

**USER MANUAL**  
**OF**  
**USER MATERIAL SUBROUTINE (UMAT)**  
**FOR**  
**A NEW COUPLED MATERIAL MODEL FOR WOVEN FABRICS,**  
**INCLUDING A COUPLED NON-ORTHOGONAL HYPOELASTIC**  
**CONSTITUTIVE MODEL INTEGRATED WITH A NEW WRINKLING**  
**CRITERION**

**By:**

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*Submitted to 32<sup>nd</sup> ASC Technical Conference, Purdue University*

*Oct 2017*

## 1. Introduction

This manual contains review and evaluation of an ABAQUS User Subroutine UMAT\_Woven Fabric. UMAT\_Woven Fabric is a Fortran based code for the purpose of the numerical implementation of a new material model for woven fabrics in ABAQUS. This version of UMAT\_Woven Fabric is based on the material model presented in the paper submitted to ASC 2017 Conference (*Toward Enhanced Forming Simulation of Woven Fabrics using a Coupled Non-Orthogonal Hypoelastic Constitutive Model, Integrated with a New Wrinkling Onset Criterion*).

Section 2 of this manual provides context for the subroutine function, input and output parameters. Furthermore, an example is provided in Section 3 of this user manual to illustrate the effectiveness and validity of the newly developed UMAT compared to currently available material model for woven fabrics in ABAQUS. Eventually, the potential capabilities of the subroutine are discussed in Section 3.

## 2. Description of the Subroutine

### 2.1 Overview

One of the advantages of woven fabrics over unidirectional prepregs is their superior formability thanks to the large shear deformation capability. There exists, however, a limit on the shear deformation of woven fabrics, namely the wrinkling. Applying tension to delay wrinkling during forming processes, a consequence of the inherent coupling in woven fabrics, is widely known to the industry. Yet, inherent coupling – change in the effective material properties of a given direction of the fabric due to the applied deformation in other directions - has not been fully understood and implemented in the forming simulations of fabric reinforcements. Coupling should be incorporated in numerical optimization routines to accurately predict the deformation of the material under complex forming set-ups, and more importantly to predict a realistic yarn tension level that can suppress wrinkles. Towards this goal, a new coupled non-orthogonal model which predicts not only the stress-strain path, but also the critical point (shear wrinkling) of the woven fabrics should be proposed and implemented in a numerical simulation. The theory of the material model is thoroughly discussed in the submitted article. In summary, a coupled non-orthogonal hypoelastic constitutive model along with a criterion for wrinkling in terms of coupling are offered. To show its application, the model is implemented in ABAQUS via a UMAT code to predict the stress and strain fields as well as the onset of wrinkling under large shear deformations.

The popular user subroutine to study the solid mechanics of woven fabrics is VFabric subroutine, which is deeply explained in ABAQUS (DS Simulia) user manual. In spite of taking non-orthogonality into account, the inherent coupling is ignored in the VFabric subroutine. The code provided here is able to take the inherent coupling into consideration, resulting in more accurate prediction of the response of woven fabrics under forming processes. Moreover, the VFabric subroutine is efficient for dynamic explicit analyses and cannot be used for dynamic implicit and static FE simulations. Although the explicit analysis is more practical for studying

forming process; its precision, in particular for wrinkling prediction, is much weaker than implicit analysis. Also, coupled thermal-mechanical analysis to consider the effect of temperature on the mechanical properties of woven fabrics can be performed using the implicit approach. Hence, the presented subroutine has been written for the implicit solutions in ABAQUS. When ABAQUS calls the UMAT\_Woven Fabrics, the subroutine is provided with the inputs such as current stress as well as strain components and solution dependent state variables (SDV). The Subroutine undertakes the stress analysis and predicts the response of woven fabrics in the current time increment by providing outputs at the end of the increment. These outputs (stress, strain and SDV) will be used as next increment inputs.

## 2.2 Definitions of Subroutine Functions, Inputs and Outputs

The subroutine includes four basic modules. In the first module the stress and strain arrays are transformed from the orthogonal global coordinate system to the local non-orthogonal coordinate system. The material stiffness matrix will be determined and assembled in the second module based on local strain array. The third module transfers the local stiffness matrix to the global stiffness matrix. Finally, the last module computes the global stress array and SDVs for next increment.

### 2.2.1 Subroutine Parameters

The subroutine was written in accordance ABAQUS User Material Manual Guide. The following lines were imported from ABAQUS to provide proper context for UMAT coding:

```

SUBROUTINE UMAT(STRESS,STATEV,DDSDDE,SSE,SPD,SCD,
1 RPL,DDSDDT,DRPLDE,DRPLDT,
2 STRAN,DSTRAN,TIME,DTIME,TEMP,DTEMP,PRED,DPRED,CMNAME,
3 NDI,NSHR,NTENS,NSTATV,PROPS,NPROPS,COORDS,DROT,PNEWDT,
4 CELENT,DFGRD0,DFGRD1,NOEL,NPT,LAYER,KSPT,JSTEP,KINC)
C
INCLUDE 'ABA_PARAM.INC'
C
CHARACTER*80 CMNAME
DIMENSION STRESS(NTENS),STATEV(NSTATV),
1 DDSDDE(NTENS,NTENS),DDSDDT(NTENS),DRPLDE(NTENS),
2 STRAN(NTENS),DSTRAN(NTENS),TIME(2),PRED(1),DPRED(1),
3 PROPS(NPROPS),COORDS(3),DROT(3,3),DFGRD0(3,3),DFGRD1(3,3),
4 JSTEP(4)

```

We also need to define the dimension of arrays used in the subroutine code (the STRANL and STRESSL are local strain and stress arrays, the rest of the arrays have been defined in the code, Appendix 1):

**DIMENSION** DOBAR(4), DCBAR(4), DOSBAR(1), DCSBAR(1),  
**1**STRANL(NTENS),STRESSL(NTENS),TRANSF(NTENS,NTENS),  
**2**TRANSFT(NTENS,NTENS),  
**3**DBAR(NTENS,NTENS),DSTRANL(NTENS),DENTAL(NTENS,NTENS),  
**4**DDSDDDET(NTENS,NTENS), DELTA(1)

The coefficient and constant numbers that are used to define the stiffness function are coded as follows:

**PARAMETER**(ONE=1.D0, TWO=2.D0, THREE=3.D0, FOUR=4.D0, FIVE=5.D0,  
**1** AII=0.4518D12, BII=3.2748D10, CII=7.3238D8, DII=6.6648.D0,

.....

The basic arrays in this subroutine are:

TRANSF- the transformation matrix from local coordinate to global

TRANSFT- the transformation matrix from global coordinates to local

DDSDDDET- the local material stiffness matrix

STATEV(1)- $\epsilon_1$

STATEV(2)-  $\epsilon_2$

STATEV(3)-  $\epsilon_{12}$

STATEV(4)-  $\sigma_1$

STATEV(5)-  $\sigma_2$

STATEV(6)-  $\sigma_{12}$

STATEV (NSTATV)- the wrinkling indicator (1=wrinkled, 0=Normal)

### 2.2.2 Transformation from global orthogonal coordinate system to local non-orthogonal coordinate system

The details of the transformation between the orthogonal and non-orthogonal coordinate systems are presented in the submitted article. The transformation matrix can be written as:

$$T = \begin{bmatrix} \cos^2 \alpha & \cos^2(\alpha + \theta) & 2 \cos \alpha \cos(\alpha + \theta) \\ \sin^2 \alpha & \sin^2(\alpha + \theta) & 2 \sin \alpha \sin(\alpha + \theta) \\ \sin \alpha \cos \alpha & \sin(\alpha + \theta) \cos(\alpha + \theta) & \sin(2\alpha + \theta) \end{bmatrix}$$

This matrix implemented in the code as:

TRANSF(1,1)=COS(STRAN(3)/TWO)\*\*TWO  
TRANSF(1,2)=SIN(STRAN(3)/TWO)\*\*TWO  
TRANSF(1,3)=SIN(STRAN(3))  
TRANSF(2,1)=SIN(STRAN(3)/TWO)\*\*TWO

```

TRANSF(2,2)=COS(STRAN(3)/TWO)**TWO
TRANSF(2,3)=SIN(STRAN(3))
TRANSF(3,1)=SIN(STRAN(3)/TWO)*COS(STRAN(3)/TWO)
TRANSF(3,2)=SIN(STRAN(3)/TWO)*COS(STRAN(3)/TWO)
TRANSF(3,3)=ONE

```

### 2.2.3 Stiffness functions in the non-orthogonal coordinate system

The coupled stiffness functions in the local non-orthogonal coordinate system are determined based on the local strains. For a new woven fabric, a user needs only to modify this section based on the characterization results. As a matter of fact, this section of the code is the main body of the code which presents the coupled constitutive model in the nonorthogonal coordinate system.

```

DOBAR(1)=-AII*STATEV(1)**FIVE+BII*STATEV(1)**FOUR
1 -CII*STATEV(1)**THREE+DII*STATEV(1)**TWO-EII*STATEV(1)+FII
IF(STATEV(1).GT.EPSILONII) THEN
DOBAR(1)=-AII*EPSILONII**FIVE+BII*EPSILONII**FOUR
1 -CII*EPSILONII**THREE+DII*EPSILONII**TWO-EII*EPSILONII+FII
END IF
IF(STATEV(1).LT.ZERO) THEN
DOBAR(1)=FII
END IF
.....
.
.
.
.
.
.
.
.
.
.
C
IF(STATEV(1).LT.ZERO.OR.STATEV(1).EQ.ZERO) THEN
IF(STATEV(2).LT.ZERO.OR.STATEV(2).EQ.ZERO) THEN
DOSBAR(1)=ZPTZS
IF(STATEV(3).GT.ZPZS)Then

```

```

        DOSBAR(1)=DOSBAR(1)-ZPOZE
    END IF
    DCSBAR(1)=ONE
END IF
END IF

```

C

### 2.2.4 Generating the local stiffness matrix

The local stiffness matrix is assembled as follows :

```

    DBAR(1,1)=DOBAR(1)*DCBAR(1)
    DBAR(1,2)=DOBAR(3)*DCBAR(3)
    DBAR(1,3)=ZERO
    DBAR(2,1)=DOBAR(4)*DCBAR(4)
    DBAR(2,2)=DOBAR(2)*DCBAR(2)
    DBAR(2,3)=ZERO
    DBAR(3,1)=ZERO
    DBAR(3,2)=ZERO
    DBAR(3,3)=DCSBAR(1)*DOSBAR(1)

```

### 2.2.5 Stress Analysis in Global Coordinate System

After transferring stiffness matrix from local to global coordinate system (by using TRANSF array), the stress analysis and updating the state variables are as follows:

```

DO K1=1, NTENS
    DO K2=1, NTENS
        DDSDDDE(K1,K2)=DDSDDDET(K1,K2)
    END DO
END DO

DO K1=1, NTENS
    DO K2=1, NTENS
        STRESS(K1)=STRESS(K1)+DDSDDDE(K1,K2)*DSTRAN(K2)
    END DO
END DO

```

Updating SDVs:

```

DO K1=1, NTENS
  STATEV(K1)=STRANL(K1)
  STATEV(NTENS+K1)=STRESSL(K1)
END DO

```

### 2.2.6 Wrinkling prediction

The criterion for prediction of wrinkling occurrence is developed in the submitted article. The last step of the material subroutine is to predict wrinkling initiation which is coded as:

```

DELTA(1)=ABS(STRESSL(1))**(ONE+HALF)/(ABS(STRANL(1))**(ONE+HALF)+
1 STRANL(1)**HALF-TANH(ABS(STRANL(1))**HALF))
IF(STATEV(3).LT.-ACOS(ANGS).OR.STATEV(3).GT.DELTA(1)) THEN
  STATEV(3*NTENS)=ONE
END IF
STATEV(NSTATV)=DELTA(1)

```

## 3. Use of the Subroutine in ABAQUS

To use the UMAT\_Woven Fabrics, it is required to select the user material in the defining properties section. Afterwards, when it comes to define the job, we should provide the path for the Fortran file of UMAT\_Woven Fabrics in the special section of the Job bar.

### 3.1 An example to demonstrate the advantage of the new subroutine

To prove the effectiveness and validity of the written subroutine, an example is provided below. Firstly, 2% tension in the transverse direction is applied and then tension in the main direction is applied up to 4%. Figure 1 demonstrates the higher accuracy of the coupled model (the new model) compared to the uncoupled model (The currently available woven fabrics subroutine in ABAQUS - VFabric).

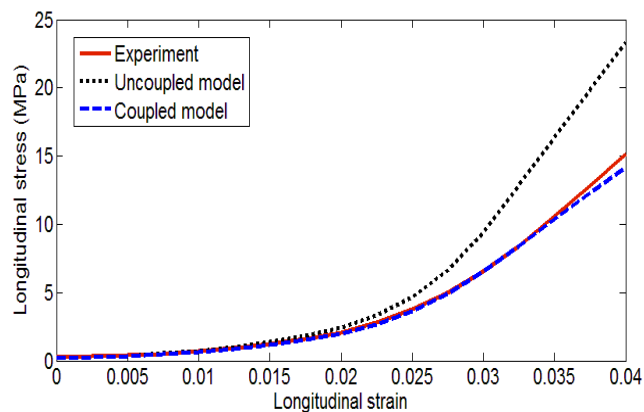


Figure 1. Comparison between the newly developed material model for woven fabrics and the currently available woven fabrics model – VFabric - in ABAQUS.

### **3.3 Potential Application**

The results showed the advantages of the presented UMAT for woven fabrics over the VFabric subroutine which is available in ABAQUS. In fact, the new subroutine predicts the behavior of woven fabrics under forming processes with higher precision in comparison with VFabric subroutine. Moreover, thanks to taking the inherent coupling into account, the manufacturing process parameters such as the applied pressure on the blank holder can be optimized using this new UMAT to prevent wrinkling. Furthermore, since the presented code was written for implicit solutions, coupled thermal-mechanical analyses to capture the effect of temperature on the mechanical properties is also possible. The last, but not least, it can be used for various types of woven fabrics by just modifying the stiffness functions section based on the characterization results of the given woven fabrics.



## Appendix 1 : UMAT\_WOVEN FABRICS

```
C *****
c *****
C           UMAT FOR Woven Fabrics
c           UMAT_WOVEN FABRICS
c           User Material Subroutine for Coupled Non-Orthogonal
c           Constitutive Modeling of Woven Fabrics
c
c           January, 2017
c           Version 1.1
c *****
c Applications:
c           2-D Planer Stress/Strain
c           Conventional Shell Element S4
c *****
c Authors:
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c
c           Department of Mechanical Engineering,
C           University Of British Columbia, Canada
c
C
C -----
C
C SUBROUTINE UMAT(STRESS,STATEV,DDSDDE,SSE,SPD,SCD,
1 RPL,DDSDDT,DRPLDE,DRPLDT,
2 STRAN,DSTRAN,TIME,DTIME,TEMP,DTEMP,PREDEF,DPRED,CMNAME,
3 NDI,NSHR,NTENS,NSTATV,PROPS,NPROPS,COORDS,DROT,PNEWDT,
4 CELENT,DFGRD0,DFGRD1,NOEL,NPT,LAYER,KSPT,JSTEP,KINC)
C
C INCLUDE 'ABA_PARAM.INC'
C
C CHARACTER*80 CMNAME
C DIMENSION STRESS(NTENS),STATEV(NSTATV),
1 DDSDE(NTENS,NTENS),DDSDDT(NTENS),DRPLDE(NTENS),
2 STRAN(NTENS),DSTRAN(NTENS),TIME(2),PREDEF(1),DPRED(1),
3 PROPS(NPROPS),COORDS(3),DROT(3,3),DFGRD0(3,3),DFGRD1(3,3),
4 JSTEP(4)
c
C LOCAL ARRAYS
C -----
C DOBAR - PURE TENSILE STIFNESS MODULES
```

C DCBAR - COUPLING TENSILE STIFNESS INDUCED MODULUS  
 C DOSBAR- PURE SHEAR STIFNESS MODULES  
 C DCSBAR- COUPLING SHEAR STIFNESS INDUCED MODULUS  
 C TRANSF- TRANSFORMATION MATRIX  
 C DBAR- JACOBIAN NON-ORTHOGONAL MATRIX  
 C STRANL- NON-ORTHOGONAL STRAIN ARRAY  
 C STRESSL- NON-ORTHOGONAL STRESS ARRAY  
 C

**DIMENSION** DOBAR(4), DCBAR(4), DOSBAR(1), DCSBAR(1),  
 1 STRANL(NTENS), STRESSL(NTENS), TRANSF(NTENS,NTENS),  
 2 TRANSFT(NTENS,NTENS),  
 3 DBAR(NTENS,NTENS), DSTRANL(NTENS), DENTAL(NTENS,NTENS),  
 4 DDSDDT(NTENS,NTENS), DELTA(1)

C  
 C

**PARAMETER**(ONE=1.D0, TWO=2.D0, THREE=3.D0, FOUR=4.D0, FIVE=5.D0,  
 1 AII=0.4518D12, BII=3.2748D10, CII=7.3238D8, DII=6.6648D6,  
 2 EII=1.3571D4, FII=0.3344D2, GII=27.47D0,  
 3 EPSILONII=0.03D0, OPTH=1.3D0, ZPSF=0.83D0, ZPTE=0.28D0,  
 4 ZPTF=0.025D0, FIFFOUR=54.D0, TPS=2.6D0, THUND=1200.D0,  
 5 ZPZT=0.02D0, THOO=211.D0, ZPTZS=0.1587D0, ZPOZE=0.1071D0,  
 6 ZPZS=0.07D0, SVHU=1700.D0, TAH=2.5D0, OFFN=14597.D0,  
 7 OPTT=1.22D0, TWTH=12.D3, HALF=0.5D0, OFSZ=1570.D0,  
 8 OFSE=15685.D0, OOTF=11353.D0, ZPZOS=0.017D0, ZERO=0.D0,  
 9 TOL=1.D-10, ANGS=0.921D0)

C  
 C  
 C-----  
 C

TRANSF(1,1)=**COS**(STRAN(3)/TWO)\*\*TWO  
 TRANSF(1,2)=**SIN**(STRAN(3)/TWO)\*\*TWO  
 TRANSF(1,3)=**SIN**(STRAN(3))  
 TRANSF(2,1)=**SIN**(STRAN(3)/TWO)\*\*TWO  
 TRANSF(2,2)=**COS**(STRAN(3)/TWO)\*\*TWO  
 TRANSF(2,3)=**SIN**(STRAN(3))  
 TRANSF(3,1)=**SIN**(STRAN(3)/TWO)\***COS**(STRAN(3)/TWO)  
 TRANSF(3,2)=**SIN**(STRAN(3)/TWO)\***COS**(STRAN(3)/TWO)  
 TRANSF(3,3)=ONE

**DO** K1=1, NTENS  
**DO** K2=1, NTENS  
 TRANSFT(K1,K2)=TRANSF(K2,K1)  
**END DO**  
**END DO**

C

C TRANSFORMING TO NONORTHOGONAL STRANL AND DSTRANL

C -----

C

```
DO K1=1, NTENS
  STRANL(K1)=ZERO
  DSTRANL(K1)=ZERO
END DO
```

```
DO K1=1, NTENS
  DO K2=1, NTENS
    STRANL(K1)=STRANL(K1)+TRANSFT(K1,K2)*(STRAN(K2)+DSTRAN(K2))
  END DO
END DO
```

```
DO K1=1, NTENS
  DO K2=1, NTENS
    DSTRANL(K1)=DSTRANL(K1)+TRANSFT(K1,K2)*DSTRAN(K2)
  END DO
END DO
```

C

C

C -----

C -----

C -----

C FUNCTIONS DOBAR DCBAR DOSBAR DC

C -----

C DOBAR..

C -----

```
DOBAR(1)=-AII*STATEV(1)**FIVE+BII*STATEV(1)**FOUR
1 -CII*STATEV(1)**THREE+DII*STATEV(1)**TWO-EII*STATEV(1)+FII
IF(STATEV(1).GT.EPSILONII) THEN
  DOBAR(1)=-AII*EPSILONII**FIVE+BII*EPSILONII**FOUR
1 -CII*EPSILONII**THREE+DII*EPSILONII**TWO-EII*EPSILONII+FII
END IF
IF(STATEV(1).LT.ZERO) THEN
  DOBAR(1)=FII
END IF
```

C

```
DOBAR(2)=-AII*STATEV(2)**FIVE+BII*STATEV(2)**FOUR
1 -CII*STATEV(2)**THREE+DII*STATEV(2)**TWO-EII*STATEV(2)+FII
IF(STATEV(2).GT.EPSILONII) THEN
  DOBAR(2)=-AII*EPSILONII**FIVE+BII*EPSILONII**FOUR
1 -CII*EPSILONII**THREE+DII*EPSILONII**TWO-EII*EPSILONII+FII
END IF
IF(STATEV(2).LT.ZERO) THEN
```

DOBAR(2)=FII  
**END IF**

C

DOBAR(3)=(TPS+THUND\*STATEV(2)-THUND\*(STATEV(2)-ZPZT))/OPTH  
**IF**(STATEV(2).LT.ZPZT) **THEN**  
DOBAR(3)=(TPS+THUND\*STATEV(2))/OPTH  
**END IF**  
**IF**(STATEV(2).LT.ZERO) **THEN**  
DOBAR(3)=TPS  
**END IF**

C

DOBAR(4)=(TPS+THUND\*STATEV(1)-THUND\*(STATEV(1)-ZPZT))/OPTH  
**IF**(STATEV(1).LT.ZPZT) **THEN**  
DOBAR(4)=(TPS+THUND\*STATEV(1))/OPTH  
**END IF**  
**IF**(STATEV(1).LT.ZERO) **THEN**  
DOBAR(4)=TPS  
**END IF**

C

C

C DCBAR

C

-----  
DCBAR(1)=ONE-GII\*(STATEV(2)+TOL)\*\*OPTH  
**IF**(STATEV(1).GT.ZPTF) **THEN**  
DCBAR(1)=DCBAR(1)-ZPSF\*(STATEV(2)+TOL)\*\*ZPTE  
**END IF**  
**IF**(STATEV(2).GT.ZPTF) **THEN**  
**IF**(STATEV(1).LT.EPSILONII.OR.STATEV(1).EQ.EPSILONII) **THEN**  
DCBAR(1)=DCBAR(1)+(TWO-FIFFOUR\*STATEV(1))  
**END IF**  
**IF**(STATEV(1).GT.EPSILONII) **THEN**  
DCBAR(1)=DCBAR(1)+(TWO-FIFFOUR\*EPSILONII)  
**END IF**  
**END IF**

DCBAR(2)=ONE-GII\*(STATEV(1)+TOL)\*\*OPTH  
**IF**(STATEV(2).GT.ZPTF) **THEN**  
DCBAR(2)=DCBAR(2)-ZPSF\*(STATEV(1)+TOL)\*\*ZPTE  
**END IF**  
**IF**(STATEV(1).GT.ZPTF) **THEN**  
**IF**(STATEV(2).LT.EPSILONII.OR.STATEV(2).EQ.EPSILONII) **THEN**  
DCBAR(2)=DCBAR(2)+(TWO-FIFFOUR\*STATEV(2))  
**END IF**  
**IF**(STATEV(2).GT.EPSILONII) **THEN**

```
DCBAR(2)=DCBAR(2)+(TWO-FIFFOUR*EPSILONII)
END IF
END IF
```

```
DCBAR(3)=ONE+THOO*STATEV(1)
```

```
DCBAR(4)=ONE+THOO*STATEV(2)
```

```
IF(STATEV(2).LT.ZERO.OR.STATEV(1).LT.ZERO) THEN
DO K1=1, 4
DCBAR(K1)=ONE
END DO
END IF
```

```
C -----
```

```
C -----
```

```
C
```

```
C DOSBAR AND DCSBAR
```

```
C -----
```

```
DOSBAR(1)=ZPTZS
```

```
DCSBAR(1)=ONE+SVHU*(STATEV(1)+STATEV(2))
```

```
IF(STATEV(1).GT.ZPTF) THEN
```

```
DCSBAR(1)=DCSBAR(1)+OFFN*(STATEV(1)-ZPTF)**OPPT
```

```
END IF
```

```
IF(STATEV(2).GT.ZPTF) THEN
```

```
DCSBAR(1)=DCSBAR(1)+OFFN*(STATEV(2)-ZPTF)**OPPT
```

```
IF(STATEV(1).GT.ZPTF) THEN
```

```
DCSBAR(1)=DCSBAR(1)
```

```
1 +TWTH*((STATEV(1)-ZPTF)*(STATEV(2)-ZPTF))**HALF
```

```
END IF
```

```
END IF
```

```
IF(STATEV(3).GT.ZPZOS) THEN
```

```
DCSBAR(1)=DCSBAR(1)-OFSZ*(STATEV(1)+STATEV(2))
```

```
IF(STATEV(1).GT.ZPTF) THEN
```

```
DCSBAR(1)=DCSBAR(1)-OFSE*(STATEV(1)-ZPTF)**OPPH
```

```
END IF
```

```
IF(STATEV(2).GT.ZPTF) THEN
```

```
DCSBAR(1)=DCSBAR(1)-OFSE*(STATEV(2)-ZPTF)**OPPH
```

```

        IF(STATEV(1).GT.ZPTF) THEN
            DCSBAR(1)=DCSBAR(1)
1       -OOTF*((STATEV(1)-ZPTF)*(STATEV(2)-ZPTF)**HALF
C
        END IF
        END IF
        END IF
C
        IF(STATEV(1).LT.ZERO.OR.STATEV(1).EQ.ZERO) THEN
            IF(STATEV(2).LT.ZERO.OR.STATEV(2).EQ.ZERO) THEN
                DOSBAR(1)=ZPTZS
                IF(STATEV(3).GT.ZPZS)Then
                    DOSBAR(1)=DOSBAR(1)-ZPOZE
                END IF
                DCSBAR(1)=ONE
            END IF
        END IF
C
C -----
C -----
C
C
C ASSEMBLING NONORTHOGONAL JACOBIAN MATRIX
C -----
        DBAR(1,1)=DOBAR(1)*DCBAR(1)
        DBAR(1,2)=DOBAR(3)*DCBAR(3)
        DBAR(1,3)=ZERO
        DBAR(2,1)=DOBAR(4)*DCBAR(4)
        DBAR(2,2)=DOBAR(2)*DCBAR(2)
        DBAR(2,3)=ZERO
        DBAR(3,1)=ZERO
        DBAR(3,2)=ZERO
        DBAR(3,3)=DCSBAR(1)*DOSBAR(1)
C-----
C-----
C-----
C CALCULATING ORTHOGONAL JACOBIAN MATRIX AND STRESS
C -----
C
        DO K1=1, NTENS
c       STRESS(K1)=ZERO
        DO K2=1, NTENS
            DDSDDT(K1,K2)=ZERO

```

```
DENTAL(K1,K2)=ZERO
END DO
END DO
```

```
DO K1=1, NTENS
  DO K2=1, NTENS
    DO K3=1, NTENS
      DENTAL(K1,K2)=DENTAL(K1,K2)+TRANSF(K1,K3)*DBAR(K3,K2)
    END DO
  END DO
END DO
```

```
DO K1=1, NTENS
  DO K2=1, NTENS
    DO K3=1, NTENS
      DDSDET(K1,K2)=DDSDET(K1,K2)+DENTAL(K1,K3)*TRANSFT(K3,K2)
    END DO
  END DO
END DO
```

```
DO K1=1, NTENS
  DO K2=1, NTENS
    DDSDE(K1,K2)=DDSDET(K1,K2)
  END DO
END DO
```

```
DO K1=1, NTENS
  DO K2=1, NTENS
    STRESS(K1)=STRESS(K1)+DDSDE(K1,K2)*DSTRAN(K2)
  END DO
END DO
```

### C CALCULATING NONORTHOGONAL STRESSES

```
C-----
DO K1=1, NTENS
  STRESSL(K1)=ZERO
END DO
DO K1=1, NTENS
  DO K2=1, NTENS
    STRESSL(K1)=STRESSL(K1)+TRANSFT(K1,K2)*STRESS(K2)
  END DO
END DO
```

```
C-----
C-----
```

```

C
  DO K1=1, NTENS
    STATEV(K1)=STRANL(K1)
    STATEV(NTENS+K1)=STRESSL(K1)
  END DO
C WRINKLE PREDECTION
C-----
  DELTA(1)=ABS(STRESSL(1))**(ONE+HALF)/(ABS(STRANL(1))**(ONE+HALF)+
  1 STRANL(1)**HALF-TANH(ABS(STRANL(1))**HALF))
  IF(STATEV(3).LT.-ACOS(ANGS).OR.STATEV(3).GT.DELTA(1)) THEN
    STATEV(3*NTENS)=ONE
  END IF
  STATEV(NSTATV)=DELTA(1)
C-----
C-----
C
  RETURN
  END

```



## Appendix 2: The Input File of the Example

\*Heading

\*\* Job name: picturrame Model name: Model-1

\*\* Generated by: Abaqus/CAE 6.14-2

\*Preprint, echo=NO, model=NO, history=NO, contact=NO

\*\*

\*\* PARTS

\*\*

\*Part, name=Part-1

\*Node

1,	0.,	0.
2,	1.,	0.
3,	2.,	0.
4,	3.,	0.
5,	4.,	0.
6,	5.,	0.
7,	0.,	1.
8,	1.,	1.
9,	2.,	1.
10,	3.,	1.
11,	4.,	1.
12,	5.,	1.
13,	0.,	2.
14,	1.,	2.
15,	2.,	2.
16,	3.,	2.
17,	4.,	2.
18,	5.,	2.

19,	0.,	3.
20,	1.,	3.
21,	2.,	3.
22,	3.,	3.
23,	4.,	3.
24,	5.,	3.
25,	0.,	4.
26,	1.,	4.
27,	2.,	4.
28,	3.,	4.
29,	4.,	4.
30,	5.,	4.
31,	0.,	5.
32,	1.,	5.
33,	2.,	5.
34,	3.,	5.
35,	4.,	5.
36,	5.,	5.

\*Element, type=CPS4

1, 1, 2, 8, 7  
2, 2, 3, 9, 8  
3, 3, 4, 10, 9  
4, 4, 5, 11, 10  
5, 5, 6, 12, 11  
6, 7, 8, 14, 13  
7, 8, 9, 15, 14  
8, 9, 10, 16, 15  
9, 10, 11, 17, 16

10, 11, 12, 18, 17

11, 13, 14, 20, 19

12, 14, 15, 21, 20

13, 15, 16, 22, 21

14, 16, 17, 23, 22

15, 17, 18, 24, 23

16, 19, 20, 26, 25

17, 20, 21, 27, 26

18, 21, 22, 28, 27

19, 22, 23, 29, 28

20, 23, 24, 30, 29

21, 25, 26, 32, 31

22, 26, 27, 33, 32

23, 27, 28, 34, 33

24, 28, 29, 35, 34

25, 29, 30, 36, 35

\*Nset, nset=Set-1, generate

1, 36, 1

\*Elset, elset=Set-1, generate

1, 25, 1

\*\* Section: Section-1

\*Solid Section, elset=Set-1, material=Material-1

1.3,

\*End Part

\*\*

\*\*

\*\* ASSEMBLY

\*\*

```
*Assembly, name=Assembly
**
*Instance, name=Part-1-1, part=Part-1
*End Instance
**
*Nset, nset=Set-1, instance=Part-1-1, generate
  1, 31, 6
*Elset, elset=Set-1, instance=Part-1-1, generate
  1, 21, 5
*Nset, nset=Set-2, instance=Part-1-1, generate
  1, 6, 1
*Elset, elset=Set-2, instance=Part-1-1, generate
  1, 5, 1
*Nset, nset=Set-3, instance=Part-1-1, generate
  6, 36, 6
*Elset, elset=Set-3, instance=Part-1-1, generate
  5, 25, 5
*Nset, nset=Set-4, instance=Part-1-1, generate
  31, 36, 1
*Elset, elset=Set-4, instance=Part-1-1, generate
  21, 25, 1
*Nset, nset=Set-5, instance=Part-1-1
  1,
*End Assembly
**
** MATERIALS
**
*Material, name=Material-1
```

```
*Depvar
  30,
*User Material, constants=1
1.,
** -----
**
** STEP: Step-1
**
*Step, name=Step-1, nlgeom=NO
*Static
0.0625, 1., 1e-05, 0.0625
**
** BOUNDARY CONDITIONS
**
** Name: BC-3 Type: Displacement/Rotation
*Boundary
Set-3, 2, 2, 1.
** Name: BC-4 Type: Displacement/Rotation
*Boundary
Set-4, 1, 1, 1.
** Name: BC-5 Type: Displacement/Rotation
*Boundary
Set-5, 1, 1
Set-5, 2, 2
**
** OUTPUT REQUESTS
**
*Restart, write, frequency=0
```

\*\*

\*\* FIELD OUTPUT: F-Output-1

\*\*

\*Output, field, variable=PRESELECT

\*\*

\*\* HISTORY OUTPUT: H-Output-3

\*\*

\*Output, history

\*Element Output, elset=Part-1-1.Set-1

E11, E12, E22

\*\*

\*\* HISTORY OUTPUT: H-Output-2

\*\*

\*Element Output, elset=Part-1-1.Set-1

S11, S12, S22, SDV

\*\*

\*\* HISTORY OUTPUT: H-Output-1

\*\*

\*Output, history, variable=PRESELECT

\*End Step