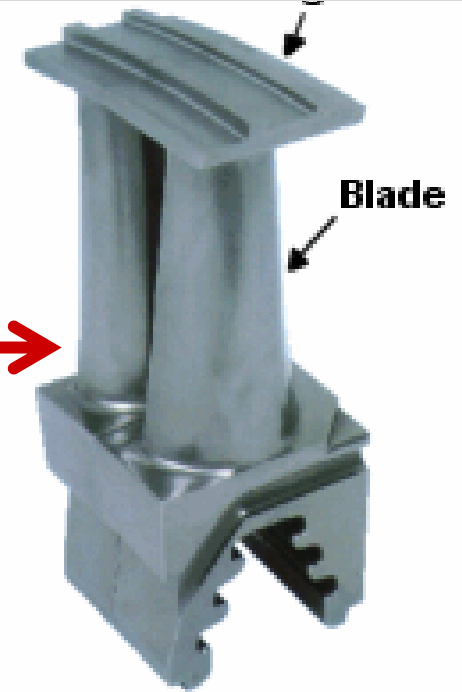
- Mechanically attached (Peened) shrouds
- Constant profile vanes
- Short band groups
- Thru-Penetration tie wires
- 1960 Design standard



To :

- Optimized aerodynamic vane
- Continuously coupled integral tip shroud.
- Improved Dovetail /Fastener (as required)

Continuously coupled integral cover.



Typical Reliability Upgrade Opportunity

From :

- ◆ Obsolete constant cross section vane
- ◆ Hard peened tenons
- ◆ Short band groups
- ◆ Through drilled tie-wire
- ◆ Stress concentration at tie-wire penetration
- ◆ Challenging wet and corrosive operating environment



Typical Tall Blade Modernizations

Original Design

- ◆ Rigid / Brazed Lashing lugs & tie wires
- ◆ Reduced damping
- ◆ Repetitive maintenance



Fully Modernized design

- ◆ Integral (not through drilled) mid-span lugs
- ◆ Inserted (loose) mid-span couplings for increased damping and vibration suppression.

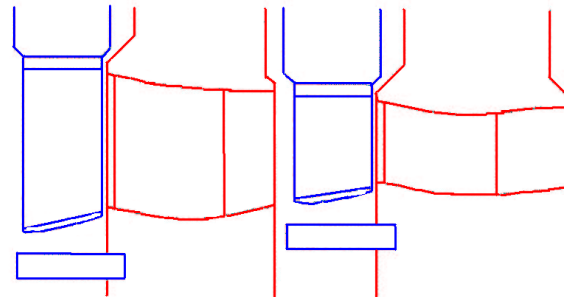
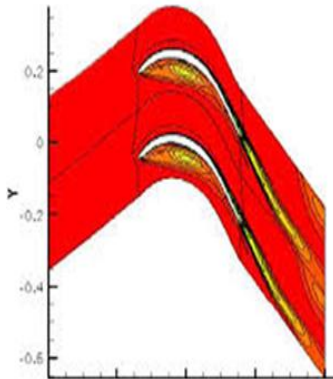
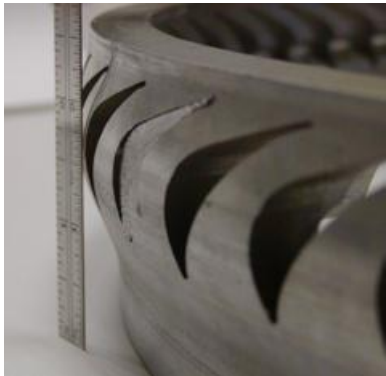
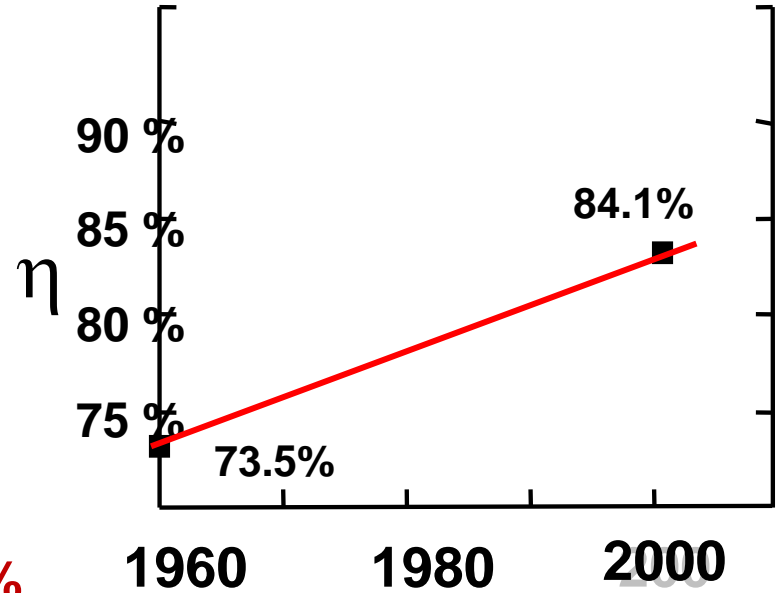


Performance Improvements Summary

Design Feature **% GAIN**

<i>Stationary Blades</i>	0.5-0.75
<i>Profiled End Wall</i>	1.0-1.5
<i>Rotating Blades</i>	0.5-0.75
<i>Steam Path Surface Finish</i>	0.5-3.0
<i>Brush Packing</i>	0.5-1.0
<i>Throttling Losses</i>	1.0-2.0
<i>Stage Leakage Control</i>	<u>0.25-1.0</u>

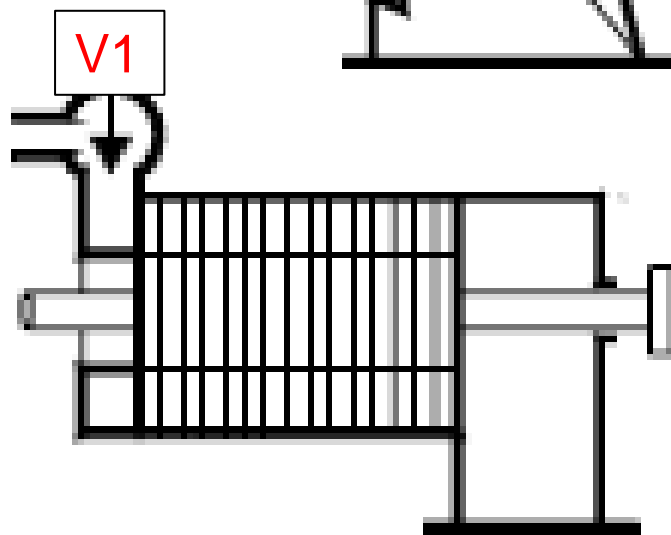
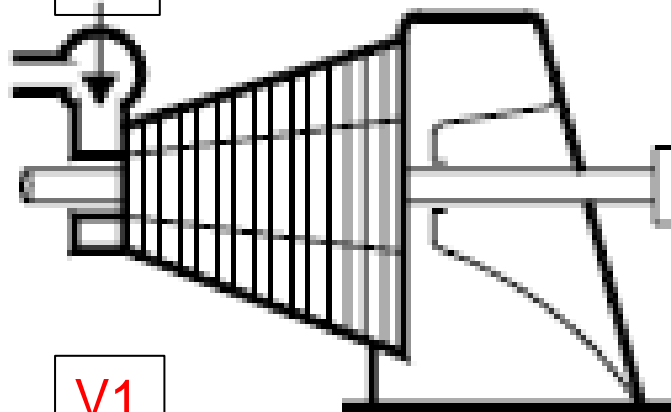
TOTAL **4.25 - 10.0 %**



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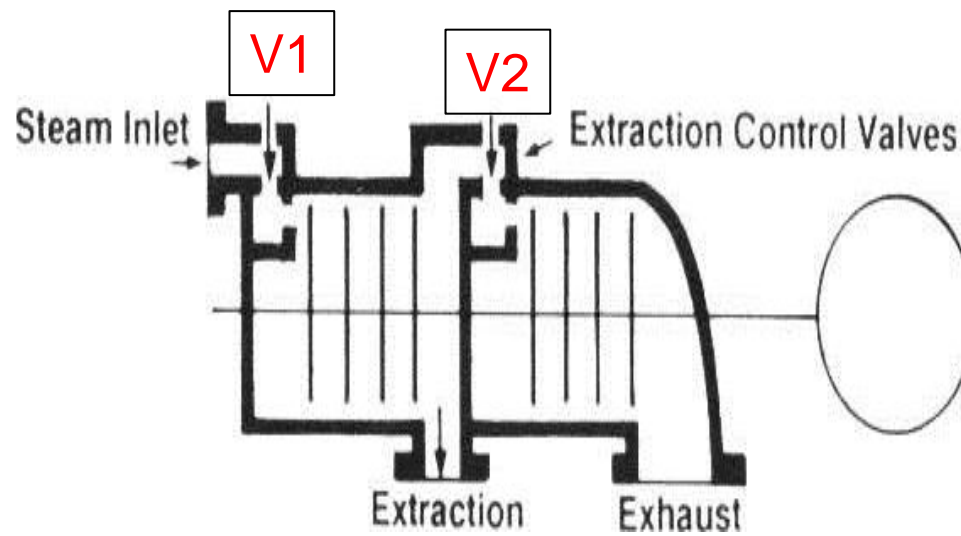
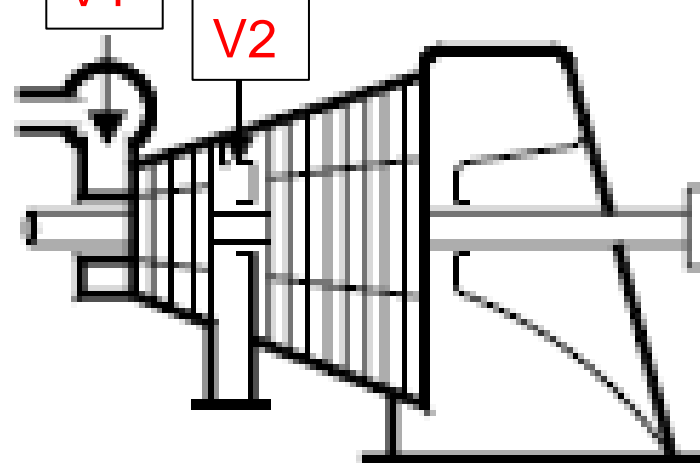
Steam Turbine Types & Opportunities

V1 Straight Through units



- Changes in inlet Flow / power/ efficiency
- Add extract or admission
- Convert to non- condensing

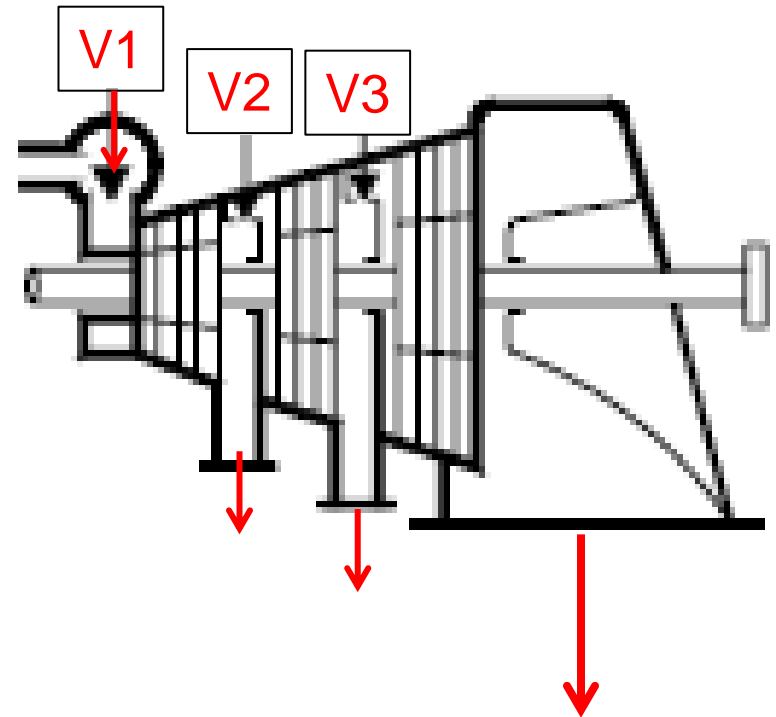
V1 V2 Extraction units



Multiple options for Changes in inlet / extract flow / pressure & output

Steam Turbine Types & Opportunities

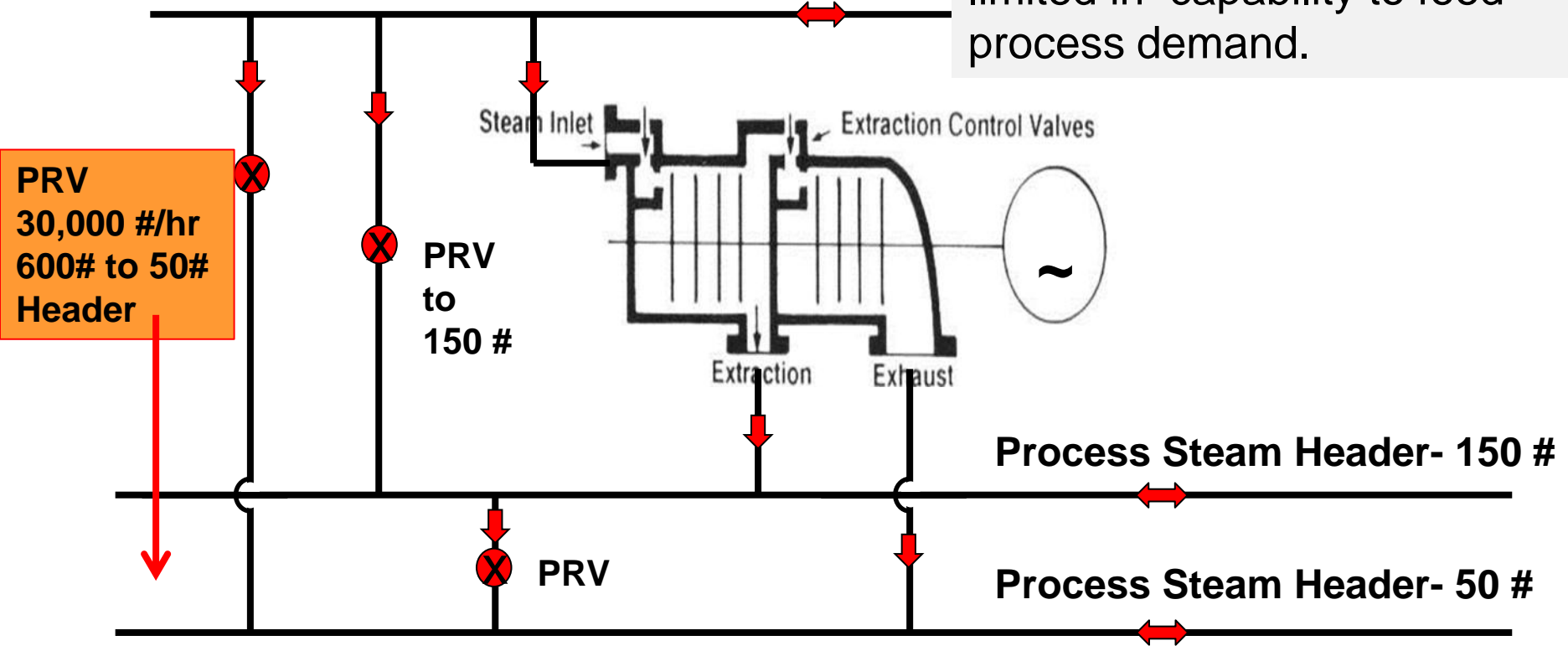
- ◆ Double auto extraction condensing (DAXC)
 - Changes in extraction pressures
 - Changes in section flows
 - Eliminate need to PRV of steam for process
 - Changes typically driven by revised process requirements
 - Common Paper Mill / Refinery Configuration



Typical Industrial Cycle Optimization

Main Steam Header
600 psig / 750F

Head End and Exh sections limited in capability to feed process demand.



- Customer PRV 30,000 #/hr 600# steam to 50 psig to feed process
- Benefits not apparent as process requirements satisfied
- **Power Gain = + 1.45 MW / Estimated cost savings = \$900 K / year**

Optimizing Turbine Sections

- ◆ **Turbines originally designed by specification to provide:**
 - Maximum capability at nameplate/max-rating
 - Optimum efficiency at “Guarantee” point
 - AE’s always provide additional flow capacity to account for degradation and compressor margin, API 612 requirement

- ◆ **But turbine sections must also accommodate:**
 - All maximum and minimum swings in section flow and pressure
 - Variable summer and winter process demands
 - Specified power and section flow requirements
 - Need for supplemental PRV steam for process use
 - Provide optimum performance at “normal” operating point

As a result many turbine sections actually operate at significantly off peak design flow and efficiency

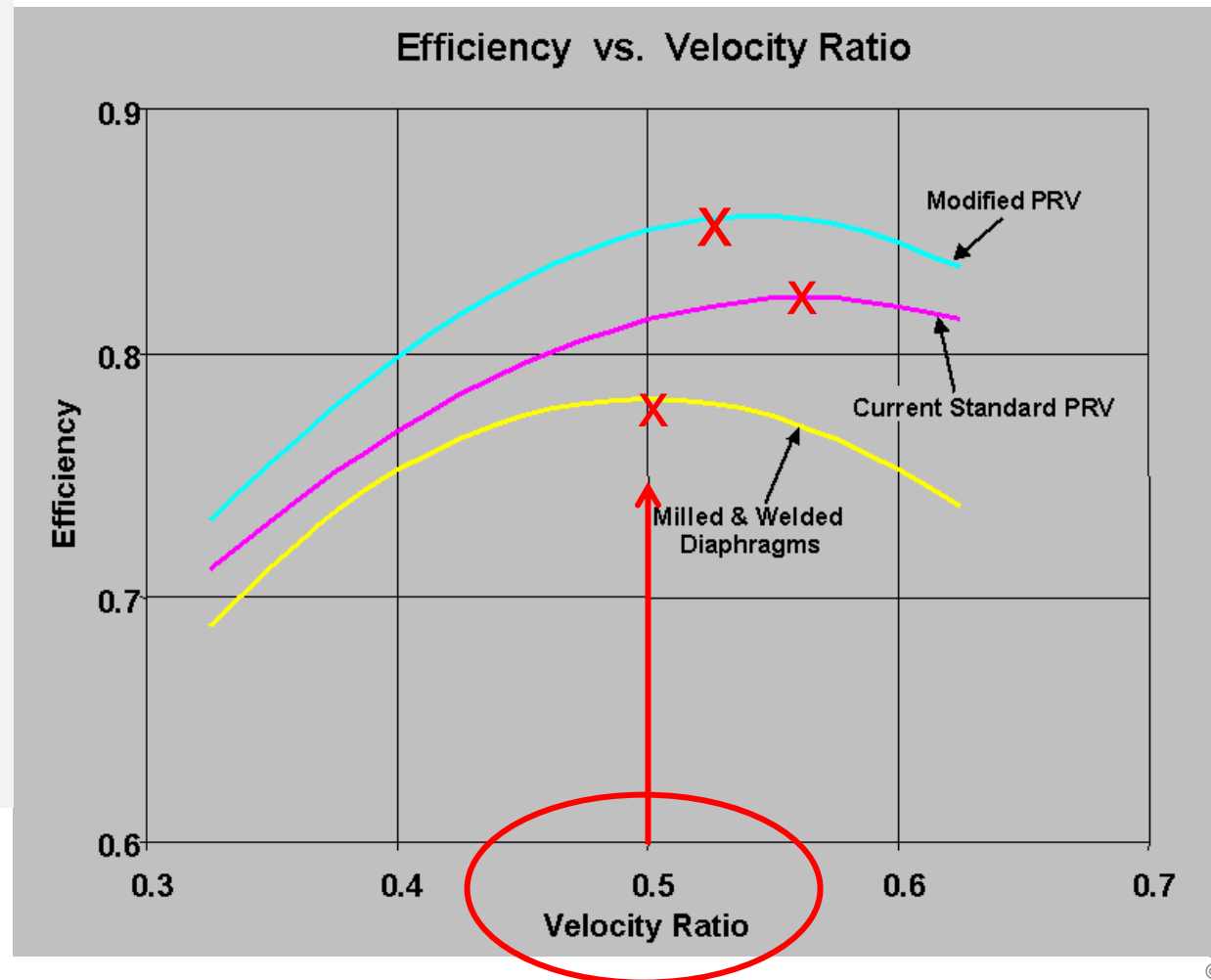
The Key Thermodynamic Design Parameter.

Velocity Ratio

$$W/V_0 = \frac{\text{Wheel Speed}}{\text{Steam Jet Velocity}}$$

Key Characteristics

- Max Eff @ turbine design point
- Efficiency Max when Wheel speed = $\frac{1}{2}$ Steam Jet Velocity
- W/V_0 directly Influenced by off-design section flows.
- Efficiency reduced when operated off peak.



Stage loading characteristics @ part load operation.

Inlet Press **1250 #**

Stage P1/P2

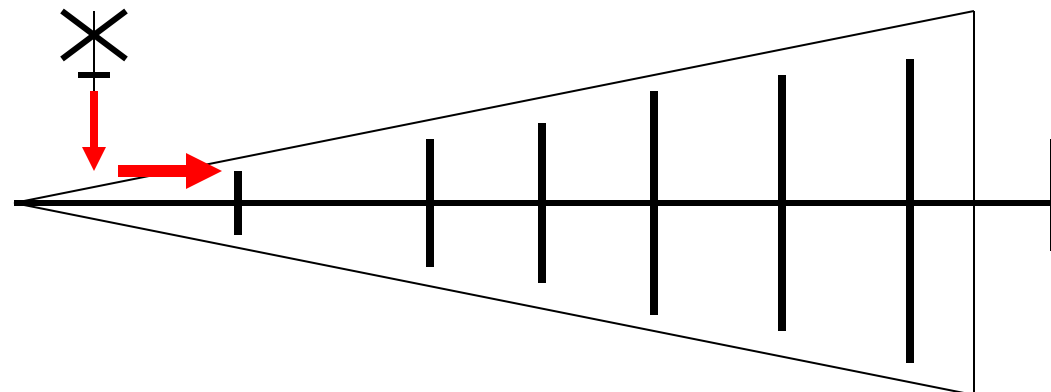
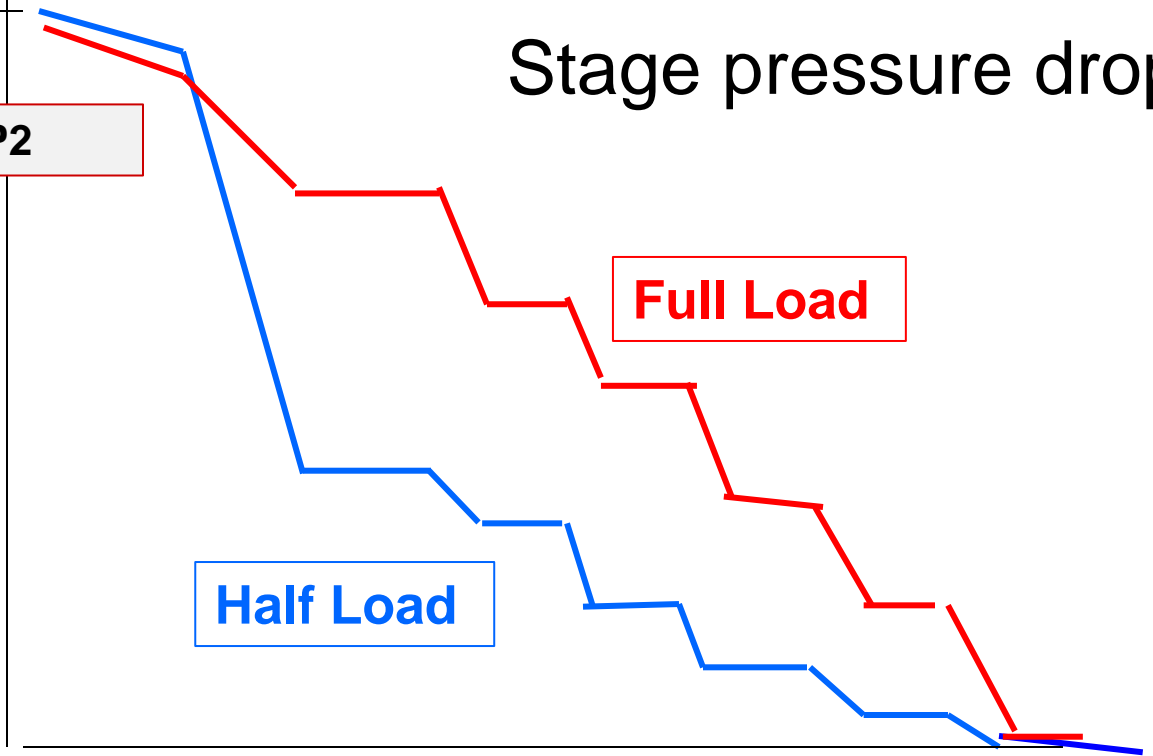
Unit design
250 K#/hr inlet
Straight condensing

Operation @ Half load
125 K #/hr inlet - Cond.

Impact on operation

- Increased Stg # 1 loading
- Stgs 2-6 starved for flow
- Off peak performance on Stg #1 & Stgs 2-6
- Losses and potential heating issues on Exhaust stages @ low flows .

Stage pressure drop



Stg # 1 2 3 4 5 6

Ex - Optimization For Part Load Operation

- ◆ **Problem** : Turbine operating @ part load / well below design Conditions.
- ◆ **Result** Significant efficiency loss
- ◆ **Solution:** De-rate individual turbine section or entire steam path to fit new cycle requirements
- ◆ **Example**
 - Turbine running @ 20% design throttle flow.
 - Original efficiency = 80%. Efficiency @ 20% throttle flow = 40%.
 - Inlet = 900 psig / 900-F/ Exhaust - 50 psig.
 - Original max flow = 1,200 K#/hr / new flow is 240 K #/hr
 - Purchased Power costs \$.07/kw-hr
 - De-rated unit can achieve Approx. 70 + % overall efficiency
 - Output at 40% efficiency = **3,980 kW's**
 - Output @ 70% efficiency = **6,964 kW's** (Post Rerate)
 - Power Gain from Re-Rate = **2,984 KW**
 - **Savings** = **\$1,835,000 /year**

Seemingly insignificant changes can often yield surprisingly large benefits!

Part Load Operation - Auto Extraction Turbine

Inlet Press **1250 #**

Stage pressure drop

Unit design

250 K#/hr inlet

100 K #/hr Extract @ 650#

Max Exh flow = 150 K #/hr

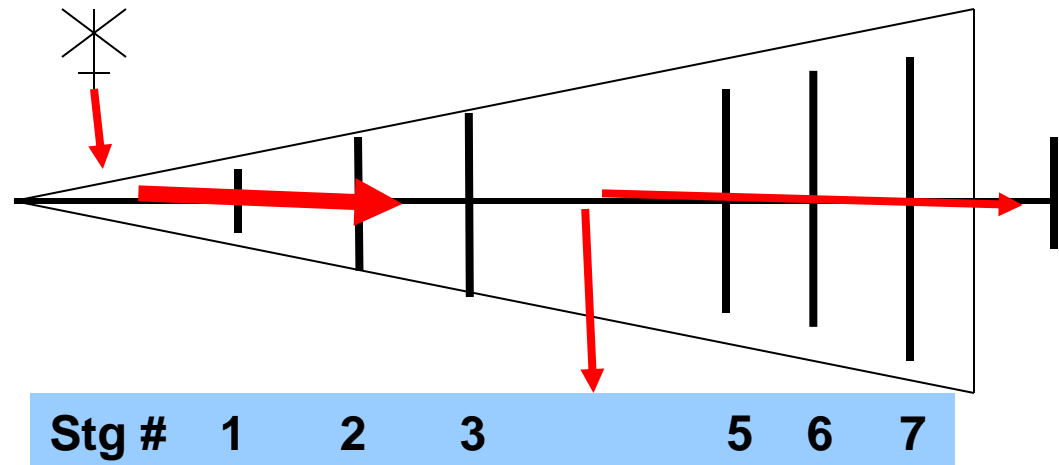
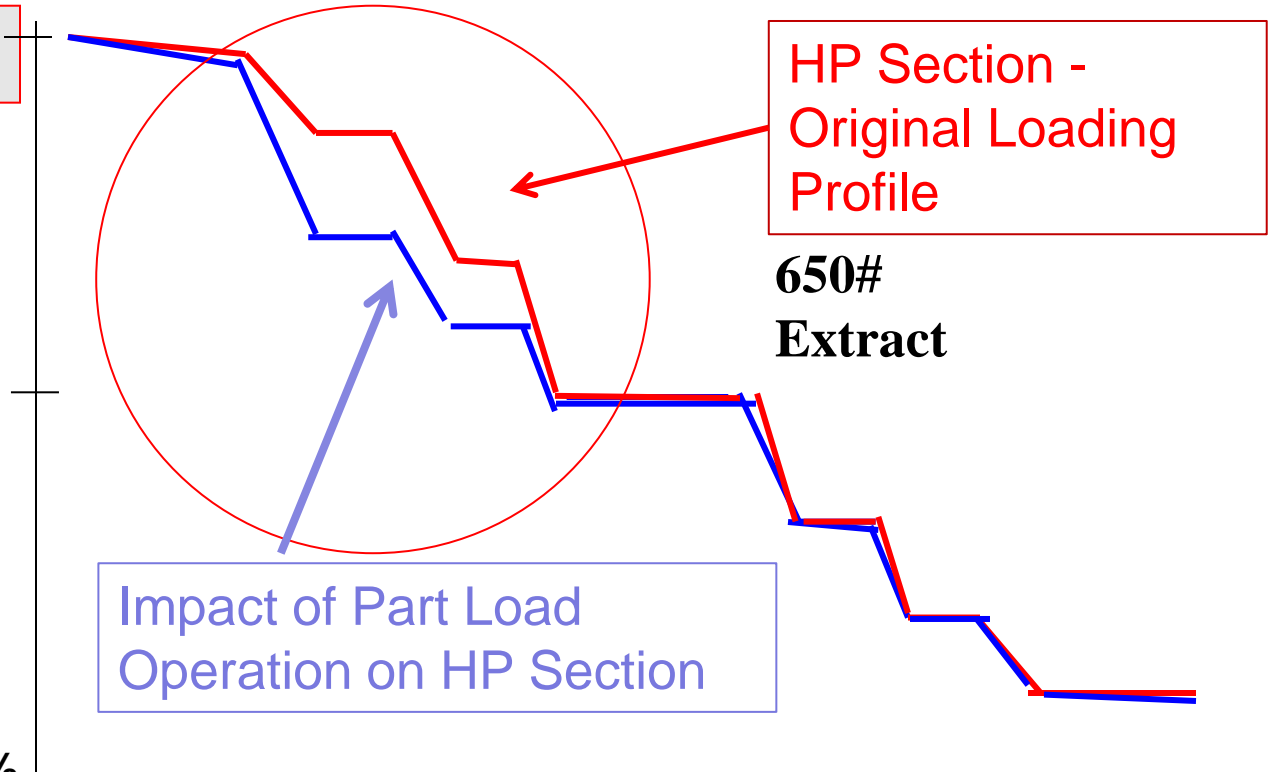
When run @ Zero Extract

150 K #/hr inlet

150 K #/hr Exh

Impact on operation

- Head end running at 60% Design flow.
- Result = off peak HP section Efficiency
- Increased Stg #1 loading
- Loss of overall MW output



Impact of Changes in Extract Pressure on Section Loading

1250 #

Inlet Press

Observations :

- Increased energy range on head end
- Reduced energy range on Exhaust

Required Changes :

- Add stage to HE
- Remove stage in Exh

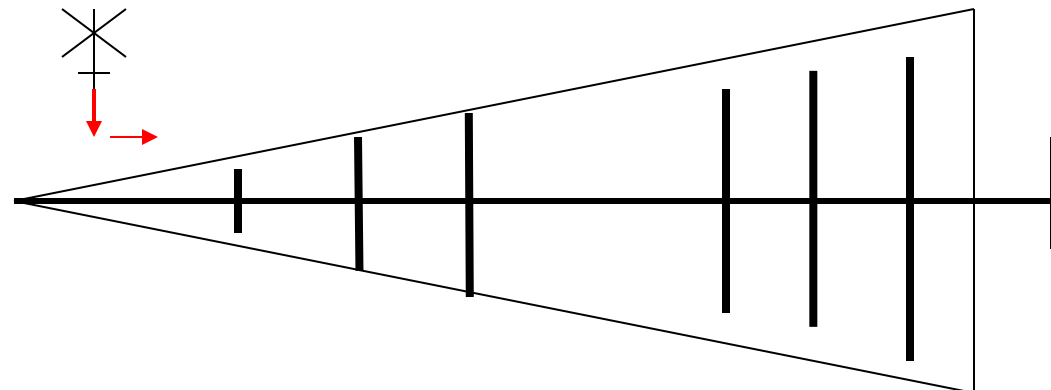
Impact on operation

- HE oversized for flow
- Increased stg #1 loading
- HE Needs additional stage
- LP section severely undersized . Needs fewer stages

Stage pressure drop

650 # Extr

150#
Extract



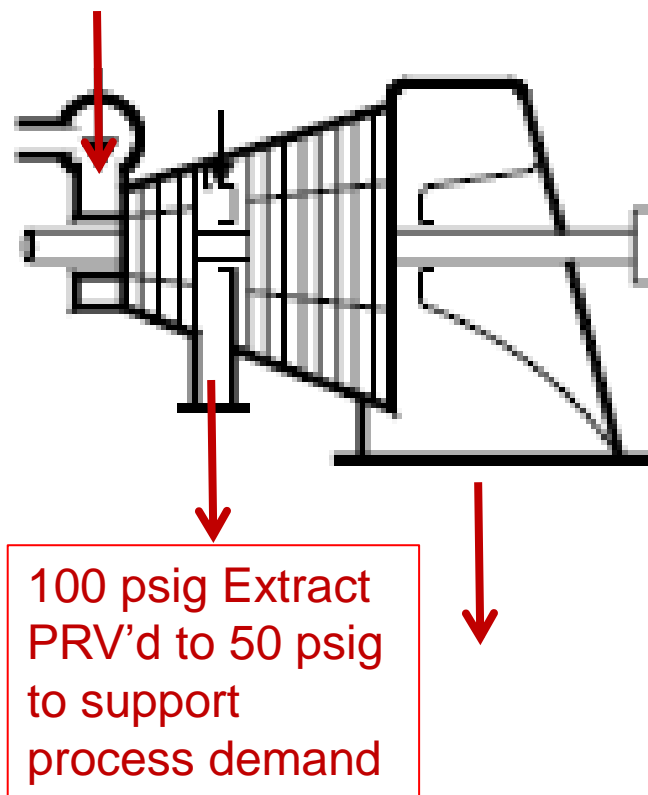
Stg # 1 2 3 5 6 7

Typical Revamp Scenario-Cycle Change

- ◆ Single-Auto-Extract-Condensing unit (SAXC)
- ◆ Process changes require change in extraction pressure from 100 to 50 psig
- ◆ Process demand currently satisfied by PRV of 200,000 #/hr of 100 psig steam to 50 psig level
- ◆ Loss of flow through turbine HP section results in loss of 2,700 KW output
- ◆ Assume Electricity valued \$.07 cent per kw-hr.

Potential Economic Benefit = \$ 1.6M/year

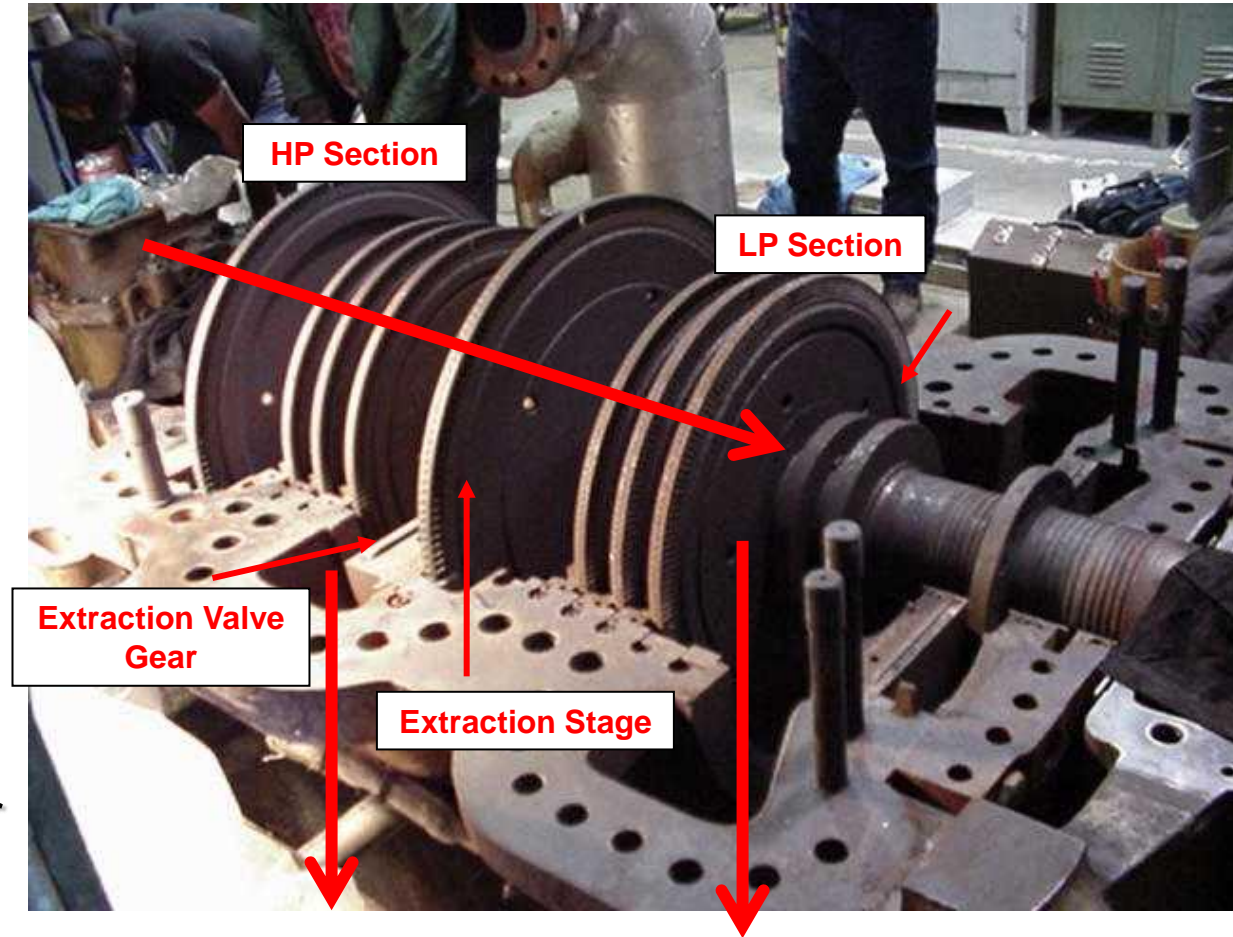
Seemingly insignificant changes can often yield surprisingly large benefits!



Ex- Impact of Eliminating Extraction

Define Objective:

- ◆ 5 MW GE -SAXNC
- ◆ Change in plant process eliminates need for extraction steam
- ◆ Loss of extraction flow results loss of available MW output.
- ◆ LP section unable to pass additional flow required to make up for MW shortfall.



Current Turbine Section Flow Limits

Unit capability as shown on Original Extraction Map

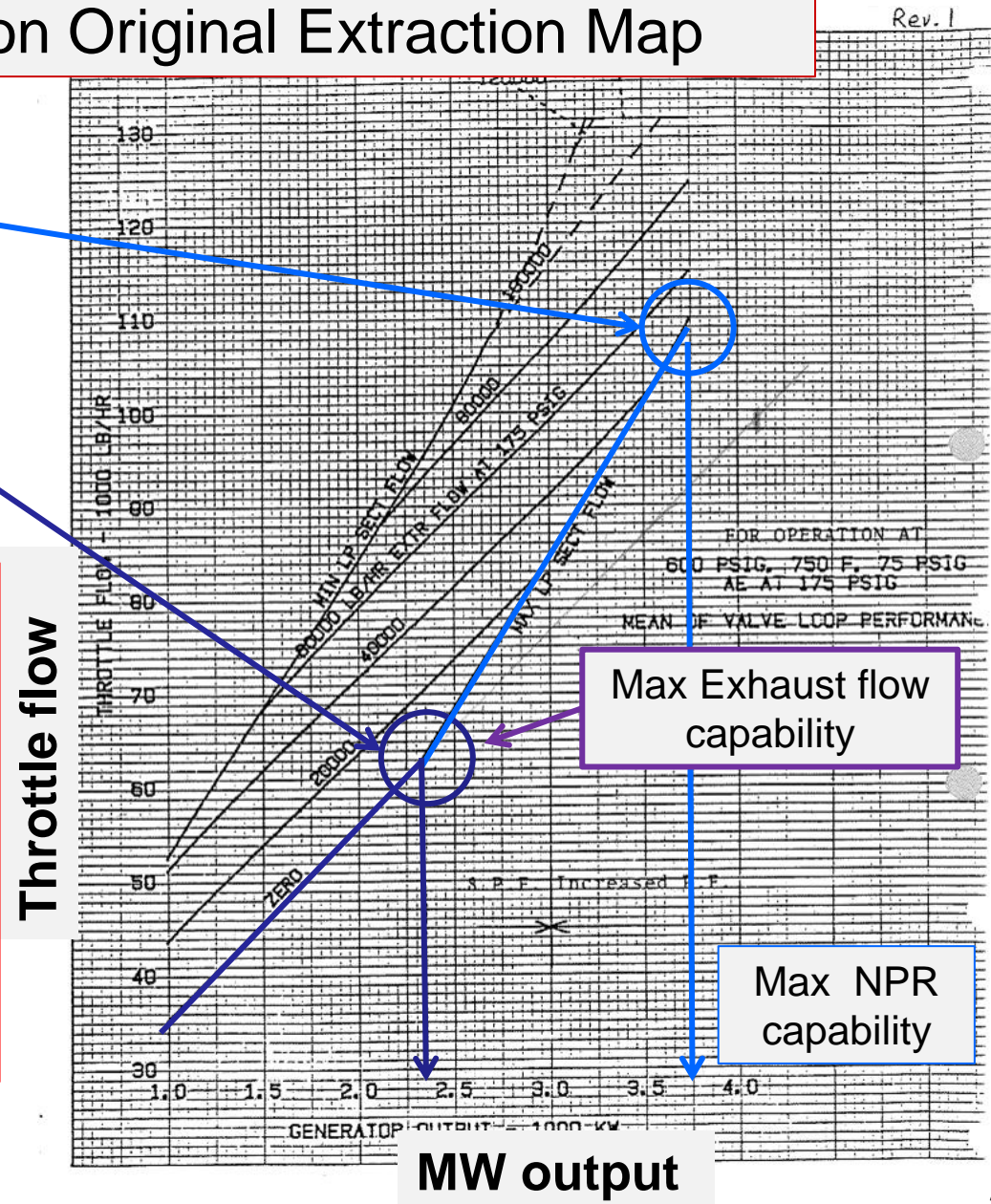
- Max capability w:
- Max Exh flow
 - And 50K #/hr Extract
 - 3.75 MW

- Max capability w/
Zero Extract flow
- 2.35 MW

Change in plant process eliminated need for extract flow

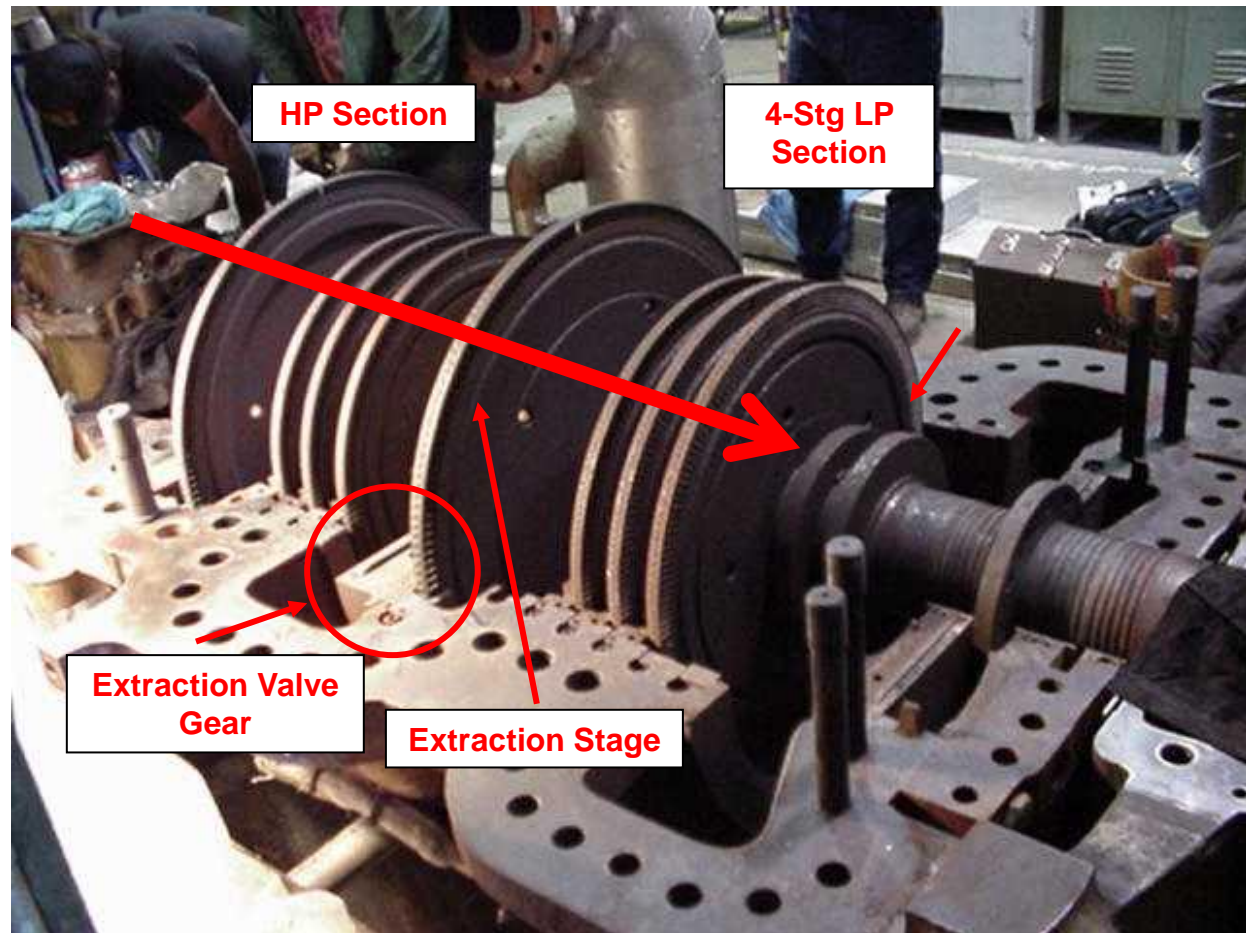
Boiler capacity available & process requires additional MW output.

Max Cond output = 2.35 MW



Observations on Unit limitations:

- ◆ Extraction valve gear restricts flow capability to the LP section.
- ◆ Pressure @ Stg # 5 no longer limited by Extraction
- ◆ LP stages 5-8 under-sized for HP flow capability.
- ◆ Straight Non-Condensing operation without flow path changes results in loss of MW output.



Uprate Changes & Benefits

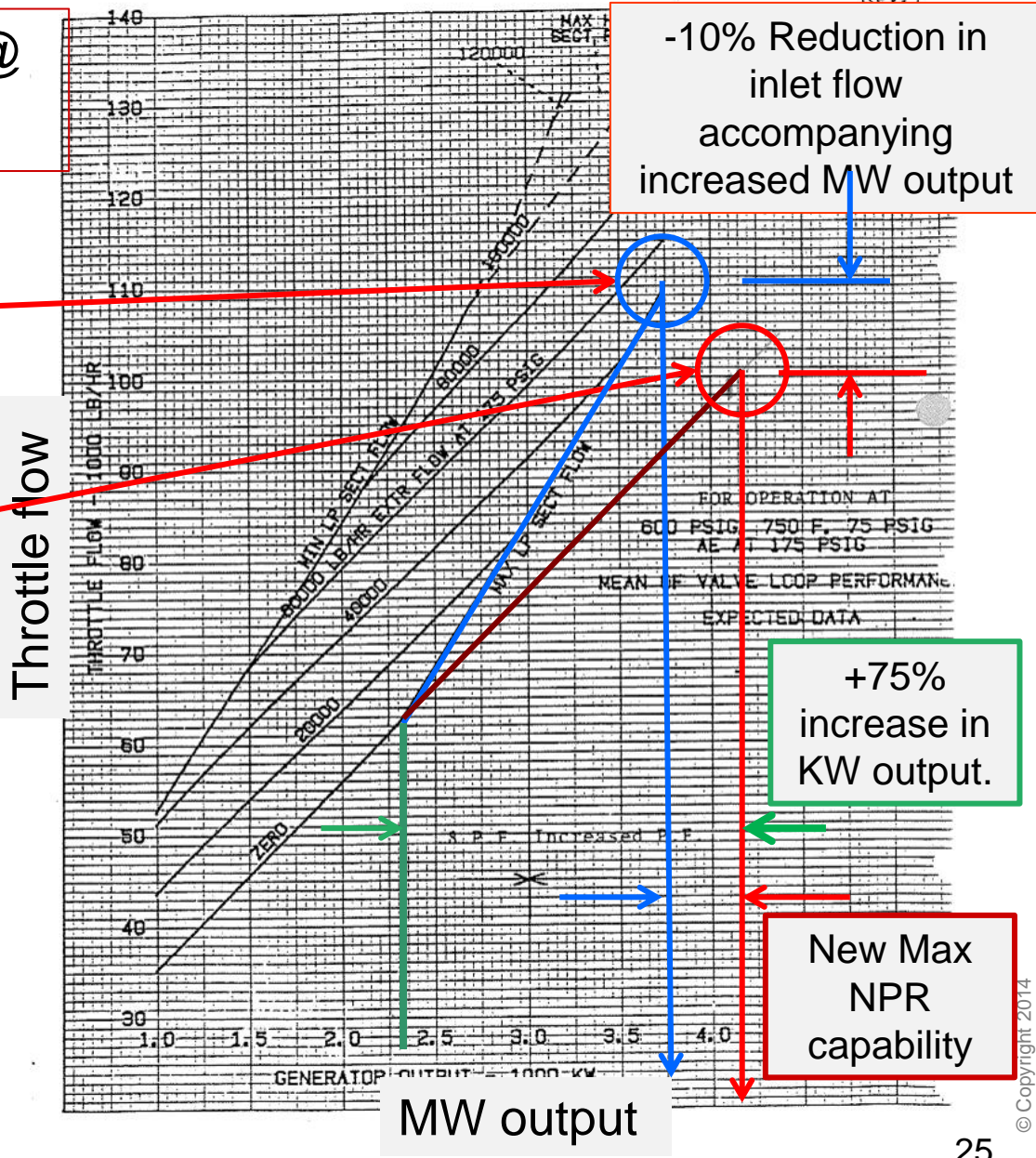
Rev. 1

Post Uprate Performance @ new operating conditions

Original Max MW output required additional 50K #/hr Extract flow

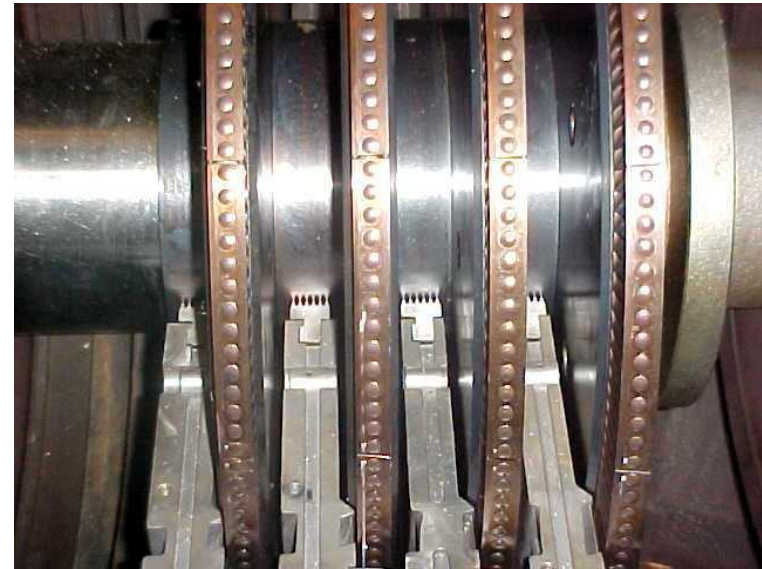
New capability as reconfigured for Straight-Condensing operation.

- SC Output capacity increased from:
- 2.35 MW to 4.15 MW (+75%)
 - @ 10% less inlet flow
 - & limited hardware changes.
 - Eliminated need to vent Extract steam to generate needed MW.



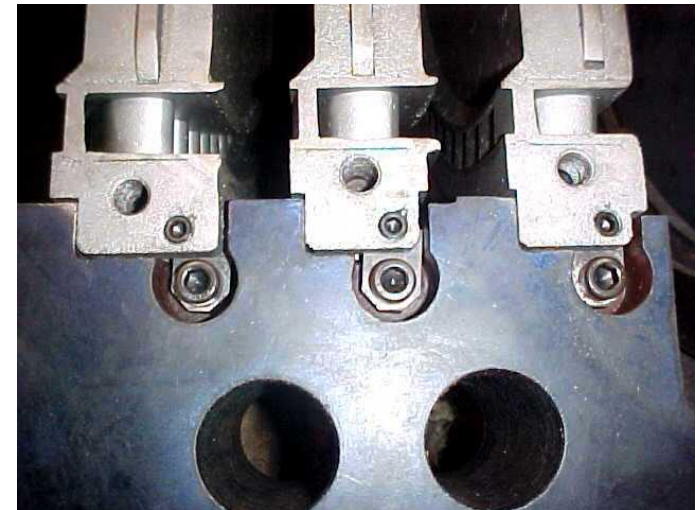
LP Rerate Modifications

- ◆ LP Section Modifications:
 - New forged wheels with high performance blades
 - New high efficiency diaphragms
 - Blank Casing Extraction Ports
 - Remove Extraction valve gear
 - Low leakage Inter-stage & gland seal labyrinths



Net Benefit

- ◆ Increased :
 - L.P section flow capability,
 - Efficiency,
 - MW output.



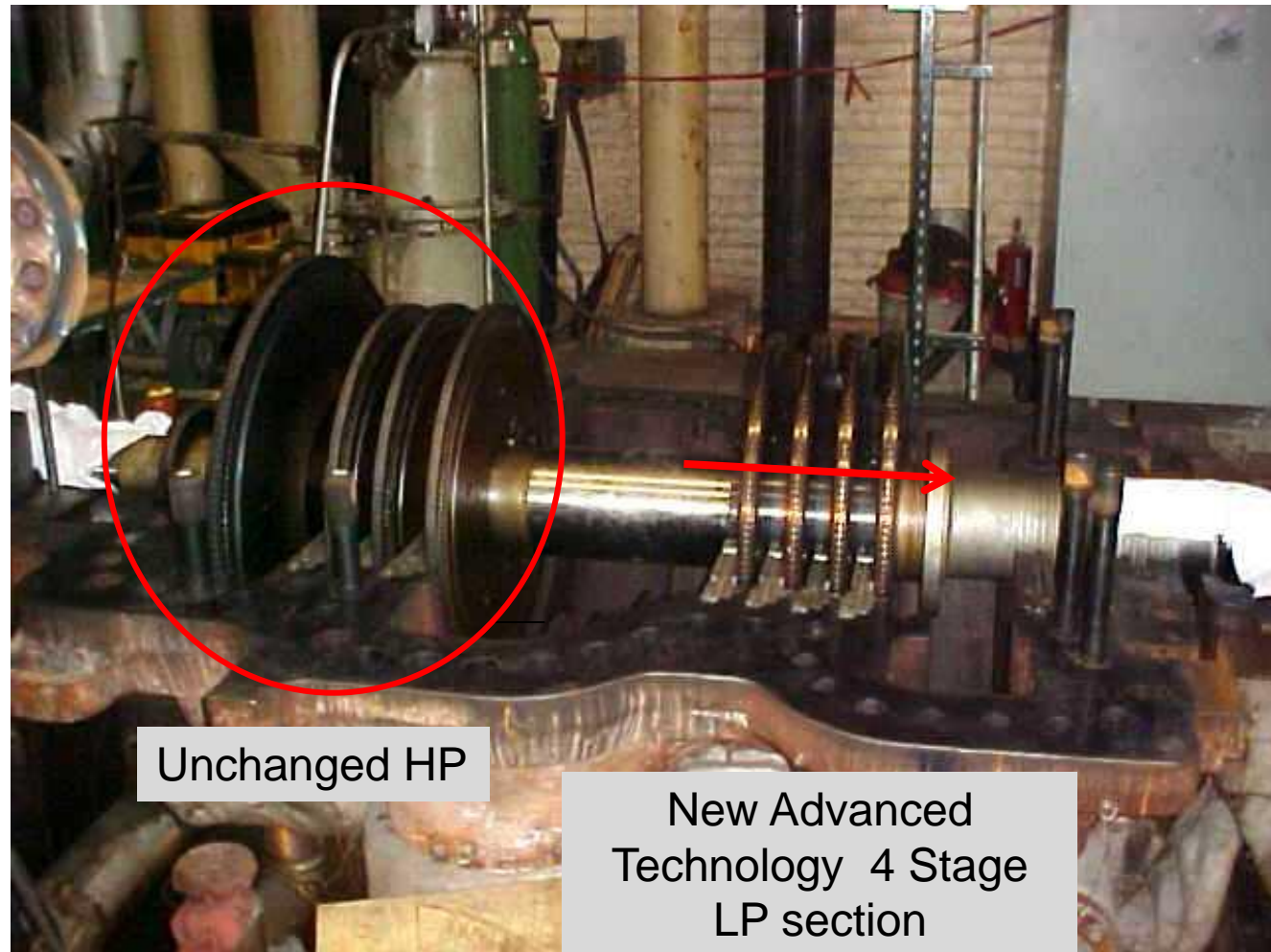
Post Uprate Steam Path configuration

Revamp Solution:

- ◆ Convert unit to straight through Non-Condensing operation
- ◆ Remove Extraction
 - Nozzle box
 - Valve gear
 - Reduces losses
 - Simplifies controls

New LP staging to Increase :

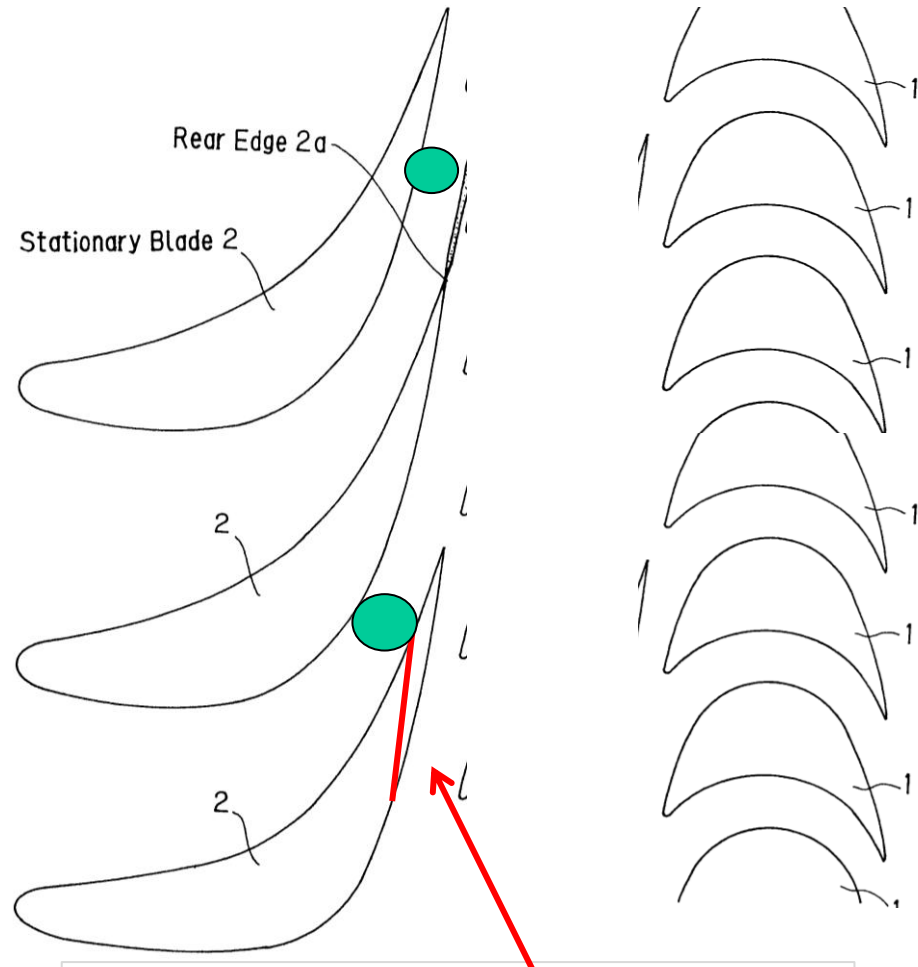
- Efficiency
- Flow capability
- MW Output



Reduced scope rerate options

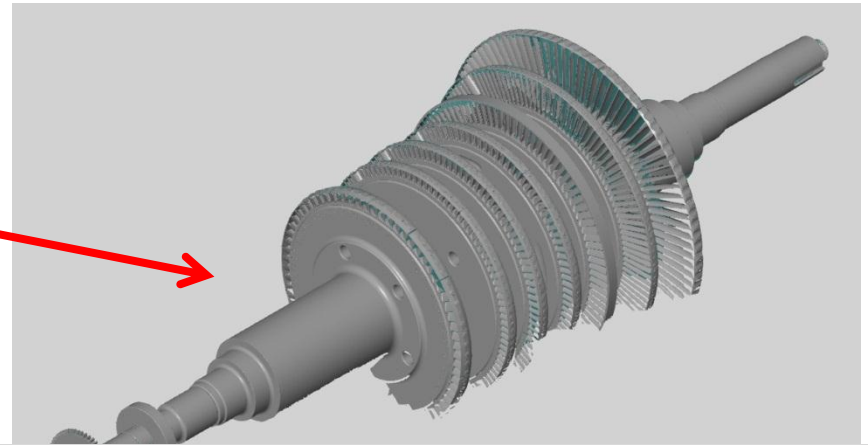
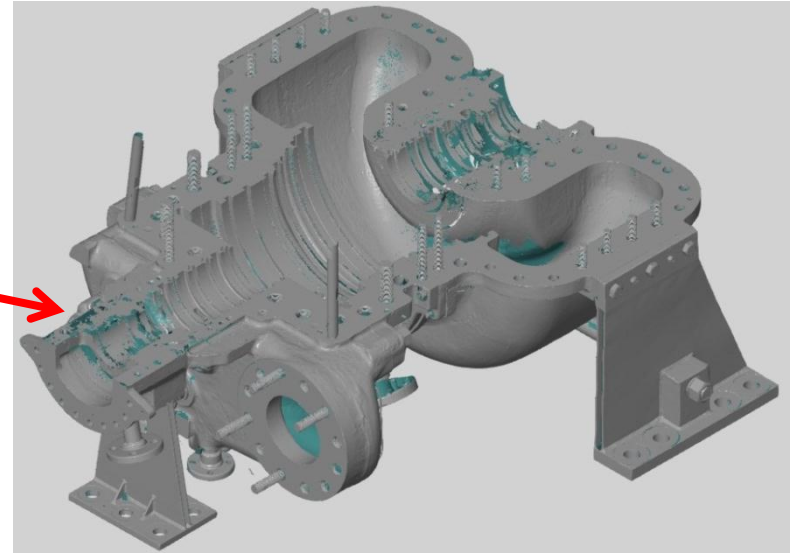
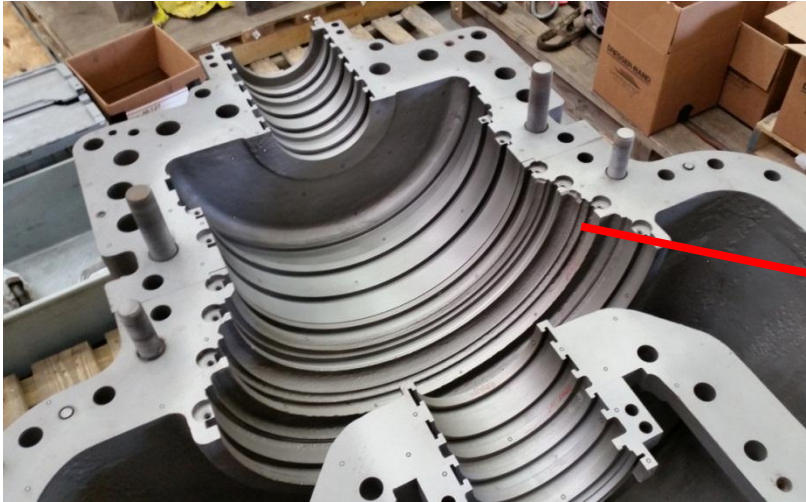
Limited Increases in flow passing capability possible with minor changes.

- Modify diaphragm flow passing area by grinding
- Increase d flow capability with some limited reduction in performance.
- Mods typically implemented as a short term fix during outage, with follow up optimization



Reconfigured nozzle exit edge.

Advanced optical scanning technology can provide critical data despite limited access to original design records.

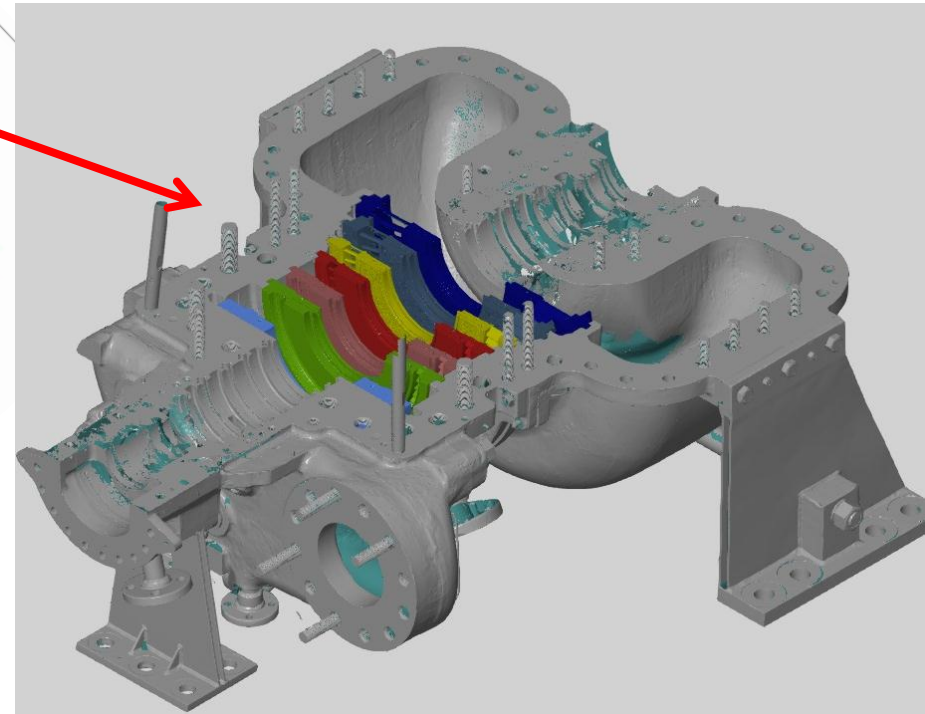
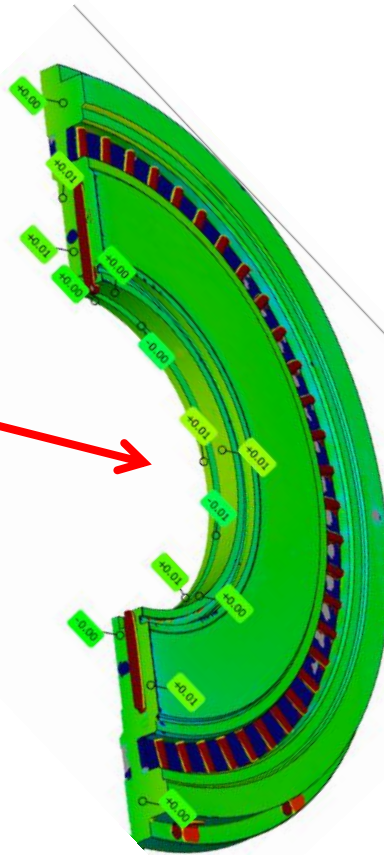
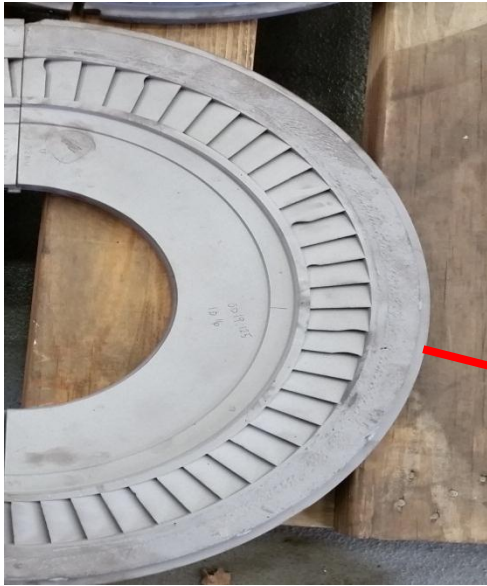


Detailed Casing & Rotor Scans can be completed on site with minimum impact on outage schedule

3-D models can be used to analyze and Verify Critical Component Assembly & Interfaces,

Solid models applied to support:

- Dimensional verification
- Detailed stress analysis
- Short cycle / flexible mfg.

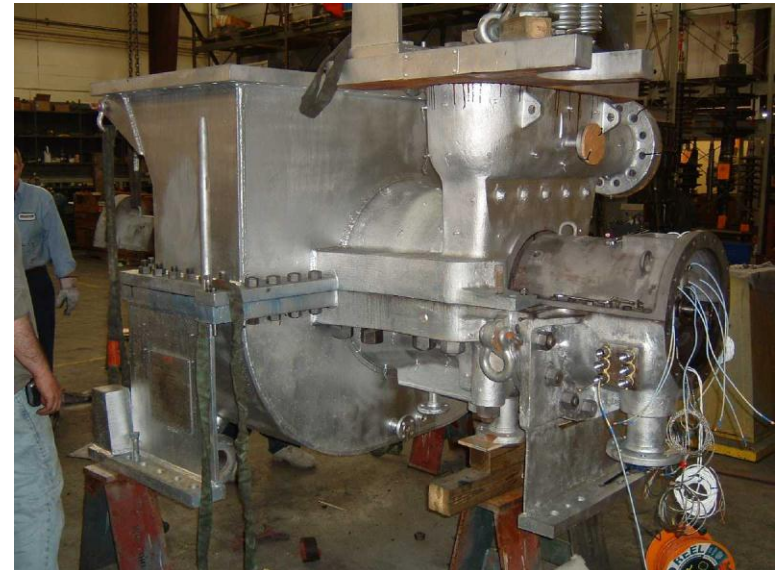


- Identify potential limitations w/ design.
- Facilitate installation
- Reduce outage cycle & risk
- Confirm revamp design prior to installation

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Reapplied Turbines Can = Excellent Value

- ◆ Non-Reheat units have near unlimited service life w/ proper inspection and upgrade.
- ◆ Steam Path measurement can identify potential uprate options.
- ◆ Cost effective modifications often possible to accommodate new service conditions.
- ◆ Older units often more robust than modern designs.
- ◆ New units less than 50MW often Geared vs Direct coupled TG sets.



Benefits of Uprate vs New Unit

- ◆ Reduced delivery cycle – 30wks vs 12-18 months
- ◆ Reduced initial Capital Investment / Cost.
- ◆ Typically Installed during normal turbine outage.
- ◆ Minimizes BOP / Foundation changes and Installation costs
 - These can be 3-10X the cost of comparable revamp
- ◆ Minimum power interruption / delayed installation
- ◆ Uprated units w/ modern Steam path components can deliver comparable new Unit performance @ significantly reduced cost
- ◆ Assured of a Proven Reliable Design.
- ◆ New unit start up delays can extend installation time
- ◆ New 30-60 MW Turbines are often less rugged - High Speed Geared sets.



Uprates Typically Far More Cost Effective Than a New Replacement Turbine

Tailoring Turbine Uprates to Enhance Overall Cycle Efficiency

Optimizing Cycle Efficiency is the most Significant Optimization Opportunity.

Cost Effective Revamps Not Always Obvious

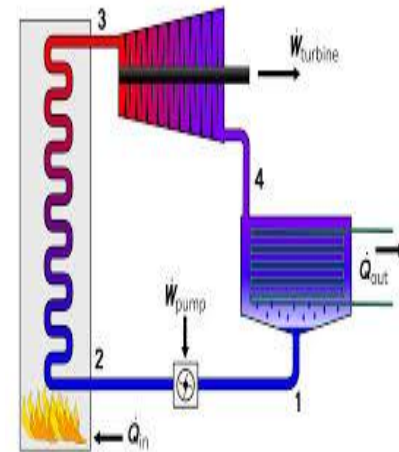
- Impact of Off-Optimum operation seldom recognized
- Customers experts on their process but not always on turbine design.
- OEM's typically concerned with their equipment / not overall cycle benefits.

Significant Optimization Opportunities Include:

- ◆ Optimizing the turbine to fit the cycle
- ◆ Resizing turbine section flow to meet cycle demands.
- ◆ Adding Extractions / removing process and cycle bottlenecks
- ◆ Reconciling original design and actual operating conditions.
- ◆ Fully understanding turbine's role in cycle and unit operation
- ◆ Compensating for loss in MW output due to Addition of Emissions Equip

Optimizing Overall Plant Performance Requires a Focus on Both Cycle and Individual Turbine Efficiency.

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Common Misconceptions about Turbine Uprates

- ◆ They are always complex
- ◆ That they are prohibitively expensive
- ◆ They cannot be accommodated during a normal turbine outage
- ◆ Paybacks are difficult to commercially justify
- ◆ They require major balance of plant changes.



Turbine Uprates are often overlooked in cycle optimization studies, and can represent a cost effective way to significantly improve overall system performance, and reliability.

Thanks For Your Attention !

◆ Questions ?