

# Undamped Vibration of a Beam

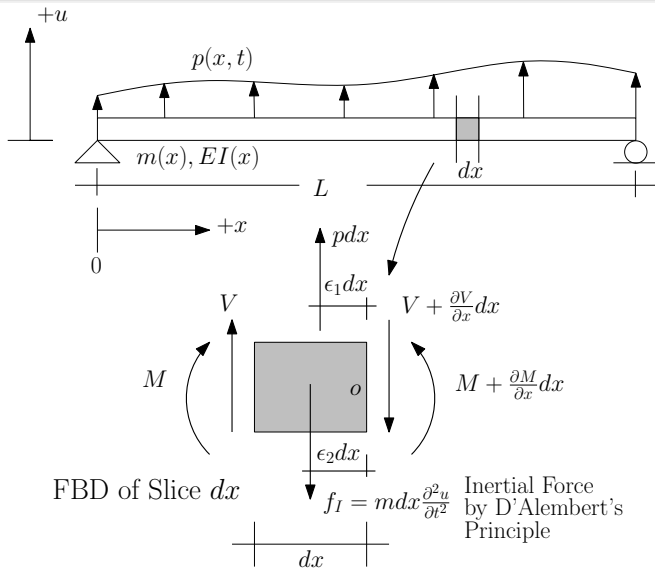
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# Problem - Undamped Transverse Beam Vibration



# Derivation of PDE

- Sum Forces Vertically, choosing + up

$$V - \left( V + \frac{\partial V}{\partial x} dx \right) + p(x, t) dx - m(x) dx \frac{\partial^2 u}{\partial t^2} = 0 \quad (1)$$

- Sum Moments about o, choosing CCW as + rotation

$$-M - V dx + p(x, t) \epsilon_1 dx^2 + m(x) \epsilon_2 dx^2 \frac{\partial^2 u}{\partial t^2} + M + \frac{\partial M}{\partial x} dx = 0 \quad (2)$$

Simplifying (1), and in (2) ignoring higher order terms in the limit as  $dx \rightarrow 0$  gives

$$\frac{\partial V}{\partial x} = p(x, t) - m(x) \frac{\partial^2 u}{\partial t^2}, \quad \text{and} \quad V = \frac{\partial M}{\partial x} \quad (3)$$



# Derivation of PDE

- From mechanics of materials class, moment curvature relation (given here to save time)

$$M = EI(x) \frac{\partial^2 u}{\partial x^2} \quad (4)$$

- Substituting equation two of (3) and equation (4) into equation one of (3) and rearranging yields

$$m(x) \frac{\partial^2 u}{\partial t^2} + \frac{\partial^2}{\partial x^2} \left[ EI(x) \frac{\partial^2 u}{\partial x^2} \right] = p(x, t) \quad (5)$$

- Equation (5) is the PDE governing the motion  $u(x, t)$ , subject to the external forcing function  $p(x, t)$ .



# Solving the PDE

- Analytical solution difficult or impossible to obtain due to  $m(x)$  and  $I(x)$ .
- Numerical methods such as Finite Element Method or Finite Differences can solve the PDE.
- Can simplify the PDE to demonstrate analytical methods by the following assumptions:
  - $m(x) = m = \text{constant}$  along the beam length
  - $I(x) = I = \text{constant}$  along the beam length
  - $p(x, t) = 0$ , ie, no forcing function



- After simplifying assumptions the governing PDE (5) becomes

$$m \frac{\partial^2 u}{\partial t^2} + EI \frac{\partial^4 u}{\partial x^4} = 0 \quad (6)$$

- To make things pretty at the end, define  $a^2 = EI/m$ , so that

$$\frac{\partial^2 u}{\partial t^2} + a^2 \frac{\partial^4 u}{\partial x^4} = 0 \quad (7)$$

# Solving the PDE

- Assume a solution of the following form

$$u(x, t) = \phi(x)q(t) \quad (8)$$

- Substitute (8) into the PDE (7) to get

$$\phi \frac{\partial^2 q}{\partial t^2} + a^2 q \frac{\partial^4 \phi}{\partial x^4} = 0 \quad (9)$$

- By separation of variables, observe that l.h.s and r.h.s must equal a constant,  $\beta^4$

$$-\frac{1}{a^2 q} \frac{\partial^2 q}{\partial t^2} = \frac{1}{\phi} \frac{\partial^4 \phi}{\partial x^4} = \beta^4 \quad (10)$$



# Solving the PDE

- From (10), two ODE's are obtained

$$\frac{\partial^4 \phi}{\partial x^4} - \beta^4 \phi = 0 \quad (11)$$

$$\frac{\partial^2 q}{\partial t^2} + \beta^4 a^2 q = 0 \quad (12)$$

- The respective solutions are

$$\phi(x) = A \sinh \beta x + B \cosh \beta x + C \sin \beta x + D \cos \beta x \quad (13)$$

$$q(t) = E \sin a\beta^2 t + F \cos a\beta^2 t \quad (14)$$

- $\therefore$  solution of the PDE (6) is  $u(x, t) = \phi(x)q(t)$ .





# Solving a Boundary Value Problem (BVP)

- To solve a realistic problem, boundary conditions must be specified
- The six boundary conditions (BC's) are
  - 1 @  $x = 0$ ,  $u(0, t) = \phi(0)q(t) = 0$
  - 2 @  $x = L$ ,  $u(L, t) = \phi(L)q(t) = 0$
  - 3 @  $x = 0$ ,  $u''(0, t) = \phi''(0)q(t) = 0$
  - 4 @  $x = L$ ,  $u''(L, t) = \phi''(L)q(t) = 0$
  - 5 @  $t = 0$ ,  $\dot{u}(x, 0) = \phi(x)\dot{q}(0) = 0$
  - 6 @  $t = 0$ ,  $u(x, 0) = Gx(L - x)$ ,  $G$  specified constant

# Applying the boundary conditions to $\phi(x)$

- Applying the first four boundary conditions yield the following results

①  $\phi(0) = B + D = 0$

②  $\phi(L) = A \sinh \beta L + B \cosh \beta L + C \sin \beta L + D \cos \beta L = 0$

③  $\phi''(0) = B\beta^2 - D\beta^2 = 0 \Rightarrow B - D = 0$

④  $\phi''(L) = A\beta^2 \sinh \beta L + B\beta^2 \cosh \beta L$   
 $- C\beta^2 \sin \beta L - D\beta^2 \cos \beta L = 0$

- From BC's (1) and (3),  $B = 0$  and hence  $D = 0$

# Applying the boundary conditions to $\phi(x)$

- From BC's (2) and (4)  
 $A \sinh \beta L + C \sin \beta L = 0$   
 $A \sinh \beta L - C \sin \beta L = 0$
- The above results imply

$$A \sinh \beta L = 0 \quad \text{and} \quad C \sin \beta L = 0 \quad (15)$$

- From the first expression of (15),  $A = 0$ . If  $A = 0$  is not chosen,  $\beta = 0$  is required and this leads to  $\phi(x) = 0$  for all  $x$  which is the at rest condition (not very interesting).
- Using the remaining case (since  $A = 0$ ), either  $C = 0$  or  $\sin \beta L = 0$ . Choosing  $C = 0$  isn't an option since that leads to  $\phi(x) = 0$  for all  $x$  which is the uninteresting at rest condition.

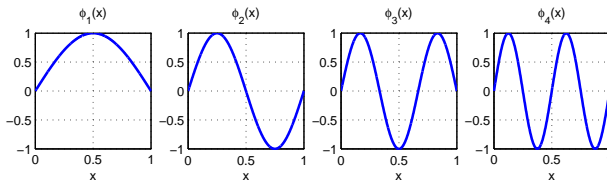


# Applying the boundary conditions to $\phi(x)$

- Therefore, must have  $\sin \beta L = 0$ , which implies  $\beta L = n\pi$ .
- After solving for  $\beta$ , the  $n$  solutions (which satisfy the B.C.'s) for  $\phi(x)$  are

$$\phi_n(x) = C_n \sin \frac{n\pi x}{L} \quad (16)$$

- This implies that the beam vibrates in the following natural mode shapes for  $n = 1, 2, 3, 4\dots$



# Applying boundary condition (5)

- Applying BC (5) yields

$$\dot{q}(0) = -a\beta^2 E \sin a\beta^2 0 + a\beta^2 F \cos a\beta^2 0 = 0 \quad (17)$$

- The sine term equals zero and hence  $F = 0$ . As a result

$$q(t) = E \cos a\beta^2 t \quad (18)$$

- In light of the fact that  $\beta = n\pi/L$

$$q_n(t) = E_n \cos \frac{an^2\pi^2 t}{L^2} \quad (19)$$



# Applying boundary condition (6)

- Combining (16) and (19) and defining  $b_n = C_n E_n$  yields

$$u_n(x, t) = \phi_n(x)q_n(t) = b_n \sin \frac{n\pi x}{L} \cos \frac{an^2\pi^2 t}{L^2} \quad (20)$$

- Equation (20) satisfies the PDE and the first 5 BC's for any value of  $n$  and arbitrary constants  $b_n$ . As a result, any linear combination of (20) also satisfies the requirements so that

$$u(x, t) = \sum_{n=1}^{\infty} b_n \sin \frac{n\pi x}{L} \cos \frac{an^2\pi^2 t}{L^2} \quad (21)$$

# Applying the boundary condition (6)

- To satisfy BC (6) the following must be true

$$u(x, 0) = Gx(L - x) = \sum_{n=1}^{\infty} b_n \sin \frac{n\pi x}{L} \quad (22)$$

- Hence, the  $b_n$  are the sine Fourier coefficients for  $Gx(L - x)$ . That is

$$b_n = \frac{2}{L} \int_0^L Gx(L - x) \sin \frac{n\pi x}{L} dx \quad (23)$$

$$= \frac{8GL^2}{n^3\pi^3} \quad \text{for } n \text{ odd} \quad (24)$$

$$= 0 \quad \text{for } n \text{ even} \quad (25)$$



# Final solution of the BVP

- Using the results of (21) and (23) gives the final solution of the BVP.

$$u(x, t) = \sum_{n=1,3,5,\dots}^{\infty} \frac{8GL^2}{n^3\pi^3} \sin \frac{n\pi x}{L} \cos \frac{an^2\pi^2 t}{L^2} \quad (26)$$

- Comments:

- Recall  $a^2 = EI/m$  which is known
- $G$  specifies initial amplitude at  $t = 0$ , hence is known
- By observing the cosine term of (26) it is concluded that the natural frequencies for the beam are

$$\omega_n = \frac{n^2\pi^2}{L^2} \sqrt{\frac{EI}{m}}$$

- Reference: Miller, Kenneth S., "Partial Differential Equations in Engineering Problems", Prentice-Hall, Englewood Cliffs, NJ, 1953.

